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M. Aslam, W. G. Godden, and D. T. Scalise

August 1978

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SLOSHING OF WATER IN TORUS

PRESSURE-SUPPRESSION POOL OF BOILING WATER REACTORS

UNDER EARTHQUAKE GROUND MOTIONS

A Report of an Analytical and Experimental Study of Sloshing Response in Axisymmetric Tanks Under Earthquake Ground Motions

by

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August 1978

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LIST OF SYMBOLS

- a Outer radius of tank
- b Inner radius of tank
- Bl Liquid-solid interface boundary
- B2 Free surface boundary
- EB1 Element boundary B1
- EB2 Element boundary B2
- EV Element Volume

F Load vector (Solid-liquid interface matrix)

- g Acceleration of gravity
- h Water depth
- J Jacobian matrix

K Fluid matrix (equivalent to stiffness matrix)

M Free surface matrix (equivalent to mass matrix)

- N Shape functions or a number
- p Pressure
- r Local coordinate
- s Local coordinate or surface
- t Time
- v Normal velocity
- X,Y,Z Rectangular coordinates
- Velocity potential
- ϕ $d\phi/dt$

∆t Time step

ρ Mass density

δ Free surface displacement or a parameter

v

ABSTRACT

-vi-

This report presents an analytical and experimental investigation into the sloshing of water in torus tanks under horizontal earthquake ground motions. This study was motivated because of the use of torus tanks for pressure-suppression pools in Boiling Water Reactors. Such a pressure-suppression pool would typically have 80 ft and 140 ft inside and outside diameters, a 30 ft diameter section, and a water depth of 15 ft.

A general finite element analysis was developed for all axisymmetric tanks and a computer program was written to obtain time-history plots of sloshing displacements of water and dynamic pressures. Tests were carried out on a 1/60th scale model under sinusoidal as well as simulated earthquake ground motions. Tests and analytical results regarding natural frequencies, surface water displacements, and dynamic pressures were compared and a good agreement was found within the range of displacements studied. The computer program gave satisfactory results within a maximum range of sloshing displacements in the full-size prototype of 30 in. which is greater than the value obtained under the full intensity of the El Centro earthquake (N-S component 1940). The range of linear behavior was studied experimentally by subjecting the torus model to increasing intensities of the El Centro earthquake. The general computer program was also used for comparison with a previous study on the sloshing of water in annular tanks, and the previous annular tank solution was also used as an approximate solution in the torus tank

problem showing that sloshing response is not very sensitive to the precise cross-section geometry of an axisymmetric tank.

Tests were also conducted to study the effect of vertical ground motions on the dynamic response of the fluid. These showed a negligible effect on sloshing displacements.

KEY WORDS

Sloshing Response, Pressure-suppression pool, Torus tanks, Annular tanks, Earthquake response, Dynamic pressures, Finite element solution.

1. INTRODUCTION

1.1 Objective

This report presents the results of a study into the sloshing response of water in torus tanks under the action of earthquake-induced ground motions, and is a continuation of a previous investigation into the sloshing response of water in annular tanks [1].

Torus tanks are used as pressure-suppression pools in certain designs of boiling water reactors (BWR) (e.g., General Electric Mark I), and a knowledge of the dynamic response of the water and particularly of the resulting water surface elevations is important in evaluating the effectiveness of the system under earthquake conditions. A typical torus suppression pool as used in the GE Mark I reactor has an outside diameter of 140 ft and a section height of 30 ft. A 1/60th scale idealized model of such a pool is shown in Fig. 2-la.

Rather than deriving a particular analytical solution to the torus tank problem, it was decided at the outset to undertake a general study into the sloshing of water in axisymmetric tanks; this would be applicable to the torus tank, the annular tank previously studied, and to all tanks with a constant section of revolution. References [1] through [16] are some of the previous studies related to the sloshing of fluids in tanks.

1.2 Scope of the Investigation

This study includes the testing of a 1/60th scale model of GE Mark 1 torus, the development of a general finite element theory for axisymmetric tanks, and a computer code to implement the finite element theory. A comparison of test results with an approximate analysis based on the annular tank theory is also given in Chapter 2.

The test model was constructed by cementing together wedge-shaped lengths of 6 in. lucite tubing as shown in Fig. 2-1. This was tested under harmonic and simulated earthquake ground motions. The quantities measured included the sloshing frequencies and free surface displacements.

The finite element equations (Chapter 3) were derived using the Galerkin principle and a linearized small displacement theory was used. The velocity potential ϕ was taken as the field variable and the sloshing displacements as well as impulsive pressures were derived from ϕ . The finite element equations were first derived for a general three dimensional sloshing problem under arbitrary ground motions, and then it was specialized to an axisymmetric tank subjected to horizontal earth-quake ground motions only.

A computer code named 'SLOSH2' was developed to implement the finite element theory, and a comparison of sloshing displacements and pressures as predicted by the computer program was made with the test results from annular [1] and torus tanks. These comparisons show that the finite element program can successfully predict the sloshing displacements as well as impulsive pressures in an axisymmetric tank under horizontal ground motions within the range of linear behavior.

2.1 Introduction

The objective of the tests on the torus tank model was to obtain experimental data to compare with the results obtained from approximate analysis based on a previous study on annular tanks [1], and also to check the accuracy of the finite element solution developed and described in Chapter 3.

Test data on sloshing displacements was obtained for harmonic as well as simulated seismic-type ground motions. Details of the test procedure, experimental data, and comparison with results of approximate analysis based on annular tank theory are discussed in this chapter.

2.2 Model Description and Instrumentation

A simplified 1/60th scale model of a Mark 1 suppression pool was constructed as shown in Figs. 2-la and 2-lb. The model was fabricated from 16 short lengths of 6 in. diameter clear plastic tubing cemented together to approximate complete torus. The internal details of the prototype, including the 'Headers' and 'Downcomers', were not reproduced in the model and not taken into account in the analysis. However, for reference purposes, the Header and Downcomer configuration is shown in Fig. 2-lC for a 1/30th scaled model designed for proof-tests by the reactor manufacturer. The mean diameter of the 1/60th scale model was 22 in. The normal operating water depth was 3 in.; that is, the water

surface at the section diameter, though water surface elevations above and below this level were also studied. The tank was mounted on a plywood base which was in turn prestressed to the shaking table.

The model was instrumented with one displacement gage located at a distance of 3/8 in. from the inside wall. This gage was of the same type as used in the annular tank tests [1].

2.3 Test Procedure and Experimental Data

2.3.1 Tests on the small shaking table

Sloshing frequencies of the 1/60th scale model of the Mark 1 torus and the steady state sloshing response under sinusoidal table motions were measured using the 3×4 ft shaking table described in Ref. [1] Sec. 4.4. The test set-up is shown in Fig. 2-2 and is similar to that described in Ref. [1] Sec. 4.5, the only difference being that this time a spectrum analyzer was used to determine the sloshing frequencies.

The test procedure to determine the steady state sloshing response under sinusoidal motions was the same as described in Ref. [1] Sec. 4.5 and was measured at the gage location shown in Fig. 2-la. To determine the sloshing frequencies, the frequency of the table motion was continuously changed and the water displacement signal was fed to the spectrum analyzer which was arranged to produce an averaged frequency spectrum. In this way the sloshing frequencies were read directly.

Tests on the small shaking table were not only conducted at the normal depth of 3 in., but also at depths of 2, 2.5, 3.5 and 4 in.

Test data regarding the sloshing frequencies and the wave heights are presented in Tables 2-1 and 2-2 respectively. Table 2-1 shows the mode number, the depth of water, and the measured as well as the analytical sloshing frequencies as approximated by using annular tank theory. Table 2-2 gives the steady-state response for harmonic ground motion, and tabulates the frequency of table motion, depth of water in the tank, amplitude of the table acceleration, and measured as well as the predicted values of the amplitudes of wave heights at the gage location. Two approximate solutions are given based on annular tank theory and are described in Sec. 2.4.

2.3.2 Tests under simulated earthquakes

Tests under simulated earthquake motion using the El Centro (1940) record were made at the University of California's Earthquake Simulator Laboratory. The test set-up of the 1/60th scale model of the Mark 1 torus is shown in Fig. 2-3 where the model is shown mounted on the 20×20 ft shaking table. Details of the shaking table facility and data acquisition system are given in Ref. [1] Chapter 5. The testing procedure was similar to that described in Ref. [1] Sec. 5.6. The prototype of the El Centro earthquake record was reduced by a factor of $\sqrt{60}$ to meet the similitude requirements of a 1/60th scale model.

In each test the quantities that were recorded, digitized and stored on magnetic tape included the horizontal and vertical table accelerations and displacements, and the water surface displacements in the model at the gage location shown in Fig. 2-1. The peak values of these quantities together, with the test numbers are shown in Table 2-3. The maximum and minimum water displacements in this table represents the

peak upward and downward displacements respectively. These tests were carried out both with and without the vertical component of the recorded ground motion. They were made at increasing amplitudes of ground motion in order to determine the range of linear response. Some selected results are plotted in Figs. 2-4 through 2-8. The center graph in Figs. 2-4 through 2-7 also shows a comparison of the measured displacement with approximate results from annular tank theory under horizontal ground motion only. Figure 2-8 shows that the vertical ground motion alone does not produce sloshing displacements as could be expected.

It should be remembered that a shaking table acts as a low-pass filter and will not fully reproduce motions at frequencies above the natural frequency of the system. The small time scale ($T_r = \sqrt{60} = 7.75$) used in this study resulted in the acceleration peaks associated with high frequency ground motion being filtered out by the table. Hence the intensity of the earthquake given in Table 2-3, and measured by peak table acceleration, cannot be compared directly with the intensity of the original El Centro record. However the sloshing response is produced mainly by the low frequency components of the ground motion and these are correctly reproduced. It is estimated that the simulated earthquake ground motion in Table 2-3 with a peak acceleration of 0.34 g is equivalent to approximately 2.0 times the actual intensity of the El Centro earthquake in the significant low frequency components.

(Note: The recorded peak acceleration in the actual 1940 El Centro earthquake is 0.32 g.)

2.4 Comparison of Test Results with Approximate Theory

Approximate analyses were carried out using the computer program 'SLOSH' which is based on the annular tank theory developed in Ref. [1]. For carrying out the analyses the measured table acceleration was used and the tank radii a and b were taken as the horizontal distances from the axis of symmetry to the outer and inner walls where the free water surfaces come in contact with the solid boundaries. Two approximations were then made for the water depth as follows:

(1) In the first approximation the depth of the water (h) in the torus tank solution was taken as the actual depth from the bottom of the torus to the free surface. This is called the 'Annular Tank' approximation in Tables 2-1 and 2-2.

(2) In the second approximation, the depth of water used in the annular solution was adjusted to make the volume of water in the annular tank (with the same values of a and b as the torus tank) equal to the actual volume in the torus. This is called the 'Equivalent Volume' approximation in Tables 2-1 and 2-2.

Using these two approximations, analyses were carried out using the SLOSH program and the comparison of test and computer results for the sloshing frequencies and displacements is given.

2.4.1 Comparison of natural sloshing frequencies

Table 2-1 shows the comparison between the approximate results based on annular tank theory and the test values. It can be seen that for the first four modes, and within the range of water depth considered, the two approximations give very similar results. This could be expected

as in an annular tank over this range the natural frequencies are not very sensitive to water depth. Also, the approximate solutions compare well with the measured torus values in the 2 in. to 4 in. depth range indicating that frequencies are not very sensitive to the actual shape of cross-section of the tank.

2.4.2 <u>Comparison of sloshing displacements under steady state harmonic</u> ground motion

Table 2-2 gives a comparison of the measured and computed results from approximate annular tank theory for water depths ranging from 2.5 in. to 3.5 in. under steady state sinusoidal table motions varying in frequency from 1.50 Hz to 2.55 Hz. It can be observed in Table 2-2 that the annular tank theory gives satisfactory results for this range of water depth and the 'Equivalent Volume' approximation gives better results in most cases. It was also found that when the torus tank is nearly empty or nearly full (water depths less than 2 in. or greater than 4 in. in the model) as could be expected the approximate theory does not give satisfactory results and should not be used.

2.4.3 Comparison of sloshing displacements under simulated earthquake ground motions

Figures 2-4 through 2-7 show the time-history plots of the water surface displacements at a distance of 3/8" from the inside wall under increasing intensities of the simulated El Centro earthquake motion (horizontal component only). The depth of water in the torus tank was 3 in. in all these tests and the comparison of measured and approximate analytical displacements is given in the middle plot in each case. In Fig. 2-4

the analysis was carried out using approximation (1) (i.e., depth of water in the annular solution was taken as 3 in.). It can be seen that the measured and computed results differ by about 30% in displacement amplitude, and the measured frequency of response oscillation is somewhat lower. In Fig. 2-5 the 'Equivalent Volume' approximation was used (h = 2.36 in.) for the same ground motion and it can be seen that the agreement between the test and approximate analysis is better as regards displacement amplitude.

Figures 2-6 and 2-7 show a comparison at higher intensities of the ground motion where the analysis in both cases was done using the 'Equivalent Volume' annular tank approximation. It can be seen that even at these relatively large displacements the approximate theory, where a modified depth based on an equivalent volume is used in the annular tank solution, gives reasonably satisfactory results. This approximate theory should however be used with caution especially when the water depth is outside the range of that used in these tests.

2.5 Dynamic Pressures

The dynamic pressures were not measured on account of their small values and therefore no comparison is available between measured and analytical results. It is however anticipated that the annular tank theory should not be expected to give satisfactory results for the dynamic impulsive pressures in a torus tank and should not be used for this purpose. If the pressures are required in the torus, the Finite Element Method described in the next chapter should be used.

2.6 Linearity Range

The range of linear behavior of sloshing response was tested experimentally by subjecting the model to increasing intensities of the simulated El Centro earthquake (horizontal component only), and measuring the maximum (upward) and minimum (downward) peak water surface displacements. The displacements associated with this ground motion are primarily in the first radial mode, and hence the following comments on linearity are primarily related to displacements in this mode. The results are plotted in Fig. 2-9 and are tabulated in Table 2-3, together with some tests which included the vertical component of ground motion.

For displacements less than 0.1 in. in the model, linearity holds within 1%. But as the amplitude of displacement increases, nonlinearity also increases and becomes approximately 10% at displacements in the order of 0.4 in. estimated on the basis of both upward and downward displacements. For practical purposes it may be assumed that the linear theory gives satisfactory results as long as the displacements are less than 0.5 in. in the model (or 30 in. in the prototype Mark 1 suppression pool). It may be seen in Fig. 2-6 that the modified annular tank theory (Equivalent Volume approximation) gives quite satisfactory results although the displacements are of the order of 0.5 in.

2.7 Summary of Important Observations on Sloshing in Torus Tank Model

1) The overall sloshing behavior of water in the torus tank model studied in this report was very similar to that in the annular tank described in Ref. [1].

2) The effect of vertical ground motion on the sloshing displacements is negligible in the linear range (compare Test No. 220378.2 with Test No. 220378.10 in Table 2-3) but becomes more significant in the nonlinear range (compare Test No. 220378.6 with Test No. 220378.13 in Table 2-3).

3) Modified annular tank theory gives satisfactory results for sloshing displacements in the torus tank. However this approximate theory cannot be applied for the determination of dynamic pressures.

4) A non-linearity of approximately 10% could be expected at displacements of the order of 0.4 in. in the model (or 24 in. in the prototype). Linear theory gives satisfactory results even under strong ground motions such as the El Centro earthquake of 1940.

Mode	Depth of		Natural Frequer	ncies (Hz)						
No.	Water		Theory							
	(in.)	Test	Annular Tank	Equivalent Volume						
1	3	0.45	0.49	0.44						
2	3	2.15	2.18	2.12						
3.	3	3.02	3.20	3.19						
4	3	3.95	3.93	3.92						
1	4	0.55	0.56	0.52						
2	. 4	2.37	2.34	2.31						
. 3	4	3.15	3.31	3.30						
4	4	4.15	4.04	4.04						
1	2	0.35	0.40	0.34						
2	2	2.00	2.13	1.91						
3	2	3.20	3.27	3.16						
4	2	3.92	4.04	4.00						

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TABLE 2-1. Natural sloshing frequencies of small scale model (1/60-scale model).

Frequency	Depth	Table	Displacement at Inner Wall (in.)						
(Hz)	(in.)	(g)	Test	Theory					
				(Equivalent Volume)	Annular Tank				
1.5	3.0	.0109	.069	.069	.0685				
1.8	3.0	.00392	.040						
1.8	3.0	.00785	.079						
1.8	3.0	.0118	.119						
1.8	3.0	.0157	.157	.157	.139				
1.8	3.0	.0196	.179						
1.9	3.0	.00438	.058	•					
1.9	3.0	.00875	.116						
1.9	3.0	.0131	.179						
1.9	3.0	.0175	.240	.240	.190				
1.9	3.0	.0218	.271						
2.4	3.0	.0242	.246	.205	.319				
1.50	2.5	.0118	.083	.078	.073				
1.50	2.5	.0233	.165						
1.80	2.5	.0163	.216	.210	.148				
1.80	2.5	.0326	.444						
1.80	2.5	.0245	.330						
1.50	3.5	.0118	.072		.074				
1.80	3.5	.0163	.150	.140	.132				
1.80	3.5	.0326	.310						
2.55	3.5	.0325	.252	.226	.275				

TABLE 2-2. Sloshing response of water in torus tank under sinusoidal ground acceleration (1/60th scale model) a = 14", b = 8".

	Peak Table Ac (g)	celeration	Peak Table Di (inche	splacement s)	Water Dis (in	splacement nches)
Test No	Horizontal	Vertical	Horizontal	Vertical	Maximum (up)	Minimum (down)
220378.2	0.115	0.0	0.048	0.0	0.246	0.231
220378.3	0.164	0.0	0.073	0.0	0.409	0.346
220378.4	0.237	0.0	0.100	0.0	0.614	0.458
220378.5	0.284	0.0	0.125	0.0	0.826	0.563
220378.6	0.338	0.0	0.149	0.0	0.999	0.667
220378.7	0.049	0.0	0.023	0.0	0.109	0.111
220378.8	0.0	0.083	0.0	0.05	0.003	0.006
220378.9	0.0	0.260	0.0	0.160	0.014	0.012
220378.10	0.107	0.067	0.046	0.036	0.249	0.245
220378.11	0.170	0.103	0.075	0.063	0.485	0.378
220378.12	0.232	0.163	0.100	0.098	0.758	0.552
220378.13	0.358	0.217	0.151	0.136	1.26	0.924
230378.1	0.477	0.0	0.201	0.0	1.22	0.921

TABLE 2-3. Extreme values in torus tank model tests under simulated El Centro 1940 earthquake (time scale (= $\sqrt{60}$ = 7.75, depth of water = 3 inches).



ELEVATION

XBL 789-2279

FIG.2-10 1/60 SCALE MODEL OF MARK I PRESSURE SUPPRESSION POOL



XBB 785-5492

FIG. 2-16 CONSTRUCTION OF 1/60 SCALE MODEL OF MARK I TORUS PRESSURE SUPPRESSION POOL



XBL 789-2280

FIG. 2-IC I/30 SCALE MODEL OF MARK I SUPPRESSION POOL TORUS WITH HEADERS AND DOWNCOMERS



CBB 779-8988 FIG.2-2 I/60 SCALE MODEL OF TORUS TANK ON SMALL SHAKING TABLE



XBB 785-5344 FIG.2-3 I/60 SCALE MODEL ON 20ft X 20ft SHAKING TABLE



RADIUS = 14 IN., DEPTH OF WATER IN A TOROS TANK (INNER RADIUS = 8 IN., OUTER RADIUS = 14 IN., DEPTH OF WATER = 3 IN.) UNDER SIMULATED EL CENTRO 1940 EARTHQUAKE, TIME SCALE = 7.7, TEST NO. 220378.2. PEAK SHAKING TABLE ACCEL-ERATION = 0.115g HORIZONTAL, 0.0g VERTICAL.

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XB1 789-11310



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XBL 789-11311



RADIUS = 14 IN., DEPTH OF WATER IN A TOROS TANK (INNER RADIUS - 8 IN., OUTER RADIUS = 14 IN., DEPTH OF WATER = 3 IN.) UNDER SIMULATED EL CENTRO 1940 EARTHQUAKE, TIME SCALE = 7.7, TEST NO. 220378.6. PEAK SHAKING TABLE ACCEL-ERATION = 0.338g HORIZONTAL, 0.0g VERTICAL.

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XBL 789-11312



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XBL 789-11313

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XBL 789-2276

3. FINITE ELEMENT ANALYSIS OF EARTHQUAKE INDUCED SLOSHING IN AXISYMMETRIC TANKS

3.1 Introduction

The finite element analysis [18] has become a powerful tool in solving complex engineering problems. Since the finite element method is completely general, it was decided that instead of looking for a closed form solution to predict the sloshing displacements and hydrodynamic pressures due to earthquake ground motions in a torus tank, the finite element method would be a better alternative in that it would be more general and thus can be applied to tank shapes other than cylindrical and toroidal.

Previous work on the finite element analysis of sloshing in tanks was done by Edwards [19] in which the shell theory was used for the prediction of seismic stresses and displacements in a cylindrical tank filled with liquid, but the sloshing was not considered in this analysis.

