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### Publication Date

1970-06-01

REPORT NO.  
UC SESM 70-10

STRUCTURES AND MATERIALS RESEARCH  
DEPARTMENT OF CIVIL ENGINEERING

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# COMPUTER PROGRAM FOR CELLULAR STRUCTURES OF ARBITRARY PLAN GEOMETRY

by

K. J. WILLAM

and

A. C. SCORDELIS

Report to the Sponsors: Division of Highways, Department of Public Works, State of California, and the Bureau of Public Roads, Federal Highway Administration, United States Department of Transportation.

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SEPTEMBER 1970

COLLEGE OF ENGINEERING  
OFFICE OF RESEARCH SERVICES  
UNIVERSITY OF CALIFORNIA  
BERKELEY CALIFORNIA

Structures and Materials Research  
Department of Civil Engineering  
Division of Structural Engineering  
and  
Structural Mechanics

UC-SESM Report No. 70-10

COMPUTER PROGRAM FOR CELLULAR STRUCTURES OF  
ARBITRARY PLAN GEOMETRY

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to

the Division of Highways  
Department of Public Works  
State of California  
Under Research Technical Agreement  
No. 13945-14423

and

U.S. Department of Transportation  
Federal Highway Administration  
Bureau of Public Roads

College of Engineering  
Office of Research Services  
University of California  
Berkeley, California

September 1970

ABSTRACT

A computer program is presented for the analysis of cellular structures of constant depth with arbitrary geometry in plan view. The development is based on the finite element method and uses two different element types designed to capture the main behavior of deck and web components. The structure may be subjected to a variety of force and displacement boundary conditions, such as distributed dead and live-loads in addition to concentrated nodal loads and prescribed nodal displacements. The well established direct stiffness method is used for the element assembly. After solving for the unknown nodal displacements and reactions, internal forces are computed at the center and at nodes of deck and web elements selected by the user.

KEY WORDS

Box girder bridges, multi-cell bridges, skew bridges, interchange structures, orthotropic folded plates, anisotropic folded plates, elastic analysis, structural analysis, structural design, finite elements, direct stiffness method.

TABLE OF CONTENTS

	<u>Page</u>
ABSTRACT AND KEY WORDS . . . . .	i
TABLE OF CONTENTS . . . . .	ii
1. INTRODUCTION . . . . .	1
2. METHOD OF ANALYSIS	
2.1 Finite Element Idealization . . . . .	4
2.2 Direct Stiffness Method . . . . .	4
3. DESCRIPTION OF FINITE ELEMENTS	
3.1 Deck Elements . . . . .	6
3.2 Web Elements . . . . .	9
4. DESCRIPTION OF PROGRAM CELL	
4.1 Nature of Program . . . . .	12
4.2 Coordinate Systems and Sign Convention . . . . .	13
4.3 Elastic Analysis . . . . .	16
4.4 Method of Solution . . . . .	17
4.5 Capabilities and Restrictions . . . . .	18
5. PROGRAMMING INFORMATION	
5.1 Program Structure . . . . .	20
5.2 Program Decks . . . . .	20
5.3 File Usage . . . . .	25
5.4 Input Specifications . . . . .	26
5.5 Commentary on Generation Options . . . . .	36
5.6 Output Description . . . . .	40
5.7 General Remarks . . . . .	43

	<u>Page</u>
6. EXAMPLES	
6.1 Example 1 - In Plane Analysis of Skewed Sheet . . . . .	49
6.2 Example 2 - In Plane Analysis of Cantilever . . . . .	51
6.3 Example 3 - Plate Bending Analysis of Rhombic Plate . . . . .	53
6.4 Example 4 - Two Cell Box Girder Bridge on Right Supports . . . . .	56
6.5 Example 5 - Two Cell Box Girder Bridge on Skewed Supports . . . . .	63
6.6 Example 6 - Two Cell Box Girder Highway Branch . . . . .	70
7. ACKNOWLEDGEMENTS . . . . .	77
8. REFERENCES . . . . .	78
APPENDIX A Source Listing of Computer Program CELL . . . . .	A-1
APPENDIX B Listing of Data for Example 5 - Two Cell Box Girder Bridge on Skewed Supports . . . . .	B-1

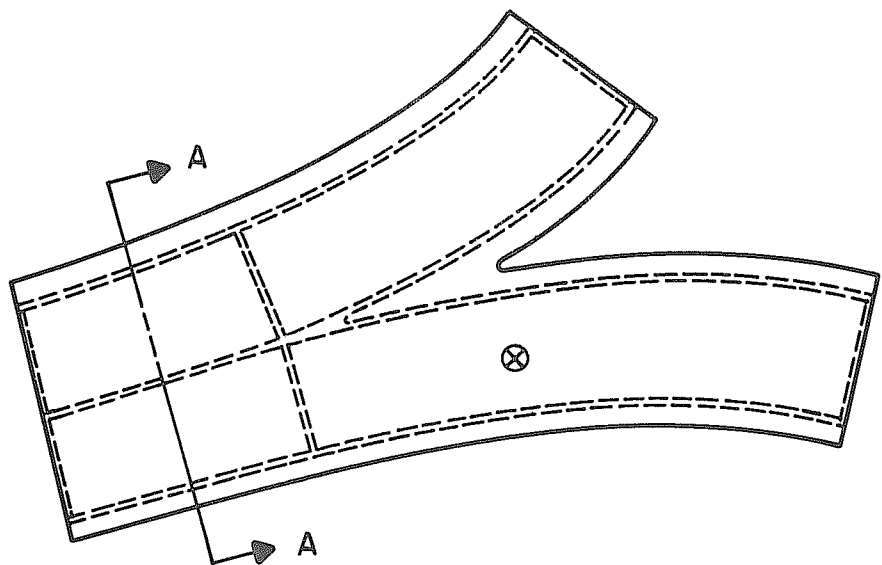
## 1. INTRODUCTION

Cellular systems are used extensively for various types of civil engineering structures, such as box girder bridges, buildings, aircraft and hydraulic structures. A variety of methods have been presented for the analysis of prismatic box girders [1,2,3,4] but an analytical tool for cellular structures with arbitrary geometric configuration has not been introduced yet except within the context of a general shell analysis.

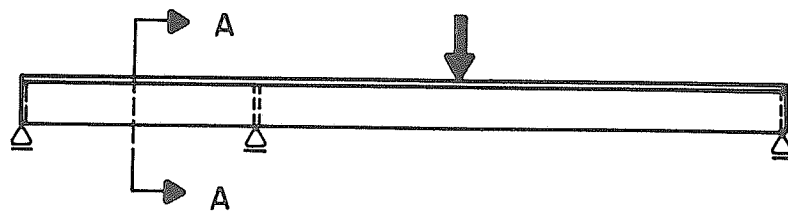
This report describes a finite element computer program for the analysis of cellular structures of constant depth with arbitrary geometry in plan view. The well known finite element method of analysis is ideally suited for computer applications combining versatility and efficiency in an optimum fashion. A variety of special elements have been developed in reference [5] for the analysis of box structures. These elements were assembled in two cellular programs developed to compare efficiency and accuracy. The program using elements without midside nodes, which was identified by the name CELL, has been further developed for general usage as a box girder bridge program and will be presented in this report. Its present version is capable of treating box structures with arbitrary plan view, subjected to any type of loading and boundary conditions. There is still one limitation on the geometric configuration of the cellular structure, it must have vertical web components of constant height. This restriction could be easily removed by changing the input format and by including more general element transformations as all elements can be of either quadrilateral or triangular shape. The structure and input format of

program CELL takes advantage of the simplicity of geometry and topology found in most box girder bridges. Cases with completely arbitrary geometry should be analyzed by other less convenient and less efficient shell programs. A typical box structure, which can be analyzed by program CELL is illustrated in Fig. 1. All structural components exhibit in plane and flexural stiffness which are accounted for in the analysis.

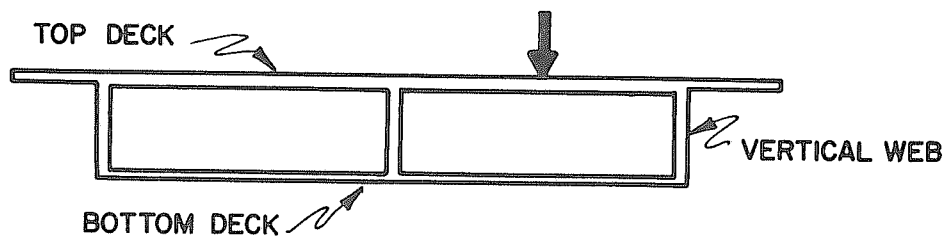




PLAN VIEW



ELEVATION



SECTION A - A

FIG.1 BOX GIRDER BRIDGE EXAMPLE OF A CELLULAR STRUCTURE

## 2. METHOD OF ANALYSIS

The finite element method of analysis is summarized very briefly since it has been described extensively in numerous publications.

### 2.1 Finite Element Idealization

The structure is idealized by means of general quadrilateral or triangular elements which are interconnected at the corner nodes by 5 degrees of freedom,  $U$ ,  $V$ ,  $W$ ,  $\theta_X$  and  $\theta_Y$ , omitting the global rotation  $\theta_Z$  from the general 6 degrees of freedom system used in shell analysis. In reference [5] results are presented for the analysis of single cell box girders which are discretized by a variety of nodal configurations. The 5 degrees of freedom system appeared the most promising tool for the analysis of box structures both in terms of accuracy and efficiency. Different types of displacement models and mixed models are utilized to represent the in plane and plate bending behavior of the deck and web components. Their nodal configurations and formation is briefly described in Chapter 3 while a detailed derivation is given in reference [5].

### 2.2 Direct Stiffness Method

Compatibility of the kinematic nodal quantities is enforced by the well known direct stiffness method used for the assembly of elements. As only 5 degrees of freedom exist complete continuity of displacements cannot be maintained along the element interfaces. Therefore, a bound of the results cannot be guaranteed from a theoretical point of view but convergence is assured as all elements satisfy the constant energy criterion.

The resulting system of simultaneous equations is solved for the unknown nodal displacements taking advantage of the symmetric, positive definite, and banded character of the structural stiffness matrix. Sparsity is accounted for to increase the efficiency of the direct solution scheme.

Finally, the internal forces are determined from the resulting element displacement field by using the standard stress displacement relationships.

### 3. DESCRIPTION OF FINITE ELEMENTS

A variety of finite elements are used to account for the different behavior of the two main structural components, the horizontal deck and the vertical webs. They have been described and tested in reference [5]. All elements satisfy the constant energy criterion and maintain full continuity as long as adjacent elements are coplanar.

#### 3.1 Deck Elements

The deck is idealized by quadrilateral or triangular elements having 5 DOF per node. As the local coordinates of the deck elements  $x, y, z$  coincide with the global coordinates of the box structure,  $X, Y, Z$ , no transformation is required, a rearrangement of the in-plane and plate bending contributions suffices to form the global element stiffness  $\underline{K}_D$  ( $20 \times 20$ ).

##### a) Plane Stress Behavior.

The in-plane action is represented by the mixed model Q8D11 having 8 external DOF and 3 internal ones. This quadrilateral element degenerates to the constant strain triangle by letting the coordinates of the first and fourth node coincide. The nodal configurations of both, the quadrilateral element Q8D11 and the triangular element, CST, are illustrated in Fig. 2.

The mixed model is constructed using separate expansions for the displacement and the strainfield. The variation of the components  $u$  and  $v$  of the displacement field is approximated by the standard bilinear expansion. The normal strain components  $\epsilon_x$  and  $\epsilon_y$  are derived from the displacement field by the linearized strain displacement relationships. The shear-strain variation is assumed to be constant.

It has been shown in reference [5] that this choice for the field variables provides a stiffness matrix which yields more flexible and better results than the associated displacement model. With the help of an extension of the Hu-Washizu Variational Principle the stiffness is formed using a two point Gaussian numerical integration formula. The three internal DOF, the  $u$  and  $v$  components of the center node and the generalized coordinate associated with the constant shear strain variation are eliminated by an internal static condensation process.

Due to the lack of invariance of the stiffness matrix with regard to coordinate rotations, this element is formed in the convected coordinate system  $\bar{x}$ ,  $\bar{y}$ , illustrated in Fig. 2, with the  $\bar{x}$  axis being defined by nodes 1 and 2. The  $\bar{y}$  axis is now fully defined in order to provide a right hand Cartesian coordinate system. The element stiffness is generated in this convected coordinate system and subsequently transformed into the local  $x$ ,  $y$  coordinates.

b) Plate Bending Behavior.

The flexural action is represented by the Q19 quadrilateral two-way bending element having 12 external DOF and 7 internal ones. This quadrilateral displacement model has been described in detail in reference [6]. It degenerates to the LCCT-9 triangular plate bending element if the first and the last node in the nodal array coincide. The nodal configurations of the quadrilateral element Q19 and the triangular element LCCT-9 are illustrated in Fig. 3.

The displacement model Q19 is assembled from four LCCT-11 triangles with 11 bending DOF at 5 nodal points. The 7 internal DOF are eliminated by an internal static condensation process. The linear curvature compatible triangle, denoted by LCCT, uses complete

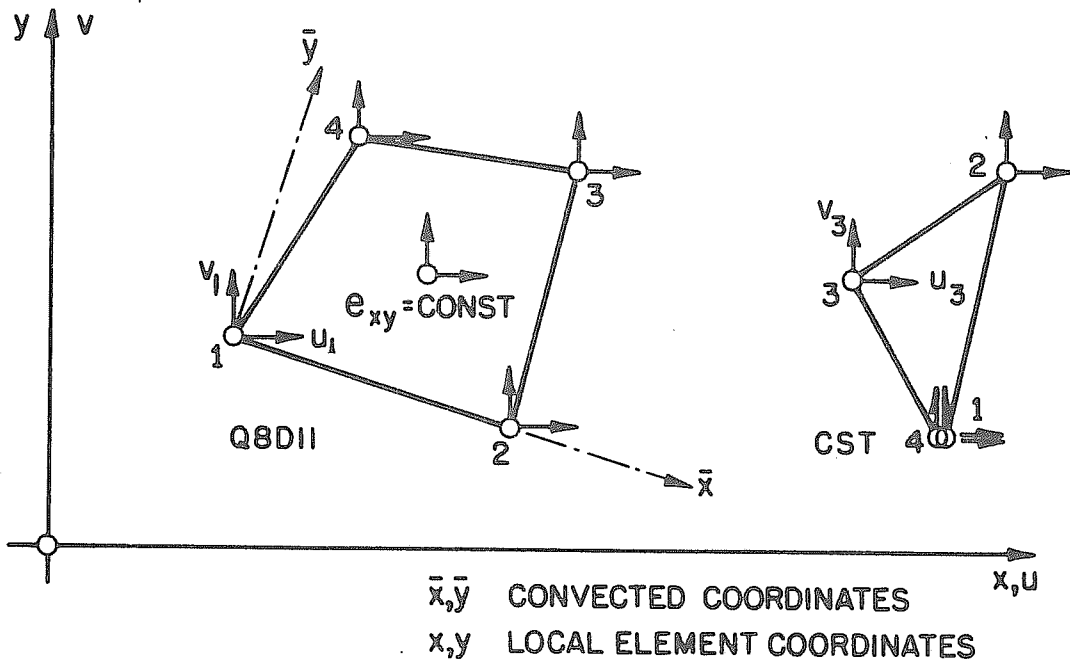


FIG. 2 NODAL CONFIGURATION OF PLANE STRESS ELEMENTS Q8DII AND CST

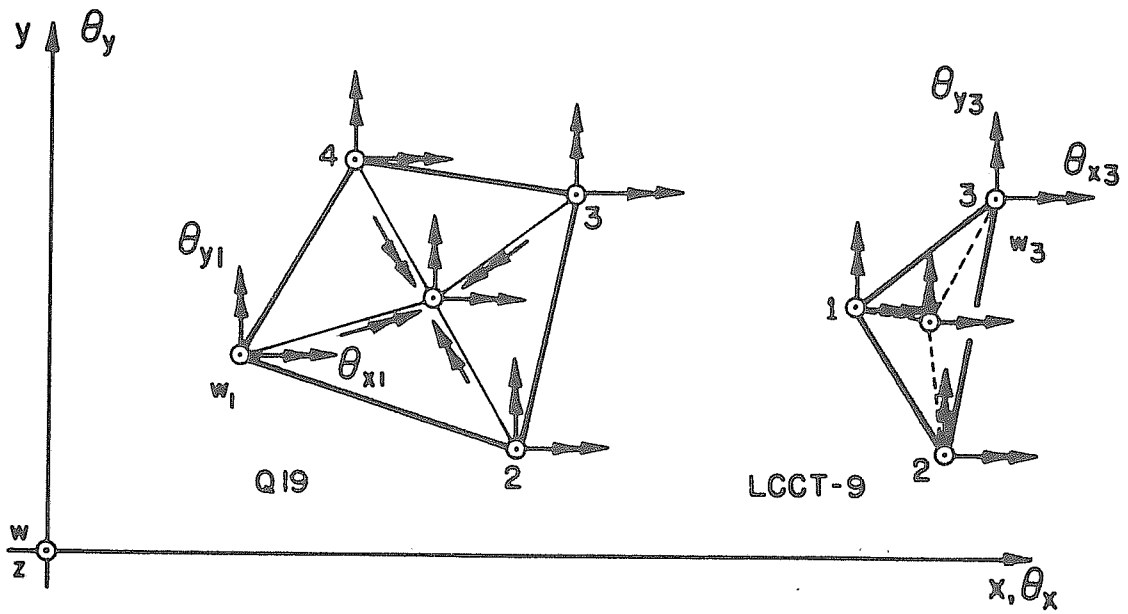


FIG. 3 NODAL CONFIGURATION OF PLATE BENDING ELEMENTS Q19 AND LCCT-9

cubic expansions for the transverse displacement field  $w$  over each of the triangular subregions. Enforcing continuity between the subregions yields the element LCCT-12 having 12 DOF with a quadratic variation of normal slopes along the element edges. Note that second derivatives are piecewise continuous within each subregion of the triangle. This element degenerates to the LCCT-11, LCCT-10 or LCCT-9 simply by using kinematic constraint conditions to enforce a linear variation of normal slopes along the edges.

### 3.2 Web Elements

The web is idealized by quadrilateral elements having 5 DOF per node. They are assumed to lie in a vertical plane with arbitrary orientation with respect to the global X-Y coordinates. The transformation involves simple rotations of the local nodal quantities  $u$ ,  $w$  and  $\theta_x$ ,  $\omega$  into the global DOF  $U$ ,  $V$  and  $\theta_X$ ,  $\theta_Y$ , and after rearrangement of the in-plane and plate bending contributions the global element stiffness  $\underline{K}_w$  ( $20 \times 20$ ) is developed.

#### a) Plane Stress Behavior.

The in-plane action is represented by the displacement model QUSP12 having 12 fundamental DOF. The nodal configuration of this element which has been described in detail in reference [5] is illustrated in Fig. 4. This displacement model is constructed using different expansion for the  $u$  and  $v$  components of the displacement field in order to capture the beam behavior of the web components by single elements over the depth. A bilinear expansion for  $u$  and  $v$  assures that the element contains rigid body modes and constant energy states. Moreover, a cubic variation in the  $x$ -direction of the  $v$  component

can be described introducing the nodal rotations  $\omega = \frac{\partial v}{\partial x}$  normal to the plane of the element. This makes it possible to represent the beam behavior by a single plane stress element. Due to the different variation of the displacement components  $u$  and  $v$  the stiffness depends on the orientation of the coordinate system and lacks invariance. Hence, as a general rule governing the numbering of nodal points the edge 1-2 and 4-3 have to be approximately parallel to the local  $x$ -axis. Since the present program is restricted to box structures of constant depth the spar elements remain rectangular. Hence, no problem arises choosing a convected coordinate system to minimize the lack of invariance.

b) Plate Bending Behavior.

The flexural action is represented by the element ONEW, a quadrilateral one-way bending element having 8 fundamental DOF. The nodal configuration of this beam type of element is illustrated in Fig. 5. This element is constructed using cubic beam type expansions to approximate the flexural action by one-way bending in the  $y$ -direction between nodes 1-4 and 2-3. Due to the assumption of one-way bending there is no coupling between nodes 1-4 and 2-3. The width  $B$  of the equivalent beams is computed as average of the  $x$  differences of nodes 1-2 and 3-4, and the spans equal the difference of  $y$ -coordinates between nodes 1-4 and 2-3.



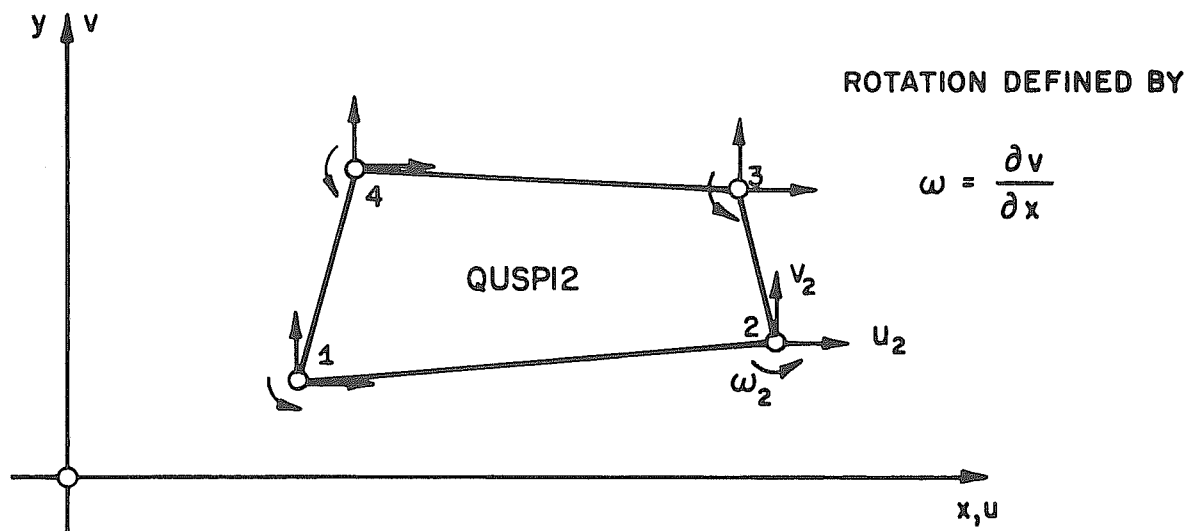


FIG. 4 NODAL CONFIGURATION OF PLANE STRESS "SPAR" ELEMENT QUSPI2

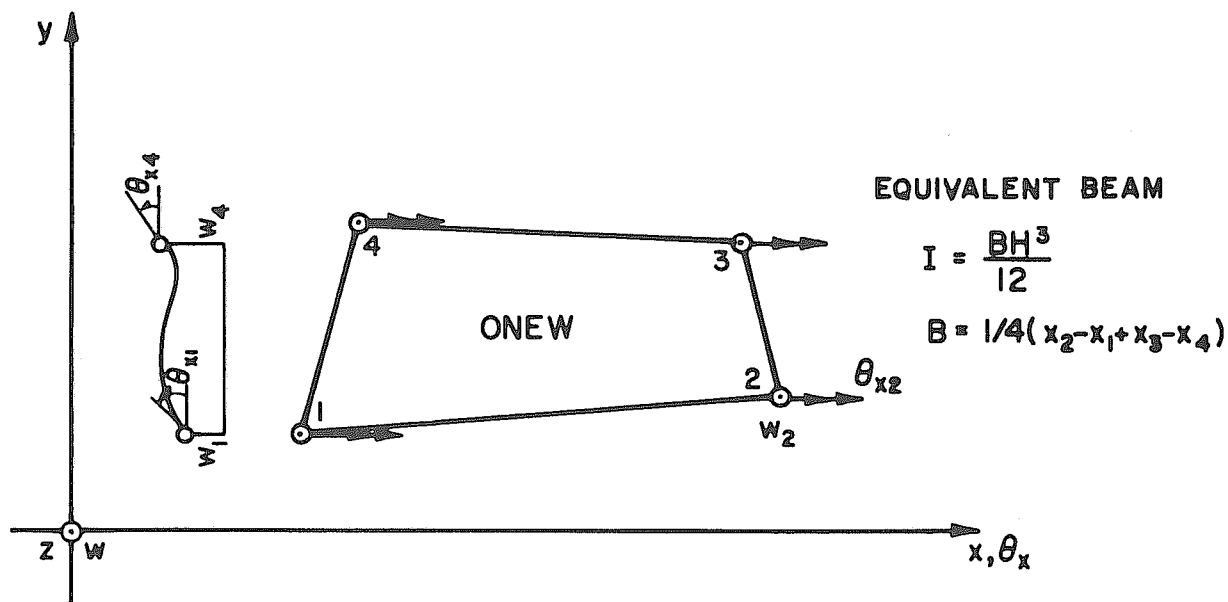


FIG. 5 NODAL CONFIGURATION OF ONE WAY PLATE BENDING ELEMENT ONEW

#### 4. DESCRIPTION OF PROGRAM CELL

##### 4.1 Nature of Program

This program is capable of analyzing cellular structures consisting of top and bottom decks which are interconnected by vertical web elements. At this stage the depth of these box structures is assumed constant. Such a cellular system is illustrated in Fig. 1.

The structure is idealized by different types of finite elements which are interconnected at the nodes by the 5 global degrees of freedom  $U, V, W, \theta_X, \theta_Y$ . Advantage is taken of the special behavior of these box structures by omitting  $\theta_Z$  from the 6 DOF system commonly used for the finite element analysis of arbitrary shells. All elements exhibit multi-component action possessing both in-plane and flexural stiffness. While the flexural action of deck elements is represented by two-way bending elements that of vertical web elements has to be idealized by one-way bending due to the lack of the  $\theta_Z$  DOF. The various types of elements which may be of arbitrary quadrilateral or triangular shapes have been described in Chapter 3.

The elastic material constitution of each element can be specified in the form of an orthotropic stress-strain law or equivalent anisotropic laws relating stress resultants and moments to strains and curvatures. This makes it possible, if desired, to eliminate bending contributions or to consider equivalent material laws for stiffened plate structures. The thickness is assumed constant over each element. A more detailed description of the anisotropic elastic analysis is given in Section 4.3.

The structure can be subjected to a variety of loading conditions such as concentrated nodal loads, live loads uniformly

distributed over the area of each top deck element and dead loads which are all determined using the tributary area concept. Arbitrary displacement boundary conditions can be imposed along the global DOF of each node and in any skewed direction within the X-Y plane.

The input is so arranged that the problem regarding mesh layout and nodal coordinates is fully defined by describing the top deck. Hence, the data is arranged similarly to a typical plate bending or plane stress program. Additional information is only needed for the position and location of vertical web elements. The output of resulting nodal displacements, nodal forces and internal forces is so arranged that the static quantity of the top deck is output together with its corresponding value in the bottom deck. A variety of output options have been incorporated to provide the user with enough flexibility to print out particular quantities at selected points of interest.

#### 4.2 Coordinate System and Sign Conventions

The cellular structure is referred to a global right-handed Cartesian system X, Y, Z, with X-Y lying in the plane of the top deck, as illustrated in Fig. 6. U, V, W are linear nodal displacements which are positive in the global directions X, Y, Z.  $\theta_X$ ,  $\theta_Y$  are nodal rotations about the X, Y axes whose positive direction are given by the right hand rule. Each element is formed in a local right handed Cartesian system x, y, z with x, y lying in the plane of the element as illustrated in Fig. 6. For the deck elements only, the global and local systems are identical. The sign convention of the internal forces is referred to these local coordinates and is indicated in Fig. 7.