Finite element analysis for liquid sloshing problems by Luck [20] gives only the mode shapes and frequencies in an elastic container. His analysis is based on the variational principle suggested by Tong [21].

In this investigation our main concern is to study the sloshing effects in pressure-suppression pools of boiling water reactors, namely the Mark I torus and the Mark III annular suppression pools. Such structures can be considered as effectively rigid for the sloshing problem and thus the coupled effect of water-structure interaction is neglected in this analysis. Also the nonlinear sloshing problem has been linearalized [13] for this analysis.

The finite element equations were first derived for a completely general three dimensional problem and then were specialized to an axisymmetric tank subjected to arbitrary horizontal ground motions. The finite element equations were derived using the Galerkin principle [22]. More background information on Galerkin method may be found in References [18] and [23-26].

3.2 Mathematical Formulation:

3.2.1 Equation of motion

Consider a tank of arbitrary shape with rigid walls filled with a liquid whose free surface area is B2 as shown in Fig. 3-1. Bl represents the surface area of liquid in contact with the solid boundary of the container. V is the volume of the liquid and δ is the surface water displacement with respect to the undisturbed liquid surface. Using the same assumptions as in Ref. [1] Sec. 2.2, the velocity potential ϕ exists at every point in V and must satisfy the Laplace equation which in rectangular coordinates can be written as:

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} + \frac{\partial^2 \phi}{\partial z^2} = 0$$
 (3-1)

where $\phi = \phi$ (x, y, z, t).

Equation (3-1) will be solved by the finite element method subject to the appropriate time dependent boundary conditions as specified below. 3.2.2 Boundary conditions

Let $v_n(t)$ be the velocity of the tank wall along its outward normal to the boundary at any pount, then:

$$\frac{\partial \Phi}{\partial n} = v_n(t) \quad \text{on Bl}$$
 (3-2)

where n is the outward normal to the solid boundary and v_n is a function of time t.

It can also be shown that a liquid particle on the free surface B2 must satisfy the following two conditions [13]

$$\frac{\partial \phi}{\partial x} \frac{\partial \delta}{\partial x} - \frac{\partial \phi}{\partial z} + \frac{\partial \phi}{\partial y} \frac{\partial \delta}{\partial y} + \frac{\partial \delta}{\partial t} = 0$$
(3-3)

and

$$g\delta + \frac{\partial \phi}{\partial t} + \frac{1}{2} \left[\left(\frac{\partial \phi}{\partial x} \right)^2 + \left(\frac{\partial \phi}{\partial y} \right)^2 + \left(\frac{\partial \phi}{\partial z} \right)^2 \right] = 0 \qquad (3-4)$$

where g is the acceleration of gravity.

Equations (3-3) and (3-4) which represent the non-linear free surface boundary conditions can be simplified and combined into one boundary condition by neglecting higher order terms and eliminating δ . This single linearized boundary condition [13] can be written as follows.

$$\frac{1}{g} \frac{\partial^2 \phi}{\partial t^2} + \frac{\partial \phi}{\partial z} = 0 \quad \text{on B2.}$$
(3-5)
3.3 Finite Element Formulation

3.3.1 Derivation of finite element equations

In the finite element analysis, the continuum is divided into discrete elements or subregions which are interconnected at a finite number of points called nodes. The necessary formulation follows the Galerkin principle where we let the unknown field variable ϕ , throughout the solution domain, be approximated as

$$\phi = \sum_{j=1}^{N} N_{j}(x, y, z) \phi_{j}(t)$$
 (3-6)

in which N_j are the shape functions defined piecewise, element by element, and ϕ_j (t) are the time dependent nodal values of the field variable i.e., the velocity potential in this case. In the summation process an appropriate function for the particular point in space must be used. The N nodal values ϕ_j are obtained by solving a set of N simultaneous equations each derived by equating the boundary and interior residuals calculated by multiplying with a weighting function and integrating over the domain. In the Galerkin approach, the shape functions are taken as the weighting functions and for a typical node i substituting Eq. (3-6) into Eqs. (3-1), (3-2) and (3-5) and equating the weighted and integrated interior and boundary residual, we have:

$$\int_{V} N_{i} \left\{ \frac{\partial^{2}}{\partial x^{2}} + \frac{\partial^{2}}{\partial y^{2}} + \frac{\partial^{2}}{\partial z^{2}} \right\} \sum_{i}^{N} N_{j} \phi_{j} dv = \int_{B1} N_{i} \frac{\partial}{\partial n} \sum_{i}^{N} N_{j} \phi_{j} ds$$

$$-\int_{B1} N_{i} v_{n} ds + \int_{B2} \left\{ \frac{N_{i}}{g} \frac{\partial^{2}}{\partial t^{2}} \sum_{i}^{N} N_{j} \phi_{j} + N_{i} \frac{\partial}{\partial z} \sum_{i}^{N} N_{j} \phi_{j} \right\} ds$$

$$(3-7)$$

in which $\int dv$ and $\int ds$ represent the integrals over the volume and appropriate surfaces respectively. Consider the first term on the left hand side of Eq. (3-7) and write it in the following form

$$\int_{\mathbf{v}} N_{\mathbf{i}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial^{2} N_{\mathbf{j}}}{\partial \mathbf{x}^{2}} \phi_{\mathbf{j}} d\mathbf{v} = \int_{\mathbf{v}} \frac{\partial}{\partial \mathbf{x}} \left(N_{\mathbf{i}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial N_{\mathbf{j}}}{\partial \mathbf{x}} \right) \phi_{\mathbf{j}} d\mathbf{v} - \int_{\mathbf{v}} \frac{\partial N_{\mathbf{i}}}{\partial \mathbf{x}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial N_{\mathbf{j}}}{\partial \mathbf{x}} \phi_{\mathbf{j}} d\mathbf{v} .$$
(3-8)

Applying the Divergence theorem on the first integral on the right hand side of Eq. (3-8), we can rewrite it in the following form.

$$\int_{\mathbf{v}} \mathbf{N}_{\mathbf{i}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial^{2} \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{x}^{2}} \phi_{\mathbf{j}} \, d\mathbf{v} = \int_{\mathbf{B}} \mathbf{N}_{\mathbf{i}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{x}} \ell_{\mathbf{x}} \phi_{\mathbf{j}} \, d\mathbf{s} - \int_{\mathbf{v}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{i}}}{\partial \mathbf{x}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{x}} \phi_{\mathbf{j}} \, d\mathbf{v}$$
(3-9)

in which B = BI + B2 and ℓ_x is the direction cosine in the x-direction of the outward normal n. Similar expression can be written for the second and third terms in Eq. (3-7). Substituting Eq. (3-9) and similar expressions into the right hand side of Eq. (3-7), we get

$$\begin{split} &\int_{B} N_{i} \left(\sum_{1}^{N} \frac{\partial N_{j}}{\partial x} \, \lambda_{x} \phi_{j} + \sum_{1}^{N} \frac{\partial N_{j}}{\partial y} \, \lambda_{y} \phi_{j} + \sum_{1}^{N} \frac{\partial N_{j}}{\partial z} \, \lambda_{z} \phi_{j} \right) \, ds \\ &- \int_{V} \left(\frac{\partial N_{i}}{\partial x} \sum_{1}^{N} \frac{\partial N_{j}}{\partial x} \, \phi_{j} + \frac{\partial N_{i}}{\partial y} \sum_{1}^{N} \frac{\partial N_{j}}{\partial y} \, \phi_{j} + \frac{\partial N_{i}}{\partial z} \sum_{1}^{N} \frac{\partial N_{j}}{\partial z} \, \phi_{j} \right) \, dv \qquad (3-10) \\ &= \int_{B1} N_{i} \sum_{1}^{N} \frac{\partial N_{j}}{\partial n} \, \phi_{j} \, ds - \int_{B1} N_{i} v_{n} \, ds + \frac{1}{g} \int_{B2} N_{i} \sum_{1}^{N} N_{j} \frac{\partial N_{j}}{\partial s} \, ds + \int_{B2} N_{i} \sum_{1}^{N} \frac{\partial N_{j}}{\partial z} \phi_{j} \, ds \\ & \text{in which } \ddot{\phi} = d^{2} \phi / dt^{2}. \end{split}$$

The boundary integral on the left hand side of Eq. (3-10) can be substituted by

$$\int_{B} N_{i} \sum_{l}^{N} \frac{\partial N_{j}}{\partial n} \phi_{j} ds \qquad (3-11)$$

or by

$$\int_{B1} N_{i} \sum_{l}^{N} \frac{\partial N_{j}}{\partial n} \phi_{j} ds + \int_{B2} N_{i} \sum_{l}^{N} \frac{\partial N_{j}}{\partial n} \phi_{j} ds. \qquad (3-12)$$

Replacing the boundary integral of left hand side of Eq. (3-10) with Eq. (3-12) and using the approximation (small slopes)

$$\frac{\partial N_i}{\partial z} \approx \frac{\partial N_i}{\partial n}$$
 on B2, Eq. (3-10)

can be simplified to the following form.

$$\int_{\mathbf{v}} \left[\frac{\partial \mathbf{N}_{\mathbf{i}}}{\partial \mathbf{x}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{x}} \phi_{\mathbf{j}} + \frac{\partial \mathbf{N}_{\mathbf{i}}}{\partial \mathbf{y}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{y}} \phi_{\mathbf{j}} + \frac{\partial \mathbf{N}_{\mathbf{i}}}{\partial \mathbf{z}} \sum_{\mathbf{l}}^{\mathbf{N}} \frac{\partial \mathbf{N}_{\mathbf{j}}}{\partial \mathbf{z}} \phi_{\mathbf{j}} \right] d\mathbf{v}$$
(3-13)

$$+ \frac{1}{g} \int_{B2} N_{i} \sum_{l}^{N} N_{j} \ddot{\phi}_{j} ds = \int_{B1} N_{i} v_{n} ds$$

or

$$\frac{M}{\Phi} + \frac{K}{\Phi} = \underline{F}$$
 (3-14)

in which the elements of \underline{M} , \underline{K} and \underline{F} are given by

$$M_{ij} = \frac{1}{g} \sum_{EB2} \int_{EB2} N_{i} N_{j} ds \qquad (3-15)$$

$$\kappa_{ij} = \sum \int_{EV} \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} + \frac{\partial N_i}{\partial z} \frac{\partial N_j}{\partial z} \right) dv$$
(3-16)

$$F_{i} = \sum \int_{EB1} N_{i} v_{n} ds \qquad (3-17)$$

where summation for M_{ij} covers only the elements on the free surface boundary and the integral is carried out on the free surface of each element EB2. Summation for K_{ij} covers the contribution of each element and EV is the element region. EBl refers only to the elements which lie on the solid boundary, Bl, and the loading term thus is associated with the elements that lie on the tank wall boundary.

The free surface matrix \underline{M} and the fluid matrix \underline{K} are comparable to the mass and stiffness matrices respectively used in structural dynamics. It is interesting to note that the free surface matrix \underline{M} gets the contribution only from the free surface elements. \underline{M} and \underline{K} are symmetric matrices and Eq. (3-14) is a set of second order linear differential equations which can be solved either by direct integration or by mode superposition.

3.3.2 <u>Isoparametric formulation for axisymmetric tank under arbitrary</u> horizontal ground motions

The following analysis will be restricted to rigid tanks which are symmetrical about the z - axis and are subjected to arbitrary horizontal ground motions alone. Since Eq. (3-13) involves only the first derivatives of shape functions, a 4-node quadrilateral element with linear interpolation functions will satisfy the convergence requirements. However, the more recently developed 4-to-8 variable node isoparametric element [27] has greater flexibility in accommodating the curved boundaries and is convenient for numerical integration. Therefore a variable 4-to-8 node, 2-dimensional isoparametric element will be used in the present formulation. Such an element can be used to model an axisymmetric problem or a two dimensional problem such as sloshing in a rectangular tank.

3-7

Figure 3-2(a) shows such a 4-to-8 variable node element lying in the x-z plane where z is the axis of symmetry. Any of the mid-side nodes 5 through 8 may or may not be present and can be eliminated if desired. Such a curvilinear 4-to-8 node element can be obtained by using an isoparametric mapping from a bi-unit square which has a local r-s coordinate system as shown in Fig. 3-2(b). The local coordinates r and s vary between -1 and +1. The nodes 1 through 4 are the corner nodes and the nodes 5 through 8 are the mid-side nodes corresponding to Fig. 3-2(a). Node 1 has the coordinates (1,1), The mapping between the local coordinate system (r,s) and the global (x,z) coordinate system must be unique in order to carry out the transformations properly. The coordinate transformation between the bi-unit square and the curvilinear element is given by;

$$x_{m}(r,s) = \sum_{i=1}^{8} h_{i}(r,s) x_{im}$$
 (3-18)
 $z_{m}(r,s) = \sum_{i=1}^{8} h_{i}(r,s) z_{im}$ (3-19)

in which (x_{im}, z_{im}) are the global coordinates of node i in element m and h_i are the interpolation functions in local coordinates corresponding to node m. The interpolation functions h_i for any element m are defined as follows:

$$h_{1} = \frac{1}{4} (1+r) (1+s) - \frac{1}{2} h_{5} - \frac{1}{2} h_{8}$$

$$h_{2} = \frac{1}{4} (1-r) (1+s) - \frac{1}{2} h_{5} - \frac{1}{2} h_{6}$$

$$h_{3} = \frac{1}{4} (1-r) (1-s) - \frac{1}{2} h_{6} - \frac{1}{2} h_{7}$$

$$h_{4} = \frac{1}{4} (1+r) (1-s) - \frac{1}{2} h_{7} - \frac{1}{2} h_{8}$$

$$h_{5} = \frac{1}{2} (1-r^{2}) (1+s)$$

$$h_{6} = \frac{1}{2} (1-r) (1-s^{2})$$

$$h_{7} = \frac{1}{2} (1-r^{2}) (1-s)$$

$$h_{8} = \frac{1}{2} (1+r) (1-s^{2}).$$
(3-20)

Since a horizontal ground motion will excite only the antisymmetric modes (in the linearized case) of sloshing in an axisymmetric tank, we can approximate the distribution of the velocity potential within an element m in terms of the velocity potential at node 1 through 8 and the interpolation functions given by Eq. (3-20)

$$\phi_{m}(\mathbf{r},\mathbf{s},\theta,\mathbf{t}) = \sum_{n=1}^{\infty} \sum_{i=1}^{8} h_{i}(\mathbf{r},\mathbf{s}) \cos \theta \phi_{im}(\mathbf{t}) \qquad (3-21)$$

in which $\varphi_{\mbox{im}}$ is the value of the velocity potential at node i of element m.

It is obvious that if this shape function given by Eq. (3-21) is used in Eq. (3-17) to calculate the loading vector, the integral between the limits 0 and 2π will be non-zero only when n = 1 in the case of horizontal ground motion only, because v_n in such a case varies as a function of $\cos\theta$ and the $\int_0^{2\pi} \cos\theta \cdot \cos\theta \, d\theta = 0$ for any $n \neq 1$. Therefore Eq. (3-21) can be written as

$$\phi_{m}(\mathbf{r},\mathbf{s},\theta,\mathbf{t}) = \sum_{i=1}^{8} h_{i}(\mathbf{r},\mathbf{s}) \cos\theta \cdot \phi_{im}(\mathbf{t}) \qquad (3-22)$$

anđ

$$\frac{\partial}{\partial r} \phi_{m} (r,s,\theta,t) \sum_{i=1}^{8} \frac{\partial}{\partial r} h_{i} \cos\theta \cdot \phi_{im}(t) \qquad (3-23)$$

$$\frac{\partial}{\partial s} \phi_{m} (r, s, \theta, t) = \sum_{i=1}^{8} \frac{\partial}{\partial s} h_{i} \cos \theta \cdot \phi_{im}(t) \qquad (3-24)$$

or in matrix form, the above two equations can be written as

$$\begin{pmatrix} \frac{\partial \phi}{\partial \mathbf{r}} \\ \frac{\partial \phi}{\partial \mathbf{s}} \\ \frac{\partial \phi}{\partial \mathbf{s}} \\ m \end{pmatrix}^{\mathbf{m}} = \cos\theta \left[\begin{array}{c} \frac{\mathbf{p}_{\mathbf{m}}}{2 \times 8} & (\mathbf{r}, \mathbf{s}) \\ \frac{\partial \phi}{2 \times 8} \\ \frac{\partial \phi}{8 \times 1} \\ \end{array} \right] \frac{\phi_{\mathbf{m}}(\mathbf{t})}{8 \times 1}$$
(3-25)

in which $\underline{p}_{\underline{m}}(r,s)$ contains the derivatives of interpolation functions derived from Eq. (3-20). Using the chain rule of differentiation, we can relate the global derivatives to the local derivatives.

$$\begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{r}} \\ \frac{\partial \phi}{\partial \mathbf{s}} \\ \frac{\partial \phi}{\partial \mathbf{s}} \end{bmatrix}_{\mathbf{m}} = \begin{bmatrix} \frac{\partial \mathbf{x}}{\partial \mathbf{r}} & \frac{\partial \mathbf{z}}{\partial \mathbf{r}} \\ \frac{\partial \mathbf{x}}{\partial \mathbf{s}} & \frac{\partial \mathbf{z}}{\partial \mathbf{s}} \end{bmatrix}_{\mathbf{m}} \begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{x}} \\ \frac{\partial \phi}{\partial \mathbf{z}} \\ \frac{\partial \phi}{\partial \mathbf{z}} \end{bmatrix}_{\mathbf{m}}$$
(3-26)

Jacobian matrix

by inverting

$$\begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{x}} \\ \\ \frac{\partial \phi}{\partial z} \end{bmatrix}_{\mathbf{m}} = \mathbf{J}_{\mathbf{m}}^{-1} \begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{r}} \\ \\ \\ \frac{\partial \phi}{\partial s} \end{bmatrix}_{\mathbf{m}}$$
(3-27)

in which $J_{\underline{m}}^{-1}$ is the inverse of the Jacobian matrix in Eq. (3-26). Substituting for $\partial\phi/\partial r$ and for $\partial\phi$ ∂s from Eq. (3-25) into Eq. (3-27), we obtain the relationship between the global derivatives and nodal values of ϕ .

$$\begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{x}} \\ \frac{\partial \phi}{\partial z} \end{bmatrix}_{\mathbf{m}} = \frac{\mathbf{J}^{-1}}{\frac{\mathbf{m}}{2 \times 2} \times \frac{\mathbf{p}}{2 \times 8}} (\mathbf{r}, \mathbf{s}) \frac{\phi_{\mathbf{m}}(t) \cdot \cos\theta}{8 \times 1}$$
(3-28)

or

$$\begin{bmatrix} \frac{\partial \phi}{\partial \mathbf{x}} \\ \\ \frac{\partial \phi}{\partial z} \end{bmatrix}_{\mathbf{m}} = \frac{\mathbf{B}}{\mathbf{m}} \frac{\phi_{\mathbf{m}}(t) \cos\theta}{(3-29)}$$

in which

 $\underline{\mathbf{B}}_{\underline{\mathbf{m}}} = \underline{\mathbf{J}}_{\underline{\mathbf{m}}}^{-1} \underline{\mathbf{p}}_{\underline{\mathbf{m}}} .$

(3-30)

Thus

$$\begin{pmatrix} \frac{\partial N}{\partial x} \\ \\ \frac{\partial N}{\partial y} \end{pmatrix}_{m} = \frac{B_{m}}{m} \cos\theta \qquad (3-31)$$

3.3.3 Free surface (mass) matrix for axisymmetric element

The complete free surface matrix for the system is formed by direct summation of individual element matrices i.e.,

$$\underline{M} = \frac{1}{g} \sum_{m=1}^{n} \underline{M}_{m}$$
(3-32)

where n is the total number of free surface elements, and the element matrix \underline{M}_{m} is a 2 × 2 matrix given by

$$M_{m} = \int_{EB2} \frac{N_{m}^{T}N}{m-m} ds. \qquad (3-33)$$

In case of axisymmetric case the surface integral can be transformed to a line integral. Consider a free surface element with nodes i and j (Fig. 3-3) and let

r = local coordinates for free surface boundary element
R(r) = radius to any point r between node i and j
L = length of the elements
L =
$$\sqrt{(x_i - x_j)^2 + (z_i - z_j)^2}$$
. (3-34)

Assuming that x-axis coincides with the horizontal plane, we can write the transformation.

$$R(\mathbf{r}) = \left[(1 - \mathbf{r}/L) \quad \mathbf{r}/L \right] \left\{ \begin{array}{c} \mathbf{x}_{i} \\ \mathbf{x}_{j} \end{array} \right\}.$$
(3-35)

$$ds = R(r) dr d\theta \qquad (3-36)$$

where θ is defined in Fig. 3-2

ds =
$$\begin{bmatrix} 1 - r/L \\ r/L \end{bmatrix} \begin{pmatrix} x_i \\ x_j \end{pmatrix} dr d\theta$$
. (3-38)

Thus

$$\underline{\mathbf{N}} = \begin{bmatrix} \mathbf{1} - \mathbf{r}/\mathbf{L} & \mathbf{r}/\mathbf{L} \end{bmatrix} \cos\theta.$$
 (3-39)

Therefore

$$\underline{M}_{m} = \int_{0}^{2\pi} \int_{0}^{L} \left\{ \begin{array}{c} 1 - r/L \\ r/L \end{array} \right\} \left[(1 - r/L) & r/L \right] R(r) \cos^{2}\theta \cdot drd\theta.$$
(3-40)

$$\underline{M}_{m} = \pi \int_{0}^{L} \left\{ \begin{array}{c} 1 - r/L \\ r/L \end{array} \right\} \left[(1 - r/L) & r/L \right] R(r) \, dr d\theta.$$
(3-41)

Evaluation of the above matrix gives

$$M_{ii} = \frac{\pi L}{4} \left[x_i + \frac{x_j}{3} \right]$$
(3-42)

$$M_{jj} = \frac{\pi L}{4} \left[\frac{x_j}{3} + x_j \right]$$
(3-43)

$$M_{ij} = \frac{\pi L}{12} \left[x_i + x_j \right]$$
(3-44)

3.3.4 Evaluation of fluid (stiffness) matrix for axisymmetric element

The complete system fluid matrix is formed by direct summation of element matrices

$$\underline{K} = \sum_{m=1}^{n} \underline{K}_{m}$$
(3-45)

where n is the number of liquid elements and the element matrix $\frac{K}{m}$ is obtained by using Eq. (3-31) and Eq. (3-16).

$$\underline{K}_{m} = \int_{EV} \cos^{2}\theta \ \underline{B}_{m}^{T} \ \underline{B}_{m} \ dv \qquad (3-46)$$

In case of an axisymmetric element, dv = R $d\theta dr,$ the volume integral then becomes

$$\underline{\mathbf{K}}_{\mathbf{m}} = \int_{0}^{2\pi} \int_{\mathbf{A}_{\mathbf{m}}} \cos^{2}\theta \quad \mathbf{R} \; \underline{\mathbf{B}}_{\mathbf{m}}^{\mathrm{T}} \; \underline{\mathbf{B}}_{\mathbf{m}} \, \mathrm{d}\mathrm{A}\mathrm{d}\theta \qquad (3-47)$$

where R is the radius of any point and A is the area.

In the natural coordinate system $dA = \begin{vmatrix} J_m \end{vmatrix} dsdr$ where $\begin{vmatrix} J_m \end{vmatrix}$ is the determinant of the Jacobian matrix, therefore

$$\underline{K}_{m} = \pi \int_{-1}^{1} \int_{-1}^{1} \mathbb{R} \underline{B}_{m}^{T} \underline{B}_{m} |J_{m}| dsdr. \qquad (3-48)$$

The above integral can be evaluated numerically using Gaussian quadrature as follows [18]

$$K_{m} = \sum_{j=1}^{N} \sum_{i=1}^{N} H_{i} H_{j} f(r_{i}, s_{i})$$
(3-49)

where N refers to the order of integration; H and H are the weighting factors and

$$\underline{f}(\mathbf{r}_{i},\mathbf{s}_{i}) = \underline{B}_{m}^{T}(\mathbf{r}_{i},\mathbf{s}_{i}) \underline{B}_{m}(\mathbf{r}_{i},\mathbf{s}_{i}) | \mathbf{J}_{m}(\mathbf{r}_{i},\mathbf{s}_{i}) | \mathbf{R}(\mathbf{r}_{i},\mathbf{s}_{i})$$
(3-49)

3.3.5 Load vector for axisymmetric element

The loading vector \underline{F} for the complete system is the sum of the contribution of individual elements and using Eq. (3-17) we can write

$$\underline{\mathbf{F}} = \sum_{m=1}^{n} \int_{\text{EB1}} N_{m}^{\text{T}} \mathbf{v}_{n} \, d\mathbf{s} = \sum_{m=1}^{n} \underline{\mathbf{F}}_{m}$$
(3-50)

where

$$\underline{F}_{m} = \int_{EB1} N_{m}^{T} v_{n} ds \qquad (3-51)$$

where summation is over all the n elements which are at the liquid-solid interface.

Consider a typical liquid-solid boundary element with nodes i and j and let r be a local coordinate as shown in Fig. 3-4. Let R(r) be the radius at any point r and L be the length of the element where

$$L = \sqrt{(x_{i} - x_{j})^{2} + (z_{i} - z_{j})^{2}}.$$

$$R(\mathbf{r}) = \left[(1-\mathbf{r}/\mathbf{L}) \quad \mathbf{r}/\mathbf{L} \right] \left\{ \begin{array}{c} \mathbf{x}_{i} \\ \mathbf{x}_{j} \end{array} \right\}.$$
(3-52)

$$\underline{N} = \left[(1-r/L) \quad r/L \right] \cos\theta \qquad (3-53)$$

ds = $R(r) dr d\theta$.

If v_x = horizontal ground velocity in the x-direction then $v_n = v_x \cos\theta \cdot \cos\psi$ where $\psi = \tan^{-1} \left(\frac{x_i - x_j}{z_i - z_j}\right)$; then $\frac{F}{m} = v_x \int_0^L \int_0^{2\pi} \left(\frac{(1-r/L)}{r/L}\right) \left[(1-r/L) r/L\right] \left(\begin{array}{c} x_i \\ x_j \end{array}\right) \cos^2\theta \cdot \cos\psi \, d\theta \, dr$ (3-54)

integrating we get the following element load vector

$$\underline{\mathbf{F}}_{\mathbf{m}} = \frac{\pi \mathbf{L} \mathbf{v}_{\mathbf{x}} \cos \psi}{6} \left\{ \begin{array}{c} 2\mathbf{x}_{\mathbf{i}} + \mathbf{x}_{\mathbf{j}} \\ \mathbf{x}_{\mathbf{i}} + 2\mathbf{x}_{\mathbf{j}} \end{array} \right\} = \left\{ \begin{array}{c} \mathbf{F}_{\mathbf{i}} \\ \mathbf{F}_{\mathbf{j}} \end{array} \right\}.$$
(3-55)

The contribution to the loading matrix comes from those elements which lie at the liquid-solid interface.

3.4 Numerical Solution of Finite Element Equations

The discretization of the continuum into finite elements and the assemblage of free surface, liquid and loading element matrices results in a set of linear, coupled, second order ordinary differential equations. Since these equations are linear they can be uncoupled by an orthogonal transformation and the solution can be obtained using mode superposition [28, 29] or they can be solved by direct step-by-step integration. In this study Newmark's step by step integration method [30,29] which is based on the following expressions was used.

$$\dot{\phi}_{t+\Delta t} = \dot{\phi}_{t} + \Delta t \quad (1-\delta) \quad \dot{\phi}_{t} + \Delta t \quad \delta \phi_{t+\Delta t}$$
(3-56)

$$\phi_{t+\Delta t} = \phi_t + \Delta t \dot{\phi}_t + \Delta t^2 (\frac{1}{2} - \alpha) \dot{\phi}_t + \Delta t^2 \alpha \dot{\phi}_{t+\Delta t}$$
(3-57)

in which Δt is step size. α and δ are parameters which are selected to produce the desired stability and accuracy. In all the sample analyses carried out in this investigation, Newmark's constant-average-acceleration method ($\delta = 1/2$ and $\alpha = 1/4$) was used, which is an unconditionally stable method without numerical damping.

This is an implicit method and satisfies the equilibrium equations at time t + Δ t, i.e.,

$$\underline{\mathbf{M}} \quad \underline{\boldsymbol{\phi}}_{\mathbf{t}+\Delta \mathbf{t}} + \underline{\mathbf{K}} \quad \underline{\boldsymbol{\phi}}_{\mathbf{t}+\Delta \mathbf{t}} = \underline{\mathbf{F}}_{\mathbf{t}+\Delta \mathbf{t}} \tag{3-58}$$

The above three equations can be combined into a step-by-step algorithm which involves the solution of a set of equations at each time step of the form.

$$\underline{K}^{*} \underline{\phi}_{t+\Delta t} = \underline{F}^{*}$$
(3-59)

In this analysis \underline{K}^{\star} is independent of time and is formed and triangularized only once. To make the numerical algorithm more general, the option of combining the Wilson Theta method [31] with the Newmark method was incorporated in the computer program. The Wilson Theta method was first applied to Newmark's linear accelerator method in order to improve stability and to damp out high frequency oscillations which often develop in step by step integration. A summary of the Newmark-Wilson algorithm used in the computer program is given below.

3.4.1 The Newmark-Wilson algorithm for linear step-by-step

integration

INITIAL CALCULATIONS

- 1. Initialize ϕ_0 and $\dot{\phi}_0$ (taken to be zero) (ϕ is a vector)
- 2. Form the free surface and fluid matrices $(\underline{M}, \underline{K})$

3. Specify algorithm parameters $\alpha,~\delta$ and θ

4. Calculate integration constants.

$$\tau = \theta \Delta t \qquad a_3 = \frac{1}{2\alpha} - 1 \qquad a_7 = \Delta t \delta$$

$$a_0 = \frac{1}{\alpha \tau^2} \qquad a_4 = \frac{\delta}{\alpha} - 1 \qquad a_8 = \Delta t^2 (\frac{1}{2} - \alpha)$$

$$a_1 = \frac{\delta}{\alpha \tau} \qquad a_5 = \frac{\tau}{2} (\delta/\alpha - 2) \qquad a_9 = \alpha \Delta t^2$$

$$a_2 = \frac{1}{\alpha \tau} \qquad a_6 = \Delta t (1 - \delta)$$

5. Form
$$\underline{K}^* = \underline{K} + \underline{a}_0 \underline{M}$$

6. Triangularize $\underline{K}^* : \underline{K}^* = \underline{LDL}^T$

FOR EACH TIME STEP

1. Calculate the effective load vector \underline{F}^{\star} at time t+T

$$\underline{F}^{*} = \underline{F}_{t+\tau} + \underline{M} (a_0 \phi_t + a_2 \dot{\phi} + a_3 \phi_t).$$

2. Solve for velocity potential ϕ at t + T:

$$\underline{\text{LDL}}^{T} \phi_{t+\tau} = F^{*}$$

3. Calculate the velocity potential ϕ and its derivatives at time $t + \Delta t$: $\begin{array}{c} & & & & & \\ & & & & \\ & & & & \\ & & &$

$$\phi_{t+\Delta t} = \phi_t + \Delta t \dot{\phi}_t + a_8 \phi_t + a_9 \phi_{t+\Delta t}$$

4. Determine the sloshing displacements and hydrodynamic impulsive pressures at time t + Δt Sloshing displacement $\delta_{t+\Delta t} = -\frac{1}{g}\dot{\phi}_{t+\Delta t}$ Impulsive pressure $p_{t+\Delta t} = -\rho \dot{\phi}_{t+\Delta t}$

3.5 Computer Program 'SLOSH2'

The program 'SLOSH2' developed to implement the finite element theory of the sloshing phenomenon in axisymmetric tanks is coded in standard FORTRAN IV language. The basic set up of SLOSH2 is the same as that of the computer program DOT [32] because of the fact that the finite element formulation of the sloshing problem is similar in certain respects to that of the heat conduction equations.

The earthquake input can either be as an accelerogram or a displacement-time history, digitized in the appropriate format. The program derives the velocity-time history by integration or differentiation depending upon the type of ground motion input. The earthquake input must be properly adjusted for base-line correction such that at the end of earthquake as acceleration goes to zero, the ground velocity and displacement also go to zero.

The 'effective' equilibrium equations (Eq. (3-59) are solved using the linear equation solver COLSOL [33]. This subroutine processes only those elements which are within the skyline of \underline{K}^* , thus minimizing the storage requirements as well as the number of operations. This subroutine is based on Gauss elimination and requires a symmetrical positive-definite system of equations. A compact storage scheme is used in the computer program whereby a one-dimensional array is used to store only those elements of K^* which are within its skyline. In the actual implementation of the computer program a lumped mass parameter system was used which not only simplifies the analysis considerably, but also minimizes the storage requirements. The three finite element groups namely, the free surface elements, the fluid elements and the liquid-solid interface elements, are processed in blocks and then stored on the disc for later use in order to increase the maximum capacity of the program.

It should be noted that the free surface elements and the liquidsolid interface elements which contribute to \underline{M} and \underline{F} respectively, are only two node elements for the axisymmetric case, whereas the liquid continuum itself is discretized by two-dimensional 4-to-8 node isoparametric elements which contribute to the fluid matrix K.

In this computer program, a variable dimension is used for dynamic allocation of primary storage into a single array in blank common. The lower primary storage locations are used for the storage of each block of element group data which is read in from the secondary storage (disc) as required during the solution phase. The user has to supply the maximum estimated number of storage locations required to store any individual element group in the lowest primary storage. Usually this number is determined by the number of liquid elements that form the \underline{K} matrix and not the free surface or liquid-solid interface elements. An estimation of the C.P.U. time required to run the program on CDC 6400 will be indicated in the next article. A user's manual and Fortran listing for SLOSH2 are given in Appendix Al and A2 respectively.

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3.6 <u>Sample Analyses and Comparison with Test Data from Annular and</u> Torus Tanks

Figures 3-5 and 3-6 show the finite element mesh layout for the 8 ft diameter annular tank and the 28 inches diameter torus tank models respectively which were studied in Ref. [1] and in Chapter 2 of this report. The annular tank chosen for this analysis is a simplified 1/15th scale model of pressure-suppression pool of Boiling Water Reactor GE Mark III (see Ref. [1] Figs. 5-3 and 5-7a) whereas the torus tank represents the 1/60th scale model (see Fig. 2-1) of the GE Mark I pressuresuppression pool. These analyses were carried out to check the accuracy of the finite element model against precise test data. A finer mesh size has been used near the free water surfaces because that is where the maximum sloshing displacements occur.

3.6.1 Annular tank

In Fig. 3-5 the x-axis is taken at the bottom of the tank and z-axis as the axis of symmetry of the annular tank. There are a total of 25, 10 and 17 elements in group Nos. 1, 2 and 3 respectively. Group No.1 contains 4-to-8 node elements whereas group Nos. 2 and 3 contain the free surface elements and liquid-solid interface elements respectively with two nodes each. There are a total of 52 nodes in this mesh. The finite element analysis was done using the digitized accelerogram recorded in Test No. 211276.1 of Ref. [1] Chapter 5 and a comparison of measured and predicted results is given in Figs. 3-7 and 3-8.

Figure 3-7 shows a comparison of the sloshing displacements at node 2 between the finite element solution and the test data under the

3-20

same ground motion. The results are shown for the first six seconds and it can be seen that there is close agreement between the test and finite element results.

Figure 3-8 shows a comparison of impulsive pressures between the measured and finite element results, and it can be seen again that the agreement between the two is excellent with test results consistently 5-10% higher compared with the finite element solution indicating a possible error of calibration of pressure gage and some error due to electrical noise. This comparison is given at node 52 (Fig. 3.5).

3.6.2 Torus tank

The finite element mesh layout for the torus tank model (Fig. 3-6) has a total of 24, 12 and 24 elements in group Nos. 1, 2 and 3 respectively. The z-axis is taken as the axis of symmetry of the torus and the x-axis at the bottom of the tank. The total number of nodes in this case is 73.

The earthquake input used for the finite element analysis was the recorded shaking table displacement instead of acceleration because the displacements gave zero velocity and acceleration on differentiation at the end of the earthquakes whereas the integration of the accelerogram usually gives finite amount of velocity and displacement without a base line correction.

In comparing the finite element solution with the test data for the torus tank it should be remembered that the two systems are not exactly the same: the finite element solution is for a perfectly round tank, whereas the test tank was made of a set of 16 straight segments as shown in Fig. 2-1. Figures 3-9, 3-10 and 3-11 show the comparison of sloshing displacements at node No.2 for increasing intensity of ground motion. In Figs. 3-9 and 3-10 the agreement between the two results during the first half of the earthquake is excellent and for the rest of the earthquake quite satisfactory. In Fig. 3-11, although the sloshing response is relatively large, the linear finite element analysis still gives satisfactory results for practical purposes. The hydrodynamic pressures in this case were not measured on account of their relatively small magnitude in this small scale model.

The C.P.U. time required by the CDC 6400 to obtain the time-history response in the torus tank model analysis (Fig. 3-9) with the mesh size shown in Fig. 3-6 was 126 seconds.

3.7 <u>Sample Analysis of Mark I Prototype Torus Tank Under El Centro</u> 1940 Earthquake

Figure 3-12 shows the sloshing response in the prototype of Mark I torus under the full intensity of the 1940 El Centro earthquake (N-S component). The mesh layout for this analysis was similar to that shown in Fig. 3-6 except the overall dimensions which in this case are 60 times larger. The sloshing displacement shown in Fig. 3-12 is at node #2 with a maximum value of 24.3".







(a) TWO-DIMENSIONAL ELEMENT IN GLOBAL X-Z SYSTEM



(b) BI-UNIT SQUARE IN LOCAL r-S SYSTEM FIG.3-2 TWO-DIMENSIONAL MAPPING OF AN ISOPARAMETRIC ELEMENT







FIG.3-4 BOUNDARY ELEMENT AT LIQUID-SOLID INTERFACE (AXISYMMETRIC CASE)







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FIG. 3-7 COMPARISON OF SLOSHING DISPLACEMENTS BETWEEN THE TEST AND FINITE ELEMENT RESULTS AT NODE #2 (TEST #211276.1) IN ANNULAR TANK MODEL (INNER RADIUS = 33.2 IN., OUTER RADIUS = 48.0 IN., DEPTH OF WATER = 16 IN.) UNDER SIMULATED EL CENTRO 1940 EARTHQUAKE, TIME SCALE = $\sqrt{15}$ = 3.9, PEAK SHAKING TABLE ACCELERATION = 0.24g HORIZONTAL, 0.0g VERTICAL.



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FIG. 3-8 COMPARISON OF IMPULSIVE PRESSURES BETWEEN THE TEST AND FINITE ELEMENT RESULTS AT NODE #52 (TEST #211276.1) IN ANNULAR TANK MODEL.

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FIG. 3-9 COMPARISON OF SLOSHING DISPLACEMENTS BETWEEN THE TEST AND FINITE ELEMENT RESULTS AT NODE #2 (TEST #220378.2) IN TORUS TANK MODEL (INNER RADIUS = 8 IN., OUTER RADIUS = 14 IN., DEPTH OF WATER = 3 IN.)

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4. CONCLUSIONS

(1) A comparison of data from shaking table tests on annular and torus tanks confirms that the finite element analysis presented in this report can successfully predict hydrodynamic pressures and free surface displacement in rigid axisymmetric tanks under strong motion earthquakes. The finite element program is applicable not only to axisymmetric tanks but may also have possible application to offshore structures under seismic conditions.

(2) As sloshing response is not very sensitive to the precise geometry of the tank section, a modified annular tank solution gives satisfactory results in predicting the sloshing frequencies and displacements in torus tanks under horizontal ground motions.

(3) The validity of the theory developed herein is independent of the size of the model that was analyzed and tested. In the 1/60-scale torus model the sloshing response is produced mainly by the low frequency components of the reference earthquake ground motion, and these are correctly reproduced on the 20-foot shaking table.

ACKNOWLEDGMENTS

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APPENDIX A1

SLOSH2 USER'S MANUAL

SLOSH2: A linear finite element program which determines the sloshing displacements and impulsive pressures in axisymmetric rigid tanks under arbitrary horizontal ground motions.

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- Al.l Program Description
- Al.2 Program Capacity
- Al.3 Input Data
 - I. Problem Initiation and Title
 - II. Master Control Card
 - III. Nodal Coordinates
 - IV. Solution Time and Step Size
 - V. Earthquake Input
 - VI. Element Data

Type 1: Two Dimensional Finite Elements Type 2: Free Surface Elements Type 3: Solid-Liquid Interface Elements

- VII. New Problem Data
- VIII. Termination Card

Al.4 C

Output

Al.l Program Description

The computer program SLOSH2 has been developed to predict sloshing displacements and impulsive pressures in a liquid filled axisymmetric container subjected to only horizontal ground motion. The tank is assumed to be rigid and fixed at the base. The program requires the following three types of elements.

> Two dimension 4-to-8 node axisymmetric elements idealizing the liquid.

2. Two node free surface elements.

3. Two node elements representing the liquid-solid interface.

Al.2 Program Capacity

The program uses a variable dimensioning in order to make an optimum use of high speed storage. Element group data is stored block wise on the disc. The program capacity can be varied through two Fortran statements in the main program.

```
COMMON (n)
MTOT = n
```

The total memory n required can be estimated by the following formula.

n = M + 2 * NPTM + 10 * NUMNP

in which

M = NELl * (4 * MXNODS - 2)
NELl = Number of elements in group 1
MXNODS = Maximum nodes in any element of group 1
NPTM = Number of points of earthquake input
NUMNP = Total number of node points

Al-2

Al.3 Program Input Data

The following format should be followed for the necessary input data.