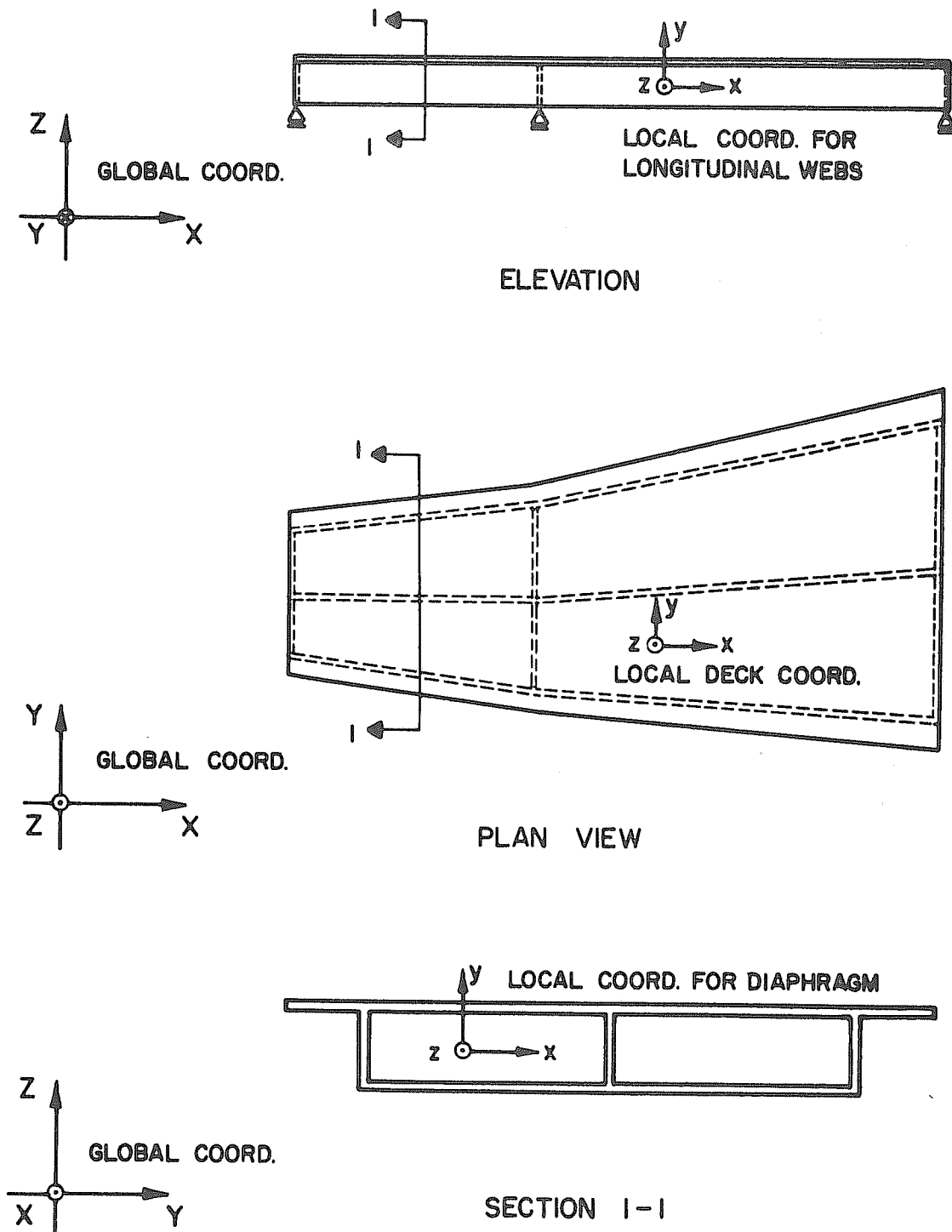
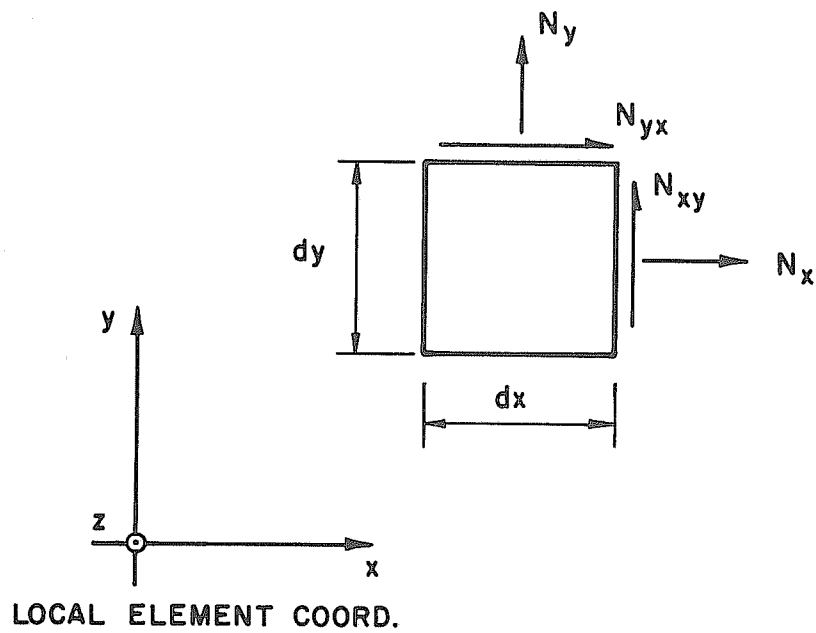
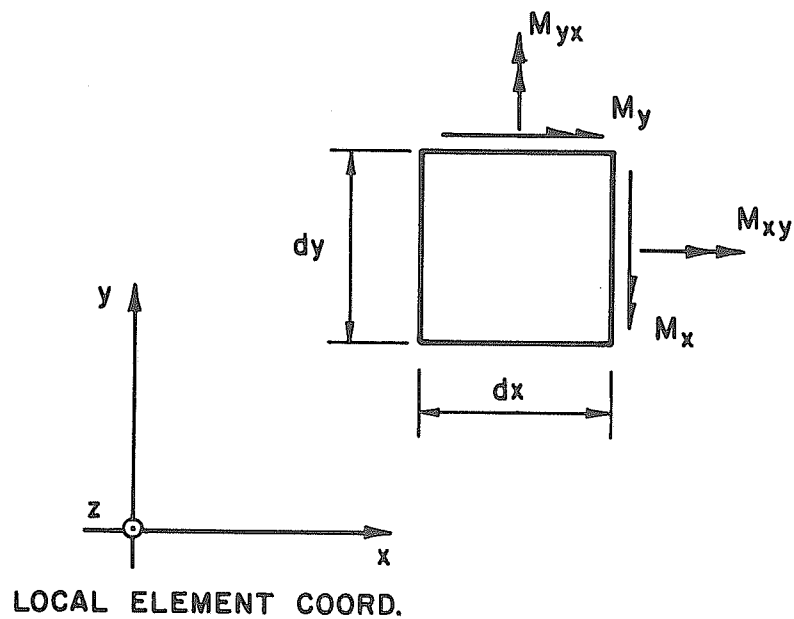


FIG. 6 SIGN CONVENTIONS OF LOCAL AND GLOBAL COORD. SYSTEMS



### IN PLANE STRESS RESULTANTS



### PLATE BENDING MOMENT RESULTANTS

FIG. 7 SIGN CONVENTION OF INTERNAL FORCES

### 4.3 Elastic Analysis

The material properties can be specified at two levels, a) a plane stress law relating stresses to strains, and b) a law relating internal forces to strains and curvatures.

#### a) Orthotropic Plane Stress Material.

The principal axes of orthotropy for each element are denoted by  $\bar{x}_1, \bar{y}_1, \bar{z}_1$  where  $\bar{x}_1$  forms an angle  $\varphi$  with the local element x-axis. The orthotropic stress-strain relation refers to these principal axes and is defined by

$$\begin{Bmatrix} \bar{\tau}_{xx} \\ \bar{\tau}_{yy} \\ \bar{\tau}_{xy} \end{Bmatrix} = \begin{bmatrix} E_1/\chi & E_1 \nu_{12}/\chi & 0 \\ E_2 \nu_{21}/\chi & E_2/\chi & 0 \\ 0 & 0 & G_{12} \end{bmatrix} \begin{Bmatrix} \bar{\epsilon}_{xx} \\ \bar{\epsilon}_{yy} \\ \bar{\gamma}_{xy} \end{Bmatrix}$$

$E_1, E_2$

Elastic moduli in the  $\bar{x}$  and  $\bar{y}$  direction.

$\nu_{12}, \nu_{21}$

Poisson's ratio with  $E_1 \nu_{12} = E_2 \nu_{21}$  for symmetry.  $\nu_{12}$  is defined as the ratio of the strain in the  $\bar{x}$ -direction to the strain in the  $\bar{y}$ -direction due to a uniaxial stress in the  $\bar{y}$ -direction.

$\chi = 1 - \nu_{12} \nu_{21}$

Denominator in material law.

$G_{12}$

Shear modulus.

The corresponding material law in the x, y coordinates requires a tensor transformation eliminating zero coefficients if  $\varphi \neq 0$ . For isotropic material the constants degenerate to  $E_1 = E_2 = E$ ,

$\nu_{12} = \nu_{21} = \nu$ ,  $G_{12} = G$  and the angle  $\varphi$  is not needed. The in-plane stress resultants strain relation is obtained by multiplying this matrix by h, where h is the plate thickness. The moment-curvature

relation is obtained by multiplying this matrix by  $\frac{h^3}{12}$ .

b) Anisotropic Stress Resultant-Strain and Moment-Curvature Relations.

In order to increase the versatility of the program direct access is available to the material law relating stress resultants to strains and moments to curvatures. This capability makes it possible to take account for instance of stiffeners arranged along two skewed directions or of various levels of flexural action, such as two-way bending, one-way bending or no bending action altogether.

In-plane stress resultants are related to the strains by

$$\underline{N} = \underline{DS} \underline{\epsilon}$$

$$\begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix} = \begin{bmatrix} DS(1) & DS(4) & DS(5) \\ & DS(2) & DS(6) \\ \text{Sym.} & & DS(3) \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \gamma_{xy} \end{Bmatrix}$$

Internal moments are related to the curvatures by

$$\underline{M} = \underline{DM} \underline{\kappa}$$

$$\begin{Bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{Bmatrix} = \begin{bmatrix} DM(1,1) & DM(1,2) & DM(1,3) \\ & DM(2,2) & DM(2,3) \\ \text{Sym.} & & DM(3,3) \end{bmatrix} \begin{Bmatrix} \kappa_{xx} \\ \kappa_{yy} \\ \kappa_{xy} \end{Bmatrix}$$

The coefficients of the material matrices  $\underline{DS}$  and  $\underline{DM}$  can be determined either by integration of the associated plane stress material law over the thickness as it is done in a) or by performing appropriate experiments to evaluate these coefficients.

#### 4.4 Method of Solution

The solution is based on the direct stiffness method which has been described extensively in the literature. The unknown kinematic

quantities are the 5 nodal displacements  $U, V, W, \theta_X, \theta_Y$ . They are matched for all elements meeting at a node to enforce interelement continuity. Full conformity is achieved between all coplanar elements. At junctures of deck and web elements full continuity of deformations cannot be maintained. Due to the different functional variation of in-plane deck displacements and transverse web displacements, conformity of displacement is violated in the X-Y plane. Due to the one-way bending representation of the flexural web action no slope continuity can be enforced except for  $\theta_X$  at the nodes. It has been shown in reference [5] that this lack of conformity is of minor influence. The numerical results of the proposed finite element analysis tend to remain on the stiff side which implies that the displacements obtained are smaller than the true values in the continuum. But from a theoretical point of view one cannot claim an upper bound to the total energy due to the lack of continuity of displacements and slopes and due to the use of mixed models in the finite element development.

#### 4.5 Capabilities and Restrictions

The program in its present form imposes the following restrictions onto the geometry, material and the discretization of the problem considered.

##### a) Geometry of Box Structure.

The geometry of the cellular structure is restricted to box structures of constant depth and vertical webs interconnecting the horizontal top and bottom deck. The geometry can be arbitrary in plan view.



b) Material Constitution.

The material properties are specified for each element type. An elastic anisotropic material law describes the constitution within each element. Similar to the constant element thickness the material properties are assumed constant within each element domain.

c) Programming Limitations.

There are a few restrictions on the discretization of the cellular structure by finite elements which originate in the programming.

First, the vertical web is idealized by a single element over the whole depth.

Second, there is a restriction on the largest bandwidth, 130, and hence the connectivity of the element layout. This limitation can be eliminated by incorporating into the direct block solution technique a capability which allows the bandwidth to be larger than the block length.

Third, there is an additional restriction on the largest number of DOF, 200, which may be affected by kinematic constraints.

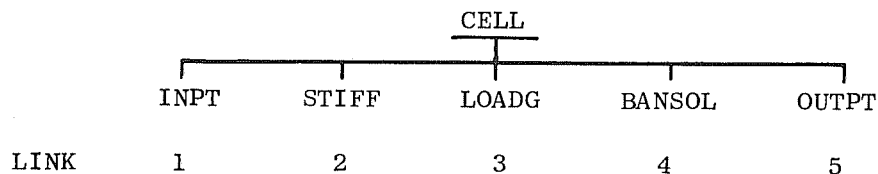
No restrictions on the number of elements, number of nodal points, number of material types etc. are imposed since the program features dynamic storage allocation coupled with an automatic field length reduction. This capability adjusts the field length automatically during execution to the current storage requirements. An explicit formula for the hand calculation of the required field length is given in Section 5.7. There is no limitation on the number of load cases which can be treated without repeated decomposition of the stiffness matrix.

## 5. PROGRAMMING INFORMATION

The program is written in FORTRAN IV and has been tested on the CDC 6400 at the Berkeley Computer Center.

### 5.1 Program Structure

The program uses the Overlay feature. The main program CELL is in permanent residence while 5 links of primary level are called consecutively into residence from the main program CELL. Schematically it can be shown as follows:



### 5.2 Program Decks

The program contains the following decks which need not be in sequence within each link as long as they are contained in the appropriate link.

#### MAIN PROGRAM

PROGRAM CELL

SUBROUTINE FL (in COMPASS)

CELL monitors the calling sequence of the primary overlays, links 1 to 5. FL has two purposes: If called LWA(N) it retrieves the last word address N of the program during execution, and if called RFL(N) it resets the field length to N. This program is not written in standard FORTRAN IV, but its equivalent should be available at any computer center. Otherwise a fixed amount of storage has to be calculated by hand using formulas given in Section 5.7. This storage

has to be reserved in the area of blank COMMON.

LINK 1

PROGRAM INPT

SUBROUTINE INPU

SUBROUTINE FORMC

SUBROUTINE Q8D11

SUBROUTINE QUSP12

SUBROUTINE SPLATE

SUBROUTINE SLCCT

SUBROUTINE SONEW

SUBROUTINE DECK

SUBROUTINE WEB

INPT reads in the input control, computes the required field length for this link and resets it if necessary.

INPU reads in the data, generates the finite element mesh, the nodal coordinates, the material properties and forms the element stiffnesses for both deck and web elements in the global coordinate system.

FORMC determines the anisotropic plane stress material law in the element coordinates from given orthotropic constants which are specified in their principal directions.

Q8D11 computes the in-plane stiffness of the deck elements Q8D11 (8x8) in local coordinates.

QUSP12 computes the in-plane stiffness of the web elements QUSP12 (12x12) in local coordinates.

SPLATE assembles the plate bending stiffness for the quadrilateral deck element Q19 (12x12) in local coordinates.

SLCCT computes the plate bending stiffness of the triangular subelement LCCT-9 (9×9) or LCCT-11 (11×11).

SONEW computes the one-way plate bending stiffness for the web element ONEW (8×8) in local coordinates.

DECK assembles the in-plane and plate bending stiffness into the global deck element (20×20).

WEB assembles the in-plane and plate bending stiffness into the global web element (20×20) after transforming it into global coordinates.

#### LINK 2

PROGRAM STIFF

SUBROUTINE BIGK

STIFF computes the required field length for this link and resets it if necessary.

BIGK assembles the global stiffness matrices of the deck and web elements into the structural stiffness matrix in block form and imposes displacement boundary conditions on the structural level.

#### LINK 3

PROGRAM LOADG

SUBROUTINE LOAD

LOADG computes the required field length for this link and resets it if necessary.

LOAD reads in the loading information for the concentrated nodal loads, the live loads and the dead loads. In addition, non-zero displacement boundary conditions may be input in the global directions which are converted into equivalent loads.

LINK 4

PROGRAM BANSOL

SUBROUTINE BAND

BANSOL computes the required field length for this link and resets it if necessary.

BAND has two entries: First, the stiffness matrix is decomposed block by block, a single load vector is reduced with the total vector in core and is overwritten after back substitution by the total solution. Second, with the reduced stiffness available from a previous load case each load vector is reduced separately and overwritten by the corresponding solution after back substitution. Two blocks of the stiffness matrix and the total load vector need to be in core at a time. A block is defined by NBLKL rows and IBANDW columns.

IBANDW equals the bandwidth which depends on the connectivity of the finite element mesh layout and NBLKL is the block length. The block length NBLKL can be specified on the input control card. Otherwise it is calculated from the maximum available core in LINK 2 or LINK 4, the smaller of which determines the length of NBLKL:

$$\text{LINK 2} \quad \text{NBLKL} = (120,000 \text{ B} - 4*\text{NUMEL} - 2*\text{NBEAM}) / (2*\text{IBANDW})$$

$$\text{LINK 4} \quad \text{NBLKL} = (120,000 \text{ B} - \text{NEQ}) / (2*\text{IBANDW} + 1)$$

where B indicates the octal base.

This out of core symmetric band solver takes advantage of variable bandwidth with 45 degrees shadows and accounts for zeros within the band. These small alterations of the standard Gauss algorithm for symmetric matrices without pivoting increase tremendously the efficiency of the algorithm in combination with the use of single

subscripted arrays. Note that only two files are needed since the total load vector is in core at a time leading to considerable savings in peripheral processing. The storage requirement remains the same since the buffer requirement for two additional tapes offsets eventual reductions by treating the load vector block by block. The present version imposes the following limitation upon the shape of each block:

$$IBANDW \leq NBLKL$$

This limitation can be removed introducing two additional files for temporary storage and requires a substantial change in the element assembly.

#### LINK 5

PROGRAM OUTPT

SUBROUTINE OTPT

SUBROUTINE PSD11

SUBROUTINE PLSP12

SUBROUTINE FPLATE

SUBROUTINE FLCCT

SUBROUTINE FONEW

OUTPT reads in the output control, computes the required field length for this link and resets it if necessary.

OTPT outputs first the global displacement components at each node of the top and bottom deck. Then the nodal forces and reactions and the internal forces at the center of deck elements are computed and printed if the appropriate output controls are activated. Furthermore, the internal forces are computed and averaged at NSTO nodes

of the deck elements and web elements. Their element contributions to each node and averaged nodal values are output separately for both types of structural components together with their principal values.

PSD11 determines the in-plane stress resultants in local coordinates at the four nodes and at the centroid of the plane stress element Q8D11.

PLSP12 determines the in-plane stress resultants in local coordinates at the four nodes and at the centroid of the spar element QUSP12.

FPLATE determines the moments in local coordinates at the four nodes and at the centroid of the plate bending element Q19.

FLCCT determines the curvatures at the nodes of each triangular plate bending element LCCT.

FONEW determines the transverse moments  $M_y$  in local coordinates at the four nodes of the one-way bending element ONEW.

### 5.3 File Usage

FORTRAN logical units 1, 3, 8 are used as binary tapes for temporary storage. File unit 7 is available for binary punched output.

On-line input-output is done via FORTRAN statements READ n, list and PRINT n, list. On the CDC 6400 these conventions are used instead of the standard IBM commands READ (i,n) list and WRITE (i,n) list.

#### 5.4 Input Specifications

The input data is key punched on cards as specified below. The sequential order of the input cards must be strictly adhered to and consistent units must be used throughout.

##### INPUT DATA FOR IDEALIZATION

1. Start Card for Problem (A6)

Col. 1 to 5 - CHECK, each problem has to begin with this card with the word "START" in the first 5 columns which initiates the input of each new problem.

2. Title Card (13A6)

Col. 1 to 78 - HED (13), alphanumeric information to identify problem.

3. Input Control Card (6I4, F10.2)

Col. 1 to 4 - NUMEL, number of elements in top deck

Col. 5 to 8 - NUMNP, number of nodal points in deck

Col. 9 to 12 - NBEAM, number of vertical web elements

Col. 13 to 16 - NMAT, number of different material types which include the element thickness.

Col. 17 to 20 - NUMBC, total number of top and bottom deck nodes with specified kinematic constraints; these must be imposed separately for top and bottom deck nodes.

Col. 21 to 24 - NBLKL, specified block length of structural stiffness matrix. If left blank the block length is computed internally using the current bandwidth and the maximum available core storage of 140000B (B indicates octal base) to reduce the number of



blocks, and hence the tape handling.

Col. 25 to 34 - AH, height of the box structure; this is assumed constant for entire structure.

4. Deck Element Array (10I4)

Col. 1 to 4 - L, element number.

Col. 5 to 20 - NP(4) element nodal array, must be listed counter-clockwise (viewed from above).

Col. 21 to 24 - MA, material number of top deck element.

Col. 25 to 28 - MB, material number of bottom deck element.

One way generation is activated if previous element numbers have been omitted, in which case following further data must then be given.

Col. 29 to 32 - NDIF, difference in corresponding nodal numbers of consecutive elements. If it is left blank NDIF is set to 1 for the one way generation.

If two way generation is desired the following data must be added.

Col. 33 to 36 - MOD, modular difference in the element number of adjacent element strings.

Col. 37 to 40 - NLIM, number up to which two way generation is to be activated.

5. Web Element Array (7I4)

Col. 1 to 4 - L, element number

Col. 5 to 12 - NBP(2), element nodal array, list nodes in increasing numerical order.

Col. 13 to 16 - MC, material number.

One way generation is activated if previous elements are omitted, in which case the following further data must then be given.

Col. 17 to 20 - NDIF, difference in corresponding nodal numbers of consecutive elements. If it is left blank NDIF is set to 1 for the one way generation.

If two way generation is desired the following data must be added.

Col. 21 to 24 - MOD, modular difference in the element number in adjacent element strings.

Col. 25 to 28 - NLIM, element number up to which two way generation is to be activated.

6. Nodal Coordinate Array (I4, 2F10.2, 2I4)

Col. 1 to 4 - N, nodal point

Col. 5 to 14 - XORD(N), global x-coordinate of node N

Col. 15 to 24 - YORD(N), global y-coordinate of node N

One way generation is activated if previous nodes are omitted, in which case NDIF = 1 in all cases.

If two way generation is desired the following data must be added.

Col. 25 to 28 - MOD, modular difference in the nodal numbers of adjacent nodal strings.

Col. 29 to 32 - NLIM, number of node up to which two way generation is activated.

Note, for a detailed description of the generation options with some examples, see Section 5.5.

7. Material Array (I4, 6F10.2)

The element material properties must be input totally as a constitutive law relating stresses to strains or as a law relating in plane stress resultants and moments to strains and curvatures.

- a) An elastic orthotropic material law is specified in the local  $\bar{x}_1 - \bar{y}_1$  frame of the principal directions of orthotropy. It relates stresses to strains for each material type of the elements and incorporates the element thickness.

Col. 1 to 4 - N, material number, must be nonzero and ordered sequentially.

Col. 5 to 14 - E1(N), elastic modulus in principal direction of orthotropy  $\bar{x}_1$ .

Col. 15 to 24 - E2(N), elastic modulus in principal direction of orthotropy  $\bar{x}_2$ .

Col. 25 to 34 - G12(N), shear modulus

Col. 35 to 44 - PR(N), mean Poisson's ratio  $\nu = \sqrt{\nu_{12} \nu_{21}}$

Col. 45 to 54 - ANG(N), angle in degrees between local element x-coordinate axis and principal axis of orthotropy  $\bar{x}_1$ . Use right hand rule for + direction.

Col. 55 to 64 - TH(N), thickness of element assumed to be uniform over the element.

To increase the versatility of the program an alternative input of the material constitution can be specified for each element type.

- b) An elastic anisotropic material law relating moments to curvatures and in plane stress resultants to strains is specified for each material type of the elements in the local x,y element coordinate system.

First set of cards describes plate bending action (I4, 6F10.2).

Col. 1 to 4 - N, normally set to material number is now always set to zero to activate input of material properties in the form 7b instead of 7a. Materials must be ordered sequentially.

Col. 5 to 14 - DM(I,1,1), element 1,1 of (3 × 3) material law I.

Col. 15 to 24 - DM(I,2,2), element 2,2 of (3 × 3) material law I.

Col. 25 to 34 - DM(I,3,3), element 3,3 of (3 × 3) material law I.

Col. 35 to 44 - DM(I,1,2), element 1,2 of (3 × 3) material law I.

Col. 45 to 54 - DM(I,1,3), element 1,3 of (3 × 3) material law I.

Col. 55 to 64 - DM(I,2,3), element 2,3 of (3 × 3) material law I.

Second set of cards describing in plane action (I4, 6F10.2).

Col. 1 to 4 - N, material number, must be ordered sequentially.

Col. 5 to 14 - DS(I,1), element 1,1 of (3 × 3) material law I.

Col. 15 to 24 - DS(I,2), element 2,2 of (3 × 3) material law I.

Col. 25 to 34 - DS(I,3), element 3,3 of (3 × 3) material law I.

Col. 35 to 44 - DS(I,4), element 1,2 of (3 × 3) material law I.

Col. 45 to 54 - DS(I,5), element 1,3 of (3 × 3) material law I.

Col. 55 to 64 - DS(I,6), element 2,3 of (3 × 3) material law I.

Note for definition of the above constants, see Section 4.3 in Chapter 4.

#### 8. Displacement Boundary Condition Array (I4, 6L2, 2F10.2)

Kinematic constraints, 3 translations and two rotations can be specified on NUMBC cards for each top and bottom node. A separate card must be used to impose boundary conditions at each of the top and bottom nodes constrained. Note that a sufficient number of kinematic constraints have to be imposed to remove possible rigid

body motion of the structure and the singularity of the stiffness matrix.

Col. 1 to 4 - M, nodal point number

Col. 5 to 6 - J1, set true, T, if U-component constrained

Col. 7 to 8 - J2, set true, T, if V-component constrained

Col. 9 to 10 - J3, set true, T, if W-component constrained

Col. 11 to 12 - J4, set true, T, if  $\theta_X$  component constrained

Col. 13 to 14 - J5, set true, T, if  $\theta_Y$  component constrained

If above specifications apply to a top node, no further data is needed. For a bottom node the following data need to be added.

Col. 15 to 16 - JB, set true, T, if constraints are applied

at the nodes of the bottom deck.

Omit subsequent data if the above kinematic constraints act in the global coordinate direction. However, if a displacement or a rotation component is constrained in a direction which differs from the global X,Y axes then an angle must be specified separately for each of these constraints. The first angle is denoted by ALF for the displacement constraint V and is taken from the + global X axis to the direction along which the node remains free to displace  $\bar{U}$ . The second angle is denoted by BET for the rotational constraint  $\theta_Y$  and is taken from the + global X-axis to the axis about which the node remains free to rotate,  $\bar{\theta}_X$ . Use right hand rule for + direction of both angles. The corresponding orthogonal components  $\bar{V}$  and  $\bar{\theta}_Y$  are set to zero if the constraint for V or  $\theta_Y$  was set true, T.

Col. 17 to 26 - ALF, angle of skew for constraint  $\bar{V}$  in degrees

Col. 27 to 36 - BET, angle of skew for constraint  $\bar{\theta}_Y$  in degrees.

Nonzero displacement boundary conditions can be imposed by applying the corresponding kinematic constraints and by specifying the magnitude of given displacements in the input for nodal loads.

INPUT DATA FOR LOADING AND OUTPUT CONTROLS

Repeat subsequent set of cards for each load case.

9. Start Card for Loading (A6)

Col. 1 to 4 - CHECK, each load case has to begin with a card having the word "LOAD" in the first 4 columns which initiates the input of the loading and output controls. Any number of load cases can be treated.

10. Loading Control Card (3I4, 2F10.2)

Col. 1 to 4 - NLD, number of nodes with concentrated forces and moments. These must be applied separately at top and bottom deck nodes.

Col. 5 to 8 - NLL, number of different live load intensities acting on the top deck elements in the z-direction, leave blank if no live load included.

Col. 9 to 12 - NDL, set to 1 if dead load is included, otherwise leave blank.

If  $NLL > 0$  the following data must be added.

Col. 13 to 22 - PLL, live load intensity [force/area] for  $NLL = 1$  which is assumed uniform over the area of all top deck elements.

If  $NLL > 1$  the remaining live load intensities have to be specified together with the top deck elements over which they act on cards specified in paragraph 12.

If  $NDL = 1$  the following data must be added.

Col. 23 to 32 - PDL, specific weight for dead load [force/volume].

Leave blank if  $NDL = 0$ .

11. Concentrated Nodal Loads (I4, 2L2, 2x, 5F10.2)

If  $NLD = 0$  these cards are omitted.