I. Problem Initiation and Title (A5, 3X, 18A4) - one card

Columns	Variable	Description
1 - 5	MODE	Punch the word 'START'
9 - 80	HED	Title of the problem

II. Master Control Card (415) - one card

Columns	Variable	Description
1 - 5	NUMNP	Total number of nodes
6 - 10	NEG	Number of element groups
11 - 15	NUMEST	Estimated number of storage locations required (M _l) for element group l. Zero or blank: defaults to 3000
16 - 20	MODEX	Execution mode. Specify (a) Zero: data check only (b) 1: execution

III. Nodal Coordinates (I5, 5X, 2F 10.0, I5)

As many cards as needed to generate total number of nodes

NUMNP and their coordinates

Columns	Variable	Description
1 - 5	N	Node number See Note 1
11 - 20	X (N)	X coordinate
21 - 30	Y(N)	Z coordinate
35 - 35	KN	Node number difference between suc- cessive generated nodes (given on first card in a sequence).
		Specify. Zero: No generation.

See Note 2.

Note:

(1) Node cards may not be in numerical order. Eventually, how-

ever, all nodes must be identified.

- (2) The mesh generation parameter KN must appear on the first card of a series of nodal points to be generated. The intermediate nodes to be generated between nodes (say N1 and N2) will be located at equal intervals along the straight line joining the two nodes. KN is the nodal increment to be added to previous node number. The node difference N2-N1 must be exactly divisible by KN.
- IV. Solution Time and Step Size (215, 3F10.0, 3I5)

Columns	Variable	Description
1 - 5	NDT	Number of solution time steps. T Specify
		Zero: defaults to 1 step
6 - 15	DT	Step size
16 - 20	NPRINT	Time interval for printout of nodal displacements and pressures expressed as a multiple of the integration time step. Specify
		Zero: defaults to l

V. Earthquake Input

A. Control Information (215) - one card

Columns	Variable	Description
1-5	NBCF	Number of ground input components (use l)
6 - 10	NPTM	Maximum number of points to describe the earthquake input. See Note 1.

Note:

(1) NPTM is the number of [f(t),t] pairs used to define the earthquake ground motion which could be either an acceleration or displacement-time history record. At least two points are required to describe the input. B. Earthquake Input Data

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For one component of ground motion (horizontal in this case) a control card followed by as many cards as needed to define the earthquake.

1. Control Card (215, F10.0, I5) - First card

Columns	Variable	Description
1 - 5	NC	Function number. Specify equal to 1 in this case
6 - 10	NPTS (NC)	Number of time points used to describe the earthquake input (GE.2 and EQ. to NPTM)
11 - 20	FOM	Multiplication factor for conversion to right units. See Note l
21 - 25	INPUT	Specify
		 If acceleration is ground input If displacement is the given ground input.

2. [f(t), t] Earthquake Data (8F10.0)

As many cards as needed to define NPTS (NC) pairs of points [TFN(NC,I), FN(NC,I)]; four pairs per card. See Note 2.

Columns	Variable	Description
1 - 10	TFN(NC,1)	Time at point 1 : t ₁
11 - 20	FN (NC,1)	Acceleration or displacement value at point l : f (t _l)
21 - 30	TFN(NC,2)	t ₂
31 - 40	FN(NC,2)	f(t ₂)
41 - 50	TFN(NC,3)	t ₃
51 - 60	FN (NC,3)	f(t ₃)
61 - 70	TFN (NC,4)	t ₄
71 - 80	FN (NC,4)	f(t ₄)
Next card(s) –	as many as needed to define the earth- quake input.

Note:

- Factor of multiplication if necessary to make the units of ground input compatible with the units of the tank dimensions. This option is available only if input is in the form of accelerogram. In case of displacement history the units must be in inches, seconds and pounds.
- (2) Time values at successive points are assumed to increase in magnitude. Values of ground input other than TFN(NC,I) are calculated within the program using a linear interpolation.

VI. Element Data

Elements are divided into three groups (NEG). An element group is a series of elements of a particular type.

The elements in a particular group must be numbered sequentially starting with the number of the first element as specified on the element group control card.

Following are the three types of element groups used in this program.

Type 1 - Two Dimensional Finite Elements

These are 4-to-8 node axisymmetric isoparametric elements which lie in the global X-Z plane and are used to model the liquid continuum. Z-axis has been taken as the axis of revolution for the axisymmetric tank.

Type 2 - Free Surface Elements

These are 2 node axisymmetric elements which have been . used to represent the free surface of the liquid. These elements lie in the X-Z global plane where Z is the axis of revolution of the tank. These elements contribute to mass matrix.

Type 3 - Liquid-Solid Interface Elements

These are two node axisymmetric elements and lie in the global X-Z plane. These elements contribute to the loading vector.

Type 1 - Two Dimensional Finite Elements

A. Control Information (615) - one card.

Columns	Variable	Description
1 - 5	NGR	Element group indicator. Punch the number "1".
6 - 10	NEL1	Number of elements in group 1.
11 - 15	MFST	Element number of the first element in this. See Note 1.
16 - 20	ITYP2D	Element type code. Specify Zero: axisymmetric
21 - 25	MXNODS	Maximum number of nodes used to describe any one element. Specify Zero: defaults to 4. (GE.4 and LE.8)
26 - 30	NINT	Numerical integration order to be used in Gaussian quadrature. Specify Zero: defaults to 2 (GE.2 and LE.4) See Note 2.

Note:

- Element numbers in any group may not start from 1 if MFST is specified.
- (2) For rectangular elements, an integration order of 2 is sufficient. For non rectangular elements a higher order should be used.

B. Element Data (1115)

As many data cards as are needed in order to generate the element data for the elements (NEL1) in this group.

Columns	Variable	Description
1-5	M	Element number See Note 1
6 - 10	NOD(1)	Global node number of element node 1.
11 - 15	NOD (2)	Global node number of element node 2.
16 - 20	NOD (3)	Global node number of element node 3.
21 - 25	NOD(4)	Global node number of element node 4.
26 - 30	NOD (5)	Global node number of element node 5.
31 - 35	NOD(6)	Global node number od element node $\ddot{6}$.
36 - 40	NOD(7)	Global node number of element node 7.
41 - 45	NOD(8)	Global node number of element node 8. See Note 2
46 - 50	IEL	Number of nodes in the element. Zero: defaults to MXNODS
51 - 55	KG	Node number increment for element generation (given on lst card in a sequence) Zero: defaults to l See Note 3

Note:

- Elements must be input in increasing sequence, with MFST being the 1st element. Cards for the first and last element must be included.
- (2) If an element has less than 8 nodes (i.e., IEL.LT.8), input a zero or blank corresponding to the missing node location. For example, for a 6 node element with nodes 6 and 8 missing, the element node number array would be NOD(I) = [X X X X 0 X 0] where X entries represent the global node numbers.

(3) The node generation parameter KG must appear on the first element card of a sequence and is used to determine the node numbers for a group of missing elements in that sequence. If M is the first element of the sequence and the elements [M+1, M + 2, M + J] are the missing J elements, then the node numbers of the successive J elements are automatically incremented by the value KG given for the element M. Only the nonzero node numbers appearing on the M-th element card are incremented in this automatic generation. In the printout of the element data, generated elements are marked with an asterisk.

Type 2 - Free Surface Boundary Elements

A. Control Information (415) - one card.

Columns	Variable	Description
1-5	NGR	Element group number. Punch the number '2'.
6 - 10	NEL2	Number of elements in group 2
11 - 15	MFST	Number of the first element in group 2 (need not start with l)
16 - 20	ITYP	Element type. Specify Zero: axisymmetric this is the only option available.

B. Element Data (415)

As many cards as needed to generate NEL2 elements.

Columns	Variable	Description
1 - 5	М	Element number See Note (1)
6 - 10	NOD(1)	Global node number of element node I
11 - 15	NOD(2)	Global node number of element node J
16 - 20	KG	Node number increment for element generation. Zero: defaults to l. See Note (2)

Note:

- All elements must be input in ascending numerical order, starting with element number MFST. Cards for the first and last element must be included.
- (2) The node generation parameter KG must be given on the first element card prior to a group of missing elements. In the print out of the element data, generated elements are prefixed by an asterisk.

Type 3 - Liquid-Solid Interface Elements

A. Control Information (415) - one card.

Columns	Variable	Description
1 - 5	NGR	Element group number Punch the number '3'
6 - 10	NEL3	Number of elements in group 3
11 - 15	MFST	Number of first element in group 3
16 - 20	ITYP	Element type : Specify Zero: axisymmetric (the only option available)

A1-10

B. Element Data (315, 5X, F10.0)

As many cards as the number of elements NEL3 in group 3

Columns	Variable	Description					
1 - 5	М	Element number					
6 - 10	NOD(1)	Global node number of element node I					
11 - 15	NOD(2)	Global node number of element node J					
21 - 30	COSS	X-direction cosine of the outward normal to the element.					

VII. New Problem Data

A new problem may now be solved by adding data starting with Section I. Any number of problems can be solved in one run.

VIII. Termination Card (A4) - one card

Column	s	Variable	Description				
1 -	4	MODE	Punch	the	word	'STOP'.	

Al.4 Output

Output includes the nodal displacements and impulsive pressures. Displacements are meaningful only for the free surface nodes. С С

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PROGRAM SLOSH2(INPUT, OUTPUT, PUNCH, TAPE5=INPUT, TAPE6=OUTPUT, SLOSH SLOSH + TAPE1, TAPE2, TAPE3=PUNCH) SLOSH SLOSH SLOSH2----A FINITE ELEMENT PROGRAM TO DETERMINE THE SLOSHING RESPONSE UNDER EARTHQUAKE GROUND MOTIONS IN AN AXI-SYMMETRIC RIGID TANK SLOSH SLOSH SLOSH DEVELOPED BY--- MOHAMMAD ASLAM, DEPARTMENT OF CIVIL SLOSH ENGINEERING, UNIVERSITY OF CALIFORNIA, BERKELEY SLOSH **AUGUST 1978** SLOSH SLOSH 19 SLOSH 11 COMMON /CNTRL1/ NUMNP, NEG, MODEX, NPAR(10), NG, KBC SLOSH 12 COMMON /CNTRL2/ KST.NDT.DT.TSTART.TAMB.NPRINT.NTSREF.TIME.KP SLOSH 13 COMMON /DIM / N1.N2.N3.N4.N5.N6.N7.N8.N9.N10.N11.N12.N13.N14.N15SLOSH 14 CONTION /ELSTOR/ NUMEST, MIDEST, MAXEST CONTION /JUNK / HED(18), MTOT, NLINE SLOSH 15 SLOSH 16 / NHBC, NBCF, NPTM CONMON /NBC SLOSH 17 COMMON /WORK / WORK (200) SLOSH 18 CONTRON/CONST /A0, A1, A2, A3, A4, A5, A6, A7, A8, A9, THETA, DELTA, ALPHA, PI, GSLOSH 19 +, RD SLOSH 20 SLOSH 21 SLOSH 22 SLOSH 23 DIMENSION SD(2000), PB(2000) COMMON A(19000) SLOSH 24 MTOT. = 10000 200 MAXEST = 0 SLOSH 25 SLOSH 26 SLOSH 27 SLOSH 28 SLOSH 29 INPUT PHASE ----SLOSH 30 PROGRAM MASTER CONTROL DATA SLOSH 31 32 33 CALL DOTI SLOSH INPUT ELEMENT INFORMATION SLOSH SLOSH 34 CALL ELCAL SLOSH 35 36 SLOSH SLOSH 37 -----SOLUTION PHASE SLOSH 38 SLOSH 39 ********** SLOSH 40 SLOSH 42 BLANK COMMON STORAGE ALLOCATION SLOSH 43 SLOSH 44 ARRAY -----DESCRIPTION------DIMENSION SLOSH 45 TIME VALUES AT POINTS NPTM#HBCF SLOSH 46 Н1 TFN FUNCTION VALUES AT POINTS NUMBER OF FUNCTION INPUT POINTS NPTM# BCF SLOSH 47 N2 FN N3 NBCF **SLOSH 48** NPTS FIRST DERIVATIVE OF VEL POT. 2ND DERIVATIVE OF VEL. POT. NUMNP SLOSH 48 N4 TD N5 NUMNP SLOSH 49 TDD NUMNP SLOSH 50 N6 TTAU NUMNP SLOSH 51 N7 P PRESSURE (DYNAMIC IMPULSIVE) VELOCITY POTENTIAL ADDRESSES OF XK DIAGONAL ELEMENTS EFFEUTIVE STIFFNESS MATRIX SLOSH 52 SLOSH 53 N8 Т NUMNP NUMNP+1 N9 MAXA N10 NWK. SLOSH 54 XK LOADING VECTOR MASS MATRIX SLOSH 55 SLOSH 56 Q NUMNP N11 NUMNP N12 С WATER DISPLACEMENTS AT SURFACE NUMNP SLOSH 57 N13 Ε

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C C				****	ukukakakak		****	o lokajokoko ka	xolololololokokokokolokolokokoko	SLUSH ****SLOSH	60 60
		CALL	ADRSK	((A(N11),A	(N12).H		K, MB)		SLOSH SLOSH	61 62
ç		CHIET	STOP	DACE		MINATE			τοτο	SLOSH	63
č	-	JHIP I								SLOSH	65
	5	I = 1 N12M	+ m- = N12	4XEST 2 - 1						SLUSH	66 67
		DO 10	J=N3	3, N12	М					SLOSH	68
	10	I = I	+1	,,						SLOSH	70
С		N1 =	1	+	MOVEST					SLOSH	71
		N2 =	ŇI	+	NPTMM	BCF				SLOSH	73
		N3 =	N2	+	NPTM#N	BCF				SLOSH	74
		N5=N4	+	NUMN	P					SLOSH	76
		N6=N5	+	NUMN	P					SLOSH	77
		N8=N7	+	NUMN	P					SLOSH	79
		N9 =	N8	+	NUMNP					SLOSH	80
		N10 = N11 =	N10	3 +	NUMBER	• 1				SLOSH	82
		N12 =	N11	L +	NUMNP					SLOSH	83
		N13 = N14 =	N12	2 + 3 +	NUMNP					SLUSH	84
		N15 =	N14	4 +	NUMNP	50000		OT)		SLOSH	86
С		IF (NI)	5.61.	. MIUI	JUHLL	ERRUR	(N12-UI	01)		SLUSH	88
-		IF (MO)	DEX.E	EQ.Ø)	GO TO	200				SLOSH	89
C C		INITI	AL IZE	E STI	FFNESS	MATRIX	K (XK) A	ND LOADIN	G VECTOR Q	SLUSH	90 91
č										SLOSH	92
		N12M DO 15	= N12 I=N1	2 - 1 10.N1	2M					SLUSH	93
_	15	A(D)	= 0.0	3						SLOSH	95
L L		INITI	AL 17F	- VEL			TAL VECT	TR AT AT	TT(0)=T(0)	SLUSH	96 97
č					-	012				SLOSH	98
		DO 20	I=1, N8	NUMN + T	Р - 1				·	SLOSH	99 100
		İTT =	N14	+ <u>i</u>	- ī					SLOSH	101
C	20	ACT I) = H	4(11)						SLUSH	102
č		INITI	ALIZE	E THE	TIME	STEP CO	DUNTER			SLOSH	104
C	22	KSTEP	= A							SLUSH	105
_		TIME=	a. Č							SLOSH	107
C r		INITI	91 TZE	- MOS	S MOTP		MPEN MAS	S SYSTEM		SLOSH	108
č					U 1 1 111					SLOSH	110
		N13M -	= N13 T=N1	3 - 1 12.N1	7M					SLOSH	111
_	25	Â(I)	= 0.0)	511					SLOSH	113
C r			ATE	LUNC	TONTS	DE INTE				SLOSH	114
č				20113	inije i					SLOSH	116
		PI=3.	14159 19	92654						SLOSH	117
		R0=0.0	20009	9351						SLOSH	119
		THETA	=1.0							SLOSH	120

DELTA=0.50 ALPHA=0.25 TAU=THETA*DT A0=1.0/(ALPHA*TAU*TAU) A1=DELTA/(ALPHA*TAU) A2=1./(ALPHA*TAU) A3=1./(2.*ALPHA)-1. A4=DELTA/ALPHA-1. A5=TAU*(DELTA/ALPHA-2.0)/2. A6=DT*(1.-DELTA) A7=DT*DELTA A8=DT*DT*(0.5-ALPHA) A9=ALPHA*DT*DT C C C C ASSESBLE THE EFFEECTIVE SYSTEM STIFFNESS MATRIX(K*) 30 CALL ASSEMK C C C FORM THE EFFECTIVE K AND CALL IT XK CALL KSTAR(A(N9),A(N10),A(N12)) C C INITIALIZE VELOCITY POTENTIAL AND ITS DERIVATIVES N6M=N6-1 DO 36 I=N4,N6 36 A(I)=0. TRIANGULARIZE THE EFFECTIVE CONDUCTIVITY MATRIX, (K*) С Ĉ 40 KTR = 0CALL COLSOL (A(N10), A(N11), A(N9), NUMNP, MB, NUK, KTR) С Ĉ ------Č C TIME MARCHING LOOP INITIALIZE Q N12M=N12-1 DO 44 I=N11,N12M 44 A(I)=0. TX=THETA*DT C Č C 100 KSTEP = KSTEP + 1 TTH=TIME+TX TIME = TIME + DT FORM THE LOAD VECTOR CALL FORMOC (TTH) COMPUTE EFFECTIVE LOAD VECTOR CALL QEFF(A(N11),A(N12),A(N8),A(N4),A(N5),NUMNP) C C UPDATE (TT) VECTOR ē DO 82 I=1, NUMNP IT = N8 + I - 1ITT = N14 + I - 182 A(ITT) = A(IT)000 SOLVE THE EQUILIBRIUM EQUATIONS FOR VELOCITY POTENTIAL 84 KTR = 2

SLOSH121

SLOSH122

SL05H123

SL05H124

SLOSH125

SLOSH126 SLOSH127

SL0SH128

SLOSH129

SLOSH130

SLOSH131

SLOSH132

SLOSH133

SLOSH134

SLOSH135 SLOSH136 SLOSH137

SLOSH138

SLOSH139 SLOSH140 SLOSH141

SLOSH142

SLOSH143 SLOSH144

SLOSH145

SLOSH146

SLOSH147

SLOSH148 SLOSH149

SLOSH150

SLOSH151

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SLOSH153 SLOSH154

SLOSH155

SLOSH156

SLOSH157

SLOSH158

SLOSH159

SLOSH160

SLOSH161

SLOSH162

SLOSH163

SLOSH164 SLOSH165

SLOSH166

SLOSH167

SLOSH168 SLOSH169 SLOSH170

SLOSH171

SLOSH172

SLOSH173

SLOSH174

SLOSH175 SLOSH176

SLOSH177 SLOSH178

SLOSH179

SLOSH180

SLOSH181 SLOSH182

SLOSH183

CALL COLSOL (A(N10), A(N11), A(N9), NUMNP, MB, NUK, KTR) SLOSH184 SLOSH185 CCC Q-VECTOR IS NOW T-VECTOR. SET T(I)=Q(I) AND Q(I)=0. SLOSH186 SLOSH186 SLOSH187 DO 85 I=1.NUMNP SLOSH188 SLOSH189 A(IT) = A(IQ)SLOSH190 SLOSH191 85 A(IQ) = 0.0С С SLOSH192 CALCULATE VEL. POTENTIAL AND ITS DERIVITIVE AT TIME+DT SLOSH193 CALL CALCU(A(N8), A(N14), A(N4), A(N5), A(N7), A(N13), NUMNP) SLOSH194 С SLOSH195 PRINT AND/OR PUNCH THE NODAL DISPLACEMENTS AND PRESSURES. IF REQUESTED, AT THIS TIME STEP SLOSH196 000 SLOSH197 SLOSH198 SLOSH199 K = MOD(KSTEP, NPRINT) IF(K.NE.0) GO TO 90 SLOSH200 CALL OUT(A(N13),NUMNP,TIME,KSTEP) SLOSH201 CALL OUP(A(N7),NUMNP,TIME,KSTEP) 90 IF(KP.EQ.0) GO TO 92 SLOSH202 SLOSH203 L = MOD(KSTEP, KP)SLOSH204 IF(L.NE.0) GO TO 92 CALL PTEMP (A(N8),TIME,NUMNP) SLOSH205 SLOSH206 C 92 SLOSH207 SLOSH208 CONTINUE SLOSH209 SD(1)=0. NN=KSTEP+1 SLOSH210 CHECK FOR FINAL TIME STEP SLOSH211 C C SLOSH212 IF(KSTEP.LT.NDT) GO TO 100 SLOSH213 SLOSH214 GO TO 200 С SLOSH215 SLOSH216 END SUBROUTINE DOTI DOTI С DOTI COMMON /CNTRL1/ NUMNP, NEG, MODEX, NPAR(10), NG, KBC COMMON /CNTRL2/ KST, NDT, DT, TSTART, TAMB, NPRINT, NTSREF, TIME, KP DOTI DOTI / N1, N2, N3, N4, N5, N6, N7, N8, N9, N10, N11, N12, N13, N14, N15DOTI COMMON /DIM COMMON /ELSTOR/ NUMEST, MIDEST, MAXEST COMMON /JUNK / HED(18), MTOT, NLINE COMMON /NBC / NNBC, NBCF, NFTM DOTI DOTI DOTI COMMON A(1) DOTI 10 DIMENSION MOD(2) DOTI DATA MOD/SHSTART, SHSTOP / DOTI 11 DOTI 12 13 DOTI DOTI READ CONTROL INFURMATION 14 15 -----DOTI 16 17 DOTI 10 READ (5,1000) MODE, HED DOTI IF(MODE.EQ.MOD(2)) STOP DOTI 18 IF(MODE.EQ.MOD(1)) GO TO 20 DOTI 19 WRITE(6,3000) DOTI 20 21 DOTI GO TO 19 С 22 DOTI 20 READ (5,1001) HUNNP, NEG, NUMEST, MODEX DOTI 23 C DOTI 24 25 IF (NUMEST.E0.0) NUMEST = 4000DOTI 26 DOTI IF (NUMMP.GT.0) GO TO 30 WRITE(6,3001) DOTI 27 28 STOP DOTI 30 IF(NEG.GT.0) GO TO 40 DOTI 29