Global nodal forces, 3 linear force components  $P_X$ ,  $P_Y$ ,  $P_Z$  and two moment components  $M_X$  and  $M_Y$ , can be specified on NLD cards for each node. A separate card must be used to input nodal loads at the top or bottom deck nodes.

Col. 1 to 4 - M, nodal point number

Col. 5 to 6 - TBOT, set true, T, if nodal force acts at the node of the bottom deck.

Col. 7 to 8 - TDIS, set true, T, if a nonzero displacement boundary condition is prescribed.

Col. 11 to 60 - FORCE(5), nodal load or displacement components in the global directions. They are ordered in the following sequence. First, the linear forces  $P_X, P_Y, P_Z$  then the moments  $M_X, M_Y$ .

12. Uniform Live Loads and Dead Loads

If  $NLL \leq 1$  these cards are omitted. Uniform live loads can be specified over individual deck elements which differ from the overall live load distribution PLL. For each additional live load intensity  $NLL > 1$  the following information needs to be furnished.

First card (2I4, F10.2)

Col. 1 to 4 - I, live load number

Col. 5 to 8 - NEL, total number of deck elements subjected to this live load

Col. 9 to 12 - PLL, live load intensity [force/area]

Second set of cards (20I4)

Col. 1 to 80 - NOLL(NEL), sequential list of top deck elements subjected to live load number I having intensity PLL.

13. Output Control Card (5I4, F10.2)

Various options can be activated for computing, printing and punching of nodal displacements and nodal forces. In addition to that, internal forces can be determined and output at element center and at the nodes where all individual element contributions are averaged.

Col. 1 to 4 - T1, set to 1 if global nodal forces and reactions should be computed and printed.

Col. 5 to 8 - T2, set to 1 if internal forces at the center of the deck elements should be computed and printed

Col. 9 to 12 - T3, set to 1 if binary punched output desired. For detailed description of the punched output see Section 5.6, paragraph i.

Col. 13 to 16 - NSTO, total number of nodes at which internal forces should be computed, averaged and output together with their principal values. If  $NSTO = 0$ , output of nodal quantities is suppressed; if  $NSTO = NUMNP$  (total number of nodes), these quantities are evaluated and output at all nodes; if  $0 < NSTO < NUMNP$ , these quantities are determined and output at NSTO nodes only which have to be listed on subsequent cards.



Col. 17 to 20 - NDIA, total number of transverse web elements idealizing the diaphragms for which the averaging procedure at the nodes has to remain separate from that of the longitudinal web elements.

Col. 21 to 30 - ALF, angle in degrees from global X-axis to the direction along which the normal nodal stress resultants are to be computed and output. Use right hand rule for + direction of angle. These values override the computation of principal stress resultants at the nodes which are usually determined from the nodal averages.

If  $0 < NSTO < NUMNP$  input on the following set of cards list of nodes at which internal forces should be computed, averaged and output together with their principal values.

14. List of Nodes for Internal Forces (20I4)

Col. 1 to 80 - NPS(NSTO), list of nodes where internal forces desired.

Omit these cards if  $NSTO = 0$  or  $NSTO = NUMNP$ .

RUNNING SEQUENCE

Repeat for each subsequent load case cards specified in paragraphs 9 - 14 beginning with the start card LOAD.

Repeat for each subsequent problem cards specified in paragraphs 1 - 14 beginning with the start card START.

The program stops when a card with the word STOP in the first 4 columns is encountered.

### 5.5 Commentary on Generation Options

It is always possible to input all components of the deck and web element arrays and to specify all nodal coordinates. For mesh layouts with a certain degree of regularity, it is convenient to make use of mesh and coordinate generation options to reduce the required input data.

- a) Deck Element Generation (see paragraph 4 of input specifications in Section 5.4).

Two types of generations are available for the nodal arrays of deck elements.

1) One Way Generation: If element cards  $N+1$ ,  $N+2$  --  $N+L-1$  are omitted and columns 33 to 40 are left blank on element card  $N+L$  the missing  $(L-1)$  element arrays will be generated by increasing the nodal numbers of the preceding element by NDIF. NDIF is assumed to be 1 if it is not specified in Cols. 29 to 32 of element card  $(N+L)$ . The material numbers of the top and bottom deck elements are the same as those input for element  $N$ .

2) Two Way Generation: This option can be used when two adjacent strings of sequentially numbered elements have been defined previously. Two parameters need to be specified on the card for element  $N$ , which is the last element so far numbered:

Col. 33 to 36 - MOD, module ( $m > 0$ ) equalling the difference in the element numbers of corresponding elements in adjacent strings of elements.

Col. 37 to 40 - NLIM, largest element number up to which generation desired ( $> N$ ).

The  $i = 1, \dots, 4$  nodal numbers of element  $N+1, N+2, \dots, NLIM$  are generated by

$$i_K = i_{K-MOD} + (i_{K-MOD} - i_{K-2MOD})$$

where  $i_K$  denotes node  $i$  of the  $K^{\text{th}}$  element with  $K = N+1, N+2, \dots, NLIM$ . The material type numbers are set equal to their values for element  $(K-MOD)$ . If  $NLIM = NUMEL$  no more element cards are needed. If  $NLIM < NUMEL$  the card of node  $(NLIM + 1)$  must follow.

- b) Web Element Generation (see paragraph 5 of input specifications in Section 5.4).

Two types of generations are available for the nodal arrays of web elements similar to the generation of deck element arrays.

1) One Way Generation: If element cards  $N+1, N+2, \dots, N+L-1$  are omitted and columns 21 to 28 are left blank on element card  $N+L$ , the missing  $(L-1)$  element arrays will be generated by increasing the nodal numbers of the preceding element by  $NDIF$ .  $NDIF$  is set to 1 if it is not specified in Cols. 17 to 20 of element card  $N+L$ .

2) Two Way Generation: This option can be used when two adjacent strings of sequentially numbered elements have been defined previously. Two parameters need to be specified on the card for element  $N$ , which is the last element already numbered:

Col. 20 to 24 -  $MOD$ , module ( $m > 0$ ) equalling the difference of corresponding element numbers in adjacent strings.

Col. 25 to 28 -  $NLIM$ , largest element number up to which generation desired ( $> N$ ).

The  $i = 1, 2$  nodal numbers of element  $N+1, N+2, \dots, NLIM$  are generated by

$$i_K = i_{K-MOD} + (i_{K-MOD} - i_{K-2MOD})$$

where  $i_K$  denotes node  $i$  of the  $K^{\text{th}}$  element with  $K = N+1, N+2, \dots, \text{NLIM}$ . The material type number is the same as for element  $(K-\text{MOD})$ . If  $\text{NLIM} = \text{NBEAM}$ , no more element cards are needed. If  $\text{NLIM} < \text{NBEAM}$ , the card of element  $\text{NLIM} + 1$  must follow.

In order to make averaging of nodal quantities between coplanar web elements possible the following numbering scheme has to be adopted to identify web elements: Number first in increasing order transverse elements idealizing all transverse diaphragms, then continue numbering sequentially longitudinal elements idealizing all longitudinal web components.

- c) Nodal Coordinate Generation (see paragraph 6 of input specifications in Section 5.4).

Two types of coordinate generations are available similar to the element array generations.

1) One Way Generation: If  $(L-1)$  nodal cards for points  $N+1, N+2, N+3, \dots, N+L-1$  are omitted and Cols. 25-32 of the  $N^{\text{th}}$  card are left blank, the missing coordinates will be generated as those of  $L$  equally spaced points on a line joining  $N$  and  $(N+L)$ . That is

$$\text{DIV} = L$$

$$x_{N+K} = x_{N+K-1} + (x_{N+L} - x_N) / \text{DIV}$$

$$y_{N+K} = y_{N+K-1} + (y_{N+L} - y_N) / \text{DIV} \quad \text{for } K = 1, 2, \dots, (L-1)$$

2) Two Way Generation: This option of coordinate generation can be used after two adjacent strings of sequential nodal points have been defined previously. It is activated by specifying on the card for node  $N$ .

Col. 25 to 28 - MOD, module ( $m > 0$ ) equalling the difference of corresponding nodal numbers on the adjacent coordinate lines.

Col. 29 to 32 - NLIM, the largest node up to which the coordinate generation desired ( $> N$ ).

The x,y coordinates of nodes N+1, N+2,..NLIM is generated from

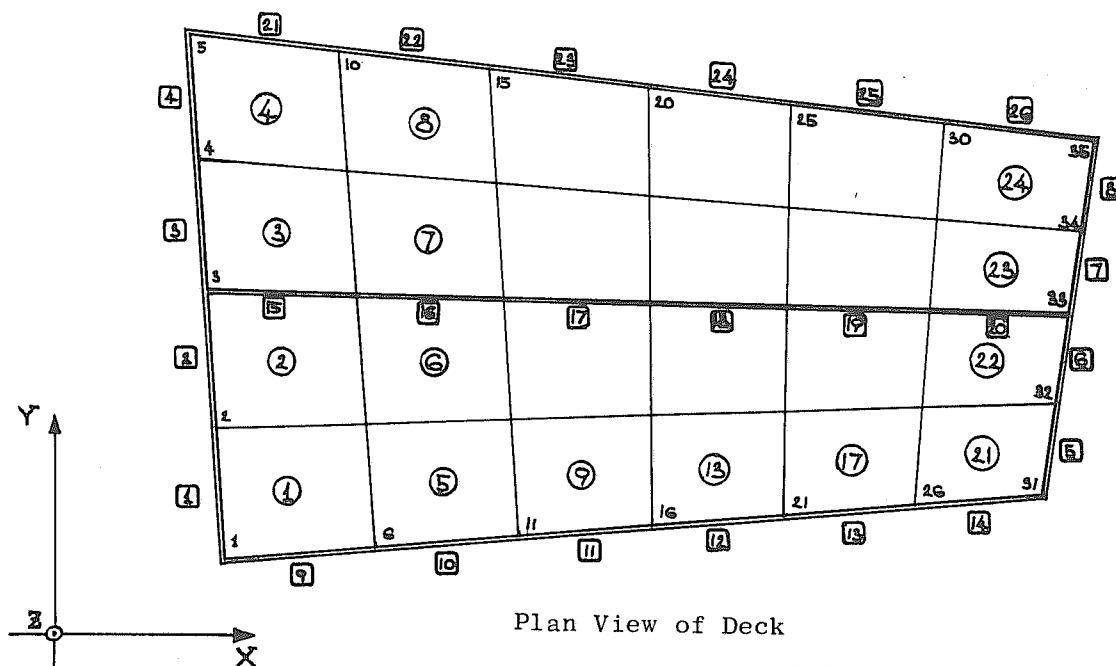
$$x_K = x_{K-MOD} + (x_{K-MOD} - x_{K-2MOD})$$

$$y_K = y_{K-MOD} + (y_{K-MOD} - y_{K-2MOD})$$

for  $K = N+1, \dots, NLIM$

If  $NLIM = NUMNP$  no more nodal cards are needed. If  $NLIM < NUMNP$ , the card of node  $(NLIM + 1)$  must follow.

d) Example for Mesh Generation



Plan View of Deck

- 1, 2, 3 ... Nodal Numbers
- ① ② ... Deck Element Numbers
- ④ ⑤ ... Web Element Numbers

In above figure each exterior edge has been subdivided equally to form mesh layout.

1) Deck Element Cards: 3 cards are needed

L	NP(1)	NP(2)	NP(3)	NP(4)	MA	MB	NDIF	MOD	NLIM
1	1	6	7	2	1	2			
5	6	11	12	7	1	2			
9	11	16	17	12	1	2		4	24

2) Web Element Cards: 5 cards are needed

L	NBP(1)	NBP(2)	MC	NDIF	MOD	NLIM
1	1	2	3			
5	31	32	3			
9	1	6	4			
15	3	8	4	5		
21	5	10	4	5	6	26

3) Nodal Coordinates: 4 cards are needed

N	X-ORD	Y-ORD	MOD	NLIM
1	$x_1$	$y_1$		
5	$x_5$	$y_5$		
6	$x_6$	$y_6$		
10	$x_{10}$	$y_{10}$	5	35

### 5.6 Output Description

The solution of each problem contains the following output information.

a) Input Echo

A printout of the input data is output with proper description

of the data.

b) Nodal Displacements

All global displacements are output at the nodes of the top and bottom deck. The 5 components  $U, V, W, \theta_X$  and  $\theta_Y$  are listed sequentially for each node.

c) Nodal Forces

The global nodal forces are computed and printed if output control T1 is set to 1. This information yields the reactions to the loading and gives an insight into the round-off error and magnitude of the residual forces at all nodes.

d) Internal Forces at Center of Deck Elements.

Both stress resultants and moments are computed and printed at the center of the deck elements if output control T2 is set to 1.

e) Internal Forces at Nodes of Deck Elements.

If output control, NSTO > 0 internal force contributions of each element to a node, their averages and associated principal values are computed and output at NSTO specified nodes of both top and bottom deck.

f) Internal Forces of Web Elements

If output control NSTO > 0 the internal force contributions of each element, their averages and corresponding principal values are computed and output at NSTO specified nodes of both, transverse and longitudinal web elements. Averaging takes place if the second node of a web element coincides with the first one of an adjacent web element. Hence, to activate averaging of coplanar web elements only, one has to provide information regarding how many web elements lie in the transverse direction, NDIA, and how many lie in the longitudinal

direction, NBEAM - NDIA. Therefore, averaging necessitates a general rule concerning the web element numbering. It is assumed that the transverse web elements in the diaphragms are numbered first increasing with the global X-direction while the longitudinal web elements follow second, increasing with the global Y-direction.

g) Direction of Internal Forces (for deck element only)

To reduce the effort for equilibrium checks in arbitrary directions, the output control ALF is available. ALF is the angle in degrees from the global X-axis to the direction along which the normal stress resultants are desired. Use right hand rule for + direction. The rotated components  $N_{nn}$ ,  $N_{ns}$  and  $M_{nn}$ ,  $M_{ns}$  are output together with the direction cosines instead of the principal values  $N_{max}$ ,  $N_{min}$  and  $M_{max}$ ,  $M_{min}$  which are usually output together with the angle in radians between the global X-axis and the direction of the maximum principal value given in radians. This capability simplifies tremendously equilibrium checks, e.g. for skewed box girders at sections parallel to the skewed support.

h) Log of Problem

The printed output of each problem is terminated by a summary listing execution times in different links and general information such as the number of degrees of freedom, the bandwidth and the block length.

i) Punched Output

With output control  $T_3$  set to 1 the following arrays are punched on cards in binary format. First, the global displacement vector  $D(NEQ)$  is always punched out as one record. The nodal components are ordered in the same way as in the stiffness matrix, namely, nodal



displacements  $U, V, W, \theta_x, \theta_y$  first at the nodes of the bottom deck and then at those of the top deck. An end of file card (789/1/7) follows. Second, the matrix  $SNP(NSTO, 12)$  of averaged internal forces in the deck is punched at NSTO nodes. The internal force components are arranged in the columns of matrix  $SNP$ , first the  $N_{xx}, N_{yy}, N_{xy}, M_{xx}, M_{yy}, M_{xy}$  components of the top deck and then those of the bottom deck. An end of file card (789/1/7) follows. Third, the matrix  $YMO(NSTO, 12)$  of the  $M_{yy}$  contributions of each deck element to the nodal averages is punched at NSTO nodes. For each of the NSTO nodes the columns contain the element contributions to that node. The first 6 columns are reserved for the top deck and the remaining 6 columns contain element contributions to the bottom deck. The number of element contributions to a node coincides with the number of columns used, increasing in order. If there are less than a maximum of 6 element contributions per node, then the remaining columns are initialized and set to zero. An end of file card (789/1/7) follows. Fourth, if the set of NSTO nodes where nodal averages are output contains also nodes of web elements, the same nodal information as in second and third is punched for the longitudinal web elements. Since they exhibit one way bending only the  $M_{xx}$  and  $M_{xy}$  components are set to zero. As averaging of internal forces takes place between coplanar web elements only a maximum of two element contributions per node is considered.

### 5.7 General Remarks

The following comments are in place regarding the usage of program CELL.

## a) Bandwidth

Number the nodes so as to minimize the bandwidth which depends on the largest difference of nodal numbers for any finite element.

## b) Interpretation of Internal Forces

Recall that the finite element method is an approximate method of analysis. It is well known that the stresses derived through the standard stress-displacement relationships from the resulting nodal displacements do not satisfy differential equilibrium. In fact element stresses exhibit usually large discontinuities if simple expansions are used to approximate the field variables within each element. Hence, stresses at the element center are considered more reliable. If nodal stresses are desired, a simple averaging of all element contributions at a node leads to the most meaningful representation. In general, these averages violate considerably the natural boundary conditions at free edges and approximate poorly local stress concentrations. Refinement of the mesh is usually required to capture steep stress gradients within the element domains or stress boundary conditions along the surfaces.

## c) Finite Element Types

Considerable computer time during formation of element stiffnesses can be saved if one considers repetitious element types. Two factors define an element type, the element geometry and the material properties. The program compares internally the geometries and material laws of consecutive elements. If both factors agree, the formation of the element stiffness is skipped together with its transformation into global coordinates. Hence, the user is able to reduce considerably the formation time by numbering consecutively elements of the same type.

## d) Storage Requirements

The required storage for a given problem is determined during execution and is allocated automatically within the program. The following formulas are useful for computing the field length by hand if it is not possible to retrieve automatically the last word address of the program and to reset the field length during execution. This estimate is based on experience with a CDC 6400 computer using the FUN compiler.

$$ST = FIX + VAR$$

describes in general the amount of storage required for a specific problem. FIX is the fixed amount of storage area which is independent of the problem being solved while VAR denotes the variable storage area which is a function of the data being processed. The program passes through its 5 links during execution, each of which requires a minimum storage area for FIX and the associated blank COMMON areas. In most cases link 2 or 4 governs the storage requirement.

Define  $AA = 6*NUMEL + 5*NBEAM + 2*NUMNP + 15*NMAT$

Link 1:  $FIX = 36267B$  (where B denotes the octal base)

$$VAR = AA + 6*NMAT$$

Link 2:  $FIX = 17573B$

$$VAR = 4*NUMEL + 2*NBEAM + 2*NBLKL*IBANDW$$

Link 3:  $FIX = 21264B$

$$VAR = AA + 2*NUMEL + 2*NEQ$$

Link 4:  $FIX = 17655B$

$$VAR = 2*NBLKL*(IBANDW + 1) + NEQ$$

Link 5:   FIX = 30316B

VAR = AA + NEQ + [NEQ + NUMEL + 7\*12\*NSTO + 2\*NSTO]

The expression in brackets should be deleted if no controls for the printed output are activated.

The following variables depend on the size of the problem and are defined by

NUMEL - number of top deck elements

NBEAM - number of web elements

NUMNP - number of nodal points

NMAT - number of different material types

NBLKL - number of equations in a block

IBANDW - half of maximum bandwidth

NEQ - total number of equations

NSTO - number of nodes at which internal forces are output

c) Execution Time

A summary of central processing times is presented below for the solution of different problems by program CELL. The total execution times are given in seconds in addition to the times spent within each link of the overlay system as obtained from the CDC 6400 computer using the FUN compiler. Moreover, parameters such as NUMEL, NBEAM, NEQ, IBAND and NSTO are listed since they govern the computational effort of the problem considered. For the purpose of identification the finite element mesh is described by the notation  $i*j/\alpha$ , where  $i$  stands for the number of deck elements in the transverse direction,  $j$  for the number of deck elements in the longitudinal direction and  $\alpha$  for the angle of skew.

TABLE 1  
EXECUTION TIMES FOR PROGRAM CELL IN SECONDS

PROBLEM MESH	NUMEL	NBEAM	NEQ	IBAND	NSTO	LINK					TOTAL
						1	2	3	4	5	
Single Cell Box 1*3/90	3	8	80	40	0	1	0	0	1	0	2
Single Cell Box 2*5/90	10	14	180	50	0	1	1	0	3	0	5
Four Cell Box 6*22/90	132	122	1610	90	8	46	17	1	71	25	162
Example 4 8*20/90	160	72	1890	110	29	14	27	1	137	28	207
Example 5 8*20/45	160	72	1890	110	49	22	23	1	118	30	193
Example 6 5*20/90	93	49	1180	90	25	38	10	1	35	17	101

It is evident that from this limited data no general formula can be constructed for the timing of program CELL. Hence, the purpose of this summary is to present execution times for a few typical problems and to suggest certain relationships between the timing and some parameters. The timing of Link 1 where the global element stiffnesses are generated depends mainly on the number of web elements, NBEAM, but no clear trend can be seen since the type generation of repetitious elements obscures the picture. The timing of Link 2 where the element stiffness is added into the structural stiffness matrix is almost proportional to the number of top deck elements, NUMEL. The timing of Link 3 where

the global load vector is formed is negligible in comparison to the other links. The timing of Link 4 where the system of equations is solved is in general proportional to the number of operations,  $0.5 \times \text{NEQ} \times \text{IBAND}^2$ , but the variable bandsolver obscures this relationship taking advantage of the sparsity of the matrix. The timing required for the solution of additional load cases amounts to 10% of the time used for decomposition of the structural stiffness matrix. The timing of Link 5 where the solution of the problem is output depends mainly on the total number of elements,  $\text{NUMEL} + \text{NBEAM}$ , and the number of nodes where the results are output,  $\text{NSTO}$ .

A very rough estimate for the total time can be obtained by assuming that the execution time is proportional to the number of degrees of freedom,  $\text{NEQ}$ .

## 6. EXAMPLES

Several examples of gradually increasing complexity have been chosen to illustrate the application of the program CELL. Whenever possible, the results obtained are compared with values from other independent solutions.

Examples 1, 2 and 3 (Figs. 8, 9 and 10) deal with the analysis of isotropic plates of rectangular or parallelogrammic shape subjected to in plane or transverse loadings. The results can be compared in the case of the skewed sheet with solutions using a refined finite element analysis, in the case of the cantilever with results obtained from refined beam theory and in the case of the rhombic plate with analytical solutions using series expansions. These independent solutions may be considered exact for the purpose of comparison.

Example 4 (Fig. 11) deals with the analysis of a non-skew, two cell box girder bridge simply supported at the end diaphragms which are perpendicular to the longitudinal axis. This structure is subjected to a concentrated load acting on the outside girder at the midspan cross-section. The results can be compared directly with those obtained by the folded plate theory using the program MULTPL, which may be considered exact for the purpose of comparison.

Example 5 (Fig. 15) deals with the analysis of the same two cell box girder bridge, but now supported on end diaphragms which are skewed  $45^\circ$  to the longitudinal axis. Since no analytical or experimental solutions are presently available for skew box girder bridges, internal equilibrium checks are performed at sections parallel to the

skewed supports providing an insight into the accuracy of the resulting stress distribution.

Example 6 (Fig. 19) is selected to illustrate the capabilities of program CELL. An idealized highway branch is chosen to show one of the complex geometric configurations the program can treat. Again, since no other method of analysis is available, internal equilibrium checks are performed to assess the accuracy of the resulting stress distribution.

### 6.1 Example 1 - In Plane Analysis of Skewed Sheet

This example has been chosen to demonstrate the accurate representation of the in plane behavior by using the deck element Q8D11. This mixed model is illustrated in Fig. 2 and described in detail in Section 3.1. The parallelogrammic structure with an angle of skew of 30 degrees is shown in Fig. 8. The sheet is subjected to two concentrated in plane loads and is supported along its skewed edges. This plane stress example was used in reference [5] to compare the accuracy of displacements and stresses obtained from various elements of the 8 DOF and the 12 DOF families.

The results from a refined finite element analysis, denoted by LSE, serve as basis of comparison because there is no exact solution available for this complex boundary value problem. A detailed formulation of this linear strain quadrilateral having 16 DOF is given in reference [7]. The LSE analysis maintains a lower bound to the exact displacements as displacement continuity is fully maintained.

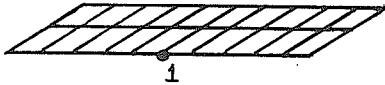
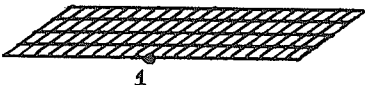
Table 2 presents a summary of V-displacements at point 1 to measure the accuracy of the solutions. The results of a coarse and



a fine mesh are given for the element Q8D11 using an "exact"  $3 \times 3$  and an "approximate"  $2 \times 2$  Gaussian integration scheme which are indicated by a (3) or (2) after the element designation Q8D11. For purpose of comparison, analogous solutions are obtained with the displacement model Q8D8 which is formed using the standard bilinear expansions for both displacement components and using a  $2 \times 2$  numerical integration scheme.

TABLE 2

EXAMPLE 1 - COMPARISON OF V-DISPLACEMENTS  
(ft  $\times 10^{-4}$ ) AT POINT 1

FINITE ELEMENT	COARSE MESH 	FINE MESH 
LSE [7]	-	54.51
Q8D8	11.48	22.16
Q8D11 (3)	30.44	42.80
Q8D11 (2)	51.49	53.74

First, one observes that all solutions lie below the lower bound obtained from the LSE analysis, hence they maintain a lower bound too.

Second, the results exhibit the tremendous effect of the internal node and the mixed formulation coupled with the relaxation of the integration rule. The combination of all these "improvements" applied to the standard element Q8D8 yields the element Q8D11(2) which performs as well as the higher order quadrilateral LSE having 16 fundamental DOF instead of 8. For these reasons this element was finally chosen to


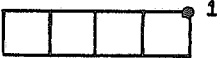
represent the in plane behavior of the deck elements in the program CELL.

## 6.2 Example 2 - In Plane Analysis of Cantilever

This example has been chosen to demonstrate the accurate representation of the in plane behavior by web element QUSP12. This displacement model is illustrated in Fig. 4 and described in Section 3.2. The structure, shown in Fig. 9, is subjected to two concentrated in plane loads at the tip of the cantilever. This plane stress example was used in reference [5] to compare displacements and stresses of various so called "spar" elements. The theoretical tip deflection is obtained from refined beam theory including shear deformations. As the root section is prevented from warping the theoretical value forms an upper bound to the exact tip deflection, but the error is less than 1/2000 as shown in reference [7]. In the finite element idealization displacement boundary conditions are prescribed to prevent warping at the fixed end section.

Table 3 presents a summary of V-displacements at the tip of the cantilever to measure the accuracy of the solutions. The results are given for a coarse and a fine mesh both idealizing the depth of the cantilever by one element. The solutions are obtained from the analysis using the web element QUSP12 with a 3 x 3 Gaussian integration scheme and the mixed model Q8D11(2) with a 2 x 2 Gaussian integration scheme. For purpose of comparison, an analogous solution is obtained using the displacement model Q8D8.

TABLE 3  
 EXAMPLE 2 - COMPARISON OF V-DISPLACEMENTS (in)  
 AT POINT 1

FINITE ELEMENT	COARSE MESH 	FINE MESH 
THEORY [5]	0.3558	
Q8D8	0.0475	-
Q8D11(2)	0.2693	0.3493
QUSP12	0.2533	0.3283

First, one observes that all solutions including that obtained from the mixed model Q8D11(2) with a relaxed integration rule lie below the theoretical results maintaining a lower bound for all practical purposes.

Second, the results exhibit again the tremendous effect of the internal node and the mixed formulation which combined with a relaxed integration rule yields the element Q8D11(2). Note the excellent performance of this element if subjected to in plane bending.

Third, results of the element QUSP12 compare very well with the theoretical solution. This element provides a cubic variation of the v-displacements along the longitudinal edges introducing a rotational  $\omega = \frac{\partial v}{\partial x}$  at each node in order to capture the in plane bending of the web. It also maintains continuity with the transverse bending displacements of adjacent deck elements simply by enforcing compatibility

of rotations. For these reasons the element QUS12 was finally chosen to represent the in plane behavior of the webs in the program CELL.

### 6.3 Example 3 - Plate Bending Analysis of Rhombic Plate

This example has been chosen to demonstrate the plate bending representation of the deck by element Q19. This displacement model is illustrated in Fig. 3 and described in detail in Section 3.1. The rhombic plate with an angle of skew of 40 degrees is shown in Fig. 9. This structure is subjected to a uniformly distributed transverse loading normal to the plane of the plate and is simply supported along its edges.

The theoretical results for transverse displacements and principal moments at the center of the plate are obtained from reference [8] which presents analytical solutions using series expansions. As this solution uses collocation to enforce equilibrium and continuity along a diagonal of the plate the results depend on sufficient control of the accuracy of boundary collocation. In general, the results of reference [8] on the flexural analysis of rhombic plates can be considered exact from a practical point of view.

Two finite element idealizations are used to discretize the structure, both having the same number of DOF and bandwidth, hence, posing the same computational problem. The first mesh layout idealizes the structure by triangular and rectangular elements while the second mesh layout uses parallelogrammic elements exclusively. Two types of boundary conditions are enforced at the corners of the idealized rhombic plate. The first clamps the corners by imposing  $W = \theta_X = \theta_Y = 0$  at the corner nodes while the second enforces only  $W = 0$  with the

rotations being free.

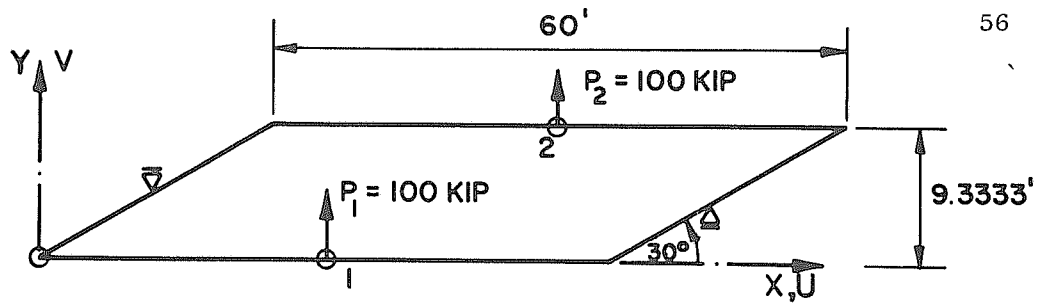
Table 4 presents a summary of various solutions for transverse displacements and principal moments at the center of the rhombic plate. The results are given for both types of mesh layouts and boundary conditions described above.

TABLE 4

EXAMPLE 3 - COMPARISON OF VERTICAL DISPLACEMENTS  $W[\text{ft} \times 10^{-2}]$  AND MOMENTS  $M[\text{k-ft/ft}]$  AT CENTER OF RHOMBIC PLATE

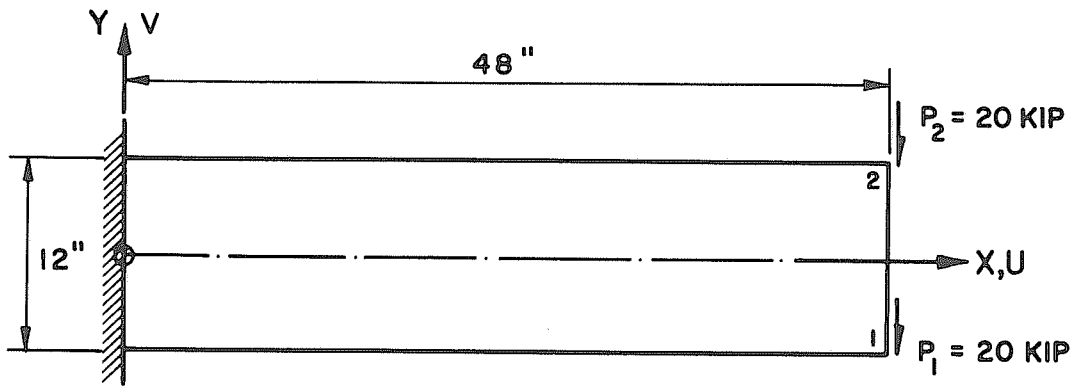
BOUND. COND. AT CORNERS	TRIANG.-RECTANG. MESH			PARALLELOGRAMMIC MESH		
	$W$	$M_{\max}$	$M_{\min}$	$W$	$M_{\max}$	$M_{\min}$
THEORY [8]	9.58	-1.80	-2.81	9.58	-1.80	-2.81
$\theta = 0$	6.95	-1.21	-2.47	6.88	-1.06	-2.29
$\theta \neq 0$	9.98	-1.95	-2.85	9.69	-2.07	-3.16

First, one observes that the solutions obtained by clamping the rotations,  $\theta = 0$ , lie far below the theoretical results. The parallelogrammic mesh yields stiffer or lower results than the rectangular mesh layout with adjustment for the skewed boundaries by triangles. It is a well known fact that the finite element approximations are too crude to capture the singularities at the obtuse corners. Since convergence of the finite element results is very slow in the case of skewed plates a highly refined mesh layout is required to provide acceptable results.



$E = 432,000$  KSF,  $\nu = 0.15$ ,  $t = 1.0$  FT.

FIG. 8 EXAMPLE 1 - IN PLANE ANALYSIS OF SKEWED SHEET



$E = 30,000$  KSI,  $\nu = 0.25$ ,  $t = 1.0$  IN.

FIG. 9 EXAMPLE 2 - IN PLANE ANALYSIS OF CANTILEVER

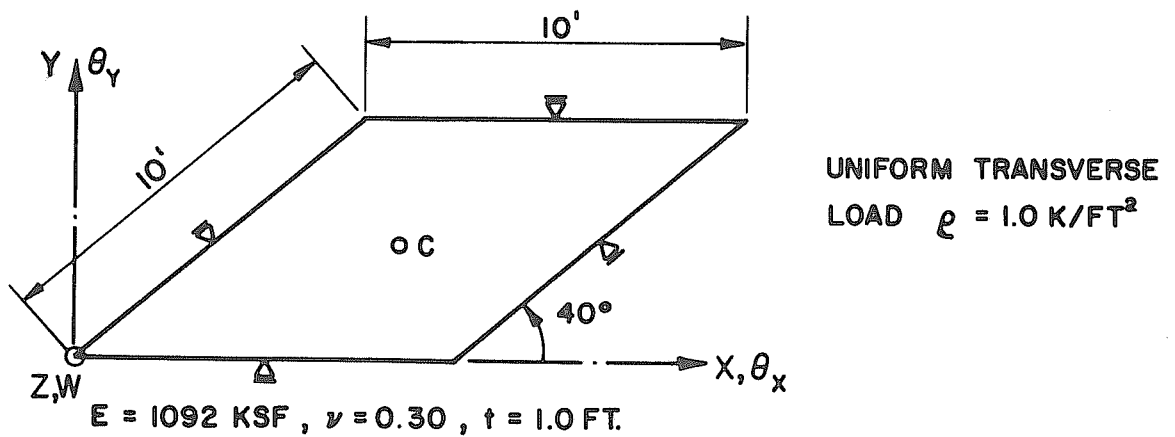


FIG. 10 EXAMPLE 3 - PLATE BENDING ANALYSIS OF RHOMBIC PLATE

Second, to speed up the slow rate of convergence the corner rotations are released from clamping. The elements adjacent to the obtuse corners are now able to approximate the steep gradients of the curvature field. One can observe that this artificial release improves the results tremendously but it destroys boundedness of the solution. Note that the rectangular mesh layout yields again displacements which are larger than those obtained from the parallelogrammic mesh but the moment resultants follow the opposite pattern.

#### 6.4 Example 4 - Two Cell Box Girder Bridge on Right Supports

This example has been chosen to demonstrate the accuracy of the finite element program CELL as applied to the analysis of cellular bridge structures. The geometry and material properties of the non-skew box girder bridge to be analyzed are described in Fig. 11 together with its loading and boundary conditions.

As this box girder is prismatic and simply supported at the end diaphragms, folded plate theory can be applied to the analysis of this structure. Program MULTPL, which was presented in reference [1], is based on folded plate theory. It is applied to the analysis of this two cell box girder and the results may be considered exact for the purpose of comparison.

The box structure is idealized by the finite element mesh layout illustrated in Fig. 12. Due to the simple geometric configuration 3 ft. X 3 ft. square elements can be used throughout with three elements idealizing the top and bottom deck of each cell in the transverse direction and one element the height of each web.

Observe that more realistic boundary conditions are imposed in the finite element layout than those assumed in folded plate theory. In the element idealization the nodes underneath the ends of the longitudinal webs are prevented from vertical movement supporting the flexible end diaphragms while the end diaphragms of folded plate theory are assumed completely rigid within their plane and do not exhibit any stiffness out of their plane. This difference of boundary conditions is negligible from a practical point of view if localized regions near the supports are excluded from the comparison.

Figure 13 illustrates the distribution of vertical displacements  $W$  at the midspan cross-section and along the top of the longitudinal outside girders. Note the excellent agreement of displacements between both solutions, the theoretical results obtained from MULTPL and the approximate finite element results obtained from CELL, maintaining throughout a relative difference below 2%.

In standard finite element analysis stresses are computed for each individual element. Recall that the resulting stress distribution neither satisfies differential equilibrium nor stress boundary conditions in a local sense. Furthermore, as long as simple expansions are used to describe the energy variation within an element, large discontinuities of stresses occur along interelement faces. In order to obtain a representative value for the resulting stress distribution it is common practice to determine nodal stresses simply by averaging all element contributions to a specific node. Averaging has to take place for the deck and web components separately.

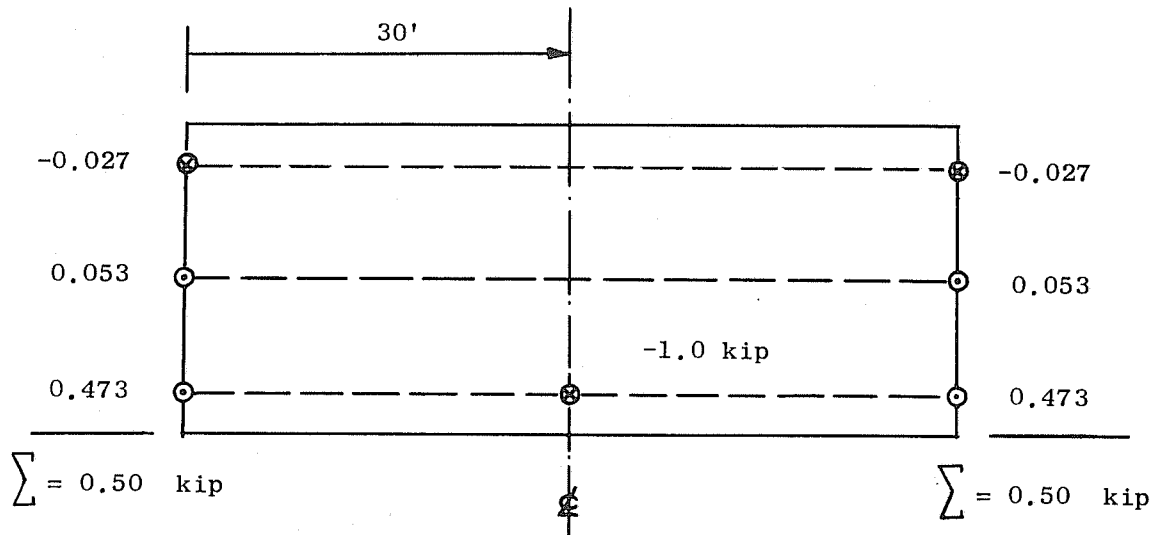


Figure 14 illustrates the distribution of the longitudinal stress resultants  $N_x$  at the midspan cross-section and along the top of the longitudinal outside girders. The theoretical solutions obtained from MULTPL agree very well with the results from CELL in regions far enough from the singularity underneath the concentrated load. In the vicinity of this point, the chosen finite element mesh layout cannot capture the steep stress gradients. Further mesh refinement is required to improve the values near the load. The longitudinal stress resultant  $N_x$  is presented along the top of the outside girders to provide some insight into the load distribution of the two cell box structure. The transverse distribution of  $N_x$  yields the information necessary for calculating the total internal moment at midspan. The ratio of this internal moment,  $M_{int}$ , determined by integration of the resulting internal stress distribution and the moment,  $M_{stat}$ , computed from overall statical considerations at a cross-section will be defined by the coefficient  $\mu$ .

$$\mu = \frac{M_{int}}{M_{stat}}$$

This coefficient gives some insight into the accuracy of the resulting stress distribution which is particularly important in cases where no theoretical solutions are available. It provides a criterion for establishing confidence in the finite element results which are approximate in nature. As long as one considers only cross-sections parallel to the supports, the magnitude of individual reactions is not required, only their sum need to be known to evaluate  $M_{stat}$ . The sum of the reactions at each support are easily computed from statics while the finite element analysis yields their individual magnitudes

which look for Example 4 as follows:



Two contributions,  $M_N$  and  $M_M$ , make up the total internal moment  $M_{int}$ . The contribution  $M_N$  to the internal moment  $M_{int}$  is established by taking moments of those forces in deck and web components which are equivalent to the internal resulting stress distribution about the neutral axis of the cross-section. The statically equivalent forces are simply determined by numerical integration of the normal stress resultants  $N_n$  at a cross-section, say by the trapezoidal rule. The factor  $M_M$  is determined by numerical integration of the normal slab moments  $M_n$  at the same section.

The contribution of internal stress and moment resultants to the gross internal moment at midspan is given by

$$M_{int} = M_N + M_M = 14.08 + 0.19 = 14.27 \text{ k-ft.}$$

The statical moment,  $M_{stat}$ , can be found directly from the external loading and reactions and equals in this case  $0.500 \times (30) = 15.0 \text{ k-ft.}$

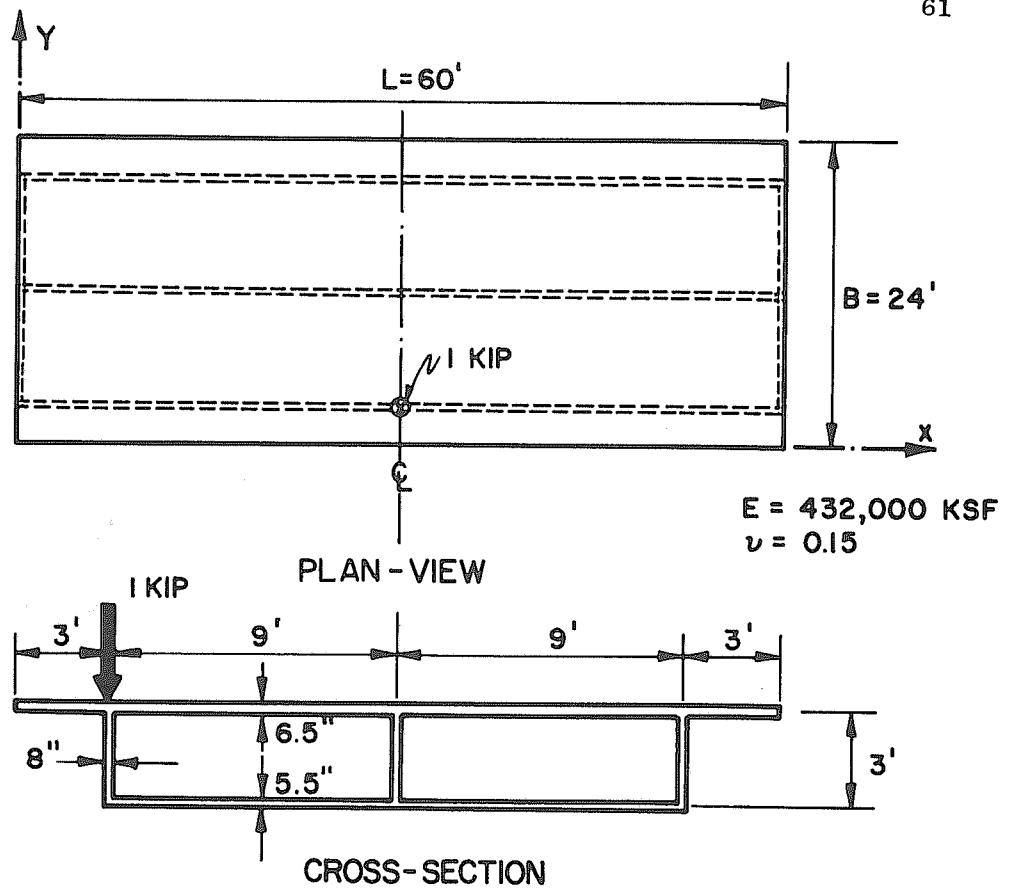


FIG. 11 EXAMPLE 4 - TWO CELL BOX GIRDER BRIDGE ON RIGHT SUPPORTS

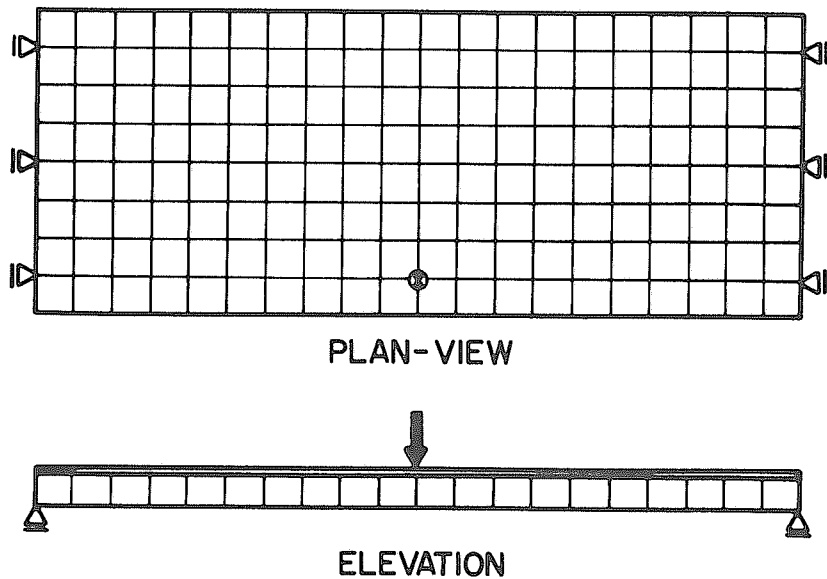
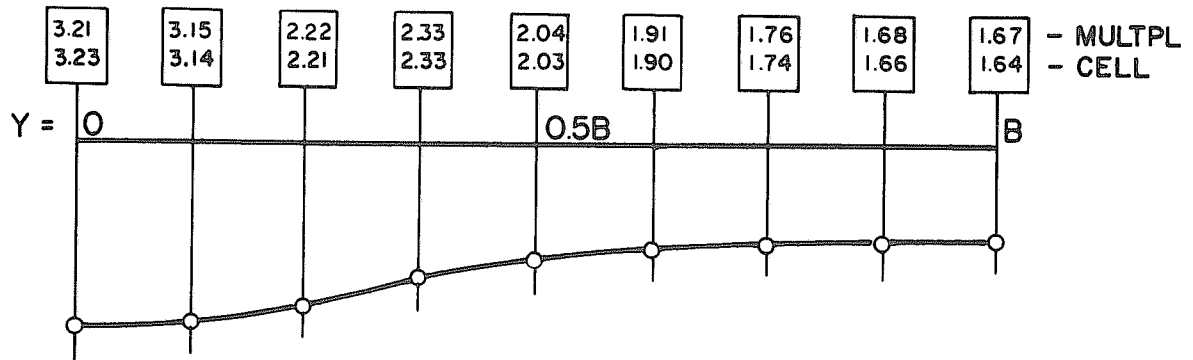
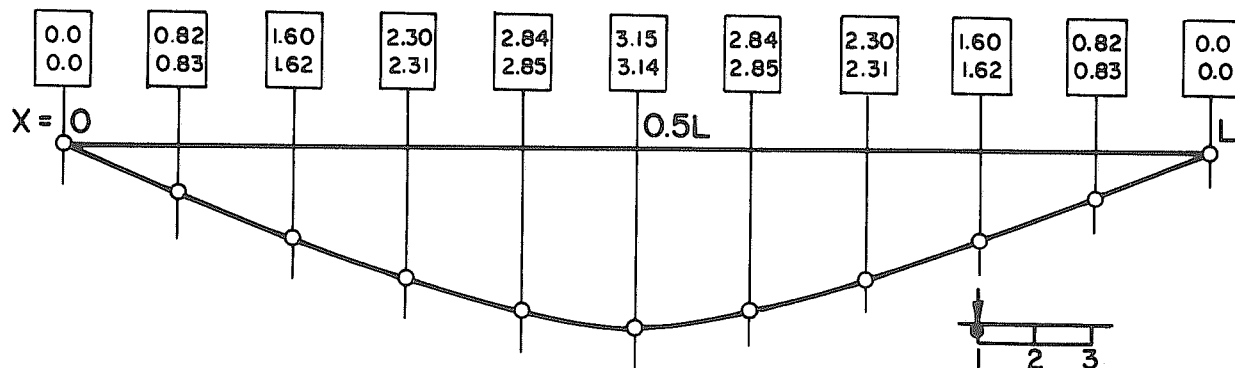


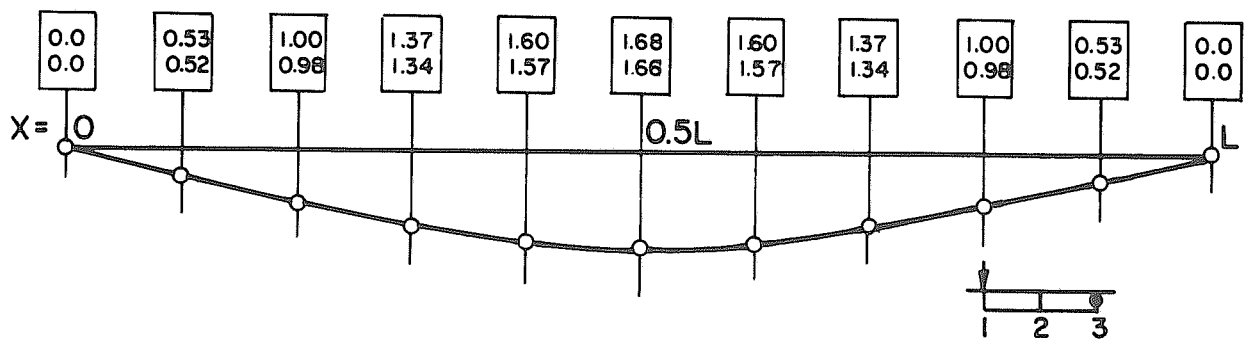
FIG. 12 EXAMPLE 4 - FINITE ELEMENT IDEALIZATION



TRANSVERSE DISTRIBUTION OF  $W$  AT TOP DECK OF MIDSPAN CROSS-SECTION

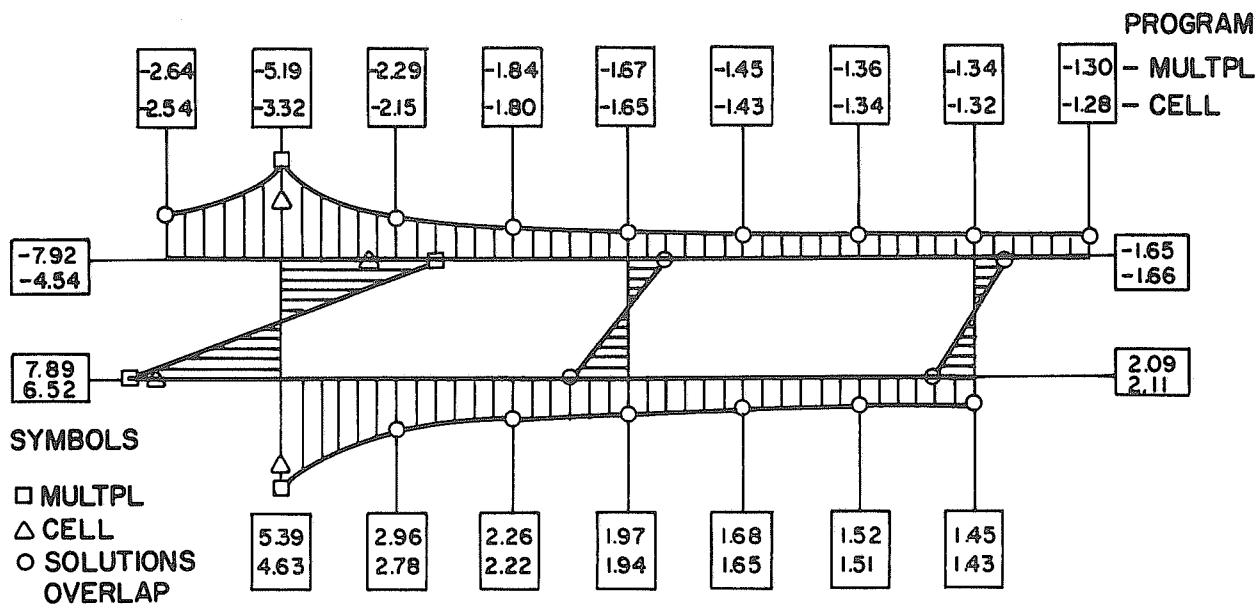


LONGITUDINAL VARIATION OF  $W$  ALONG TOP OF GIRDER 1

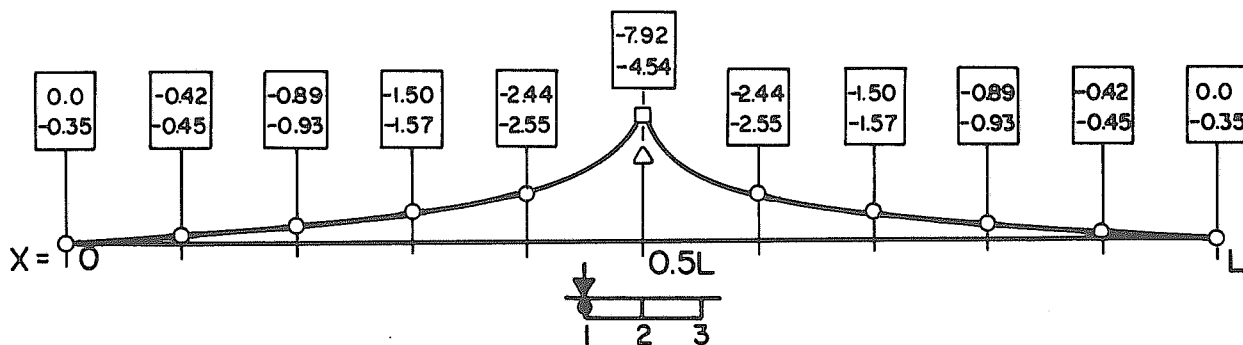


LONGITUDINAL VARIATION OF  $W$  ALONG TOP OF GIRDER 3

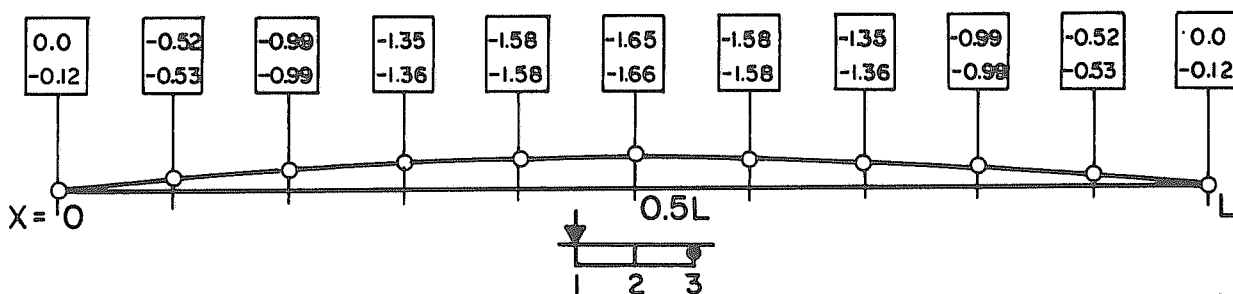
FIG. 13 EXAMPLE 4 - RIGHT BOX, VERTICAL DISPLACEMENTS  
 $W$  (ft  $\cdot 10^{-4}$ )



TRANSVERSE DISTRIBUTION OF  $N_x$  AT MIDSPAN CROSS-SECTION



LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 1



LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 3

FIG. 14 EXAMPLE 4 - RIGHT BOX, LONGITUDINAL STRESS RESULTANTS  $N_x$  (K/FT · 10<sup>-1</sup>)

The coefficient  $\mu$  is then computed from

$$\mu = \frac{M_{int}}{M_{stat}} = \frac{14.27}{15.0} = 0.95$$

Observe that the violation of equilibrium by the resulting stress distribution by 5% is negligible considering that the section is taken directly underneath the concentrated loading. This error would decrease appreciably at any other section away from stress concentrations and singularities as shown in reference [5].