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		WRITE(6,3002)	DOT	30
С			DOT	32
	40	CALL TITLE (HED) LIDITE(S 2000) NUMND NEC NUMEST, MODEX	DOT	33
		NLINE = 17	DOT	i 35
C			DOT	36
Č		*	DOT	1 37 1 38
Č		DIANK COMMON STORACE ALLOCATION	DOT	39
č		BLANK CUMMUN STURAGE ALLUCATION	DOT	40
Č		ARRAYDESCRIPTION DIMENSION	DOT	42
с С		N2 Y NODAL Y-COORDINATES NUMP	DOT	45
Č		N3 TEN TIME VALUES AT POINTS NPTMMBDF	DOT	45
с С		NA FN FUNCTION VHLUES HI PUINTS NPTIPHOBLE NS NPTS NUMBER OF FUNCTION INPUT POINTS NBCF	DOT	145 147
Ē		NG TD FIRST DERIVATIVE OF VEL POT. NUMMP	DOT	48
C r		N7 IDD 2ND DERIVATIVE OF VEL. PUI. NUMNP N8 TTALL NUMNP	DUI	149 150
č		N9 P PRESSURE (DYNAMIC IMPULSIVE) NUMNP	DOT	151
C r		N10 T VELOCITY POTENTIAL NUMMP N11 MAXA ADDRESSES OF (XK) DIACONALELTS NUMMP+1	DOT	[52 53
č		N12 MHT ACTIVE COLUMN HEIGHTS NUMNP	DOT	i 54
č		֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎֎	DOT	
Č		፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝፝ቚኯኯኯኯኯኯኯኯኯኯኯኯኯ	DOT	57
C		CONNONAL DOINT COODDINATE DATA	DOT	58
Č			DOT	1 59 1 60
С			DOT	61
		N1 = 1 + N00EST N2 = N1 + N00NP	DOT	l 63
		N3 = N2 + NUMP	DOT	64
С		IF(NS.GI.MUT) CHLL ERROR (NS-MUT)	DOT	1 65
-		CALL COORD (A(N1),A(N2),NUMNP)	DOT	67
L C		#=====================================	DOT	1 68 1 69
ŗ.		READ SOLUTION TIME AND GROUND ACCELERATION	DOT	1 70
C C			DOT	1 71
-		READ(5,1002)NDT, DT, NPRINT, KP	DOT	- 73
		KST=0 \$ TSTART=0. TAMR=0. \$ NTSRFF=0		l- 73 I- 73
С			DOT	74
		IF(NDT .EQ.0) NDT = 1 IF(NPPINT FO 0) NPPINT = 1		I 75 I 76
		CALL TITLE (HED)	DOT	i 77
		WRITE(6,2001) NDT.DT.NPRINT.KP	DOT	I- 78
С			DOT	1 80
ç			DOT	I 81
č		NENV INE ENKINGUMKE MUUELEKUGKMID Titiciaatistessessessessessessessesses	DOT	1 83
С			DOT	84
		WRITE(6,2002) NBCF,NPTM	DOT	1 85 86]
	60	NLINE = NLINE + 9	DOT	87
	20	N4 = N.5 + NPTM*NBUF N5 = N4 + NPTM*NBCF		1 91 1 92
		N6 = N5 + NBCF	DOT	i э́з

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DOTI IF (NG.GT.MTOT) CALL ERROR (NG-MTOT) 94 DOTI 95 С DOTI CALL FUNC (A(N3),A(N4),A(N5),NPTM) 96 С DOTI 101 60 CALL TITLE (HED) NLINE = 10 DOTI 102 DOTI 104 79 N7=N6+NUMNP DOTI-105 DOTI-106 N8=N7+NUMNP N9=N8+NUMNP DOTI-107 DOTI-108 N10=N9+NUMP DOTI 112 IF (N10.GT.MTOT) CALL ERROR (N10-MTOT) С DOTI 115 С DOTI 116 Ĉ INITIALISE VEL. POTENTIAL DOTI-117 DOTI 118 С **DOTI 119** С 80 N11 = N10 + NUMNP DOTI 120 IF(N11.GT.MTOT) CALL ERROR (N11-MTOT) DOTI 121 DOTI 122 С DOTI 123 CALL INITAL (A(N10), TAMB, NUMNP) DOTI 124 DOTI 125 С FORMAT STATEMENTS С DOTI 126 C 1000 FORMAT(A5,3%,18A4) DOTI 127 1001 FORMAT(415) DOTI 128 FORMAT(15,F10.0,215) DOTI-129 1002 1003 FORMAT(315) DOTI 130 DOTI 131 DOTI 132 2000 FORMAT(20(1H*)/20H CONTROL INFORMATION/20(1H*)/// NUMBER OF NODAL POINTS = IS/ NUMBER OF ELEMENT GROUPS = IS/ MAX. ELEMENT GROUP STORAGE . = IS/ 34H 1 34H DOTI 133 3 34H DOTI 134 SOLUTION MODE = 15/ EQ. 0, DATA CHECK/6X,16HEQ. 1, EXECUTION///) 4 34H DOTI 135 23H DOTI 136 Δ 2001 FORMAT(50(1H*)/38H SOLUTION TIME AND PRINT, PUNCH DOTI-137 , 12H INFORMATION/50(1H*)/// DOTI 138 112H1NFORMATION/50(1H*)///DOTI 138548HNUMBER OF SOLUTION TIME STEPS= 15/DOTI 142643HSOLUTION TIME STEP INCREMENT= F10.4//DOTI 143948HOUTPUT PRINT INTERVAL= I5/DOTI 146848HOUTPUT PRINT INTERVAL= I5///DOTI 1482002 FORMAT(25(1H*)/25HTIME DEPENDENT FUNCTIONS/25(1H*)///DOTI 148148HNUMBER OF TIME DEPENDENT FUNCTIONS/25(1H*)///DOTI 149248HMAXIMUM NUMBER OF (F(T),T) FOINTS= I5//248HMAXIMUM NUMBER OF (F(T),T) FOINTS= I5//3000 FORMAT(//51H**ERROR**PRODLEM DECK MUST & EGIN WITH START CARD)DOTI 1553001 FORMAT(//49H**ERROR**NO. OF NODAL PUINTS MUST BE .GT. ZERO)DOTI 1573002 FORMAT(//51H**ERROR**NO. OF ELEMENT GROUPS MUST BE .GT. ZERO)DOTI 1573001 FORMAT(//51H**ERROR**NO. OF ELEMENT GROUPS MUST BE .GT. ZERO)DOTI 1573001 FORMAT(//51H**ERROR**NO. OF ELEMENT GROUPS MUST BE .GT. ZERO)DOTI 1573001 FORMAT(//51H**ERROR**NO. OF ELEMENT GROUPS MUST BE .GT. ZERO)DOTI 157 1 158 C DULI RETURN DOTI 159 DULT 160 END SUBROUTINE COORD (X,Y,NUMNP) COOR С COOR 2 3 С THIS ROUTINE READS AND GENERATES THE GLOBAL NODAL POINT COORDINATE DATA FOR 4- TO 8-NODE ISOPARAMETRIC ELEMENTS С COOR 4 5 С COOR С ⋇⋻⋹⋇⋇⋇⋵⋹⋳⋳⋳⋎⋬⋺⋹⋺⋹⋇∊⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋬⋳⋳⋳⋳⋳⋷⋽⋳⋪⋺⋳⋫⋏∊⋕⋼⋹⋳⋹⋫⋫⋳⋹∊⋳⋼⋏⋇⋻⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇⋇<mark>⋇⋇⋇⋇⋇</mark>⋐**⋳⋳**ℝ 6 С COOR 7 DIMENSION X(1),Y(1) COOR 8 COMMON /JUNK / HED(18), MTOT, NLINE COOR 9 C COOR 10 READ OF GENERATE NODAL POINT DATA COOR 0 11 COOR 12 COOR 13 WRITE(6,2000) WRITE(6.2001) COOR 14 NLINE = NLINE + 12COOR 15

COOR NOLD = 0 С COOR 10 READ (5,1000) N.X(N),Y(N),KN,JPR IF(N.EQ.1) IPR-JPR IF(NLINE.LT.55) GO TO 15 COOR COOR COOR CALL TITLE (HED) WRITE(6,2001) COOR COOR COOR NLINE = 10С COOR 15 WRITE(6.2002) N.X(N).Y(N).KN NLINE = NLINE + 1 COOR COOR IF(NOLD.EQ.0) GO TO 30 COOR C C COOR CHECK IF GENERATION IS REQUIRED COOR Ē COOR COOR IF(KNOLD.EQ.0) GO TO 30 NUM = (N-NOLD) /KNOLD NUMN = NUM - 1 COOR COOR COOR RNUM = NUM = (X(N)-X(NOLD))/RNUM = (Y(N)-Y(NOLD))/RNUM COOR DX COOR DY = NOLD COOR K DU 20 J=1, NUMN COOR KK = K COOR COOR = K + KNOLDĸ COOR X(K) = X(KK) + DXY(K) = Y(KK) + DY20 COOR COOR С 30 NOLD = NCOOR KNOLD= KN COOR IF(N.NE.NUMNP) GO TO 10 COOR С COOR COOR IF(IPR.EQ.1) GO TO 200 C C COOR COOR PRINT ALL NODAL POINT DATA č COOR CALL TITLE (HED) WRITE(6,2003) NLINE = 9 COOR COOR COOR NROW = NUMNP/3 + 1 COOR NR = 0 COOR С COOR COOR DO 100 I=1, NUMP,3 NR = NR + 1IF = I + 2COOR COOR IF (NR.EQ.NROW) IP = NUMNP IF (NLINE.LT.55) GO TO 50 CALL TITLE (HED) WRITE (6,2003) COOR COOR COOR COOR NLINE = 9 50 WRITE(6,2004) (N,X(N),Y(N),N=I,IP) 100 NLINE = NLINE + 1 COOR COOR COOR COOR С č COOR FORMAT STATEMENTS COOR č 1000 FORMAT(15,5X,2F10.0,15.11) COOR 1000 FORMAT(15,5X,2F10.0,15,11) 2000 FORMAT(28(110)/28H NODAL POINT COORDINATE DATA/28(1H*)/) 2001 FORMAT(19(110)/19H A. INFUT NODE DATA/19(1H*)/// 1 4X,4HNUDE,5X,7HX-COORD,5X,7HY-COORD.5X,4HDIFF/) 2002 FORMAT(3X,15,2F12.3,3X,15) 2003 FORMAT(23(1H*)/23H B. GENERATED NODE DATA/23(1H*)/// 1 3(4X,4HNODE,5X,7HX-COORD,5X,7HY-COORD,5X)/) 2004 FORMAT(3(3X,15,2F12.3,5X)) COOR COOR COOR COOR COOR COOR COOR

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COOR 79 С 200 RETURN COOR 80 COOR END 81 SUBROUTINE FUNC (TFN, FN, NPTS, NPTM1) FUNC 1 23 FUNC DEFINE ALL BOUNDARY CONDITION FUNCTIONS FUNC 45 67 FUNC DIMENSION TEN(NPTM1,1), EN(NPTM1,1), NPTS(1) COMMON /CNTRL2/ KST,NDT,DT,TSTART,TAMB,NPRINT,NTSREF,TIME,KP COMMON /JUNK / HED(18),MTOT,NLINE FUNC 8 FUNC FUNC ĝ. NNBC, NBCF, NPTM CONTION /NBC FUNC 10 COMMON /WORK FORM(4), WORK(196) FUNC 11 С FUNC 12 WRITE(6,2001) FUNC 13 14 15 NLINE = NLINE + 3FUNC С FUNC DO 100 LL=1.NBCF FUNC-16 READ (5,1000) NC, NPTS(NC), FOM, INPUT FUNC-17 WRITE(6,2002) NC,NPTS(NC),FOM NLINE = NLINE + 1 FUNC-18 19 FUNC IF (NPTS (NC) .GE.2.AND.NPTS (NC) .LE.NPTM) GO TO 20 FUNC 20 21 22 23 WRITE(6,3000) FUNC STOP FUNC FUNC 000 READ TIME FUNCTION VERSUS TIME TABLE FUNC 24 25 26 27 28 FUNC 20 NT = NPTS(NC) FUNC READ (5,1001) (TFN(K,NC),FN(K,NC),K=1,NT) FUNC C C C FUNC 29 30 CHECK THAT TIME POINTS ARE IN INCREASING ORDER FUNC FUNC FUNC 31 TOLD = -1. DO 30 K=1.NT FUNČ 32 33 IF(TFN(K,NC).GT.TOLD) GO TO 30 FUNC WRITE(6,3001) FUNC 34 35 STOP FUNC 36 37 30 TOLD = TFN(K,NC) FUNC C C FUNC FUNC 45 46 50 DO 70 K=1.NT FUNC IF (NLINF.LT.55) GO TO 60 FUNC 47 CALL TITLE (HED) WRITE(6,2000) FUNC 48 FUNC 49 WRITE(6,2001) FUNC 50 51 HLINE = 10FUNC 60 WRITE(6,2003) K,TFN(K,NC),FN(K,NC) 70 NLINE = NLINE + 1 52 53 FUNC FUNC FUNC 54 FUNC-55 FUNC-55 INTEGRATE THE ACCELEROGRAM TO OBTAIN THE VELOCITY FUNC-56 FÜNC-57 FI=FN(1,NC) FN(1,NC)=0. FUNC-58 IF(INPUT.E0.2) GO TO 55 FUNC-59 DO 80 K=2,NT FUNC-60 FUNC-61 L=K-1 FUNC-62 FI1=FN(K,NC) FN(K,NC) =FN(L,NC)+(FI+FI1)*(TFN(K,NC)+TFN(L,NC))*FOM/2. FUNC-63 FI=FI1 FUNC-64 URITE(6,01)K, TFN(K,NC), FN(K,NC) FUNC-65 FUNC-66 31 FORMAT(60X, 15, 2F15.5)

88)	CONTINUE GO TO 100	FUNC-	-67 -68
CCC		DIFFERENTIATE DISPLACEMENT TO GET VELOCITY	FUNC-	-68 -69 -70
55	5	DO 110 I=2.NT	FUNC-	-71
		IF(I.EQ.NT) GO TO 111 K=I+1	FUNC-	-73
		A=TFN(I,NC)-TFN(J,NC) B=TFN(K,NC)-TFN(I,NC)	FUNC-	-75 -76
		C=FN(I,NC) D=(C-FI)/A	FUNC- FUNC-	-77 -78
		E=(FN(K,NC)-C)/B FN(I,NC)=(D+E)/2.	FUNC-	-79 -80
	-	FI=C URITE(6,81)I.TFN(I.NC).FN(I.NC)	FUNC-	-81 -82
11	0	CONTINUE = (FN(NT,NC) - C) / (TFN(NT,NC) - TFN(J,NC))	FUNC-	-83 -84
r	100	CONTINUE		-85 55
č		FORMAT STATEMENTS	FUNC	57 58
č	000	FORMAT(215, F10, 0, 15)	FUNC	59 - 60
1	001	FORMAT(8F10.0) FORMAT(25(1H*)/25H TIME DEPENDENT FUNCTIONS/25(1H*)//)	FUNC FUNC	61 62
2	2001	FORMAT(4X,8HFUNCTION,4X,9HNUMBER OF,6X,10HTIME POINT,4%,4HTIME, 1 4X,8HFUNCTION/5X,6HNUMBER,4X,11HTIME POINTS,7X,3HNUMBER,	FUNC	63 64
2	002	2 5X,5HVALUE,5X,5HVALUE/) FORMAT(4X,15,7X,15,40X,*MULTIPLICATION FACTOR=*,F10.4)	FUNC-	- 65
4626	000 000 000	FORMAT(//49H **ERROR** (NPTS) MUST BE .GE. 2 AND .LE. (NPTM)) FORMAT(//49H **ERROR** BC FUNCTION TIME POINTS ARE OUT OF ORDER)	FUNC	68 69
C_			FUNC	72 73
		END SUBROUTINE INITAL (T,TAMB,NUMNP)	FUNC	74
C C		**************************************	INIT KINIT	23
			VNUT- KINIT INIT	-, 4 5 6
c		DIMENSION T(NUMNP)	INIT INIT	78
		ICON=0 TAMB=0.	INIT- INIT-	- 9 - 10
	100	DO 100 I=1,NUMNP T(I) = TAMB		, 12
	300			, 35 , 36
C			ELCL	23
		CONTION /DIM / NI.N2.N3.N4.N5.N6.N7.N8.N9.N10.N11.N12.H13.N14.N15 CONTION /ELSTOR/ NUMEST.MIDEST.MAXEST	ELCL	- 4
		CONTION /JUNK / HED(18), MTOT, NLINE COMMON /WORK / NST(10), WORK(190)	ELCL	67
		COMMON A(1) DIMENSION LABEL(2,2)	ELCL	8 9
С		DHIH LHBEL/DHHAISTH, DHDEIKIU, DHM L H , DHH H K /	ELCI	10

12 THIS ROUTINE CALLS THE APPROPRIATE ELEMENT ROUTINES FOR READING, ELCL GENERATING AND STORING THE ELEMENT DATA ELCL 13 14 ELCL 15 16 17 TAPE ALLOCATION0 ELCL TAPE 1 - STORES ELEMENT GROUP DATA ELCL 18 TWO DIMENSIONAL FINITE ELEMENTS ELCL 19 ELCL 20 21 22 NPAR(1) = 1ELCL NFAR(1) = 1 NPAR(2) = NUMBER OF TWO DIMENSIONAL ELEMENTS (NEL1) NPAR(3) = NUMBER OF FIRST ELEMENT IN THIS GROUP (MFST) NPAR(4) = ELEMENT TYPE CODE (ITYP2D) E0.0, AXISYMMETRIC NPAR(5) = MAXIMUM NUMBER OF NODES (MXNODS) ELCL ELCL 23 ĒLŪL 24 25 27 ELCL ELCL NPAR(6) = NUMERICAL INTEGRATION ORDER (NINT) ELCL 28 32 ELCL-FREE SURFACE ELEMENTS 33 ELCL 34 NPAR(1) = 235 ELCL NPAR(2) = NUMBER OF FREE SURFACE ELEMENTS(NEL2) NPAR(3) = NUMBER OF FIRST ELEMENT IN THIS GROUP (MF3T) ELCL-36 ELCL 37 NPAR(4) = ELEMENT TYPE CODE (ITYP) EQ.0, AXISYMMETRIC FREE SURFACE ELEMENT ELCL 38 ELCL-39 42 SOLID BOUNDRY ELEMENTS ELCL-43 ELCL 44 NPAR(1) = 3ELCL 45 NPAR(2)=NUMBER OF SOLID BOUNDARY ELEMENTS(NEL3) NPAR(3) = NUMBER OF FIRST ELEMENT IN THIS GROUP (MFST) NPAR(4) = ELEMENT TYPE CODE (ITYP) EQ.0, AXISYMMETRIC SOLID-LIQUID BOUNDARY ELEMENTS ELCL-46 ĒLCL 47 ELEL-47 ELCL-47 48 ELCL 49 ELCL ZERO ACTIVE COLUMN HEIGHT ARRAY (MHT) 50 ELCL 51 N12 = N11 + NUMNP + 1 N13 = N12 + NUMNP 52 ELCL ELCL 53 54 IF(N13.GT.MTOT) CALL ERROR (N13-MTOT) ELCL ELCL 55 DO 5 I=N12,N13 ELCL 56 5 A(I) = 0.057 ELCL ELCL 58 **REWIND 1** ELCL 59 ELCL 60 LOOP OVER ALL ELEMENT GROUPS ELCL 61 ELCL 62 63 DO 100 NG=1, NEG ELCL CALL TITLE (HED) ELCL 64 WRITE(6,2000) NG ELCL 65 NLINE = 7ELCL 66 ELCL 67 READ (5,1000) NPAR ELCL 68 ELĈĹ 69 70 71 NGR = NPAR(1)ELCL ELCL 72 73 74 GO TO (1,2,3) NGR ELCL ELCL ELCL ELEMENT GROUP 1 ELCL 75 ELCL 76 ----------ELCL 77 ELCL 1 IF(NPAR(2).GT.0) GO TO 10 78

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	10	WRITE(6,3000) STOP IF(NPAR(6).LE.4) GO TO 20 WRITE(6.3001) STOP		79 80 81 82
ſ	20	IF(NPAR(3).EQ.0) NPAR(3) = 1 IF(NPAR(5).EQ.0) NPAR(5) = 4 IF(NPAR(6).EQ.0) NPAR(6) = 2 IF(NPAR(7).EQ.0) NPAR(7) = 1 IF(NPAR(8).EQ.0) NPAR(8) = 1 IT = NPAR(4) + 1	ELCL ELCL ELCL ELCL ELCL ELCL ELCL	84 85 86 86 88 88 88 88 88 88 88 88 88 88 88
L		WRITE(6,2001) NGR, (LABEL(I, IT), I=1,2), NPAR(2), NPAR(3), HPAR(5),	ELCL FLCL-9	91 12
-		CALL ADRS1 GO TO 50	ELCL	93 94
		ELEMENT GROUP 2		96 97 98
L	2	IF(NPAR(2).GT.0) GO TO 30 WRITE(6.3000)	ELCL 1 ELCL 1	.00 .01
30		STOP IF(NPAR(5).EQ.0) NPAR(5) = 1 IT = NPAR(4) + 1	ELCL 1 ELCL-1 ELCL 1	02
Ľ		URITE(6,2002) NGR,(LABEL(I,IT),1=1,2),NPAR(2),NPAR(3) CALL ADRS2 GO TO 50	ELCL 1 ELCL-1 ELCL 1 ELCL 1	06 07 08 09
		ELEMENT GROUP 3	ELCL 1 ELCL 1 ELCL 1 ELCL 1	10 11 12 13
C	3	IF(NPAR(2).GT.0) GO TO 40 WRITE(6.3000)	ELCL I ELCL 1 ELCL 1	14
r	40	STOP IF(NPAR(3).EQ.0) NPAR(3) = 1	ELCL 1 ELCL 1 FLCL 1	.17 .18 .19
с с		IT=NPAR(4)+1 URITE(6,2002) NGR,(LABEL(1,IT),I=1,2),NPAR(2),NPAR(3),UPAR(5) CALL ADRS3	ELCL 1 ELCL 1 ELCL 1	.07 .07 .22
L	50	IF(MIDEST.GT.MAXEST) MAXEST = MIDEST IF(MIDEST.LE.NUMEST) GO TO 60 GO TO 100	ELCL I ELCL I ELCL I	24
		STORE ALL ELEMENT GROUP INFORMATION ONTO TAPE 1		.28
د د	60	WRITE(1) MIDEST,NPAR,NST,(A(I),1=1,MIDEST)		.29
с с	100	CONTINUE	ELCL 1	32
		IF(MAXEST.LE.NUMEST) GO TO 300 WRITE(6.3002) MAXEST STOP	ELCL 1 ELCL 1 ELCL 1	34
		FORMAT STATEMENTS		38
1 2 2	000 000 001	FORMAT(1015) FORMAT(23(1H*)/20H ELEMENT DATA, GROUP,13/23(1H*)//) FORMAT(26H ELEMENT GROUP INDICATOR =13,18H (TWO DIMENSIONAL ,2A6,	ELCL 1 ELCL 1 ELCL 1 ELCL 1	40 41 42

10H ELEMENTS)/// ELCL 143 NUMBER OF ELEMENTS = 15/ NO. OF FIRST ELEMENT IN GROUP .. = 15/ MAX. NO. OF NODES PER ELEMENT .. = 15/ 2 3 ELCL 144 38H 38H ELCL 145 ELCL 146 4 38H 5 38H NUMERICAL INTEGRATION ORDER ... = 15/) FORMAT(26H ELEMENT GROUP INDICATOR = 13, 1 2H (,2A6,36H FREE SURFACE ELEMENTS 5 ELCL-147 2002 **ELCL 153** 1 1111 ELCL 154

 1
 2
 38H
 NUMBER OF ELEMENTS
 =
 IS/
 ELCL

 3
 38H
 NU. OF FIRST ELEMENT IN GROUP
 =
 IS/
 ELCL

 3000
 FORMAT(//51H
 ERROR
 NO. OF ELTS. IN GROUP MUST BE .GT. ZERO)
 ELCL

 3001
 FORMAT(//53H
 ERROR
 NO. OF INTEGRATION PTS. MUST BE .LE. FOUR)
 ELCL

 ELCL 155 ELCL 156 162 163 (NUMEST) MUST BE INCREASED TO 16) 3002 FORMAT(//41H **ERROR** ELCL 164 ſ ELCL 165 300 RETURN ELCL 166 END ELCL 167 SUBROUTINE ADRS1 ADS1 1 23 С ADS1 COMMON /CNTRL1/ NUMNP, NEG, MODEX, NPAR(10), NG, KBC ADS1 CONTMON /DIM / N1.N2.N3.N4.N5.N6.N7.N8.N9.N10.N11.N12.N13.N14.N15ADS1 COMMON /ELSTOR/ NUMEST.MIDEST.MAXEST ADS1 4 5 6 COMMON /WORK / M1, M2, M3, M4, M5, M6, M7, M8, M9, M10, WORK (198) ADS1 7 COMMON A(1) ADS1 89 ADS1 ADS1 10 BLANK COMMON STORAGE ALLOCATION ADS1 11 ADS1 12 ARRAY -----DESCRIPTION------DIMENSION. ADS1 13 ELEMENT CONNECTIVITY ARRAY MXNODE HNEL 1 ADS1 MI LM 14 2*MMNDDS#NEL1 15 M2 XΥ ADS1 M3 IELT NO. OF NODES DESCRIBING ELEMENT NEL 1 ADS1 16 MIDSIDE NODES LOCATION ARRAY ND5DIM#NEL1 ADS1 NOD5 M4 17 ADS1 25 26 27 Ē C ADS1 = NPAR(2)ADS1 28 HEL1 29 32 MXNODS = NPAR(5) ADS1 NDM. = 2*MXNODS ADS1 ND5DIM = MXNODS-4 33 ADS1 С ADS1 34 ADS1 35 M1 = 1 36 M2 = M1 + MXNODS*NEL1 ADS1 MĴ + NDM*NEL1 = 112 ADS1 37 = M3 M4 + NEL1 ADS1 38 M5 = M4 + ND5DIM*NEL1 ADS1 39 NLAST=H5-1 ADSI-40 C ADS 1 47 WRITE(6,2000) NLAST ADS1 48 MIDEST = NLAST 49 ADS1 IF (ND5DIM.EQ.0) ND5DIM = 1 50 ADS1 С ADS1 51 CALL ELGR1 (A(N1), A(N2), A(N12), A(M1), A(M2), A(M3), A(M4), UNNUS, NDM, ADSI-52 1ND5D1(4) ADSI-53 С ADS1 55 56 2000 FORMAT(38H LENGTH OF ELEMENT INFORMATION .. = 15/// ADS1 C 57 ADS1 RETURN ADS1 58 59 END ADS1 SUBROUTINE ELGRI (X,Y,MHT,LM,XY,IELT,NODS,MONODS,NDM,NI3DIM) ELG-1 3 C C C ELG1 4 INPUT INFORMATION FOR 4- TO 8-NODE ISOPARAMETRIC ELEMENTS ELG1 5 ē 6

С ELG1 7 DIMENSION X(1), Y(1), MHT(1), LM(MXNODS, 1), XY(NDM, 1), IELT(1), ELG1 8 NOD5(ND5DIM, 1) ELG1-9 1 COMMON /CNTRL1/ NUMNP,NEG,MODEX,NPAR(10),NG,KBC COMMON /JUNK / HED(18),MTOT,NLINE COMMON /WORK / DUM(10),NOD(8),NODM(8),NOD5M(8),WORK(166) ELG1 11 ELG1 12 ELG1 13 DIMENSION AST(2) DATA AST/2H .2H */ ELG1 14 15 ELG1 ELG1 16 С NEL1 = NPAR(2) ELG1 17 MEST = NPAR(3) ELG1 18 ELG1 С 19 C ELG1 20 22 23 Č ELG1 ELG1 24 CALL TITLE (HED) ELG1 00000 50 ELG1 ELG1 51 ELG1 52 READ AND GENERATE ELEMENT INFORMATION -----ELG1 53 ELG1 54 CALL TITLE (HED) WRITE(6,2003) NG ELG1 **Š**5 ELG1 56 WRITE(6,2004) (1,1=1.8) ELG1 57 NLINE = 10ELG1 58 ELG1 59 = 1 N IMEM = MEST 60 ELG1 NLAST = MFST + NEL1 - 1 ELG1 61 ELG1 С 62 100 READ (5,1002) M.NOD. IEL.KG ELG-63 С ELG1 64 65 IF(MTYP.E0.0) MTYP = 1ELG1 IF(IEL .EQ.0) IEL = MXNODS IF(KG .EQ.0) KG = 1 ELG1 66 ELG1 67 IF (MXNODS.GE. IEL) GO TO 110 ELG1 68 WRITE(6,3002) M ELG1 69 ELG1 70 STOP 71 С ELG1 1.1.2 110 IF (M-IMEM) 280, 120, 200 ELG1 72 ELG1 73 000 SAVE ELEMENT INFORMATION FOR GENERATION OF ADDITIONAL ELEMENTS ELG1 74 75 76 ELG1 120 DO 130 I=1.8 ELG1 130 NUDM(1) =NOD(1) ELG1 77 IF(IEL.EQ.4) GO TO 150 78 ELG1 ELG1 79 II=0 DO 140 I=5.8 ELG1 80 ELG1 NN=NOD(I) 81 IF (NN.EQ.0) GO TO 140 ELG1 82 ELG1 83 II=II + 1ELG1 NOD5M(II) = I84 140 CONTINUE ELG1 85 С ELG1 86 150 IELM = IEL ELG1- 87 KKK = KGASTT = AST(1) ELG1 89 ELG1 90 ELG1 91 C C C ELG1 STORE PERMONENT ELEMENT INFORMATION 92 ELG1 93 200 12 = 0 ELG1 94 DO 230 I=1. IELM IF(I.LE.4) GO TO 210 95 ELG1 96 ELG1 97 JJ = NOD5M(1-4)ELG1

Ч0, ,

II = NODM(JJ) ELG1 98 99 GO TO 220 ELG1 210 II - NODM(I) ELG1 100 $\begin{array}{r} 220 \text{ LM}(1,N) = 11 \\ 12 = 12 + 2 \end{array}$ ELG1 101 ELG1 102 XY(12-1,N) = X(11)230 XY(12,N) = Y(11)ELG1 103 ELG1 104 C ELG1 105 IELT(N) = IELMELG1 106 IF(IELM.EQ.4) GO TO 250 ELG1 108 ELG1 109 NN = IELM - 4DO 240 I=1.NN ELG1 110 240 NOD5(1,N)=NOD5M(1) ELG1 111 0000 ELG1 112 UPDATE COLUMN HEIGHTS AND BANDWIDTH ELG1 113 ELG1 114 250 CALL COLHT (MHT, IELM, LM(1,N)) ELG1 115 С ELG1 116 IF(NLINE.LT.55) GO TO 260 ELG1 117 CALL TITLE (HED) WRITE(6,2003) NG ELG1 118 ELG1 119 WRITE(6,2004) (1,1=1,8) ELG1 120 NLINE = 10ELG1 121 С ELG1 122 260 WRITE(6,2005) ASTT, IMEM, NODM, IELM ELG1-123 NLINE = NLINE + 1ELG1 124 IF (IMEM.EQ.NLAST) GO TO 300 ELG1 125 С ELG1 126 N = N + 1ELG1 127 IMEM = IMEM + 1ELG1 128 С ELG1 129 CHECK IF ELEMENT DATA IS TO BE STORED FOR CURRENT ELEMENT С ELG1 130 ē ELG1 131 IF (IMEM.EQ.M) GO TO 120 ELG1 132 С ELG1 133 C C GENERATE NODE NUMBERS FOR NEXT ELENENT ELG1 134 ELG1 135 ELG1 136 DO 270 I=1,8 IF(NODM(I).E0.0) GO TO 270 ELG1 137 NODM(I) =NODM(I) +KKK ELG1 138 270 CONTINUE ELG1 139 ELG1 140 0000 CHECK IF NEXT ELEMENT CARD IS TO BE READ ELG1 141 ELG1 142 ELG1 143 ASTT=AST(2) IF(IMEM.GT.M) GO TO 100 ELG1 144 C C ELG1 145 GENERATE INFORMATION FOR NEXT ELEMENT ELGI 146 Ċ ELG1 147 ELG1 148 GO TO 200 С ELG1 149 280 WRITE(6,3003) M ELG1 150 STOP ELG1 151 C ELG1 152 FORMAT STATEMENTS ELG1 153 C ELG1 154 1002 FORMAT(1115) ELG1-155 2003 FORMAT (30(1H*)/27H ELEMENT INFORMATION, GROUP, 13/30(1H*)//) ELG1 168 2004 FORMAT(4%, 4HELT., 3%, 10(1H-), 12HNODE NUMBERS, 10(1H-), 3%, 5H 1 6HNO, 0F/5%, 3HNO, 3%, 8(3%, 11), 4%, 3HNO, 5%, 5HNODES/) 2005 FORMAT(42, 15, 4%, 814, 10%, 15) .3X.ELG1 169 ELG1 170 ELG1-171 3002 FORMAT(//10H **ELEMENT, 15, 34H EXCEEDS MAXIMUM NUMBER OF NODES**) ELG1 174 3003 FURMAT (//26H **ERROR** ELEMENT CARD =15, 16H OUT OF SEQUENCE) ELG1 175

C 300	RETURN END SUBROUTINE ADRS2	ELG1 ELG1 ELG1 ADS2	176 177 178 1
L	COMMON /CNTRL1/ NUMNP,NEG,MODEX,NPAR(10),NG,KBC COMMON /DIM / N1,N2,N3,N4,N5,N6,N7,N8,N9,N10,N11,N12,N13,N14,N19 COMMON /ELSTOR/ NUMEST,MIDEST,MAXEST COMMON /WORK / M1,M2,M3,M4,M5,M6,M7,M8,M9,M10,WORK(193) COMMON A(1)	ADS2 ADS2 5ADS2 ADS2 ADS2 ADS2	234567
C			8
č		ADS2	10
C	BLANK CUMMUN SIURAGE ALLUCATION	ADS2 ADS2	11
C C C	ARRAYDESCRIPTION DIMENSION MI LM ELEMENT CONNECTIVITY ARRAY 2*NEL2 M2 XX ELEMENT X-COORDINATES 2*NEL2	ADS2 ADS2 ADS2	13 14 15
C C	M3 CL ELEMENT LENGTHS NEL2	ADS2	16
č		*ADS2	24
C	NEL2 = NPAR(2)	ADS2 ADS2	25 26
С		ADS2	28
	M2 = M1 + 2*NEL2	ADS2	30
	M3 = M2 + 2*NEL2	ADS2	31
	NLAST=113 + NEL2	ADS2-	- 32 - 33
С	LETTE (C. 2000) NI OCT	ADS2	40
	MIDEST = NLAST	ADS2	41
C	FOLL ELERS (0(N1) 0(N2) 0(M1) 0(M2) 0(M3))	ADS2	43
С		ADS2	44
_2000	FORMAT(38H LENGTH OF ELEMENT INFORMATION = 15///)	ADS2	47
L	RETURN	ADS2	49
	END SUBPOLITING FLORD (X Y LM XX CL)	ADS2	50
C	Sobrootine Learz (A) Demanuel	ELG2	2
C C	**************************************	*ELG2	- 3
č	***************************************	*ELG2	5
C	DIMENSION X(1), Y(1), IM(2, 1), XX(2, 1), CL(1)	ELG2	- 5
C		ELG2	9
	COMMON /CNTRL1/ NUMNP,NEG,MODEX,NPAR(10),NG,KBC COMMON /IUNK / HED(18),MTOT,NLINE	ELG2	10
	COMMON /NBC / NNBC, NBCF, NPTM	ELG2	12
	COMMON /WORK / DUM(10),NOD(2),NODM(2),WORK(186)	ELG2	13
-	DATA AST/2H .2H */	ELG2	15
С	NE(2 = NPOP(2))	ELG2	16
-	MFST = NPAR(3)	ELG2	18
C C		ELG2	19
-	KBC = 0	ELG2	24
C C		ELG2	44
č	READ AND GENERATE ELEMENT INFORMATION	ELG2	40
С		ELG2	47

С		N = 1 IMEM = MFST NLAST = MFST + NEL2 - 1 CALL TITLE (HED) URITE(6,2003) NG URITE(6,2004) NLINE = 10	ELG2 ELG2 ELG2 ELG2 ELG2 ELG2 ELG2 ELG2	48 49 50 50 50 50 50 50 50 50 50 50 50 50 50
C	100	READ (5.1003) M.NOD.KG	ELG2 ELG2-	56
с с		IF(KG .EQ.0) KG = 1 II = NOD(1) JJ = NOD(2)	ELG2 ELG2 ELG2 ELG2	60 61 62
с г		IF (M- IMEM) 280,120,200	ELG2 ELG2	64
Č		SAVE ELEMENT INFORMATION FOR GENERATION OF ADDITIONAL ELEMENTS	ELG2 ELG2	66
с с	120	NODM(1) = II NODM(2) = JJ KKK = KG ASTT = AST(1) XL = SQRT((X(JJ)-X(II))**2 + (Y(JJ)-Y(II))**2)	ELG2 ELG2 ELG2 ELG2 ELG2 ELG2	68 69 71 72 73
		STORE PERMENENT ELEMENT INFORMATION	ELG2 ELG2	75
	200 230	DO 230 I=1,2 IJ = NODM(I) LM(I,N) = IJ XX(I,N) = X(IJ)	ELG2 ELG2 ELG2 ELG2 ELG2	77 78 79 80
د د		CL(N) = XL	ELG2 ELG2	81
ſ		IF(NLINE.LT.55) GO TO 250 CALL TITLE (HED) WRITE(6,2003) NG WRITE(6,2004) NLINE = 10	ELG2 ELG2 ELG2 ELG2 ELG2 ELG2	85 86 87 88 89 89
د د	250	WRITE(6,2005) ASTT.IMEM.NODM NLINE = NLINE + 1 IF(IMEM.EQ.NLAST) GO TO 300	ELG2 ELG2 ELG2 ELG2	91 92 93
с с		N = N + 1 $IMEM = IMEM + 1$	ELG2 ELG2	95 96
č		CHECK IF ELEMENT DATA IS TO BE STORED FOR CURRENT ELEMENT	ELG2 ELG2	97 98
с г		IF(IMEM.EQ.M) GO TO 120	ELG2 ELG2	100
		GENERATE NODE NUMBERS FOR NEXT ELEMENT		102
r	270	DO 270 I=1,2 NODM(1) = PODM(I) + KKK	ELG2 ELG2 ELG2	104
č		CHECK IF NEXT ELEMENT CARD IS TO BE READ	ELG2 FLG2	107
Ū.		ASTT = AST(2) IF(IMEM.GT.M) GO TO 100	ELG2 ELG2 ELG2	109 110 111
С С		GENERATE INFORMATION FOR NEXT ELEMENT	ELG2 ELG2	112 113

•	GO TO 200		ELG2	114
C 280	WRITE(6,3002) M STOP		ELG2 ELG2 ELG2	115 116 117
Č	FORMAT STATEMENTS		ELG2 ELG2	119
1003 2003 2004	FORMAT(415) FORMAT(30(1H*)/27H ELEMENT INFORMATION, GROUP,13, FORMAT(4X,4HELT.,4X,6HI-NODE,4X,6HJ-NODE,4X,5H	/38(1H*)//) /5X.	ELG2- ELG2- ELG2 ELG2	120 121 133 134
2005 3002	FORMAT(A2, I5, 5X, I5, 5X, I5) FORMAT(//26H **ERROR** ELEMENT CARD =15, 16H OUT	OF SEQUENCE)	ELG2- ELG2 F1 62	136 139 149
ॅ 300			ELG2	141
r	SUBROUTINE ADRS3		ADS3	1
C	COMMON /CNTRL1/ NUMP, NEG, MODEX, NPAR(10), NG, KBC COMMON /ELSTOR/ NUMEST, MIDEST, MAXEST COMMON /DIM / N1, N2, N3, N4, N5, N6, N7, N8, N9, N10, N	1.N12.N13.N14.N1	ADS3 ADS3	134 4
	CONTION /WORK / M1.M2.M3.M4.M5.M6.M7.M8.M9.M10.W	JRK(190)	ADS3	67
C		n de sterne de stern	ADS3	-8
		······	ADS3	10
C	BLANK LUMMUN SIURAGE ALLU	JUHIIUN	ADS3	12
C C	M1 LM ELEMENT LOCATION ARRAY	DIMENSION NEL3	ADS3 ADS3	13 14
C C	M2 XX ELEMENT X-COORDINATES M3 CL FLEMENT LENGTHS	2×NEL3 NEL3	ADS3-	15
č	M4 SINS SINE OF ANGLE SI	NEL3	ADS3-	17
Č			ADS3	20
C		kokokokokokokoko indentato kaka	ADS3	22
С	NEL3 = NPAR(2)		ADS3 ADS3	23 24
-	M1 = 1 M2 - M1 + 2 + ME1 = 7		ADS3	25
	M3=M2+2*NEL3		ADS3-	27
	M4=M3+NEL3 M5=M4+NEL3		ADS3-	28 28
	MG=M5+NEL3		ADS3-	29
	WRITE(6,2000) NLAST		ADS3-	30
	$ \begin{array}{l} \text{MIDEST} = \text{NLAST} \\ \text{COLL FLOP3} \left(O(N1) \cdot O(N2) \cdot O(M1) \cdot O(M2) \cdot O(M3) \cdot O(M4) \right) \\ \end{array} $	2 (ME))	ADS3-	32
2000	FORMAT(38H LENGTH OF ELEMENT INFORMATION =	15///>	ADS3-	34
	RETURN		ADS3	42 43
~	SUBROUTINE ELGR3(X,Y,LM,XX,CL,SINS,COSS)		ELG3-	1
C	*** *********************************	KKKKK	ELG2	23
C C	INPUT INFORMATION FOR SOLID BOUNDARY ELEMENT	kakakakakak si Newata Nakakakakakakaka	ELG3-	45
č	᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃᠃	ֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈֈ	ELG2	ទ័
С	DIMENSION X(1),Y(1),LM(2,1),XX(2,1),CL(1),SINS(D,COSS(1)	ELG3-	79
-	COMMON /CNTRL1/ NUMPPINEG. MODEX.NPAR(10).NG.KBC		ELG2	10
	COMMON /NBC / NNBC, NBCF, NPTM		ELG2	12