The total computer time necessary for execution of this problem amounted to 3 minutes and 27 seconds. A detailed discussion of execution times as obtained on the CDC 6400 computer is presented in Section 5.7

#### 6.5 Example 5 - Two Cell Box Girder Bridge on Skewed Supports

This example has been chosen to demonstrate the capability of the finite element program CELL to analyze box girder bridges on skewed supports. In principle the same structure as in example 4 is used, the only difference being the direction of the supporting end diaphragms which are now inclined at an angle of 45 degrees with the longitudinal axis. The geometry and material properties of the skewed box girder bridge are described in Fig. 15 together with its loading and boundary conditions.

Unfortunately, no other analytical or experimental solutions are presently available to compare these results with. Only internal equilibrium checks in form of the  $\mu$  coefficients can be performed in order to assess the accuracy of the resulting stress distribution.

The box structure is idealized by the finite element mesh layout illustrated in Fig. 16. Results in reference [5] suggest that

rectangular and triangular elements should be used instead of parallelogrammic elements to idealize skewed regions. Hence, the structure is discretized by 3 ft. X 3 ft. square elements while right triangles adjust the mesh to the skewed boundaries. Similar to example 4 three elements idealize the top and bottom deck of each cell in the transverse direction and one element the height of each web.

The listing of the data for this typical example is included in Appendix B for those wishing a check case from Program CELL.

Figure 17 illustrates the distribution of vertical displacements  $W$  along the midspan cross-section and along the top of the longitudinal outside girders. The midspan section refers to the skewed section A-A parallel to the supports which passes through the center of the parallelogrammic plan view. In comparison to the associated right box the displacements are considerably smaller, especially those of the unloaded outside girder. This is no surprise since the clear span between the supports reduces from 60 ft. to 42.42 ft. and the effective moment of inertia of the skewed section increases by 1.414. Moreover, due to the continuity between longitudinal webs and diaphragms the corners are effectively restrained from rotation.

Figure 18 illustrates the distribution of the normal stress resultants  $N_n$  at the skewed midspan cross-section and of  $N_x$  along the top of the longitudinal outside girders. Again, only the averages of all element contributions to a node are given, thus smoothing stress discontinuities along interfaces. A comparison with the values obtained for the associated right box illustrates a vast reduction in the magnitude of the stress resultants due to reduction of effective

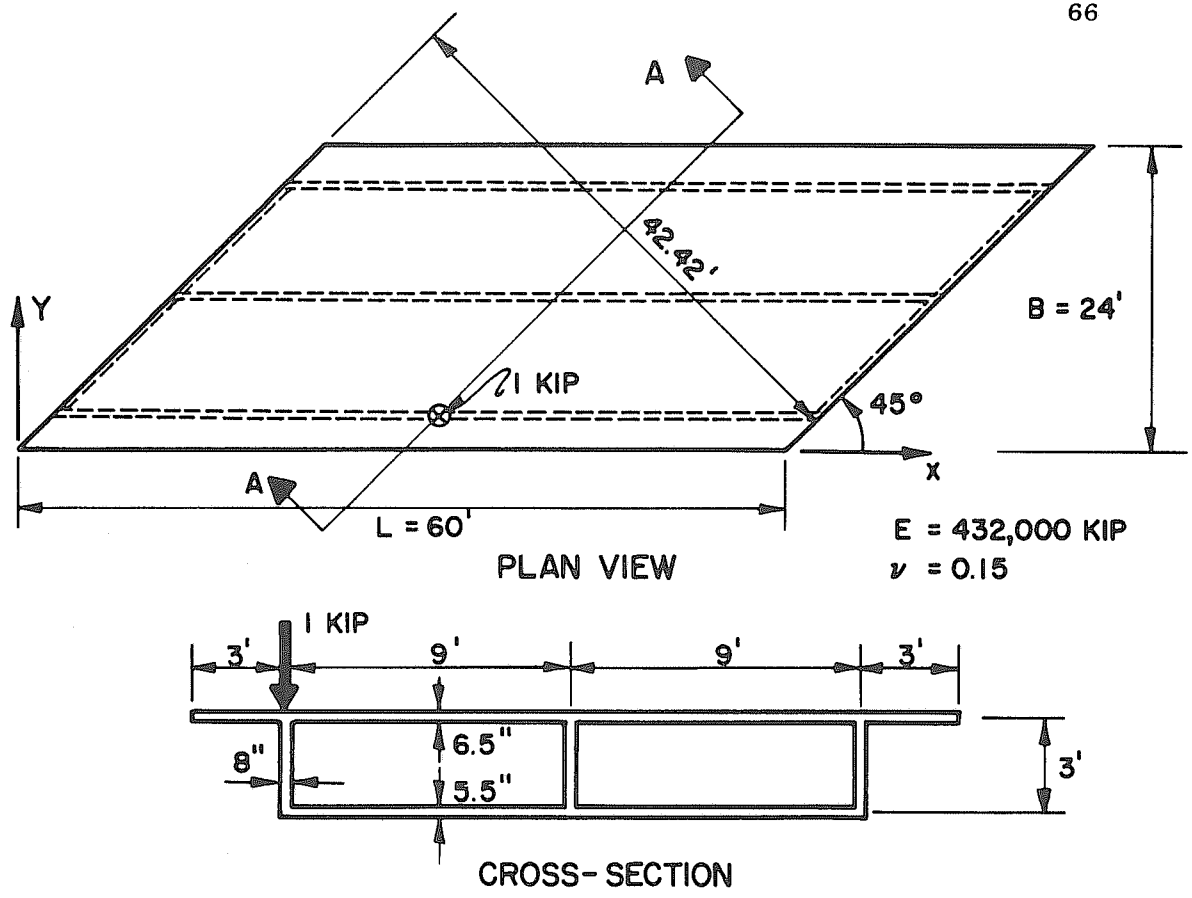


FIG.15 EXAMPLE 5 - TWO CELL BOX GIRDER BRIDGE ON SKEWED SUPPORTS

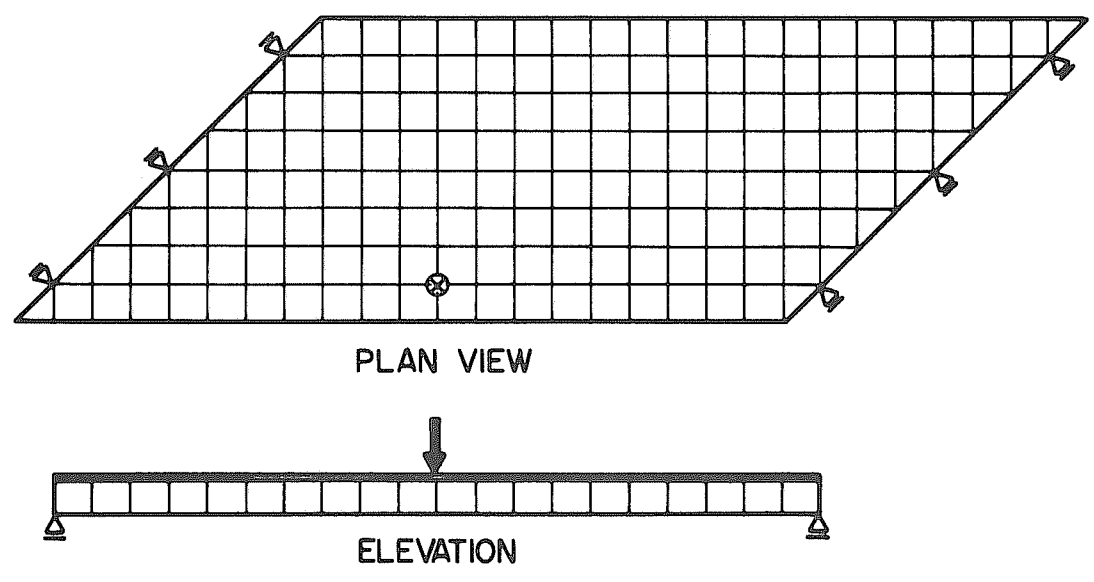
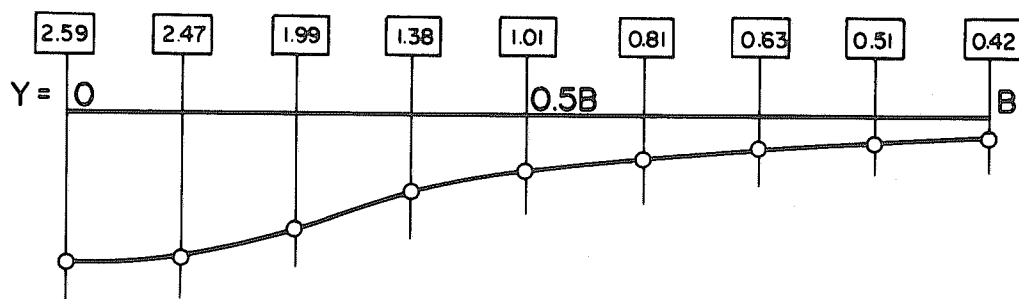
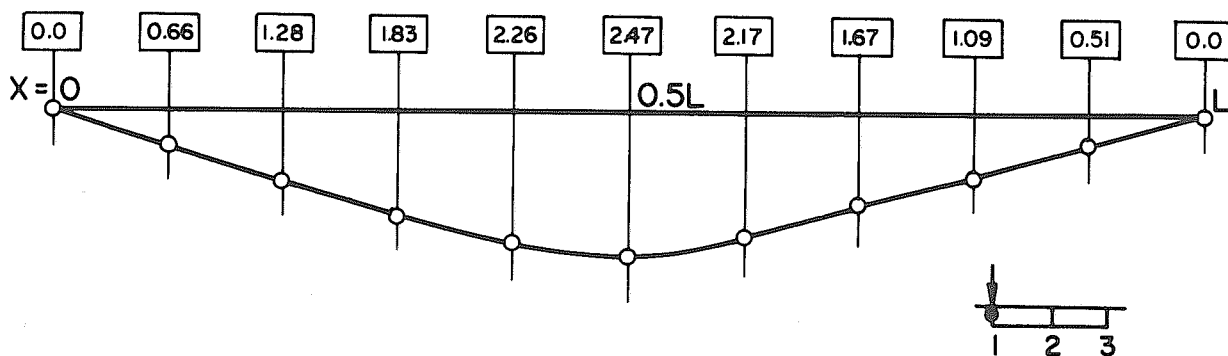


FIG.16 EXAMPLE 5 - FINITE ELEMENT IDEALIZATION

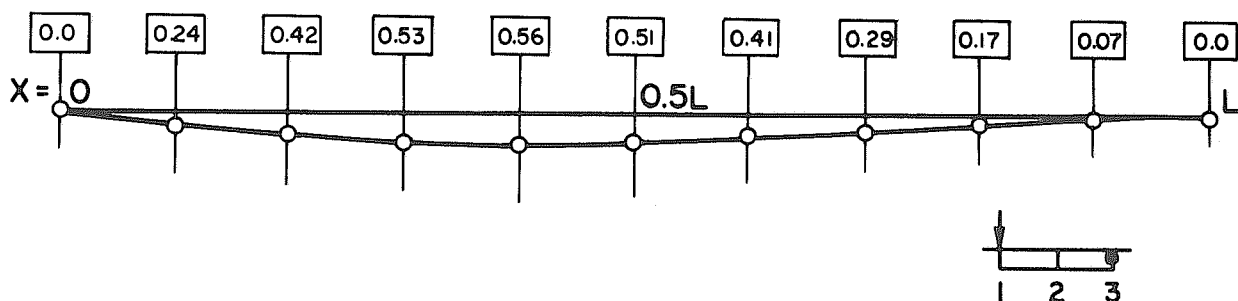




TRANSVERSE DISTRIBUTION OF W AT TOP DECK OF MIDSPAN CROSS-SECTION

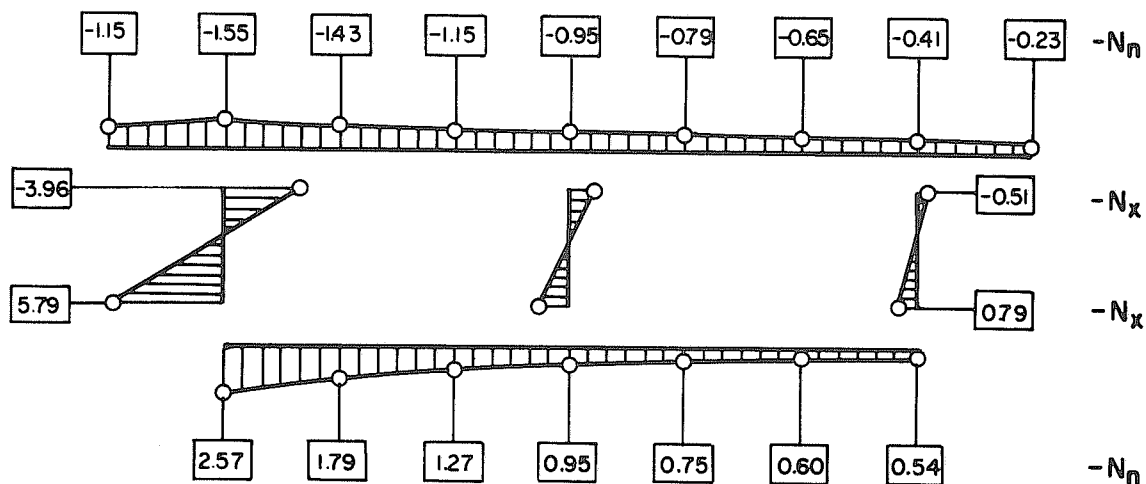


LONGITUDINAL VARIATION OF W ALONG TOP OF GIRDER 1

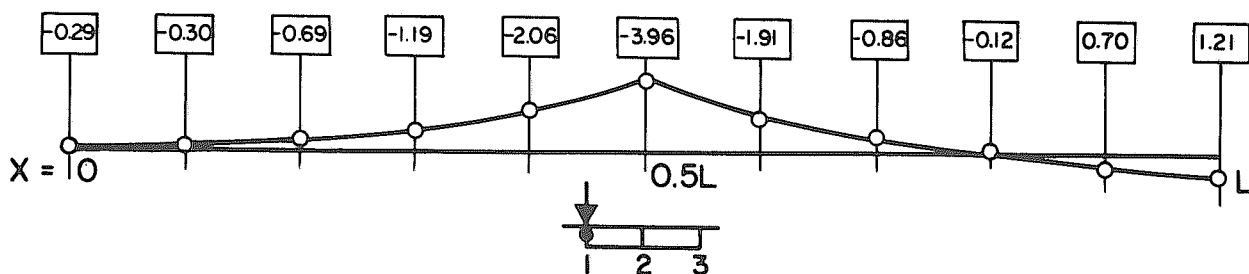


LONGITUDINAL VARIATION OF W ALONG TOP OF GIRDER 3

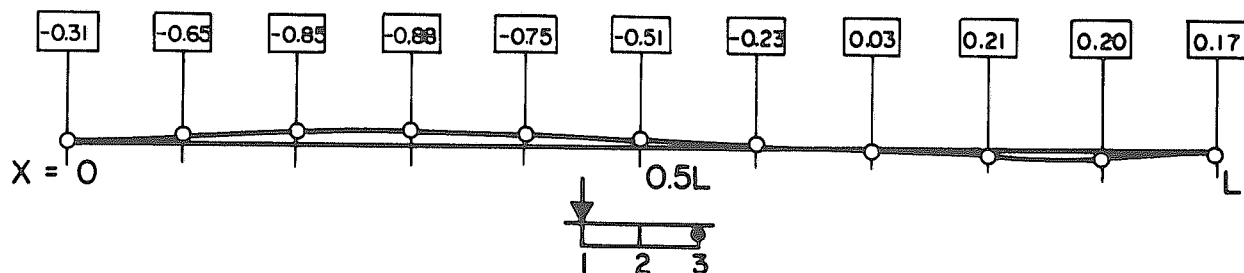
FIG. 17 EXAMPLE 5 - SKEWED BOX, VERTICAL DISPLACEMENTS  
W (ft · 10<sup>-4</sup>)



TRANSVERSE DISTRIBUTION OF  $N_n$  AND  $N_x$  AT MIDSPAN CROSS-SECTION PARALLEL TO SKEWED SUPPORTS



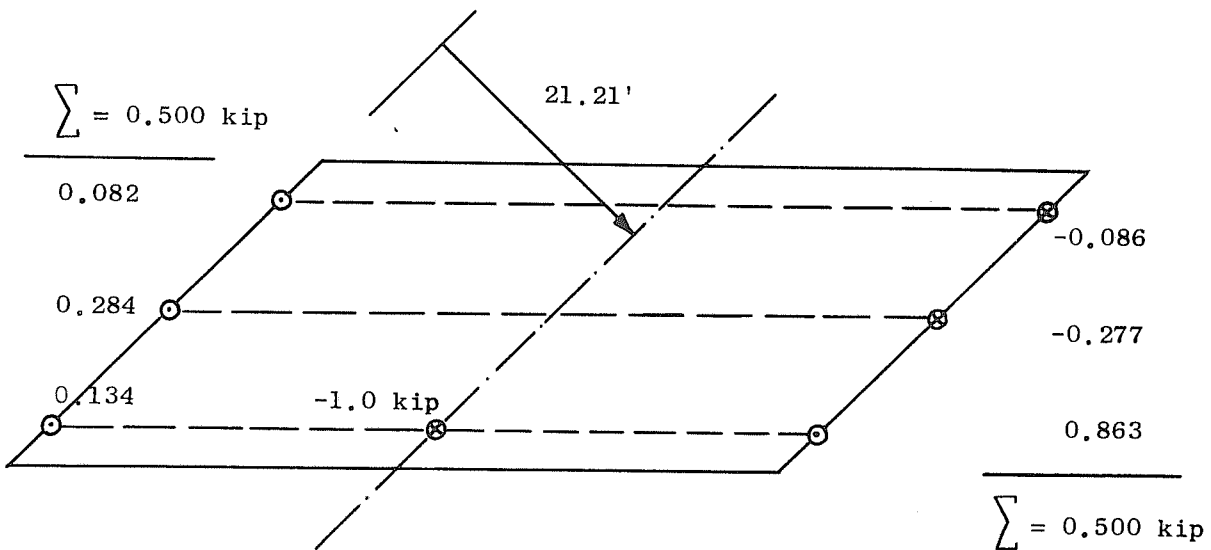
LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 1



LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 3

FIG. 18 EXAMPLE 5 - SKEWED BOX, LONGITUDINAL STRESS RESULTANTS  $N_x$  AND  $N_n$  (K/FT · 10<sup>-1</sup>)

span, the increase in effective moment of inertia and the clamping effect at the corners. This damping effect causes a change in sign of the longitudinal  $N_x$  stress resultants along the top of the outside girders. This quantity provides some insight into the load distribution of a two cell box girder on skewed supports. The transverse distribution of  $N_n$  yields the information necessary for calculating the internal moment at the midspan cross-section. As this section is taken parallel to the skewed supports the total sum of reactions at each support is known from statics and the sum of the reactions obtained from the computer output should equal this value. Hence, the gross statical moment,  $M_{stat}$ , can be easily computed for any section parallel to the skewed supports. For completeness, the magnitudes of individual reactions are given below as obtained from the finite element analysis. For this case,  $M_{stat} = 0.500 \times (21.21) = 10.61$  k-ft.



$M_N$ , the contribution of the internal stress resultants normal to the skewed midspan section, is obtained by taking moments of the resulting stress distribution about the neutral axis.  $M_M$ , the contribution of the internal moments normal to the skewed midspan section, is determined by numerical integration of the normal moments at this section. The total contribution of internal stress and moment resultants to the gross internal moment at midspan is given by

$$M_{int} = M_N + M_M = 9.92 + 0.33 = 10.25 \text{ k-ft.}$$

while the coefficient  $\mu$  is computed from

$$\mu = \frac{M_{int}}{M_{stat}} = \frac{10.25}{10.61} = 0.97$$

Observe that the resulting stress distribution at midspan satisfies equilibrium within 3%. This violation of statics is negligible if one considers that the section is taken directly underneath the concentrated loading.

The total computer time necessary for execution of this problem amounted to 3 minutes and 13 seconds. A detailed discussion of execution times as obtained on the CDC 6400 computer is presented in Section 5.7.

### 6.5 Example 6 - Two Cell Box Girder Highway Branch

This example has been chosen to demonstrate the versatility of the finite element program CELL as applied to the analysis of a cellular highway branch. The geometry and material properties of the box girder bridge to be analyzed are described in Fig. 19 together with its loading and boundary conditions.

As no other analytical tool is presently available for the solution of problems with such complex geometric configurations, no comparison with alternate results can be made. In order to assess the accuracy of the internal stress distribution equilibrium checks are performed as described in the discussion of the previous examples.

The curved box structure is idealized by the finite element mesh layout illustrated in Fig. 20. Due to symmetry of the geometry, only half of the structure need to be considered in the analysis. Two separate cases have to be treated to account for the non-symmetric loading acting on the actual structure. First, half of the structure is analyzed by imposing symmetric boundary conditions in form of  $V = \theta_X = 0$  at the nodes in the plane of symmetry, and by applying half of the actual loading. Second, the same half is analyzed by imposing anti-symmetric boundary conditions in form of  $W = \theta_Y = U = 0$  at the nodes in the plane of symmetry, and by applying half of the actual loading. The final results for the actual structure are obtained by simply superimposing the solutions of both analyses previously described. Analogous to the previous examples, each cell is idealized by three elements for top and bottom deck and by one element for the webs.

Figure 21 illustrates the distribution of vertical displacements  $W$  at the Section of  $X = 0.4L$  along which a transverse diaphragm is located and also along the top of the longitudinal outside girders. Observe the effect of this transverse diaphragm upon the transverse distribution of  $W$  at the same section enforcing a linear response. A comparison of the displacements with those obtained from the analysis of the non-skew box girder bridge in Example 4 indicates a considerable reduction in the vertical displacements  $W$ . It is interesting to note that the displacements in the loaded outside girder agree very well with those of the skew box girder bridge in Example 5 in contrast to the displacements along the unloaded outside girder. This observation is easily explained by the different load distribution in those two structures mainly due to the inclusion of a transverse diaphragm and due to the clamping effects at the corners of the skewed box girder bridge.

Figure 22 illustrates the distribution of the longitudinal stress resultant  $N_x$  at the cross-section  $X = 0.4L$  and along the top of the outside girders. The distribution along the longitudinal girders provide some insight into the load distribution of this structure. Observe the similarity with the analogous results obtained for the non-skew box girder bridge in Example 4. The improvement in the load distribution becomes apparent, which is caused mainly by the inclusion of a midspan diaphragm. The transverse distribution of  $N_x$  yields the information necessary for an internal equilibrium check. The internal moment  $M_{int}$  is determined by numerical integration of the internal forces  $N_x$  and  $M_x$  at the section of the transverse

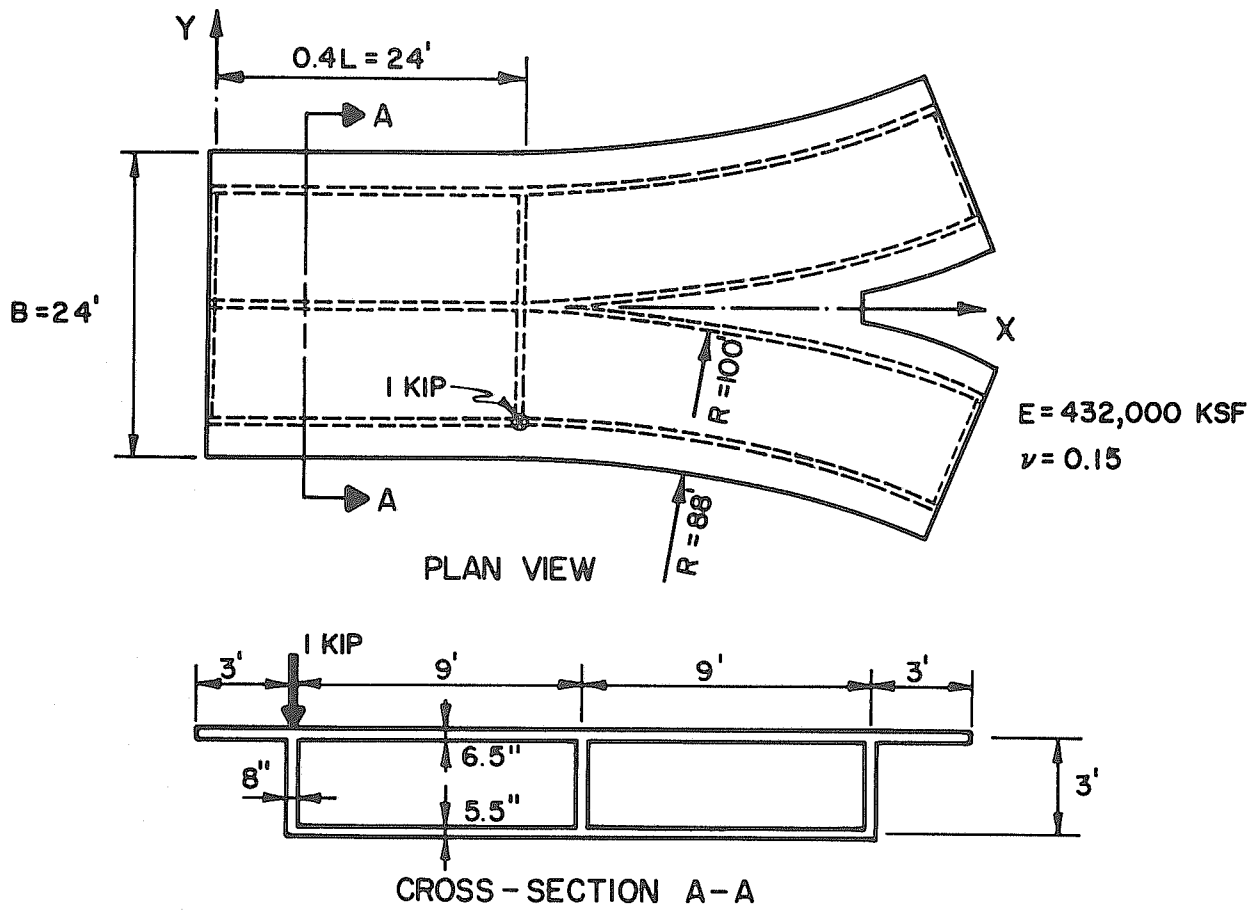


FIG. 19 EXAMPLE 6 - TWO CELL HIGHWAY BOX GIRDER BRANCH

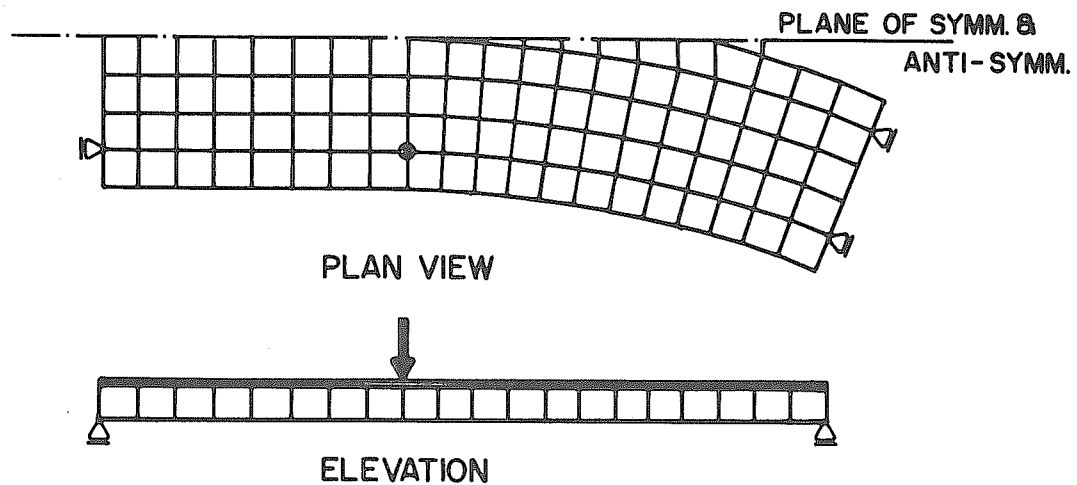
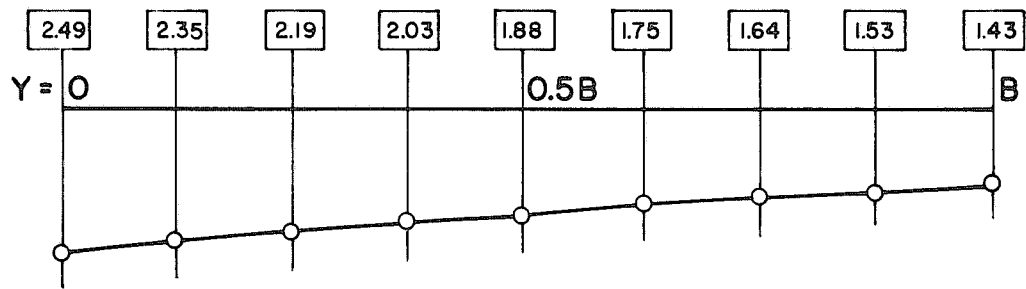
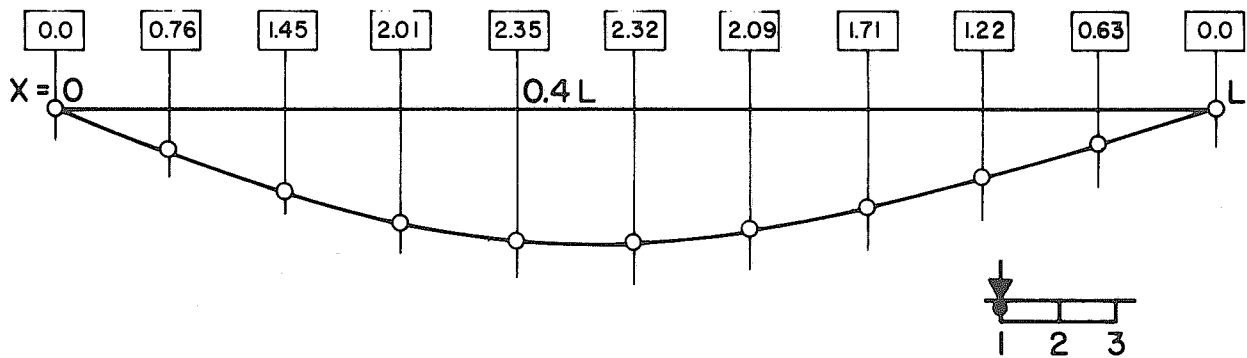


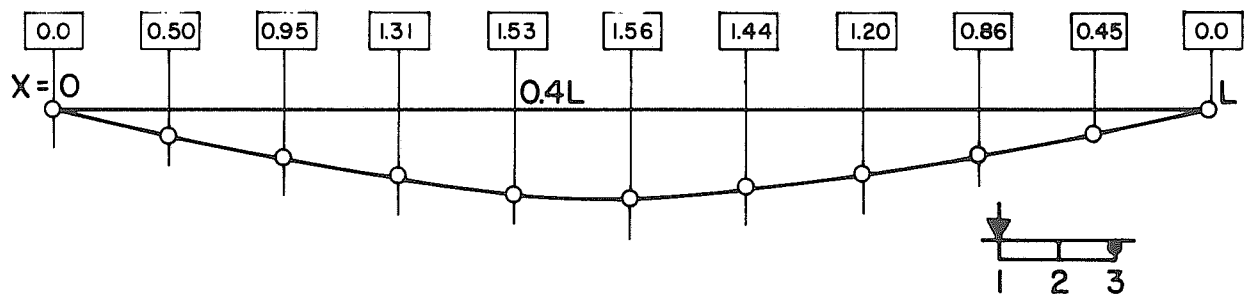
FIG. 20 EXAMPLE 6 - FINITE ELEMENT IDEALIZATION



TRANSVERSE DISTRIBUTION OF W AT THE TOP DECK OF SECTION  $X = 0.4L$



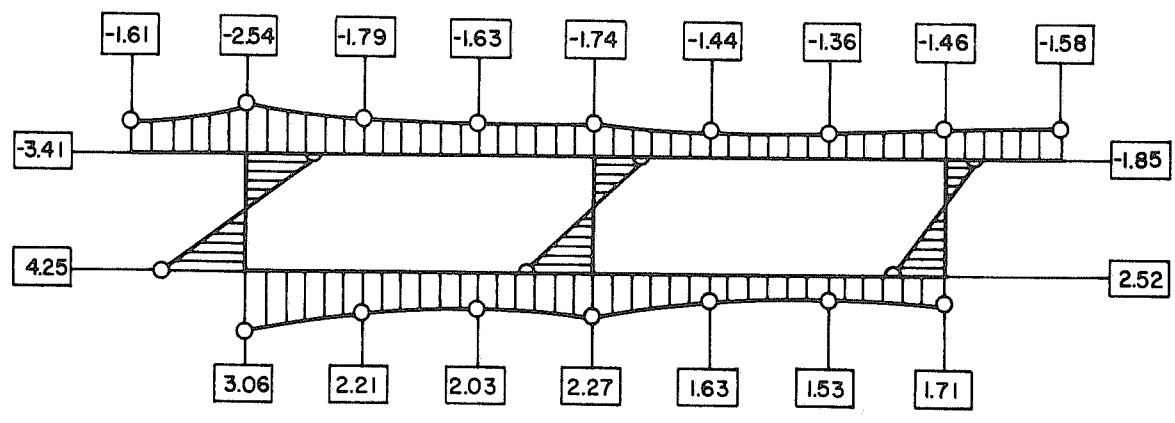
LONGITUDINAL VARIATION OF W ALONG TOP OF GIRDER 1



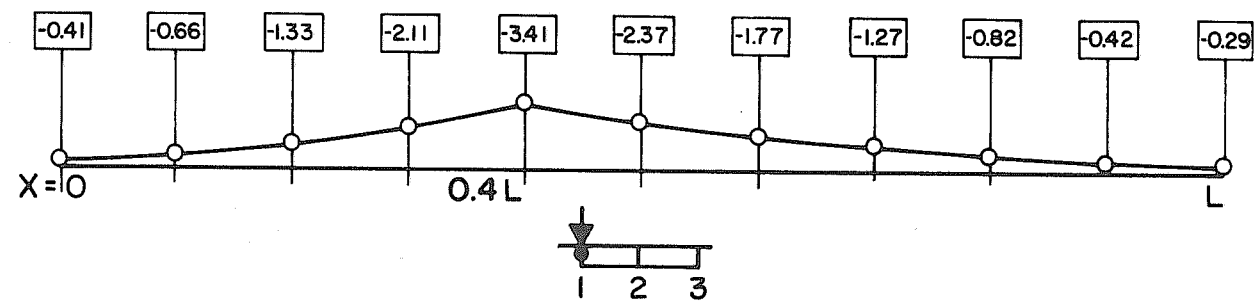
LONGITUDINAL VARIATION OF W ALONG TOP OF GIRDER 3

FIG. 21 EXAMPLE 6 - HIGHWAY BRANCH, VERTICAL DISPLACEMENTS  $W$  (FT · 10<sup>-4</sup>)

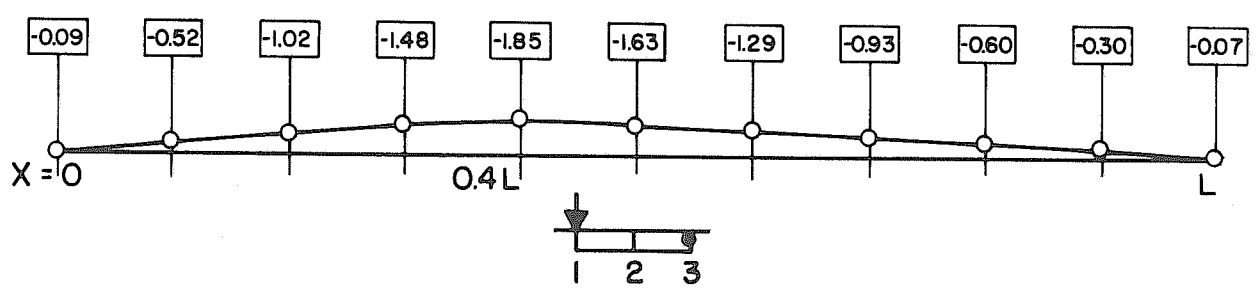




TRANSVERSE DISTRIBUTION OF  $N_x$  AT SECTION  $X=0.4L$  UNDERNEATH THE LOADING



LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 1



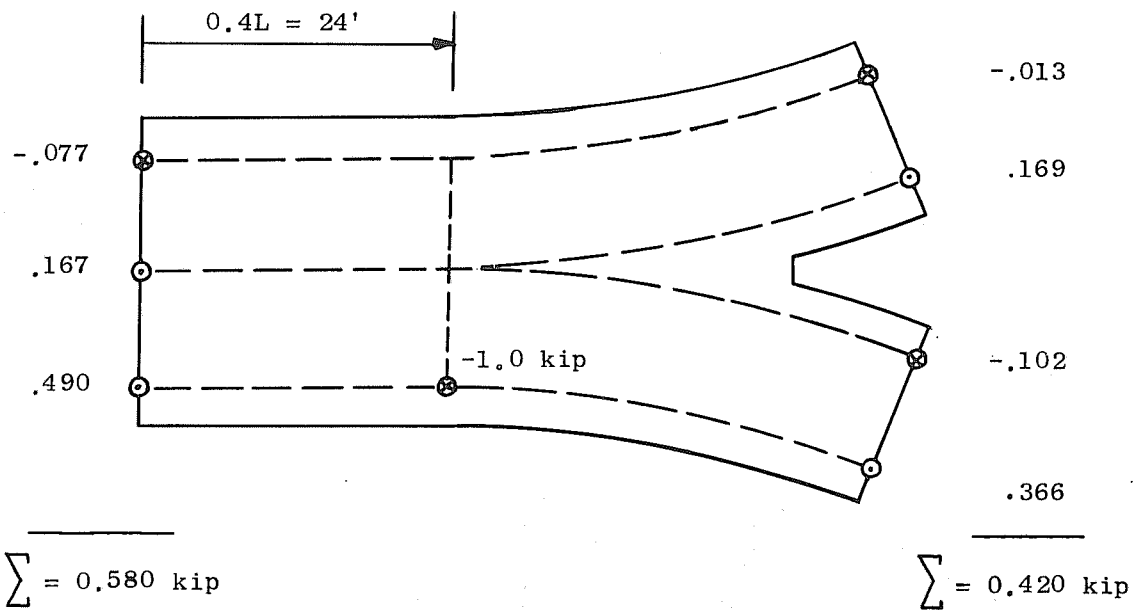
LONGITUDINAL VARIATION OF  $N_x$  AT TOP OF GIRDER 3

FIG. 22 EXAMPLE 6 - HIGHWAY BRANCH, LONGITUDINAL STRESS-RESULTANTS  $N_x$  (K/FT · 10<sup>-1</sup>)

diaphragm,  $X = 0.4L$ , where the load is applied. The gross internal moment is given by

$$M_{int} = M_N + M_M = 12.84 + 0.22 = 13.06 \text{ k-ft.}$$

The gross external moment is computed from overall statical considerations. The finite element analysis yields magnitudes of individual reactions as follows



The total static moment at the cross-section  $X = 0.4L = 24 \text{ ft.}$  is given by  $M_{stat} = 0.580 \times (24.0) = 13.92 \text{ k-ft.}$  The coefficient  $\mu$  can now be calculated to provide some insight into the accuracy of the resulting stress distribution

$$\mu = \frac{M_{\text{int}}}{M_{\text{stat}}} = \frac{13.06}{13.92} = 0.94$$

Observe that a 6% violation of equilibrium by the internal stress distribution can be neglected considering that equilibrium was checked at a section directly underneath the concentrated loading.

The total computer time necessary for execution of this problem amounted to 1 minute and 41 seconds for each of the two cases analyzed. A detailed discussion of execution times as obtained on the CDC 6400 is presented in Section 5.7.

## 7. ACKNOWLEDGEMENTS

This investigation was sponsored by the Division of Highways, Department of Public Works, State of California, and the Bureau of Public Roads, Federal Highway Administration, United States Department of Transportation. The opinions, findings, and conclusions expressed in this report are those of the authors and not necessarily those of the Bureau of Public Roads.

Mr. G. D. Mancarti, Assistant Bridge Engineer, and Mr. R. E. Davis, Senior Bridge Engineer, of the Research and Development Section, provided close liaison from the Bridge Department, Division of Highways, State of California.

The support of the Computer Center at the University of California, Berkeley, is gratefully acknowledged for providing its facilities.

## 8. REFERENCES

1. Scordelis, A. C., "Analysis of Simply Supported Box Girder Bridges," UC-SESM Report No. 66-17, University of California, Berkeley, October 1966.
2. Scordelis, A. C., "Analysis of Continuous Box Girder Bridges," UC-SESM Report No. 67-25, University of California, Berkeley, November 1967.
3. Willam, K. J. and Scordelis, A. C., "Analysis of Orthotropic Folded Plates with Eccentric Stiffeners," UC-SESM Report No. 70-2, University of California, Berkeley, February 1970.
4. Meyer, C. and Scordelis, A. C., "Computer Program for Prismatic Folded Plates with Plate and Beam Elements," UC-SESM Report No. 70-3, University of California, Berkeley, February 1970.
5. Willam, K. J., "Finite Element Analysis of Cellular Structures," Ph.D. dissertation submitted to the Division of Structural Engineering and Structural Mechanics, Department of Civil Engineering, University of California, Berkeley, December 1969.
6. Clough, R. W. and Felippa, C. A., "A Refined Quadrilateral for Analysis of Plate Bending," 2nd Conference on Matrix Methods in Structural Mechanics, Wright-Patterson AFB, Ohio, 1968.
7. Felippa, C. A., "Refined Finite Element Analysis of Linear and Nonlinear Two-Dimensional Structures," UC-SESM Report No. 66-22, University of California, Berkeley, 1966.
8. Morley, L. S. D., "Skew Plates and Structures," International Series of Monographs in Aeronautics and Astronautics, Volume 5, MacMillan Co., New York, 1963.

## APPENDIX A

### Source Listing of Computer Program CELL

Considerable time, effort and expense have gone into the development of this computer program. It is obvious that it should be used only under the conditions and assumptions for which it was developed. These are described in the report. Although the program has been extensively tested by the authors, no warranty is made regarding the accuracy and reliability of the program and no responsibility is assumed by the authors or the sponsors of this research project.

```

OVERLAY (BOX,0,0)
PROGRAM CELL (INPUT,OUTPUT,PUNCHD,TAPE1,TAPE3,TAPER,
* TAPE7=PUNCHB)
C *****
C LINEAR ELASTIC ANALYSIS OF CELLULAR SYSTEMS WITH ARBITRARY
C GEOMETRY INCLUDING TWO WAY BENDING IN DECK AND ONE WAY BENDING
C IN WEBELEMENTS, DEPTH IS CONSTANT, 5 DEGREES OF FREEDOM/NODE
C *****
C PROGRAMMED ON CDC 6400 BY KASPAR J. WILLAM
C UNIVERSITY OF CALIFORNIA, BERKELEY
C *****
C COMMON / SETUP / NUMNP,NBEM,NMAT,NRUN,NFLDL,IBANDW,NEQ,
C COMMON / POCGN / NBC(200),ANGLE(200)
C DIMENSION TIM(10)
C DATA STOPP /4HSTOP/, SFLAG /5HSTART/, PFLAG /4HLOAD/
C *****
C ASSUME BLOCKLENGTH FOR PARTITIONING OF STRUCTURAL STIFFNESS
C *****
C 500 READ 400, CHECK
C IF (CHECK.EQ.STOPP) GO TO 600
C IF (CHECK.EQ.SFLAG) GO TO 100
C IF (CHECK.EQ.PFLAG) GO TO 200
C GO TO 500
C *****
C SEQUENTIAL CALL OF FIVE PRIMARY OVERLAYS
C *****
C 100 NFLDL = 4200B
C CALL RFL (NFLDL)
C *****
C INPUT DATA AND GENERATE ELEMENT STIFFNESSES
C *****
C NRUN = 1
C NA = 1
C CALL SECOND (TIM(1))
C CALL OVERLAY (3HBOX,1,0,0)
C CALL SECOND (TIM(2))
C *****
C ASSEMBLE STRUCTURAL STIFFNESS IN BLOCKFORM
C *****
C CALL SECOND (TIM(3))
C CALL OVERLAY (3HBOX,2,0,0)
C CALL SECOND (TIM(4))
C *****
C DETERMINE LOAD VECTOR IN BLOCKFORM
C *****
C 450 READ 400, CHECK
C IF (CHECK.EQ.STOPP) GO TO 600
C IF (CHECK.EQ.SFLAG) GO TO 100
C IF (CHECK.EQ.PFLAG) GO TO 200
C GO TO 450
C *****
C 200 CALL SECOND (TIM(5))
C CALL OVERLAY (3HBOX,3,0,0)
C CALL SECOND (TIM(6))

```

```

CELL 1
CELL 2
CELL 3
CELL 4
CELL 5
CELL 6
CELL 7
CELL 8
CELL 9
CELL 10
CELL 11
CELL 12
CELL 13
CELL 14
CELL 15
CELL 16
CELL 17
CELL 18
CELL 19
CELL 20
CELL 21
CELL 22
CELL 23
CELL 24
CELL 25
CELL 26
CELL 27
CELL 28
CELL 29
CELL 30
CELL 31
CELL 32
CELL 33
CELL 34
CELL 35
CELL 36
CELL 37
CELL 38
CELL 39
CELL 40
CELL 41
CELL 42
CELL 43
CELL 44
CELL 45
CELL 46
CELL 47
CELL 48
CELL 49
CELL 50
CELL 51
CELL 52
CELL 53
CELL 54
CELL 55

```

```

DIRECT SOLUTION OF SET OF LINEAR EQUATIONS IN BLOCKFORM
CALL SECOND (TIM(7))
CALL OVERLAY (3HBOX,4,0,0)
CALL SECOND (TIM(8))
*****
OUTPUT OF NODAL DISPLACEMENTS, NODAL FORCES AND INTERNAL FORCES
CALL SECOND (TIM(9))
CALL OVERLAY (3HBOX,5,0,0)
CALL SECOND (TIM(10))
IF (NRUN.GT.1) NA=3
PRINT 310
DO 300 I=NA,5
J = 2*I
K = J-1
TIM(I) = TIM(J) - TIM(K)
300 PRINT 320,I,TIM(I)
NEQ = 10*NUMNP
PRINT 330,IBANDW, NEQ
NRUN = NRUN+1
GO TO 500
310 FORMAT (///25F1 TIME USED IN EACH LINK ///)
320 FORMAT (6F LINK I3, F10.2/)
330 FORMAT (//14F BANDWIDTH I5/
* 14H NO CF EQUAT I5//)
400 FORMAT (A6)
600 STOP
END

```

```

CELL 56
CELL 57
CELL 58
CELL 59
CELL 60
CELL 61
CELL 62
CELL 63
CELL 64
CELL 65
CELL 66
CELL 67
CELL 68
CELL 69
CELL 70
CELL 71
CELL 72
CELL 73
CELL 74
CELL 75
CELL 76
CELL 77
CELL 78
CELL 79
CELL 80
CELL 81
CELL 82
CELL 83
CELL 84
CELL 85

```





```

INPU 57      NCT = NCT+1
INPU 58      NI = N-MOD
INPU 59      N2 = NI-MCC
INPU 60      IF (NI.LE.0.OR.N2.LE.0) GO TO 54
INPU 61      DO 44 I=1,4
INPU 62      44 NPIN(I) = 2*NPIN(I) - NP(N2,I)
INPU 63      40 MAT(N) = PATIN(I)
INPU 64      NAT(N) = NATIN(I)
INPU 65      46 NI = N
INPU 66      N = N+1
INPU 67      IF (L-N) 34,22,30
INPU 68      L = N
INPU 69      IF (MOD.GT.0.AND.NI.LT.NLIM) GO TO 42
INPU 70      IF (NI-NUMEL) 20,70,56
INPU 71      C
INPU 72      C   ERROR EXITS FOR DECK ELEMENT GENERATION
INPU 73      C
INPU 74      50 PRINT 60, L
INPU 75      IFLAG = 1
INPU 76      GO TO 7C
INPU 77      52 PRINT 62
INPU 78      IFLAG = 1
INPU 79      GO TO 7C
INPU 80      54 PRINT 64
INPU 81      IFLAG = 1
INPU 82      GO TO 7C
INPU 83      56 PRINT 66, NI,NUMEL
INPU 84      60 FORMAT (//17H ELEMENT CARD N I4,16H NOT IN SEQUENCE )
INPU 85      62 FORMAT (//27H FIRST ELEMENT CARD MISSING )
INPU 86      64 FORMAT (//50H INSUFFICIENT INFORMATIC TO GENERATE DECK MESH )
INPU 87      66 FORMAT (//17H ELEMENT NUMBER I4,22H EXCEEDS GIVEN NUMEL I4)
INPU 88      68 FORMAT (//34H DECK ELEMENT GENERATION WITH MOD= I4,6H NLIM= I4)
INPU 89      C
INPU 90      C   INPUT AND GENERATION OF WEB ELEMENT ARRAY
INPU 91      C
INPU 92      C
INPU 93      70 N = 1
INPU 94      71 READ 99, L,(NPT(I),I=1,2),MC,NDIF,MOD,NLIM
INPU 95      99 FORMAT (7I4)
INPU 96      NCT = 0
INPU 97      IF (L-N) 90,75,80
INPU 98      75 DO 72 I=1,2
INPU 99      72 NBPN(I) = NPT(I)
INPU 100      MBAT(L) = MC
INPU 101      GO TO 73
INPU 102      80 IF (N.LE.1) GO TO 92
INPU 103      IF (NCF.EQ.0) NDIF=1
INPU 104      DO 76 I=1,2
INPU 105      76 NBPN(I) = NBP(N-1,I)*NDIF
INPU 106      NI = N-1
INPU 107      GO TO 74
INPU 108      82 IF (NCT.EQ.0) PRINT 98, MOD,NLIM
INPU 109      NCT = NCT+1
INPU 110      NI = N-MOD
INPU 111      N2 = NI-MCC
INPU 112      IF (NI.LE.0.OR.N2.LE.0) GO TO 54
          DD 78 I=1,2

```

```

INPU 1      SURROUTINE INPU (NP,NBP,MAT,NAT,MBAT,BSIA,BCDS,XORD,DM,DS,
INPU 2      EL,EZ,PR,GI,Z,ANG,TH,IC,IP,M,NUMBC)
INPU 3      C
INPU 4      C*****
INPU 5      THIS SUBROUTINE READS IN DATA, GENERATES NODAL MESH AND
INPU 6      COORDINATES AND EVALUATES THE PLANE STRESS AND PLATE BENDING
INPU 7      STIFFNESS MATRICES FOR THE DECK AND SPAR ELEMENTS
INPU 8      GEOMETRY PLAN ARBITRARY QUADRILATERAL BOX WITH CONST DEPTH
INPU 9      C*****
INPU 10     COMMON / SETUP / NUMNP,NUMEL,NBPA,MAT,NRUN,NFLDL,I,BANDW,NEQ,
INPU 11     NBLKL,NUMBLK,NEQBCC,MARG,AH
INPU 12     *
INPU 13     COMMON / EOCEN / XERC(ZCC),ANGLEI(200)
INPU 14     COMMON / PLSTR / XA(4),YA(4),CS(6),SPDI(2,12),RI(12),ESIG(5,3)
INPU 15     COMMON / PLBOG / XX(5),YY(5),CM(3,3),PP(5),BM(3,5),CV(3,5),
INPU 16     SPB(19,19),VALI(5)
INPU 17     *
INPU 18     DIMENSION NP(IE,4),NBP(18,2),MAT(IE),NAT(IE),MBAT(18),BSIN(18),
INPU 19     BCS(18),XORD(IP),YORD(IP),DM(IM,3,3),DS(IM,6),EI(IM),
INPU 20     EZ(IM),PR(IM),G(12,IM),ANG(IM),TH(IM)
INPU 21     *
INPU 22     DIMENSION ITE(2),IPE(4),NP(4),XD(4),YC(4),XT(4),YT(4),
INPU 23     ST(20,20),SB(20,20),SBI(19,19),SBB(19,19),S(20,20)
INPU 24     DATA ITE /1,4/, IPE /2,3,4,1/
INPU 25     LOGICAL J1, J2, J3, J4, J5, JB
INPU 26     C
INPU 27     C   INITIALIZATION
INPU 28     C
INPU 29     IFLAG = 0
INPU 30     IMAT = 0
INPU 31     MAXBC = 200
INPU 32     MAXPD = 130
INPU 33     C*****
INPU 34     READ AND ECHO GF INPUT DATAS
INPU 35     C*****
INPU 36     C
INPU 37     C   INPUT AND GENERATION OF DECK ELEMENT ARRAY
INPU 38     C
INPU 39     C
INPU 40     N = 1
INPU 41     20 READ 18, L,(NPT(I),I=1,4),PA,PB,NDIF,MCO,NLIM
INPU 42     18 FORMAT (10I4)
INPU 43     NCT = 0
INPU 44     IF (L-N) 50,22,30
INPU 45     22 DO 24 I=1,4
INPU 46     24 NP(N,I) = NPT(I)
INPU 47     MAT(L) = PA
INPU 48     NAT(L) = PB
INPU 49     GO TO 46
INPU 50     30 IF (N.LE.1) GO TO 52
INPU 51     IF (NDIF.EQ.0) NDIF=1
INPU 52     DD 32 I=1,4
INPU 53     32 NP(N,I) = NP(N-1,I)*NDIF
INPU 54     NI = N-1
INPU 55     GO TO 40
INPU 56     42 IF (NCT.EQ.0) PRINT 68, MOD,NLIM

```

```

180 PRINT 190
IFLAG = 1
GO TO 200
182 PRINT 192, N
IFLAG = 1
GO TO 200
186 PRINT 196, N, NUMNP
IFLAG = 1
112 FORMAT (14,2F10.2,3I4)
170 FORMAT (/ / 34F NODAL POINT CARD MISSING )
192 FORMAT (/ / 27F NODAL POINT CARD FOR NODE = 14, 16H NOT IN SEQUENCE)
196 FORMAT (/ / 12H NODE NUMBER 14, 22H EXCEEDS GIVEN NUMNP = 14)
C INPUT CF ELEMENT MATERIAL PROPERTIES
200 READ 210, (N, E1(I), E2(I), G12(I), PR(I), ANG(I), TH(I), I=1, NMAT)
IF (N.GT.0) GO TO 332
IMAT = 1
READ 210, (N, DOS(I, J), J=1, 6), I=1, NMAT)
DO 212 I=1, NMAT
DM(I, 1, 1) = E1(I)
DM(I, 2, 2) = E2(I)
DM(I, 3, 3) = G12(I)
DM(I, 1, 2) = PR(I)
DM(I, 1, 3) = ANG(I)
DM(I, 2, 3) = TH(I)
DM(I, 3, 1) = E1(I)
DM(I, 3, 2) = E2(I)
DM(I, 3, 3) = G12(I)
DM(I, 3, 1) = PR(I)
DM(I, 3, 2) = ANG(I)
DM(I, 3, 3) = TH(I)
DM(I, 3, 1) = E1(I)
DM(I, 3, 2) = E2(I)
DM(I, 3, 3) = G12(I)
DM(I, 3, 1) = PR(I)
DM(I, 3, 2) = ANG(I)
DM(I, 3, 3) = TH(I)
212 DM(I, 3, 2) = TH(I)
C DETERMINATION OF BANDWIDTH AND DEGREES OF FREEDOM
332 NPD = 0
DO 385 N=1, NUMEL
DO 385 I=1, 4
K = NP(IN, I)
DO 385 J=1, I
L = TABS(NP(N, J) - K)
IF (NPD.LT.L) NPD=L
385 CONTINUE
IBANDW = 10*(NPD+1)
NEQ = NUMNP*10
IF (NBLK1.NE.0) GC TO 330
NLK2 = (1200008-4*NUMEL-2*NBEAM)/(2*IBANDW)
NLK4 = (1200008-NEQ)/(2*IBANDW+1)
NBLK1 = MINO (NLK2, NLK4)
IF (NBLK1.GT.NEQ) NBLK1=NEQ
IF (IBANDW.GT.NBLK1) GO TO 1010
340 NUMBLK = (NEQ-1)/NBLK1+1
C ECHO OF INPUT INFORMATION
PRINT 363, NEQ, IBANDW, NBLK1, NUMBLK
363 FORMAT (/ / 20H NUMBER OF EQUATIONS 15/