COMMON /WORK / DUM(10),NOD(2),NODM(2),WORK(186) ELG2 DIMENSION AST(2) DATA AST/2H .2H */ С NEL3=NPAR(2) MFST = NPAR(3)С С KBC = 000000 READ AND GENERATE ELEMENT INFORMATION N = 1 IMEM = MFST NLAST=MFST+NEL3-1 CALL TITLE (HED) WRITE(6,2003) NG WRITE(6,2004) NLINE = 10 С 100 READ(5,1003)M,NOD,KG,CS,SS С IF(KG .E0.0) KG = 1 II = NOD(1)JJ = NOD(2)С IF (M-IMEM) 280,120,200 000 SAVE ELEMENT INFORMATION FOR GENERATION OF ADDITIONAL FLEXENTS 120 NODM(1) = IINODM(2) = JJYD=ABS(Y(II)-Y(JJ)) IF(YD.GT.0.00000001) GO TO 5 SI=999999999.9 GO TO 6 5 SI = (X(II) - X(JJ)) / (Y(II) - Y(JJ))KKK=KG б = AST(1)ASTT = SQRT((X(JJ)-X(II))**2 + (Y(JJ)-Y(II))**2) XL С С STORE PERMANENT ELEMENT INFORMATION Ĉ 200 DO 230 I=1.2 IJ = NODM(I) LM(I,N) = IJ230 \times (LN) = \times (L) C CL(N) = XL SI=ATAN(SI) SINS(N)=SIN(SI) COSS(N) = COS(SI) COSS(N) =CS SINS(N)=SS C IF(NLINE.LT.55) GO TO 250 CALL TITLE (HED) WRITE(6,2003) NG WRITE(6,2004) NLINE = 10250 WRITE(6,2005) ASTT, IMEM, NODM

13 ELG2 14 ELG2 ELG2 15 16 ELG2-ELG2 ELG2 -17 18 19 ELG2 20 ELG2 24 ELG2 ELG2 44 45 46 ELG2 ELG2 47 ELG2 48 49 ELG2 ELG2 50 51 ELG2-ELG2 52 ELG2 ELG2 53 54 ELG2 55 ELG2 56 ELG2-ELG2 ELG2 57 58 60 ELG2 61 ELG2 62 ELG2 63 ELG2 ELG2 ELG2 ELG2 64 65 66 67 ELG2 68 ELG2 69 ELG2-70 ELG2-ELG2-71 72 ELG2- 73 ELG2- 74 73 ELG2-ELG2 75 72 73 ELG2-ELG2 ELG2 74 75 ELG2 ELG2 ELG2 76 77 78 79 ELG2 ELG2 80 81 ELG2 ELGZ 82 ELG3-83 ELG2- 84 ELG2- 85 ELG2- 86 ELG2- 88 ELG2- 88 ELG2 84 ELG2 85 ELG2 85 ELG2 87 ELG2 88 ELG2 89 ELG2- 90

-		NLINE = NLINE + 1 IF(IMEM.EQ.NLAST) GO TO 300
L C		N = N + 1 IMEM = IMEM + 1
Č		CHECK IF ELEMENT DATA IS TO BE STORED FOR CURRENT ELEMENT
L C		IF(IMEM.EQ.M) GO TO 120
C		GENERATE NODE NUMBERS FOR NEXT ELEMENT
C	270	DO 270 I=1,2 NODM(I) = NODM(I) + KKK
C		CHECK IF NEXT ELEMENT CARD IS TO BE READ
ι Γ		ASTT = AST(2) IF(IMEM.GT.M) GO TO 100
č		GENERATE INFORMATION FOR NEXT ELEMENT
L C		GO TO 200
د د	280	WRITE(6.3002) M STOP
C		FORMAT STATEMENTS
10	03 003 004	FORMAT(415,2F10.0) FORMAT(30(1H*)/27H ELEMENT INFORMATION, GROUP,13/30(1H*)//) FORMAT(4X,4HELT.,4X,6HI-NODE,4X,6HJ-NODE,4X.5HDIREC/5X,
	005 002	FORMAT(A2,I5,5X,I5,5X,I5) FORMAT(//26H **ERROR** ELEMENT CARD =I5,16H OUT OF SEQUENCE)
с с	300	RETURN END SUBROUTINE COLHT (MHT,ND,LM)
L C		DIMENSION LM(1),MHT(1)
C		FIND SMALLEST GLOBAL NODE NUMBER (LS) FOR ELEMENT
C	80 90 100	LS=100000 DO 100 I=1.ND IF (LM(I)) 80.100.80 IF (LM(I)-LS) 90.100.100 LS=LM(I) CONTINUE
č		COMPUTE COLUMN HEIGHT ABOVE DIAGONAL (ME) AND CHECK IF MAXIMUM
	:00	DO 200 I=1.ND II=LM(I) IF (II.EQ.0) GO TO 200 ME=II - LS IF (ME.GT.NHT(II)) MHT(II)=ME CONTINUE
υ		RETURN END SUBROUTINE ERROR (N)

ELG292ELG293ELG294ELG295ELG296ELG297ELG2101ELG2102ELG2103ELG2104ELG2105ELG2107ELG2108ELG2109ELG2110ELG2111ELG2112ELG2113ELG2114ELG2115ELG2118ELG2119ELG2120 ELG2 120 ELG2-121 ELG2-133 ELG1-121 ELG2-135 ELG2-136 ELG2 139 ELG2 140 ELG2 141 ELG2 141 ELG2 142 CLHT 1 CLHT 2 CLHT 3 123456789 CLHT CLHT CLHT CLHT CLHT CLHT CLHT CLHT CLHT 10 11 12 CLHT CLHT CLHT CLHT CLHT 13 14 15 16 17 CLHT CLHT CLHT 18 19 CLHT CLHT 20 21 22 23 24 1 CLHT CLHT CLHT ERR

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С ERR 2 3 WRITE(6,2000) N ĒRR 2000 FORMAT(///31H **ERROR** STORAGE EXCEEDED BY I6) ERR 4 5 ERR STOP ERR 6 END SUBROUTINE TITLE (HED) TITL 123 С TITL DIMENSION HED(18) TITL C. TITL 4567 THIS ROUTINE PRINTS THE TITLE CARD AT TOP OF OUTPUT PAGE С TITL TITL £ WRITE(6.2000) HED TITL 2000 FORMAT(1H1, 18A4, 39X, 8HDOT 1976/) TITL 8 RETURN TITL 9 END TITL 10 SUBROUTINE ADRSK (MAXA, MHT, NUMNP, NWK, MA) ADSK 123 С ADSK 000 TO CALCULATE ADDRESSES OF DIAGONAL ELEMENTS IN A BANDED MATRIX WHOSE COLUMN HEIGHTS ARE KNOWN. ADSK. 4 ADSK 567 Ū ADSK C MA = MAXIMUM BAND WIDTH ADSK. Ĉ = ACTIVE COLUMN HEIGHTS ABOVE DIAGONAL 8 MHT ADSK ē MAXA = ADDRESSES OF DIAGONAL ELEMENTS 9 ADSK С = MAXIMUM STORAGE REQUIRED ADSK 10 NWK Ċ 11 С ADSK 12 13 DIMENSION MAXA(1), MHT(1) ADSK. С 14 15 16 17 ADSK MAXA(1) = 1ADSK. MAXA(2) = 2ADSK. MA = 0ADSK IF (NUMNP.EQ.1) GO TO 100 ADSK 18 DO 10 I=2.NUMNP 19 ADSK. IF (MHT(I).GT.MA) MA = MHT(I) ADSK. 20 21 22 23 24 10 MAXA($I \div 1$) = MAXA(I) + MHT(I) + 1 ADSK 100 MA = MA + 1 ADSK NUK = MAXA(NUMNP+1) - 1 ADSK. С ADSK RETURN 25 ADSK. 26 END ADSK SUBROUTINE ASSEMK ASMK: 1 23 С С ASMK U C ASSEMBLE THE EFFECTIVE SYSTEM STIFFNESS MATRIX (K*) ASMK-4 5 ē 7 С ASMK COMMON /CNTRL1/ NUMNP, NEG, MODEX, NPAR(10), NC, KBC ASMK. COMMON /DIM / N1.N2.N3.N4.N5.N6.N7.N8.N9.H10.N11.N12.H13.H14.N15ASMK COMMON /WORK / M1.M2.M3.M4.M5.M6.N7.M8.M9.H10.WORK(1930) ASMK 8 9 COMMON A(1) ASMK. 10 DIMENSION NST(10) ASMK. 11 EQUIVALENCE (NST(1),M1) ASMK. 12 ASMK 13 C 14 REWIND 1 ASMK REWIND 2 ASMK 15 C ASMK. 16 Ē LOOP OVER ALL ELEMENT GROUPS ASMK 17 C ASMK. 18 DO 100 NG=1, NEG ASMK. 19 С ASMK 20 21 22 READ (1) MIDEST, NPAR, NST, (A(I), I=1, MIDEST) ASMK С ASMK

_		NGR = NPAR(1)	ASMK	23
C C		GO TO (1,2,3) NGR	ASMK	24
C C		ELEMENT GROUP 1	ASMK	27 27 28
č			ASMK ASMK	29 30
	1	MXNODS = NPAR(5) NDM = 2*M%NODS	ASMK ASMK	31 34
~		ND5DIM = MXHODS-4 IF(ND5DIM.EQ.0) ND5DIM = 1	ASMK ASMK	35
L		CALL COND1 (A(M1),A(M2),A(M3),A(M4),A(N8),MSNODS,MDM,N/SDI20	ASMK-	- 37 - 38 - 40
C C			ASMK ASMK	41
Č C		ELEMENT GROUP 2	AŠMK ASMK	43 44
С 2		CALL_COND2 (A(M1),A(M2),A(M3))	ASMK ASMK-	45 46
C			ASMK ASMK OSMK	48 49 50
C C		ELEMENT GROUP 3	ASMK	50 51 52
ž	100	CONTINUE CONTINUE	ASMK- ASMK	- 53 56
С		RETURN	ASMK ASMK	57 58
c		END SUBROUTINE COND1 (LM,XY,IELT,NOD5,T,MXNOD5,HDM,ND5DIM)	CON1-	- 1
C C		**************************************	*CON1 CON1-	4
ē c		₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩	+COH1 COH1	- - - 8
		DIMENSION LM(MXNODS,1),XY(NDM.1),IELT(1),NGD5(ND5DIM,),T 1/ COMMON ZENTRL1Z NUMMP,NEG,MODEX.NPAR(10),NG.KBU	COH1- COH1	9
		COMMON ZUNIPLZZ KSILNDILDILTSIARILTABUNPZIGILNISPELITE Z COMMON ZDIM Z NILNZLNJLNJLNJLNJLNJLNJLZGILTILA COMMON ZDIM Z NEL MODE MTZELHUDE	SCCN1 CON1	12
		COMMON / WORK / DUM(10),SK(64).SC(8),HF(8).TNOD(8).WOR(102) COMMON (A(1)	CON1 CON1	15 15
С		NEL1 = NPAR(2)	CON1 CON1	17 18
C		DO 100 N=1.NEL1	CON1 CON1	20 21
		NEL = N NODS = IELT(N) NNDS = NODS = 4		22 23 25
С		NDUF = NODS*NODS	CON1 CON1	26 26 27
Č C		ZERO ELEMENT STIFFNESS MATRIX SK(NODS,NODS)	COH1- COH1	- 28 29
-	10	DO 10 I=1,NDOF SK(1) = 0.8	CON1 CON1	- 30 - 31
U C C		ZERO ELEMENT MASS VECTOE SC(NODS)	CON1-	- 32 - 33 - 41
L	4 Ā	DC 40 I=1.00DS SC(D) = 0.0	CON1 CON1	43 43
С			CONI	51

1

C		FORM ELEMENT STIFFNESS MATRIX USING GAUSS QUADRATURE		- 52
с с		CALL FORMK1 (SK,SC,XY(1,N),NOD5(1,N),NODS)	CON1-	- 54
		ASSEMBLE EFFECTIVE ELEMENT STIFFNESS MATRIX (SK*) INTO EFFECTIVE STRUCTURAL STIFFNESS MATRIX (K*)	CON1- CON1-	- 60 - 61
c c	70 100	CALL ADDBAN (A(N10),A(N9),SK,LM(1,N),NODS) CONTINUE		63 72 73
C		RETURN END SUBROUTINE FORMK1 (SK,SC,XY,NOD5,NODS)	CON1 CON1 FMK1-	74 75 - 1
C C C		**************************************	FMK1 KFMK1 FMK1-	3 4 - 5
Č		FOR AXI-SYMMETRIC ELEMENT ************************************	FMK1- FMK1 FMK1	- 6 7 8
Ū		DIMENSION SK(NODS,NODS),SC(NODS),XY(1),NOD5(1) COMMON /CNTRL1/ NUMNP,NEG,MODEX,NPAR(10),NG,KBC COMMON /CNTRL2/ KST.NDT.DT.TSTART.TAMB.NPRINT.NTSREF.TUME.KP	FMK1- FMK1 FMK1	- 9 11 12
		COMMON /WORK / DUM(98),H(8),P(2,8),B(2,8),XJ(2,2),EK(3),CORK(55) DIMENSION XG(4,4),WGT(4,4) DATA XG / A. A. A. A. A.	FMK1 FMK1 FMK1	13 14 15
		15773502691896, .5773502691896, 0., 0., 0., 27745966692415, .0000000000000, .7745966692415, 0., 3 - 8611363115941, - 3399810435849, .3399810435849, 8611363115941,	FMK1 FMK1 FMK1	16 17 18
		DATA WGT / 2.000, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0,	FMK1 FMK1	19 20
С		3.3478548451375,.6521451548625,.6521451548625,.3478548451375/	FMK1 FMK1	22 23
r		ITYP2D = NPAR(4) NINT = NPAR(6)	FMK1 FMK1 FMK1	24 25 27
č		LOOP OVER ALL INTEGRATION POINTS	FMK1	28 29
r r		VOL = 0.0	FMK1	30
C		DO 100 LX=1.NINT R = XG(LX.NINT) DO 100 LX=1.NINT	FMK1 FMK1	32 33
~		S = XG(LY,NINT) WT = WGT(LX,NINT)*WGT(LY,NINT)	FMK1 FMK1	34 35 36
		FIND INTERPOLATION FUNCTIONS (H) AND THEIR FERIVATIVES (P). FIND JACOBIAN (XJ) AND ITS DETERMINANT (DETJ).	FMK1 FMK1 FMK1	37 38 39
C c		CALL SHAPE1 (R.S.XY.H.P.NOD5.XJ.DETJ)	FMK1 FMK1	40
		EVALUATE JACOBIAN INVERSE (XJI) AND GLOBAL DERIVATIVE OPERATOR (B) AT EACH INTEGRATION POINT (R.S) WITHIN THE ELEMENT	FMK1 FMK1 FMK1	42 43 44
L C		CALL DERIVI (XY,H,P,B,XJ,DETJ,RAD,ITYP2D)	FMKI	45
ι c		FAC = WT*RAD*DETJ VOL = VOL + FAC	FMK1 FMK1 FMK1	47 48 49
		FORM SK = B(TRANSPOSE)*EK*B FOR INTEGRATION POINT (R+S)		50 77 70
L		DO 50 I=1.NODS BT1 = B(1.1)	FMK1 FMK1 FMK1	78 79 80

AZ-24	A	2-2	24
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		· A2-24	· ·	•	
50	BT2 = B(2,1) D0 50 J=1.NODS SK(1,J)=SK(1,J)+(BT1	*8(1,J)+ BT2*8(2,J))*FAC		FM
ຼັ 100	CONTINUE				Fr Fr
200	NODM = NODS - 1 DO 200 I=1,NODM II = I + 1 DO 200 J=II,NODS SK(J,I) = SK(I,J)				FT FT FT FT FT
5 600	RETURN				F1 F1
	END SUBROUTINE SHAPE1 (R	S.XY.H.P.NOD5.XJ	DETJ)		FI
	1. TO FIND INTERPOLA AND DERIVATIVES (OF A 4- TO 8-NODE 2. TO FIND JACOBIAN	OKKANANANANANANANANANANANANANANANANANANA	**************************************	GRIDOR AND	XXXXX S S S S S S S
	NOD	E NUMBERING CONVE	NTION		5 5
	2	5	1	L	S
3	0	0	0		Sh
	•		•		Sł
Č	•	S	•		SH
			•		SI
		• • • K	•)	Sł
5	•		•		Sh
	•		:		SH
2	0	0	0		SH
	3	7	4	1	Sh
	**	***	******	o <mark>lolololo</mark> la kulokuka kukokuka k	xxxxSl
-	DIMENSION XY(2,1),H COMMON /TODIM / NEL, DIMENSION IPERM(4) DATA IPERM/2,3,4,1/	(1),P(2,1),NOD5(1 NODS,MTYPE,NND5),XJ(2,2)		5 5 5 5 5 5 5 5
	INTERPOLATION FUNCTI	ONS (4-NODE ELETE	(TR		Sh
	RP = 1.0 + R				SH
	$\begin{array}{rrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrrr$				SH SH SH SH
2	$H(1) = 0.25 \times DD \times CD$				SH
	H(2) = 0.25*RM*SP				SH

SHP1 H(4) = 0.25 * RP * SM50 C C SHP1 51 LOCAL DERIVATIVES OF INTERPOLATION FUNCTIONS (4-NODE ELEMENT) SHP 1 52 Ĉ SHP 1 53 SHP 1 P(1,1) = 0.25*SP 54 SHP 1 P(1,2) = -P(1,1)55 P(1,3) = -0.25*SMSHP 1 56 SHP 1 57 P(1,4) = -P(1,3)P(2,1) = 0.25 * RPSHP 1 58 P(2.2) = 0.25*RM SHP 1 59 P(2,3) = -P(2,2)SHP 1 60 P(2,4) = -P(2,1)SHP 1 61 SHP1 С 62 č INTERPOLATION FUNCTIONS AND LOCAL DERIVATIVES FOR MIDSIDE HODES SHP1 63 SHPI 64 SHP1 65 IF(NODS.EQ.4) GO TO 50 С SHP1 66 SHP1 67 I = 0 2 I = I + 1 IF (I.GT.NND5) GO TO 40 SHPI 68 SHPI 69 SHP 1 70 NN = NOD5(I) - 471 72 GO TO (5,6,7,8) NN SHP 1 С SHP1 H(5) = 0.50*R2*SP P(1.5) = -R*SP P(2.5) = 0.50*R2 73 5 H(S) SHP1 SHP 1 74 SHP 1 75 $\begin{array}{l} \text{GO TO 2} \\ \text{H(6)} &= 0.50 \text{*RM*S2} \\ \text{P(1,6)} &= -0.50 \text{*S2} \\ \end{array}$ 76 77 SHP1 SHP1 6 H(6) SHP1 78 P(2,6) = -RM*SSHP1 79 SHP1 80 GO TO 2 7 H(7) = 0.50*R2*SM P(1,7) = -R*SM P(2,7) = -0.50*R2 SHP1 81 SHP 1 82 SHP1 83 GO TO 2 SHP 1 84 = 0.50*RP*S2 SHP1 85 8 H(8) $P(1,3) = 0.50 \times S2$ SHP1 86 SHP1 P(2,8) = -RP*S87 SHPI GO TO 2 88 SHP1 89 000 MODIFY INTERPOLATION FUNCTIONS H(1) TO H(4) AND LOCAL VERIVATIVES SHP1 90 SHPI 91 92 SHP1 40 IH = 0SHP 1 41 IH = IH + 1 93 IF(IH.GT.NND5) GO TO 50 SHP 1 94 SHPI 95 IN = NOD5(IH)I1 = IN - 4SHP1 96 I2 = IPERM(I1)SHP1 97 H(I1) = H(I1) - 0.5*H(IN)H(I2) = H(I2) - 0.5*H(IN)SHP 1 98 SHP1 99 SHP1 100 H(IH+4) = H(IN)DO 45 J=1.2 SHP1 101 P(J,I1) = P(J,I1) - 0.5*P(J,IN)P(J,I2) = P(J,I2) - 0.5*P(J,IN)45 P(J,I1) = P(J,I2) - 0.5*P(J,IN)SHP1 102 SHP1 103 SHP1 104 SHP1 105 GO TO 41 C C SHP1 106 EVALUATE THE JACOBIAN MATRIX AT POINT (R.S) SHP1 107 Ē SHP1 108 SHP1 109 50 DO 100 I=1,2 SHP1 110 DO 100 J=1.2 SHP1 111 SUM = 0.0DO 90 K-1.NODS SHP1 112