```

```

INPU 113
INPU 114
INPU 115
INPU 116
INPU 117
INPU 118
INPU 119
INPU 120
INPU 121
INPU 122
INPU 123
INPU 124
INPU 125
INPU 126
INPU 127
INPU 128
INPU 129
INPU 130
INPU 131
INPU 132
INPU 133
INPU 134
INPU 135
INPU 136
INPU 137
INPU 138
INPU 139
INPU 140
INPU 141
INPU 142
INPU 143
INPU 144
INPU 145
INPU 146
INPU 147
INPU 148
INPU 149
INPU 150
INPU 151
INPU 152
INPU 153
INPU 154
INPU 155
INPU 156
INPU 157
INPU 158
INPU 159
INPU 160
INPU 161
INPU 162
INPU 163
INPU 164
INPU 165
INPU 166
INPU 167
INPU 168
75 MAP(N, I) = 2*MAP(N1, I) - MAP(N2, I)
74 MRAT(N) = MUAT(N1)
73 N1 = N
N = N*1
IF (L=N) 79, 75, 80
79 L = N
IF (MOD.GT.0.AND.N1.LT.NLIM) GC TO 82
IF (N1-NBEAM) 71, 100, 96
C ERROR EXITS FOR WEB ELEMENT GENERATION
90 PRINT 91, L
IFLAG = 1
GO TO 100
92 PRINT 93
IFLAG = 1
GO TO 100
94 PRINT 95
IFLAG = 1
GO TO 100
96 PRINT 97, N1, NBEAM
91 FORMAT (/ / 13H WEB CARD N 14, 16H NOT IN SEQUENCE)
93 FORMAT (/ / 23H FIRST WEB CARD MISSING )
95 FORMAT (/ / 50H INSUFFICIENT INFORMATION TO GENERATE WEB MESH
97 FORMAT (/ / 20H WEB ELEMENT NUMBER 14, 22H EXCEEDS GIVEN NBEAM )
98 FORMAT (/ / 34H WEB ELEMENT GENERATION WITH MOD= 14, 6H NLIM= 14)
C INPUT AND GENERATION OF NODAL POINT COORDINATES
100 L = 0
110 READ 112, N1, XORD(N1), YORD(N1), MOD, NLIM
L1 = L+1
IF (N1.LE.L1) GC TC 120
IF (L1.LE.0) GO TO 180
DIV = N-L
DX = (XORD(N1)-XORD(L1))/DIV
DY = (YORD(N1)-YORD(L1))/DIV
DO 140 L=L1+N
XORD(L) = XORD(L-1) + DX
YORD(L) = YORD(L-1) + DY
GO TO 120
140 YORD(L) = YORD(L-1) + DY
150 L1 = N+1
PRINT 170, MOD, NLIM
160 N = N+1
N1 = N-MOD
N2 = N1-MOD
IF (N1.LE.0.OR.N2.LE.0) GO TO 182
XORD(N) = 2*XORD(N1) - XORD(N2)
YORD(N) = 2*YORD(N1) - YORD(N2)
IF (N1.LT.NLIM) GO TO 160
MOD = 0
120 L = N
IF (MOD.GT.0) GO TO 150
IF (N-NUMNP) 110, 200, 186
C ERROR EXITS FOR COORDINATE GENERATION

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INPU 281 DD 290 N=1,NUMBC
INPU 282 HEAD 240, M,J1,J2,J3,J4,J5,J8,ALF,BET
INPU 283 PRINT250, M,J1,J2,J3,J4,J5,J8,ALF,BET
INPU 284 K5 = M*10
INPU 285 K4 = K5-1
INPU 286 K3 = K4-1
INPU 287 K2 = K3-1
INPU 288 K1 = K2-1
INPU 289 L5 = K5-5
INPU 290 L4 = K4-5
INPU 291 L3 = K3-5
INPU 292 L2 = K2-5
INPU 293 L1 = K1-5
INPU 294 IF (.NOT.J1) GO TC 271
INPU 295 IF (JB) GO TO 275
INPU 296 J=J+1
INPU 297 NEBC(J) = K1
INPU 298 GO TO 271
INPU 299 275 J = J+1
INPU 300 NEBC(J) = L1
INPU 301 271 IF (.NOT.J2) GO TO 272
INPU 302 IF (JB) GO TO 276
INPU 303 J=J+1
INPU 304 NEBC(J) = K2
INPU 305 ANGLE(J) = ALF/57.29578
INPU 306 GO TO 272
INPU 307 276 J = J+1
INPU 308 NEBC(J) = L2
INPU 309 ANGLE(J) = ALF/57.29578
INPU 310 272 IF (.NOT.J3) GO TO 273
INPU 311 IF (JB) GO TO 277
INPU 312 J=J+1
INPU 313 NEBC(J) = K3
INPU 314 GO TO 273
INPU 315 277 J = J+1
INPU 316 NEBC(J) = L3
INPU 317 273 IF (.NOT.J4) GO TO 274
INPU 318 IF (JB) GO TO 278
INPU 319 J=J+1
INPU 320 NEBC(J) = K4
INPU 321 GO TO 274
INPU 322 278 J = J+1
INPU 323 NEBC(J) = L4
INPU 324 274 IF (.NOT.J5) GO TC 290
INPU 325 IF (JB) GO TO 279
INPU 326 J=J+1
INPU 327 NEBC(J) = K5
INPU 328 ANGLE(J) = BET/57.29578
INPU 329 GO TO 290
INPU 330 279 J = J+1
INPU 331 NEBC(J) = L5
INPU 332 ANGLE(J) = BET/57.29578
INPU 333 290 CONTINUE
INPU 334 NEQRG=J
INPU 335 240 FORMAT (I4,6L2,2F10.2)
INPU 336 230 FORMAT (20HBOUNDARY CONDITIONS///// 15X,

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INPU 225 20P BANDWIDTH 15/
INPU 226 20P RLCKG LENGTH 15/
INPU 227 20P NUMBER OF BLOCKS 151
INPU 228
INPU 229 PRINT 310
INPU 230 310 FORMAT (// 30P NODAL COORDINATES
INPU 231 * 5X, 12H NODAL POINT, 5X, 8H X-COORD, 4X, 8H Y-COORD //)
INPU 232 PRINT 320, (I, XGRD(I), YCRD(I), I=1, NUMNP)
INPU 233 320 FORMAT (I12, 5X, 2F12.4)
INPU 234 PRINT 382
INPU 235 PRINT 384, (N, (NP(N, I), I=1, 4), MAT(N), NAT(N), N=1, NUMEL)
INPU 236 PRINT 394, (N, (NBP(N, I), I=1, 2), MBAT(N), N=1, NBEAM)
INPU 237 382 FORMAT (// 20H DECK ELEMENT ARRAY //)
INPU 238 * 8H ELEMENT, 9X, 15H EXTERNAL NODES, 5X, 8H MAT TOP, 5X, 8H MAT BOT
INPU 239 * (//)
INPU 240 384 FORMAT (I5, I1X, 4I4, I10, I13)
INPU 241 392 FORMAT (// 20H WEB ELEMENT ARRAY //)
INPU 242 * 8H ELEMENT, 5X, 10H EXT NODES, 5X, 8H MAT
INPU 243 394 FORMAT (I5, 8X, 2I4, I10)
INPU 244 IF (IMAT.EQ.0) GO TO 222
INPU 245 * I=1, NMAT
INPU 246 PRINT 224, (I, DS(I, 1), DS(I, 2), DS(I, 3), DS(I, 4), DS(I, 5), DS(I, 6),
INPU 247 * (//)
INPU 248 PRINT 226, (I, DM(I, 1), DM(I, 2), DM(I, 3), DM(I, 4), DM(I, 5), DM(I, 6),
INPU 249 * (//)
INPU 250 224 FORMAT (// 48P MATERIAL LAM RELATING STRESS RES TO STRAINS
INPU 251 * 9H MAT. NO., 8X, 6H DS(1), 14X, 6H DS(2), 14X, 6H DS(3), 14X, 6H DS(4),
INPU 252 * 14X, 6H DS(5), 14X, 6H DS(6), // (I5, 6E20.5))
INPU 253 226 FORMAT (// 48P MATERIAL LAM RELATING MOMENTS TO CURVATURES
INPU 254 * 9H MAT. NO., 7X, 8H DM(1, 1), 12X, 8H DM(2, 1), 12X, 8H DM(3, 1), 12X,
INPU 255 * 8H DM(1, 2), 12X, 8H DM(1, 3), 12X, 8H DM(2, 3), // (I5, 6E20.5))
INPU 256 GO TO 390
INPU 257 222 PRINT 220, (I, E1(I), E2(I), G12(I), PR(I), ANG(I), TH(I), I=1, NMAT)
INPU 258 210 FORMAT (I4, 6F10.2)
INPU 259 220 FORMAT (// 20P MATERIAL PROPERTIES //)
INPU 260 * 5X, 15H ELAST MOD E1, 5X, 15H ELAST MOD E2,
INPU 261 * 5X, 15H ELAST MOD G12, 5X, 15H POISSONS RATIO, 5X, 15H PRINCIPAL
INPU 262 * 5X, 15H PL THICKNESS //
INPU 263 * (I5, 3E20.5, 3F20.5))
INPU 264
INPU 265 DETERMINATION OF DIRECTION COSINES FOR WEB ELEMENTS
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A-12

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YY(I) = YCRD(I)
PP(I) = 1.0
XA(I) = XCRD(I)
YA(I) = YCRO(I)
660 CONTINUE
C
C IF DECK ELEMENT STIFFNESS IS OF THE SAME TYPE AS PREVIOUS
C ONE ITS STIFFNESS IS NOT RECOMPUTED
C
DO 654 I=1,4
J = IPE(I)
XD(I) = XX(I)-XX(I)
YD(I) = YY(I)-YY(I)
IF (XD(I).NE.XT(I)) ITYPG=1
IF (YD(I).NE.YT(I)) ITYPG=1
XT(I) = XD(I)
YT(I) = YD(I)
654 YD(I) = YD(I)
IF (N.EQ.1) GO TO 670
IF (M.NE.PAT(N-1)) ITYPM=1
IF (ITYPG.EQ.0.AND.ITYPM.EQ.0) GO TO 656
670 DO 665 I=1,3
DO 665 J=1,3
DO 665 K=1,3
DO 675 I=1,6
675 CS(I) = DS(M,I)
CALL Q8D11 (2)
IF (NTRI.NE.3) GO TO 642
DO 646 I=1,8
SPD(I,1) = SPD(I,1) + SPD(I,7)
646 SPD(I,2) = SPD(I,2) + SPD(I,8)
DO 648 I=1,8
SPD(I,1) = SPD(I,1) + SPD(I,7)
648 SPD(I,2) = SPD(I,2) + SPD(I,8)
DO 647 J=1,2
DO 647 I=1,8
SPD(I,J+6) = 0.0
647 SPD(J+6,I) = 0.0
642 CALL SPLATE (NTRI)
CALL DECK (S)
IF (NCT.GT.0) GO TO 658
DO 662 I=1,361
662 SBT(I) = SBT(I)
DO 664 I=1,400
664 ST(I) = S(I)
GO TO 656
658 DO 666 I=1,361
666 SBT(I) = SBT(I)
DO 668 I=1,400
668 SB(I) = S(I)
680 WRITE (1) SB,SB8
680 WRITE (3) SB
GO TO 650
656 WRITE (1) ST,SBT
M = NAT(N)

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A-11

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* 12H NODAL POINT ,3X,7H ,TAG-U ,3X,7H ,TAG-V ,3X,7H ,TAG-W ,3X,
* 7H ,TAG-DX ,3X,7H ,TAG-DY ,3X,8H ,TAG-ROT,ZX+9H ,ANGLE-V ,2X,
* 9H ,ANGLE-DY ,777)
250 FORMAT (15X,I7,2X,6L10,2X,2F13.2)
C
C CHECK IF DATA WITHIN LIMITATIONS, OTHERWISE STOP
C
262 IF (NEQBC.GT.MAXBC) GO TO 1020
263 IF (IBANDM.GT.MAXPD)GO TO 1030
264 IF ((FLAG.EQ.1) STOP
C
C FORMATION OF ANISOTROPIC MATERIAL LAW
C
IF (IMAT.EC.1) GO TO 64C
DO 630 M=1,NMAT
EA = E1(M)
EB = E2(M)
XU = PR(M)
GA = G12(M)
AG = ANG(M)
TS = TH(M)
TM = TS**3/12.C
CALL FORMC (EA,EB,XU,GA,AG,CM,CS)
DO 632 I=1,3
DO 632 J=1,3
632 DM(I,J) = CM(I,J)*TM
DO 634 I=1,6
634 DS(M,I) = CS(I)*TS
630 CONTINUE
C
C STORE ELEMENT AND COORDINATE ARRAYS ON TAPE 1
C
660 REWIND 1
REWIND 3
WRITE (1) NP,NP
WRITE (1) XORD,YORD,MAT,NAT,MBAT,DM,DS,BSIN,BCOS
C*****
C DETERMINATION OF ELEMENT STIFFNESSES
C*****
C FORMATION OF GLOBAL DECK ELEMENT STIFFNESS
C
DO 652 I=1,4
XT(I) = 0.0
652 YT(I) = 0.0
DO 650 N=1,NUMEL
M = MAT(N)
NCT = 0
NTRI = 4
ITYPM = 0
IF (NP(N).EQ.NP(N+1)) NTRI=3
L = NP(N),I
XX(I) = XORD(I)

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1091 FORMAT (/// 30H LINK 1 IS COMPLETED      ///)
GO TO 1005
C *****
C ***** PROGRAM EXIT DUE TO INPUT ERRORS *****
C *****
C *****
1010 PRINT 1051, IBANDH,NBLKL
1051 FORMAT (// 10H BANDWIDTH IS,30H IS GREATER THAN BLOCK LENGTH (3)
GO TO 340
IFLAG = 1
1020 PRINT 1021, NEQ8C
1021 FORMAT (4CH2 MAX. NO. OF BC CONSTRAINTS EXCEEDED      15//)
GO TO 263
1030 PRINT 1031, IBANDH
1031 FORMAT (4CF2 MAXIMUM BANDWIDTH EXCEEDED      15//)
IFLAG=1
GO TO 264
1009 RETURN
END

```

```

INPU 449
INPU 450
INPU 451
INPU 452
INPU 453
INPU 454
INPU 455
INPU 456
INPU 457
INPU 458
INPU 459
INPU 460
INPU 461
INPU 462
INPU 463
INPU 464
INPU 465
INPU 466
INPU 467
INPU 468
INPU 469
INPU 470
INPU 471
INPU 472
INPU 473
INPU 474
INPU 475
INPU 476
INPU 477
INPU 478
INPU 479
INPU 480
INPU 481
INPU 482
INPU 483
INPU 484
INPU 485
INPU 486
INPU 487
INPU 488
INPU 489
INPU 490
INPU 491
INPU 492
INPU 493
INPU 494
INPU 495
INPU 496
INPU 497
INPU 498
INPU 499
INPU 500
INPU 501
INPU 502
INPU 503
INPU 504

NCT = NCT + 1
IF (N.EQ.1) GO TO 670
IF (M.NE.NAT(N-1)) ITPM=1
IF (ITYP.G.NE.0.OR.ITYP.M.NE.0) GO TO 67C
GO TO 680
650 CONTINUE
C
C FORMATION OF GLOBAL STIFFNESS FOR VERTICAL WEB ELEMENTS
C
ASI = 0.0
ACO = 0.0
AXL = 0.0
DO 740 N=1,NBEAM
ITYP = 0
SI = BSIN(N)
CO = BCOS(N)
L = MBAT(N)
NA = NBP(N,1)
NB = NBP(N,2)
XL = (XORG(NB)-XORG(NA))*CC + (YORD(NB)-YORD(NA))*SI
C
C IF WEB ELEMENT IS OF THE SAME TYPE AS PREVIOUS ONE, ITS
C STIFFNESS IS NOT RECCUMPTED
C
IF (SI.NE.ASI) ITP=1
IF (CO.NE.ACO) ITP=1
IF (XL.NE.AXL) ITP=1
ASI = SI
ACO = CO
AXL = XL
IF (N.EQ.1) GO TO 730
IF (L.NE.MBAT(N-1)) ITP=1
IF (ITYP.EQ.0) GO TO 720
730 DO 746 I=1,2
K = I+1
LB = ITE(I)
XX(LB) = 0.0
XX(I+1) = XL
YY(I) = 0.0
746 YY(I+2) = AXL
YA(I) = XX(I)
YA(I+1) = YY(I)
748 PP(I) = 1.0
DO 734 I=1,3
DO 734 J=1,3
734 CH(I,J) = DM(L,I,J)
736 CS(I) = DS(L,I)
CALL QUSP12 (3)
CALL SONEH
CALL WEB (S,SI,CO)
720 WRITE (1) S
740 WRITE (13) S
740 CONTINUE
PRINT 1091

```

FURM 57  
 FURM 58  
 FURM 59  
 FURM 60  
 FURM 61  
 FURM 62  
 FURM 63

CS(2) = CM(2,2)  
 CS(3) = CM(3,3)  
 CS(4) = CM(1,2)  
 CS(5) = CM(1,3)  
 CS(6) = CM(2,3)  
 RETURN  
 END

```

SUBROUTINE FURMC (E1,E2,XU,C12,ANG,CM,CS)
  FURM 1
  FURM 2
  FURM 3
  FURM 4
  FURM 5
  FURM 6
  FURM 7
  FURM 8
  FURM 9
  FURM 10
  FURM 11
  FURM 12
  FURM 13
  FURM 14
  FURM 15
  FURM 16
  FURM 17
  FURM 18
  FURM 19
  FURM 20
  FURM 21
  FURM 22
  FURM 23
  FURM 24
  FURM 25
  FURM 26
  FURM 27
  FURM 28
  FURM 29
  FURM 30
  FURM 31
  FURM 32
  FURM 33
  FURM 34
  FURM 35
  FURM 36
  FURM 37
  FURM 38
  FURM 39
  FURM 40
  FURM 41
  FURM 42
  FURM 43
  FURM 44
  FURM 45
  FURM 46
  FURM 47
  FURM 48
  FURM 49
  FURM 50
  FURM 51
  FURM 52
  FURM 53
  FURM 54
  FURM 55
  FURM 56

  C*****
  C THIS SUBROUTINE COMPUTES THE STRESS-STRAIN COEFFICIENTS
  C IN THE GLOBAL X-Y-Z SYSTEM FOR A Z-CYLINDRICALLY ORTHOTROPIC
  C MATERIAL DEFINED BY CONSTANTS REFERRED TO THE PRINCIPAL MATERIAL
  C AXES X1-X2-Z
  C*****
  C INPUT
  C E1 ELASTIC MODULUS IN X1 DIRECTION
  C E2 ELASTIC MODULUS IN X2 DIRECTION
  C G12 SHEAR MODULUS
  C XU MEAN POISSONS RATIO SQR(V12*V21)
  C ANG ANGLE BETWEEN X-AXIS AND PRINCIPAL X1-AXIS
  C OUTPUT
  C CM(3,3) PLANE STRESS MATERIAL MATRIX
  C CS(1) PLANE STRESS MATERIAL VECTOR
  C I=1,6 ELEMENTS 11,22,33,12,13,23
  C DIMENSION CM(3,3), CS(6)
  C = 1.
  S = 0.
  IF (ANG) 120,150,120
  120 PHI = ANG/57.2957795
  S = SIN(PHI)
  C = COS(PHI)
  150 C2 = C*C
  S2 = S*S
  C4 = C2*C2
  S2C2 = S2*C2
  S4 = S2*S2
  XUH = 1.-XU**2
  C11 = E1/XUH
  C22 = E2/XUH
  C12 = SQR(E1*E2)*XU/XUH
  Z = C12+2.*G12
  RZ1 = C11-Z
  RZ2 = -C22
  RZ = (RZ1-RZ2)*S2C2
  ZS2C2 = 2.*Z*S2C2
  CM(1,1) = C11*C4+C22*S4+ZS2C2
  CM(2,2) = C22*C4+C11*S4+ZS2C2
  CM(1,2) = C12*RZ
  CM(3,3) = G12*RZ
  CM(1,3) = (RZ1*C2+RZ2*S2)*SC
  CM(2,3) = (RZ2*C2+RZ1*S2)*SC
  CM(2,1) = CM(1,2)
  CM(3,1) = CM(1,3)
  CM(3,2) = CM(2,3)
  CS(1) = CM(1,1)
  
```

```

SUBROUTINE Q3D(11,INT)
C *****
C STRESS SUBROUTINE FOR 8-DF PLANE STRESS QUADRILATERAL
C % FUNDAMENTAL DEGREES OF FREEDOM AND 3 INTERNAL DEGREES OF F
C AN INTERNAL NODE AND A CONSTANT SHEAR STRAIN VARIATION
C AN ANISOTROPIC MATERIAL LAW IS USED
C *****
C INPUT
C Q8D1 10
C Q8D1 11
C Q8D1 12
C Q8D1 13
C Q8D1 14
C Q8D1 15
C Q8D1 16
C Q8D1 17
C Q8D1 18
C Q8D1 19
C Q8D1 20
C Q8D1 21
C Q8D1 22
C Q8D1 23
C Q8D1 24
C Q8D1 25
C Q8D1 26
C Q8D1 27
C Q8D1 28
C Q8D1 29
C Q8D1 30
C Q8D1 31
C Q8D1 32
C Q8D1 33
C Q8D1 34
C Q8D1 35
C Q8D1 36
C Q8D1 37
C Q8D1 38
C Q8D1 39
C Q8D1 40
C Q8D1 41
C Q8D1 42
C Q8D1 43
C Q8D1 44
C Q8D1 45
C Q8D1 46
C Q8D1 47
C Q8D1 48
C Q8D1 49
C Q8D1 50
C Q8D1 51
C Q8D1 52
C Q8D1 53
C Q8D1 54
C Q8D1 55
C Q8D1 56
C *****
C GAUSSIAN INTEGRATION RULE
C CONST LAH RELATING STRESS-RES TO STRAINS
C COMPONENTS OF D(11,22,33),D(13,23)
C GLOBAL X-COORDINATES
C GLOBAL Y-COORDINATES
C *****
C 8*8 STIFFNESS MATRIX OF PLANE STRESS ELEMENT
C IN GLOBAL XA-YA COORDINATES (FORMATION IN
C LOCAL CONVEXED X-Y COORDINATES)
C NODAL DISPLACEMENTS (U1,V1,U2,V2,...)
C CONDENSATION OF CENTER NODE DOF
C CONDENSATION OF CONSTANT SHEAR STRAIN DOF
C *****
C COMMON / PLSTR / XA(4),YA(4),D(6),S(12,12),K(12),ST(5,3)
C *****
C DIMENSION P(5,2),EC(4,2),A(12,2),L(5),Y(5),ETA(2),IPERM(2),
C * XK(4,4),WGT(4,4),X(4),Y(4),TEMP(8,4)
C *****
C DATA XK / 0., 0., 0., 0.,
C * -5773502651856, -5773502651856,
C * -7745986692415, -7745986692415,
C * -8611363115941, -3399810435849, .9611363115941, .9611363115941,
C DATA WGT / 2.000, 0.,
C * 1.0000000000000, 1.0000000000000,
C * .5555555555556, .8988888888889, .5555555555556,
C * .3478548451375, .6521451548625, .6521451548625, .3478548451375,
C DATA DC / -1., 1., 1., -1., -1., 1., 1., /, IPERM / 2, 1,
C EQUIVALENCE (A(11),A(12)), (A(21),A(22)), (A(12),A(31)), (A(22),A(41))
C *****
C INITIALIZATION
C DO 100 I=1,11
C DO 100 J=1,11
C S(I,J) = C.
C NA = 0
C NINT = INT
C *****
C TRANSFORMATION OF COORDINATES INTO LOCAL CONVEXED COOR-DINATES
C *****
C UX = XA(2) - XA(1)
C UY = YA(2) - YA(1)
C AL = S(PT(IX**2 + DY**2))
C CU = DX/4L

```

```

C *****
C STRESS SUBROUTINE FOR 8-DF PLANE STRESS QUADRILATERAL
C % FUNDAMENTAL DEGREES OF FREEDOM AND 3 INTERNAL DEGREES OF F
C AN INTERNAL NODE AND A CONSTANT SHEAR STRAIN VARIATION
C AN ANISOTROPIC MATERIAL LAW IS USED
C *****
C INPUT
C Q8D1 10
C Q8D1 11
C Q8D1 12
C Q8D1 13
C Q8D1 14
C Q8D1 15
C Q8D1 16
C Q8D1 17
C Q8D1 18
C Q8D1 19
C Q8D1 20
C Q8D1 21
C Q8D1 22
C Q8D1 23
C Q8D1 24
C Q8D1 25
C Q8D1 26
C Q8D1 27
C Q8D1 28
C Q8D1 29
C Q8D1 30
C Q8D1 31
C Q8D1 32
C Q8D1 33
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C Q8D1 40
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C Q8D1 43
C Q8D1 44
C Q8D1 45
C Q8D1 46
C Q8D1 47
C Q8D1 48
C Q8D1 49
C Q8D1 50
C Q8D1 51
C Q8D1 52
C Q8D1 53
C Q8D1 54
C Q8D1 55
C Q8D1 56
C *****
C GAUSSIAN INTEGRATION RULE
C CONST LAH RELATING STRESS-RES TO STRAINS
C COMPONENTS OF D(11,22,33),D(13,23)
C GLOBAL X-COORDINATES
C GLOBAL Y-COORDINATES
C *****
C 8*8 STIFFNESS MATRIX OF PLANE STRESS ELEMENT
C IN GLOBAL XA-YA COORDINATES (FORMATION IN
C LOCAL CONVEXED X-Y COORDINATES)
C NODAL DISPLACEMENTS (U1,V1,U2,V2,...)
C CONDENSATION OF CENTER NODE DOF
C CONDENSATION OF CONSTANT SHEAR STRAIN DOF
C *****
C COMMON / PLSTR / XA(4),YA(4),D(6),S(12,12),K(12),ST(5,3)
C *****
C DIMENSION P(5,2),EC(4,2),A(12,2),L(5),Y(5),ETA(2),IPERM(2),
C * XK(4,4),WGT(4,4),X(4),Y(4),TEMP(8,4)
C *****
C DATA XK / 0., 0., 0., 0.,
C * -5773502651856, -5773502651856,
C * -7745986692415, -7745986692415,
C * -8611363115941, -3399810435849, .9611363115941, .9611363115941,
C DATA WGT / 2.000, 0.,
C * 1.0000000000000, 1.0000000000000,
C * .5555555555556, .8988888888889, .5555555555556,
C * .3478548451375, .6521451548625, .6521451548625, .3478548451375,
C DATA DC / -1., 1., 1., -1., -1., 1., 1., /, IPERM / 2, 1,
C EQUIVALENCE (A(11),A(12)), (A(21),A(22)), (A(12),A(31)), (A(22),A(41))
C *****
C INITIALIZATION
C DO 100 I=1,11
C DO 100 J=1,11
C S(I,J) = C.
C NA = 0
C NINT = INT
C *****
C TRANSFORMATION OF COORDINATES INTO LOCAL CONVEXED COOR-DINATES
C *****
C UX = XA(2) - XA(1)
C UY = YA(2) - YA(1)
C AL = S(PT(IX**2 + DY**2))
C CU = DX/4L