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90 SUM = SUM + $P(I_*K) \times XY(J_*K)$ SHP1 113 SHP1 114 $100 \times J(I,J) = SUM$ SHP1 115 COMPUTE THE DETERMINANT OF THE JACOBIAN MATRIX AT POINT (R.S) SHP1 116 SHP1 117 SHP1 DETJ = XJ(1.1)* XJ(2.2) - XJ(2.1)* XJ(1.2) 118 DUM = ABS(DETJ)SHP 1 119 IF(DUM.GT.1.0E-8) GO TO 500 SHP 1 120 WRITE (6,3000) NEL SHP 1 121 SHP 1 STOP 122 SHP 1 С 123 3000 FORMAT(///49H **ERROR** ZERO JACOBIAN DETERMINANT FOR ELEMENT, 15) SHP1 124 C SHP1 125 500 RETURN SHP1 126 END SHP 1 127 SUBROUTINE DERIVI (XY, H, P, B, XJ, DETJ, RAD, ITYP2D) DER 1 4 С DER 1 23 C C C **токжжжески какака какака какака какака какака какака какака какака какака кака кака кака кака кака кака кака к** EVALUATION OF THE GLOBAL DERIVATIVE OPERATOR (B) AT A POINT (R.S) DERI FOR A QUADRILATERAL ELEMENT HAVING PLANAR OR AXISYMMETRIC GEOMETRYDERI 4 5 Č 6 7 С DER 1 DIMENSION XY(2,1),H(1),P(2,1),B(2,1),XJ(2.2) COMMON /TODIM / NEL,NODS,MTYPE,NND5 COMMON /WORK / DUM(145),XJI(2,2),WORK(51) 8 DER1 DER1 9 DER1 10 С DER 1 11 С COMPUTE INVERSE OF THE JACOBIAN MATRIX DER1 12 č 13 DER 1 DETJI = 1.0/DETJ DER 1 14 XJI(1.1) = XJ(2.2) * DETJIDER 1 15 XJI(1,2) = -XJ(1,2) * DETJIDER1 16 XJI(2,1) = -XJ(2,1) * DETJIDER 1 17 XJI(2,2) = XJ(1,1)* DETJI DER1 18 DER1 19 С С EVALUATE GLOBAL DERIVATIVE OPERATOR (B-MATRIX) DER1 20 Ĉ DER 1 21 DO 10 K=1.NODS DER 1 22 B(1,K) = XJI(1,1)*P(1,K) + XJI(1,2)*P(2,K)10 B(2,K) = XJI(2,1)*P(1,K) + XJI(2,2)*P(2,K) DER1 23 DER 1 24 С DER 1 25 26 27 DER1 RAD = 1.0IF(ITYP2D.NE.0) GO TO 500 DER 1 DER1 28 000 COMPUTE THE RADIUS AT POINT (R.S) FOR AXISYMMETRIC SOLID DER 1 29 30 DER1 RAD = 0.0DER 1 31 DO 50 K=1,NODS DER 1 32 33 50 RAD = RAD + H(K) * XY(1,K)DER1 DER 1 C 34 IF(RAD.GT.1.0E-8) GO TO 500 DER 1 35 WRITE(6,3000) NEL DER1 36 STOP DER 1 37 С DER1 38 3000 FORMAT(//50H **ERROR** ZERO RODIUS ENCOUNTERED IN ELEMENT HO., 15) DER1 39 С 40 DER1 41 500 RETURN DERI END DER 1 42 SUBROUTINE ADDBAN (A.MAXA.S.LM.NDOF) ADBN 1 ADBN 2 3 ASSEMBLE ELEMENT STIFFNESS INTO COMPACTED GLOBAL STIFF (200) ADBN-4 5 ADBN 6

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С		DIMENSION A(1),MAXA(1),S(1),LM(1) DO 200 J=1,NDOF JJ = LM(J)	ADBN ADBN ADBN ADBN	7 8 9 10
•	100 200	MJ = MH(IJJ) D0 200 I=1,ND0F II = LM(I) IJ = JJ - II IF(IJ) 200,100,100 KK = MJ + IJ LS = (J-1)*ND0F + I A(KK) = A(KK) + S(LS) CONTINUE	ADBN ADBN ADBN ADBN ADBN ADBN ADBN ADBN	11 12 13 14 15 16 17 18 19
ι C		RETURN END SUBROUTINE ADDC (A.SC.LM.IEL.NUMNP)	ADBN ADBN ADDC- ADDC	20 21 22 1 2
Č C C C C		жжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжж	*ADDC ADDC- *ADDC ADDC	3456
С	100	DIMENSION A(1),SC(1),LM(1) DO 100 I=1,IEL II=LM(I) A(II) = A(II) + SC(I)	ADDC ADDC ADDC ADDC ADDC	7 8 9 10 11
C		RETURN END SUBROUTINE COND2 (LM.XX.CL)	ADDC ADDC ADDC CON2-	12 13 14 1
		жжжновжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжжж	*CON2 CON2- *CON2- *CON2 CON2	43456
	-	DIMENSION LM(2,1),XX(2,1),CL(1) COMMON ZONTRE1Z NUMNP,NEG,MODEX,NPAR(10),NG,KBC COMMON ZONTRE2Z KST,NDT,DT,TSTART,TAMB,NPRINT,NTSREF,TIME,KP COMMON ZDIM Z N1,N2,N3,N4,N5,N6,N7,N8,N9,R10,N11,N12,M18,N14,N19 COMMONZONST ZA0,A1,A2,A3,A4,A5,A6,A7,A8,A9,THETA,DELT3,A5PHA,PI, +,R0	CON2- CON2 CON2 5CON2 5CON2- GCON2- CON2-	7 8 9 10 11 12
С		COMMON /WORK / DUM(10),SC(2),WORK(188) COMMON A(1) NEL2 = NPAR(2)	CON2- CON2 CON2 CON2 CON2	13 12 13 14
С		DO 100 N=1,NEL2	CON2 CON2	16 17
C		AXI-SYMMETRIC FREE SURFACE BOUNDARY ELEMENTS	CON2 CON2-	33 34
5	30	XI = XX(1.N) XJ = XX(2.N) WRITE(5.5) XI.XJ.CL(N).G.N FORMAT(50X.4F15.5.110) SC(1) = (2.*XI+XJ)*CL(N)/(6.3*G) SC(2) = (XI+2.*XJ)*CL(N)/(6.3*G)	CON2 CON2 CON2- CON2- CON2- CON2- CON2- CON2-	35 36 37 38 39 40 41
C	100	CALL ADDC(A(N12).SC.LM(1.N).2.NUMNP) CONTINUE	CON2 CON2- CON2 CON2	40 42 48
C		RETURN END	CON2 CON2	50 51

A2-28

r		SUBROUTINE KSTAR(MAXA,XK,C)	KSTAR	1
Č*	skokoka	na ka	KSTAR	3
С С*	oloiolo	THIS SUBROUTINE CALCULATES EFFECTIVE K	KSTAR	
0.0		COMMON /CNTRL1/ NUMNP, NEG, MODEX, NPAR (10), NG, KBC	KSTAR	5
		COMMON/CONST/A0,A1,A2,A3,A4,A5,A6,A7,A8,A9,THETA,DELT⊕,ALPH9,P1,0 +.R0	KSTAR	67
		DIMENSION MAXA(1),XK(1),C(1)	KSTAR	8
		DO 100 I=1.NUMNP	KSTAR	- 18 18
		XK(NNI)=XK(NNI)+C(I)*A0	KSTAR	11
10	0		KSTAR	12
		END	KSTAR	14
c		SUBROUTINE COLSOL (A,V,MAXA,NN,MA,NWA,KKK)	COLS	1
Č		*************************************	KCOLS	3
C		TO SOLVE SIMULTANEOUS EQUATIONS AX=V IN COPE, USING	COLS	4
Č		CUPACIED STURAGE AND COLDIN REDUCTION SCHERE.	COLS	6
Č		A = MATRIX STORED IN COMPACTED FORM	COLS	7
с С		MAXA = VECTOR CUNTAINING ADDRESSES OF DIAGONAL ELEMENTS OF A	COLS	89
Ĉ			COLS	10
L C		FLAG FOR TRIANGOLARIZATION (H=LO) AND/OR STAPLE FORWARD REDUCTION (LY=V) AND BACKSUBSTITUTION (UX=Y)0	COLS	11
č			COLS	13
C r	·	KKK=0 TRIANGULARIZATIUN UNLY KKK=1 TRIANGULARIZATION PLUS SOLUTION		14
č		KKK=2 FORWARD REDUCTION AND BACKSUBSTITUTION ONLY	COLS	16
C				17
č		անցիներին հանձիներին հերհնահերհանդիներին հերհնանդին հերհանդին հերհանդին հերհնանդին հերհնանդին։ Հերհն չչներ հերհ	COLS	19
r		DIMENSION A(NWA),V(1),MAXA(1)		20
L		MA1=NA - 1	COLS	22
r		IF (KKK-2) 100,700,800		23
č		222222222222	COLS	25
C		TRIANGULARIZATION		26
č			COLS	28
	100	IF(NN.ED.1) GO TO 800		29
		IF (A(1)) 80,85,110	COLS	31
	80	WRITE (6,3000) N	COLS	32
	85	WRITE (6,3001) N	COLS	-33 -34
~		STOP	COLS	35
L	110	DO 200 N=2.NN	COLS	37
		KL = MAXA(N) + 1	COLS	38
		KU≢MHXH(N+1) - 1 IF (KU-KL) 200,210,210	COLS	- 39 - 40
2	10	B=0.	COLS	41
L		KN=MAXA(N)	COLS	42
		K=N	COLS	44
		DU 220 KK=KL,KU K=K - 1	COLS	45 46
		ŘI=MAXA(K)	COLS	47
		C=A(KK)/A(K1)	COLS	48

220	B=B + C*A(KK) A(KK)=C A(KN)=A(KN) - B
222	IF (A(KN)) 222,224,226 WRITE (6,3000) N
224	STOP WRITE (6,3001) N
226	STUP MR=MIN0(MA1,NN-N) IE (MR) 200 200,228
228 C	MN = KU - KL + 1
230	DO 240 J=1,MR MJ=MAXA(N+J) + J MNJ=MAXA(N+J+1) - MJ - 1 IF (MNJ) 240,240,230 ND=MIN0(MN,MNJ) C=0. KU=KN + ND IC=MJ - KN
300	DO 300 KK=KL,KU C=C + A(KK)*A(KK+IC)
240	A(KN+IC)=A(KN+IC) - C CONTINUE
200	CONTINUE IF(KKK.EQ.0) RETURN
	FORWARD REDUCTION
700	DO 400 H=2,NN KL=MAXA(N) + 1 KU=MAXA(N+1) - 1 JE (KU=KL) 400 410 410
410	K=N C=0. D0_420_KK=KL.KU
420	K=K - 1 C=C + A(KK)*V(K) V(N)=V(N) + C
400	CONTINUE Go to 800
	BAUK SUBSTITUTION
_ 800	DO 480 N=1.NN K=MAXA(N)
480	V(N)=V(N)/A(K) IF (NH.EQ.1) RETURN N=NN DO 500 L=2.NH KL=MAXA(H) + 1 KU=MAXA(H+1) - 1 LE (H) KL 500 510 510
510	K=N D0_520_KK=KL,KU
520	κ≠κ - 1 V(K)≠V(K) - Α(KK) ×V(N)

COLS	49
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COLS	109
COLS	111

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A2-29

A2-30

500 C C C 3000	N=N - 1 FORMAT STATEMENTS FORMAT(//45H **STOP** STIFFNESS NOT POSITIVE DEFINITE/	COLS COLS COLS COLS COLS	112 113 114 115 116
3001 C	1 10%,27H NEGATIVE PIVOT IN POSITION 14) FORMAT(//33H **STOP** ZERO PIVOT IN POSITION 14) RETURN END	COLS COLS COLS COLS COLS	117 118 119 120 121
C C C C C	SUBROUTINE FORMOC(TTH) жижжиже жижиже жижиже жижие жи	FMQC- FMQC #FMQC- FMQC- #FMQC	1 2 3 4 6
Č	COMMON /CNTRL1/ NUMNP.NEG.MODEX.NPAR(10).NG.KBC COMMON /CNTRL2/ KST.NDT.DT.TSTART.TAMB.NPRINT.NTSREF.TIME.KP COMMON /DIM / N1.N2.N3.N4.N5.N6.N7.N8.N9.N10.N11.N12.N13.N14.N15 COMMON /NBC / NNBC.NBCF.NPTM COMMON /WORK / M1.M2.M3.M4.M5.M6.M7.M8.M9.M10.WORK(190) COMMON A(1) DIMENSION NST(10) EQUIVALENCE (NST(1).M1)	FMQC FMQC FMQC FMQC FMQC FMQC FMQC FMQC	7 8 9 10 11 12 13 14 15
C	REWIND 1 REWIND 2	FMQC FMQC FMQC	16 17 18 19
č	LOOP OVER ALL ELEMENT GROUPS	FMOC	20 21
r	D0 100 NG=1.NEG	FMOC	22
c c	READ (1) MIDEST, NPAR, NST, (A(I), I=1, MIDEST)	FMUC	24
c c	NGR = NPAR(1)	FMOC	26
_ 180	IF(NGR.NE.3) GO TO 100 CALL FLUX2 (A(M1),A(M2),A(M3),A(M4),A(M5),A(N1),A(N2),A(N3),A(N8) 1A(N11),A(N14),NPTM,TTH) CONTINUE	FMQC- FMQC- FMQC- FMQC FMQC	- 28 - 29 - 30 - 45 - 45
	RETURN END SUBROUTINE FLUX2 (LM.XX.CL.SINS.COSS.TEN.FN.NPTS.T.Q.TT.NPTM.TTH) COMMON/CONST /A0.A1.A2.A3.A4.A5.A6.A7.A8.A9.THETA.DELTO.ALFHA.PI.M +.RO	FMQC FMQC FLX2- FLX2- FLX2-	47 48 - 2 - 3
	www.www.www.www.www.www.www.www.www.ww	FLX2 *FLX2 FLX2- *FLX2	34 5 6
C	COMMON /CNTRL1/ NUMNP.NEG.MODEX.NPAR(10).NG.KBC COMMON /CNTRL2/ KST.NDT.DT.TSTART.TAMB.NPRINT.NTSREF.TAME.KP COMMON /NBC // NNBC.NBCF.IDUMMY DIMENSION LM(2,1).XX(2,1).CL(1).SINS(1).CDSS(1).T(1).TEN(NPTM.1). IFN(NPTM.1).NPT5(1).Q(1).TT(1)	FLX2 FLX2 FLX2 FLX2- FLX2- FLX2-	10 11 12 13 14
ι c	NEL3=NPAR(2) ITYP = NPAR(4)	FLX2-	12
10	DO 100 N=1.NEL3 II = LM(1.H) JJ = LM(2.N)	FLX2- FLX2 FLX2 FLX2	22 23 24

FLX2 = CL(N) 25 XL NPT-NPTS(1) FLX2-FLX2-FLX2-26 27 28 NC=1 CALL INTERP(TFN(1,NC),FN(1,NC),NPT,TTH,VX) FLX2-29 IF(NBCF.NE.1) GO TO 21 FLX2-FLX2-VZ=0. 30 GO TO 22 31 FLX2-21 NC = 2 32 FLX2-33 FLX2- 3 NPT=NPTS(2) CALL INTERP(TFN(1,NC),FN(1,NC),NPT,TTH,VZ) -34 FLX2-22 $VN = VX \times COSS(N) + VZ \times SINS(N)$ 35 FLX2 FLX2 FLX2 Ē 59 ē 61 25 XI = XX(1,N)62 XJ = XX(2,N)FLX2 63 VN*XL*(2.*X1+XJ)/6.0 FLX2-Q(II) = Q(II) +64 VN*XL*(XI+2.*XJ)/6.0 FLX2-65 Q(JJ) = Q(JJ) +FLX2 <u>9</u>9 100 CONTINUE С FLX2 100 FLX2 RETURN 101 END FLX2 102 SUBROUTINE INTERP (TEN, EN, NPT, TIME, VAL) INTP 1 INTP 2 3 THIS ROUTINE INTERPOLATES A GIVEN TIME-DEPENDENT FUNCTION TO FIND INTP THE VALUE OF THE FUNCTION (VAL) AT A PARTICULAR TIME POINT (TIME) INTP 4 5 6 INTP 7 DIMENSION TEN(1), FN(1) INTP 8 C INTP q INTP DO 10 N=1.NPT 10 DTIME = TFN(N) - TIME IF(DTIME.GT.0.) GO TO 15 INTP 11 INTP 12 INTE 10 CONTINUE 13 INTP С 14 15 DIFF = TFN(N) - TFN(N-1) VAL = FN(N) - (FN(N) - FN(N-1))*DTIME/DIFF INTP 15 16 17 INTP INTP С INTP 18 RETURN END INTP 19 SUBROUTINE GEFF (Q.C.T.TD.TDD.NUMNP) QEFF-1 0000 QEFF 23 QEFF-FORM THE EFFECTIVE LOAD VECTOR 4 5 Ĉ QEFF 6 7 COMMON/CONST /A0,A1,A2,A3,A4,A5,A6,A7,A8,A9,THETA,DELTA,ALAHA,PI,GQEFF-QEFF-+,RO 8 DIMENSION Q(1),C(1),T(1),TD(1),TDD(1) QEFF-9 £ QEFF 8 DO 10 I=1, NUMNP QEFF 9 QEFF-10 Q(I) = Q(I) + C(I) * (A0*T(I) + A2*TD(I) + A3*TDD(I)) 10 QEFF C 11 RETURN QEFF 12 END. QEFF 13 SUBROUTINE PTEMP (T, TIME, NUMNP) PTEM 1 Ż С PTEM Ī 0000 PTEM-PUNCH THE NODAL DISPLACEMENTS 4 5 ē PTEM 6 7 PTEM DIMENSION T(1) С FTEM 8

A2-31

c	WRITE(3,2000) TIME NCARD = NUMNP/4 + 1 NC = 0	PTEM PTEM PTEM	9 10 11
L	DO 100 I=1.NUMNP.4 NC = NC + 1 IP = I + 3	PTEM PTEM PTEM	12 13 14 15
100 C	IF(NC.EQ.NCARD) IP = NUMNP WRITE(3,2001) (N,T(N),N=I,IP)	PTEM PTEM PTEM	16 17 18
2000 2001	FURMAT(35H NODAL POINT DISPLACEMENT AT TIME = F11.4) FORMAT(4(15.5X,F10.3))	PTEM- PTEM	19 20
C		PTEM	22
	COMMON/CONST /A0,A1,A2,A3,A4,A5,A6,A7,A8,A9,THETA,DELTA ALPHA,PI, +,R0	GCAL CAL	2 3
С	COMMON /CNTRL2/ KST.NDT.DT.TSTART.TAMB.NPRINT.NTSREF.TIME.KP DIMENSION T(1).TT(1).TD(1).TDD(1).P(1).E(1)	CAL CAL CAL	456
Č	T ABOVE HAS VALUES AT TIME + TAU DO 1000 I-1,NUMNP COLOURATE SECOND DEPLYOTING DE V. P. OT TIME+ TOURTIN		78
č	T1=A0*(T(I)-TT(I))-A2*TD(I)-A3*TDD(I)		10 11
	FIND SECOND DERIVATIVE OF V.P. AT TIME+DT	CAL CAL CAL	13 14 14
C	T2=TDD(I) TUD(I)=T2+(T1-T2)/THETA	CAL CAL CAL	15 16 17
Č C	FIND FIRST DERIVATIVE OF V.P. AT TIME+DT	CAL	18 19
C	TD(I) = T3+A6*T2+A7*TDD(I)		20 21 22
C C	FIND V.P. AT TIME+DT T(I)=TT(I)+DT*T3+A8*T2+A9*TDD(I)	CAL CAL CAL	23 24 25
C C	FIND PRESSURE(P) AND SURFACE DISPLACEMENT E	CAL CAL	26 27 28
1000	E(I) =-TD(I)/G CONTINUE DETURN		29 30
500	END SUBROUTINE OUT (T.NUMNP,TIME,KSTEP)	NPBC	93 1
	**************************************		VM 45
с с >	DIMENSION T(1)		6 7 8
r	URITE(6,2000) KSTEP,TIME WRITE(6,2001) (N,T(N),N-1,NUMNP)		9 10
C C	FURMAT STATEMENTS		11 12 13
2009 2001 C	FORMAT(//28H DISPLACEMENT AT TIME STEP =15,2X,7H(TIME =511.4,1H)/ FURMAT(6(16,E14.6))	DUT DUT DUT	19 15 16

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