```

```

C *****
C STRESS SUBROUTINE FOR 8-DF PLANE STRESS QUADRILATERAL
C % FUNDAMENTAL DEGREES OF FREEDOM AND 3 INTERNAL DEGREES OF F
C AN INTERNAL NODE AND A CONSTANT SHEAR STRAIN VARIATION
C AN ANISOTROPIC MATERIAL LAW IS USED
C *****
C INPUT
C Q8D1 10
C Q8D1 11
C Q8D1 12
C Q8D1 13
C Q8D1 14
C Q8D1 15
C Q8D1 16
C Q8D1 17
C Q8D1 18
C Q8D1 19
C Q8D1 20
C Q8D1 21
C Q8D1 22
C Q8D1 23
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C Q8D1 28
C Q8D1 29
C Q8D1 30
C Q8D1 31
C Q8D1 32
C Q8D1 33
C Q8D1 34
C Q8D1 35
C Q8D1 36
C Q8D1 37
C Q8D1 38
C Q8D1 39
C Q8D1 40
C Q8D1 41
C Q8D1 42
C Q8D1 43
C Q8D1 44
C Q8D1 45
C Q8D1 46
C Q8D1 47
C Q8D1 48
C Q8D1 49
C Q8D1 50
C Q8D1 51
C Q8D1 52
C Q8D1 53
C Q8D1 54
C Q8D1 55
C Q8D1 56
C *****
C GAUSSIAN INTEGRATION RULE
C CONST LAH RELATING STRESS-RES TO STRAINS
C COMPONENTS OF D(11,22,33),D(13,23)
C GLOBAL X-COORDINATES
C GLOBAL Y-COORDINATES
C *****
C 8*8 STIFFNESS MATRIX OF PLANE STRESS ELEMENT
C IN GLOBAL XA-YA COORDINATES (FORMATION IN
C LOCAL CONVEXED X-Y COORDINATES)
C NODAL DISPLACEMENTS (U1,V1,U2,V2,...)
C CONDENSATION OF CENTER NODE DOF
C CONDENSATION OF CONSTANT SHEAR STRAIN DOF
C *****
C COMMON / PLSTR / XA(4),YA(4),D(6),S(12,12),K(12),ST(5,3)
C *****
C DIMENSION P(5,2),EC(4,2),A(12,2),L(5),Y(5),ETA(2),IPERM(2),
C * XK(4,4),WGT(4,4),X(4),Y(4),TEMP(8,4)
C *****
C DATA XK / 0., 0., 0., 0.,
C * -5773502651856, -5773502651856,
C * -7745986692415, -7745986692415,
C * -8611363115941, -3399810435849, .9611363115941, .9611363115941,
C DATA WGT / 2.000, 0.,
C * 1.0000000000000, 1.0000000000000,
C * .5555555555556, .8988888888889, .5555555555556,
C * .3478548451375, .6521451548625, .6521451548625, .3478548451375,
C DATA DC / -1., 1., 1., -1., -1., 1., 1., /, IPERM / 2, 1,
C EQUIVALENCE (A(11),A(12)), (A(21),A(22)), (A(12),A(31)), (A(22),A(41))
C *****
C INITIALIZATION
C DO 100 I=1,11
C DO 100 J=1,11
C S(I,J) = C.
C NA = 0
C NINT = INT
C *****
C TRANSFORMATION OF COORDINATES INTO LOCAL CONVEXED COOR-DINATES
C *****
C UX = XA(2) - XA(1)
C UY = YA(2) - YA(1)
C AL = S(PT(IX**2 + DY**2))
C CU = DX/4L

```

```

1000 P=PIGT 1200, GFT
1200 F=PIGT(// 30P DETERMINANT OF JACGRIAN IS E15.3//)
1400 F=PIGT 1400, (X(1), Y(1), I=1,4)
1600 STOP
      RETURN
      END

```

```

360 CONTINUE
300 CONTINUE
  NA = NA + 1
  NINT = INT - 1
  IF (NINT-1) NINT = 1
  IF (NINT-1) GC TC 500
  TRANSPOSE STIFFNESS FOR SYMPETRY
  DO *00 I=2,11
    K = I - 1
    DO *00 J = 1,K
      C=S(I,L)/PIVCT
      S(I,L)=C
    UD 700 J=I,K
      S(I,J)=S(I,J)-C*S(L,J)
    S(J,I)=S(I,J)
  700 CONTINUE
  TRANSFORM CONNECTED STIFFNESS INTO GLOBAL CARTESIAN COORDINATES
  SS = SI*SI
  CC = CO*CC
  SC = SI*CC
  DO 560 K=1,4
    IQ = K + K
    IP = IQ - 1
    DO 550 J=1,8
      IF (J.EC.IP) GC TC 550
      TEMP(J,K) = S(J,IP)*SI + S(J,IQ)*CO
      S(J,IP) = S(J,IP)*CC - S(J,IQ)*SI
      S(IQ,J) = TEMP(J,K)
      S(IP,J) = TEMP(J,K)
    S(J,IQ) = TEMP(J,K)
  550 CONTINUE
  TEMP(IQ,K) = S(IP,IP)*SS + S(IQ,IQ)*CC + 2.*S(IP,IQ)*SC
  TEMP(IP,K) = -(S(IQ,IQ)-S(IP,IP))*SC + S(IP,IQ)*CC-SS
  S(IP,IP) = S(IP,IP)*CC + S(IP,IQ)*CC-SS
  S(IP,IQ) = TEMP(IP,K)
  S(IQ,IP) = TEMP(IP,K)
  S(IQ,IQ) = TEMP(IQ,K)
  560 CONTINUE
  GU TC 1600

```





```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```

```

400 S(I,J) = S(I,J) - C*S(L,J)
610 CONTINUE
GO TO 1600
1000 PRINT 1200, DET
1200 FORMAT(' / 30th DETERMINANT OF JACOBIAN IS', E15.3//)
1400 PRINT 1400, (XCRG(I),YCRD(I),I=1,4)
1400 FORMAT (5X, 2F10.3)
STOP
1600 CONTINUE
700 RETURN
1800 END

```



```

F(N) = FN/PIVCT
DO 400 I = 1,4
C = S(I,N)/PIVCT
S(I,N) = C
F(I) = F(I) - C*FA
DO 400 J = 1,4
S(I,J) = S(I,J) - C*S(N,J)
400 S(J,I) = S(I,J) - C*S(N,J)
420 CONTINUE
500 RETURN
END

```

```

SUBROUTINE SLCT (NBF)
C*****
C STIFFNESS SUBROUTINE FOR COMPATIBLE TRIANGULAR PLATE ELEMENT
C WITH 'NBF' BENDING D.F. (NBF=12,11,10,9) - RIGHT-HAND SYSTEM -
C ORTHOTROPIC ELASTIC MATERIAL, FOR NBF=5 LCCT9=HCT
C*****
C COMMON / TRIAG / R(3),A(3),CMT(3,3),PT(3),BMT(3,3),
* ST(12,12),FT(12)
C DIMENSION P(21,12),H(21),U(21),S(3,6),HT(3),IPERM(3),TX(3),TY(3),
* NKN(4,3),NSM(4,3)
C EQUIVALENCE (CM11,CMT(1)),(CM12,CMT(2)),(CM13,CMT(3)),
* DATA IPERM/2,3,17, NKN/2,5,3,6, 8,2,9,3, 5,8,6,9/,
* NSN/2,3,5,6, 3,1,6,4, 1,2,4,5/
C INITIALIZATION
C
NDF = NBF
AREA = A(3)*B(2)-A(2)*B(3)
FAC = AREA/72.0
DO 150 I = 1,3
J = IPERM(I)
K = IPERM(J)
X = A(I)**2+B(I)**2
U(I) = -(A(I)*A(J)+B(I)*B(J))/X
X = SQRT(X)
HT(I) = 4.0*AREA/X
TY(I) = -0.5*B(I)/X
TX(I) = 0.5*A(I)/X
A1 = A(I)/AREA
A2 = A(J)/AREA
B1 = B(I)/AREA
B2 = B(J)/AREA
Q(1,1) = B1*B1
Q(2,1) = A1*A1
Q(3,1) = 2.*A1*B1
Q(1,1+3) = 2.*B1*B2
Q(2,1+3) = 2.*A1*A2
Q(3,1+3) = 2.*(A1*B2+A2*B1)
150 Q(3,1+3) = 2.*(A1*B2+A2*B1)
C FORMATION OF CURVATURE MATRIX P(21,12)
C
DO 200 I = 1,3
J = IPERM(I)
K = IPERM(J)
II = 3*I
JJ = 3*J
KK = 3*K
A1 = A(I)
A2 = A(J)
A3 = A(K)
B1 = B(I)
B2 = B(J)
B3 = B(K)

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SLCC 48  
SLCC 49  
SLCC 50  
SLCC 51  
SLCC 52  
SLCC 53  
SLCC 54  
SLCC 55  
SLCC 56



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DO 360 N = 1,19,3
U(N) = CM11*H(N) + CM12*H(N+1) + CM13*H(N+2)
U(N+1) = CM12*H(N) + CM22*H(N+1) + CM23*H(N+2)
360 U(N+2) = CM13*H(N) + CM23*H(N+1) + CM33*H(N+2)
C
C FORMATION OF STIFFNESS MATRIX ST (12,12)
C
DO 400 I = 1,J
X = 0.
DO 380 N = 1,21
380 X = X + U(N)*P(N+1)
ST(I,J) = X*FAC
400 ST(I,J) = ST(I,J)
RETURN
END

```

```

SUBROUTINE SONEW
C *****
C THIS SUBROUTINE FORMS A CNF WAY BENDING B * B STIFFNESS
C MATRIX (BENDC IN Y-DIP) IN FCRM OF A 12 * 12 MATRIX
C INPUT AND OUTPUT SAME AS FOR SPLATE
C *****
C COMMON / PLDGG / X(5),Y(5),C*(3,3),P(5),EM(3,5),CV(3,5),
* S(19,19),F(19)
C DIMENSION ITE(2,2),IPE(2),AL(2)
C DATA ITE /1,19,4,7/, IPE /0,3/
C *****
C INITIALIZATION
C
DO 100 I=1,12
DO 100 J=1,12
100 ST(I,J) = 0.0
AL(1) = Y(4) - Y(1)
AL(2) = Y(3) - Y(2)
FT = CM(1,1)*X(2)-X(1)+X(3)-X(4))/2.C
C
C FORMATION OF STIFFNESS ES OF TWO ELIVALENT BEAMS IN Y-DIRECT IONSONE
C
DO 200 I=1,2
XL = AL(I)
SIG = -1.C
DO 300 J=1,2
IA = ITE(J,I)
SIG = -SIG
ST(IA,IA) = 6.C/IXL**3)*FT
ST(IA+1,IA+1) = 2.C/IXL*FT
300 ST(IA,IA+1) = 3.C/IXL**2)*SIG*FT
IA = 1 + IPE(I)
JA = 10 - IPE(I)
ST(IA,JA) = -6.C/IXL**3)*FT
ST(IA,JA+1) = 3.C/IXL**2)*FT
ST(IA+1,JA) = -3.C/IXL**2)*FT
200 ST(IA+1,JA+1) = 1.C/IXL*FT
C
C TRANSPCSE STIFFNESS FOR SYMMETRY
C
DO 400 I=2,12
K = I - 1
DO 400 J=1,K
400 ST(I,J) = ST(J,I)
RETURN
END

```

```

SUBROUTINE SONE
C *****
C THIS SUBROUTINE FORMS A CNF WAY BENDING B * B STIFFNESS
C MATRIX (BENDC IN Y-DIP) IN FCRM OF A 12 * 12 MATRIX
C INPUT AND OUTPUT SAME AS FOR SPLATE
C *****
C COMMON / PLDGG / X(5),Y(5),C*(3,3),P(5),EM(3,5),CV(3,5),
* S(19,19),F(19)
C DIMENSION ITE(2,2),IPE(2),AL(2)
C DATA ITE /1,19,4,7/, IPE /0,3/
C *****
C INITIALIZATION
C
DO 100 I=1,12
DO 100 J=1,12
100 ST(I,J) = 0.0
AL(1) = Y(4) - Y(1)
AL(2) = Y(3) - Y(2)
FT = CM(1,1)*X(2)-X(1)+X(3)-X(4))/2.C
C
C FORMATION OF STIFFNESS ES OF TWO ELIVALENT BEAMS IN Y-DIRECT IONSONE
C
DO 200 I=1,2
XL = AL(I)
SIG = -1.C
DO 300 J=1,2
IA = ITE(J,I)
SIG = -SIG
ST(IA,IA) = 6.C/IXL**3)*FT
ST(IA+1,IA+1) = 2.C/IXL*FT
300 ST(IA,IA+1) = 3.C/IXL**2)*SIG*FT
IA = 1 + IPE(I)
JA = 10 - IPE(I)
ST(IA,JA) = -6.C/IXL**3)*FT
ST(IA,JA+1) = 3.C/IXL**2)*FT
ST(IA+1,JA) = -3.C/IXL**2)*FT
200 ST(IA+1,JA+1) = 1.C/IXL*FT
C
C TRANSPCSE STIFFNESS FOR SYMMETRY
C
DO 400 I=2,12
K = I - 1
DO 400 J=1,K
400 ST(I,J) = ST(J,I)
RETURN
END

```

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DECK 57
DECK 58
DECK 59
DECK 60
DECK 61
DECK 62
DECK 63
DECK 64
DECK 65
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DECK 67
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DECK 73
DECK 74
DECK 75
DECK 76
DECK 77
DECK 78

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SURFACTIVE DECK (S)
C *****
C THIS SUBROUTINE FORMS THE GLOBAL DECK ELEMENT STIFFNESS
C ASSEMBLING THE PLATE BEADING STIFFNESS SPB (12*12) OR (9*9)
C COMPUTED BY SPLATE AND THE IN PLANE STIFFNESS SPD(H*H) OR (6*6)DECK
C COMPUTED BY QCELL
C STRESS STIFFNESS QAD11
C *****
C COMMON / PLSTR / XX(4),YY(4),CS(6),SPD(12,12),R1(12),EST(5,3)
C COMMON / PLBDG / XX(5),YY(5),C(3,3),PP(5),BM(3,5),CV(3,5),
C * SPB(15,19),VA(15)
C DIMENSION S(20,20),IU(4),IV(4),IW(4)
C DATA IU /1,6,11,16/, IV /1,3,5,7/, IW /1,4,7,10/
C
C INITIALIZATION
C
C DO 100 I=1,400
C 100 S(I) = 0.0
C
C ADD PLANE STRESS AND PLATE BEADING STIFFNESS INTO S
C
C DO 200 I=1,4
C IA = IU(I)
C IB = IA+1
C IC = IA+2
C ID = IA+3
C IE = IA+4
C IF = IV(I)
C IG = IF+1
C IH = IF+2
C II = IF+3
C IJ = IW(I)
C IK = IJ+1
C IL = IJ+2
C IM = IJ+3
C IN = IJ+4
C IO = IU(I)
C JP = IA+1
C JQ = IA+2
C JR = IA+3
C JS = IA+4
C JT = IV(I)
C JU = JT+1
C JV = JT+2
C JW = JT+3
C JX = JT+4
C JY = IV(I)
C JZ = JY+1
C JA = JZ+1
C JB = JZ+2
C JC = JZ+3
C JD = JZ+4
C JE = JA+4
C JF = IV(J)
C JG = JF+1
C JH = JF+2
C JI = JF+3
C JJ = JF+4
C JK = JI+1
C JL = JI+2
C JM = JI+3
C JN = JI+4
C JO = JI+1
C JP = JI+2
C JQ = JI+3
C JR = JI+4
C JS = JI+1
C JT = JI+2
C JU = JI+3
C JV = JI+4
C JW = JI+1
C JX = JI+2
C JY = JI+3
C JZ = JI+4
C JA = JI+1
C JB = JI+2
C JC = JI+3
C JD = JI+4
C JE = JI+1
C JF = JI+2
C JG = JI+3
C JH = JI+4
C JI = JI+1
C JJ = JI+2
C JK = JI+3
C JL = JI+4
C JM = JI+1
C JN = JI+2
C JO = JI+3
C JP = JI+4
C JQ = JI+1
C JR = JI+2
C JS = JI+3
C JT = JI+4
C JU = JI+1
C JV = JI+2
C JW = JI+3
C JX = JI+4
C JY = JI+1
C JZ = JI+2
C JA = JI+3
C JB = JI+4
C JC = JI+1
C JD = JI+2
C JE = JI+3
C JF = JI+4
C JG = JI+1
C JH = JI+2
C JI = JI+3
C JJ = JI+4
C JK = JI+1
C JL = JI+2
C JM = JI+3
C JN = JI+4
C JO = JI+1
C JP = JI+2
C JQ = JI+3
C JR = JI+4
C JS = JI+1
C JT = JI+2
C JU = JI+3
C JV = JI+4
C JW = JI+1
C JX = JI+2
C JY = JI+3
C JZ = JI+4
C
C *****
C DO 260 J=1,4
C JA = IU(J)
C JB = JA+1
C JC = JA+2
C JD = JA+3
C JE = JA+4
C SF(IH,JA) = S(IA,IB)
C S(ID,JC) = S(IJ,IE)
C S(IE,JD) = S(IJ,IE)
C *****
C 260 CONTINUE
C RETURN
C END

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WEB 94

S(1C,JJ) = SPD(12,J3)*SI
S(1C,JE) =-SPD(12,J3)*CC
S(1B,JD) = S(1B,JD) - SPB(14,J5)*CO*CC
S(1B,JE) = S(1B,JE) - SPB(14,J5)*CO*SI
S(1A,JD) = S(1A,JD) + SPB(14,J5)*SI*CC
S(1A,JE) = S(1A,JE) + SPB(14,J5)*SI*SI
S(1A,JB) = (SPD(11,J1) - SPB(14,J4))*SI*CC
S(1D,JE) = (SPB(15,J5) - SPD(13,J3))*SI*CO
220 CONTINUE
200 CONTINUE
C
C TRANSPOSE S FOR SYMMETRY
C
DO 250 I=1,4
IA = IU(I)
IB = IA + 1
IC = IA + 2
IE = IA + 4
DO 250 J=1,4
JA = IU(J)
JB = JA + 1
JC = JA + 2
JD = JA + 3
JE = JA + 4
S(1C,JA) = S(JA,IC)
S(1C,JB) = S(JB,IC)
S(1D,JC) = S(JC,IE)
S(1E,JC) = S(JC,IE)
S(1D,JB) = S(JB,IE)
S(1E,JB) = S(JB,IE)
S(1D,JA) = S(JA,IE)
S(1E,JA) = S(JA,IE)
S(1B,JA) = S(JA,IB)
S(1E,JD) = S(JD,IE)
250 CONTINUE
RETURN
END

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SUBROUTINE WEB (S,SI,CC)
C*****
C THIS SUBROUTINE FORMS THE GLOBAL ELEMENT STIFFNESS MATRIX
C FOR THE VERTICAL WEB ELEMENTS ASSEMBLING THE ONE WAY BENDING
C STIFFNESS SPB(12*12) COMPUTED BY SCEN6, WITH THE IN PLANE
C STIFFNESS SPD(12,12) COMPUTED BY CLSP12
C*****
COMMON / PLSTR / XA(4),YA(4),CS(6),SPD(12,12),R1(12),ESIG(5,3)
COMMON / PLRDG / XX(5),YY(5),CM(3,3),PP(5),RM(3,5),CV(3,5),
*
DIMENSION S(20,20),IU(4),IN(4)
DATA IU /1,11,16,16/, IN /1,4,7,10/
C
C INITIALIZATION
C
DO 100 I=1,400
S(I,I) = 0.0
C
C ADD PLANE STRESS AND PLATE BENDING STIFFNESS INTO S AND
C APPLY TRANSFORMATION INTO GLOBAL COORDINATES
C
DO 200 I=1,4
IA = IU(I)
IB = IA + 1
IC = IA + 2
ID = IA + 3
IE = IA + 4
IF = I
I2 = I + 4
I3 = I + 8
I4 = I + 11
I5 = I + 14
DO 220 J=1,4
JA = IU(J)
JB = JA + 1
JC = JA + 2
JD = JA + 3
JE = JA + 4
J2 = J + 4
J3 = J + 8
J4 = J + 11
J5 = J + 14
S(1A,JA) = SPD(11,J1)*CC**2 + SPB(14,J4)*SI**2
S(1B,JB) = SPD(11,J1)*SI**2 + SPB(14,J4)*CO**2
S(1C,JC) = SPD(12,J2)
S(1D,JD) = SPB(15,J5)*CC**2 + SPD(13,J3)*SI**2
S(1E,JE) = SPB(15,J5)*SI**2 + SPD(13,J3)*CO**2
240 S(1A,JC) = SPD(11,J3)*CC*SI
S(1A,JE) =-SPD(11,J3)*CC*CC
S(1B,JC) = SPD(11,J2)*SI*CC
S(1B,JD) = SPD(11,J3)*SI*SI
S(1B,JE) =-SPU(11,J3)*SI*CC

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```

NL = L*IC - KSHIFT - 10
DO 4010 J=1,2
  M = NBP(N,J)
  IF (L,GT,M) GO TO 4010
  JM = 10*M - 10
  NM = M*10 - KSHIFT - 10
  MC = NM - NL + 1
  DO 4020 I=1,10
    NR = NL + I
    MC = MC - 1
  NC = MC
  IR = IL + I
  DO 4020 JJ=1,10
    NC = NC + 1
  IF (NC,LE,0) GC TC 402C
  IC = JM + JJ
  C(NR,NC) = C(NR,NC) + SPB(IR,IC)
4020 CONTINUE
4010 CONTINUE
4000 CUNTINUE
  NBP(N,1) = -NBP(N,1)
  GO TO 110
100 READ (3) SPB
110 CONTINUE
C
C***** EFFECT OF ROUND. COND. ON BIGK *****
C*****
C
DO 750 I=1,NECBC
  IF (NEBC(I),LE,0) GO TO 75C
  NL=NEBC(I)
  PHI=ANGLE(I)
  IF (PHI,EG,0.0) GC TO 78C
C
C SKEMED BOUNARY CONDITION IN UPPER BLCKC
C
  IF (NH,LT,NL) GC TO 750
  IF (NF,GT,NL) GO TO 75C
  NR=NL-KSHIFT
  NR1 = NR - 1
  CC = COS(PHI)
  SS = SIN(PHI)
  IF (NZ,LT,NL) GO TO 76C
  C(NR1,1)=C(NR1,1)*CC+2.0*C(NR1,2)*SS*CC+C(NR,1)*SS*SS
  L = NR1
  DO 744 J=3,IBANDW
    C(NR1,J) = C(NR1,J)*CC + C(NR,J-1)*SS
  L = L-1
  IF (L,LE,0) GO TO 744
  C(L,J-1)=C(L,J-1)*CC+C(L,J)*SS
744 CONTINUE
  DO 746 J=1,IBANDW
    C(NR,J)=0.0
  L=NR-J+1
  
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BIGK 113
BIGK 114
BIGK 115
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GO IC TC
440 READ (3) STO
DO 2000 I=1,4
  L=NBP(N,I)
  IL = 5*I - 5
  NL = L*IC - KSHIFT - LK
  DO 2010 J=1,4
    M=NBP(N,J)
    IF (L,GT,M) GO TO 2010
    JM = 5*J - 5
    NM = M*10 - KSHIFT - LK
    MC = NM - NL + 1
  NC = MC
  IR = IL + I
  DO 2030 JJ=1,5
    NC = NC + 1
  IF (NC,LE,0) GO TO 2030
  IC = JM + JJ
  C(NR,NC) = C(NR,NC) + SIG(IR,IC)
2030 CONTINUE
2020 CONTINUE
2010 CONTINUE
2000 CONTINUE
C
C BOTTCM DECK ELEMENTS
C
  IF (LC) GC TO 410
  LO = .TRUE.
  LK = 10
  GO TO 440
410 NP(N,1) = -NP(N,1)
  GO TO 710
700 READ (3) STQ
  READ (3) STQ
710 CONTINUE
C
C***** CONTRIBUTION OF BEAM ELEMENTS *****
C*****
C*****
C*****
DO 110 N=1,NBEAM
  IF (NBP(N,1),LT,0) GO TC 100
  DO 120 I=1,2
    LA = NBP(N,I)
    NL4 = LA*10 - 9
    IF (NF,LT,NL4) GO TO 12C
    IF (NF,LE,(NL4+9)) GO TC 14C
  120 CONTINUE
  GO TC 100
140 READ (3) SPR
  DO 4C00 I=1,2
    L = NBP(N,I)
    IL = 10*I - 10
  
```

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BIGK 157
BIGK 158
BIGK 159
BIGK 160
BIGK 161
BIGK 162
BIGK 163
BIGK 164
BIGK 165
BIGK 166
BIGK 167
BIGK 168
  
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C      CHECK FOR LAST BLOCK
C      IF (NZ.LT.NEQ) GO TO 6C
C*****
C      CHECK COMPLETENESS OF STIFFNESS MATRIX PUT ON TAPE
C*****
C      DO 450 N=1,NUMEL
C      IF (NPN,I).LT.0) GO TO 450
C      PRINT 416,N
C      416 FORMAT (25H DECKELEMANT MISSING      I4)
C      IFLAG=1
C      450 NPN(I) =-NPN(I)
C      621 DO 620 N=1,NECBC
C      IF (NEBC(N).LE.0) GO TO 62C
C      PRINT 626,N
C      626 FORMAT (25H BOUND. COND. MISSING      I4)
C      IFLAG=1
C      620 NEBC(N)=-NEBC(N)
C      DO 600 N=1,NBEAM
C      IF (NBP(N,I).LT.0) GO TO 60C
C      PRINT 606,N
C      606 FORMAT (25H BEAM MISSING      I4)
C      IFLAG = 1
C      600 NBP(N,I) =-NBP(N,I)
C      IF (NUMBLK.EQ.NBLK) GO TO 65C
C      PRINT 646, NUMBLK,NBLK
C      646 FORMAT (///25H NUMBER OF BLOCKS DIFFER      2I5)
C      IFLAG = 1
C      650 IF (IFLAG.NE.0) STOP
C      PRINT 630
C      630 FORMAT (/// 23H LINK NO 2 CPLETED      ///)
C      RETURN
C      END

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B1GK 170
B1GK 171
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B1GK 224

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IF (L.LE.0) GO TO 746
C(L,J)=0.C
746 CONTINUE
NEBC(I)=-NEBC(I)
GO TO 75C
EFFECT OF SKEWED BOUNDARY CONDITIONS IN LOWER BLOCK ONTO
EQUATIONS IN UPPER BLOCK
760 NK=NR-ND
L=ND+1
IF (NK.GE.IBANDW) GO TO 75C
IBAND = IBANDW - 1
DO 762 J=NK,IBAND
L=L-1
C(L,J)=C(L,J)*CC+C(L,J+1)*SS
GO TO 750
SIMPLE BOUNDARY CONDITION IN UPPER BLOCK
780 IF (NH.LT.NL) GO TO 75C
IF (NF.GT.NL) GO TO 75C
NR=NL-KSHIFT
DO 790 J=1,IBANDW
C(NR,J)=0.0
L=NR-J+1
IF (L.LE.0) GO TO 790
C(L,J)=0.0
790 CONTINUE
NEBC(I)=-NEBC(I)
GO TO 750
EFFECT OF SIMPLE BOUNDARY CONDITION IN LOWER BLOCK ONTO
EQUATIONS IN UPPER BLOCK
785 NJ=NR-ND+1
L=ND+1
IF (NJ.GT.IBANDW) GO TO 75C
DO 766 J=NJ,IBANDW
L=L-1
C(L,J)=0.0
766 CONTINUE
750 CONTINUE
C*****
C      WRITE UPPER BLOCK ON TAPE AND SHIFT LOWER BLOCK UP
C*****
530 WRITE (8) ((C(I,J),I=1,ND),J=1,IB)
DO 360 N=1,ND
K=N+ND
DO 360 M=1,IBANDW
C(N,M)=C(K,M)
360 C(K,M)=C.C

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OVERLAY (BOX,3,0)
PROGRAM LCADG
C*****
C THIS PROGRAM COMPUTES THE REQUIRED FIELDLENGTH AND RESETS IT
C IF NECESSARY
C*****
C COMMON / SETUP / NUMP,NUMEL,NBEAM,NMAT,NRUN,NFLDL,IBANDM,NEQ,
* NBLK,NUMBLK,NEQBC,MARG,AH
COMMON / BOCCN / NEBC(2CC),ANGLE(200)
COMMON D(1)
INTEGER FCRD
C COMPUTE AND RESET FIELD LENGTH IF NECESSARY
CALL LWA (FWCRD)
N1 = 1
N2 = N1 + 4*NUMEL
N3 = N2 + 2*NBEAM
N4 = N3 + NUMEL
N5 = N4 + NUMEL
N6 = N5 + NBEAM
N7 = N6 + NBEAM
N8 = N7 + NBEAM
N9 = N8 + NUMP
N10 = N9 + NUMP
N11 = N10 + 9*NMAT
N12 = N11 + 6*NMAT
N13 = N12 + NUMEL
N14 = N13 + NUMEL
N15 = N14 + NEQ
N16 = N15 + NEQ
LWORD = FWORD + N16
IF (LWORD.LT.NFLDL) GO TO 100
NFLDL = LWORD
CALL REL (NFLDL)
100 CALL LOAD (D(N1),D(N2),D(N3),D(N4),D(N5),D(N6),D(N7),D(N8),D(N9),
* D(N10),D(N11),D(N12),D(N13),D(N14),D(N15),NUMEL,NBEAM,
* NUMP,NMAT,NEQ)
* RETURN
END
SUBROUTINE LOAD (NP,NBP,MAT,NAT,MEAT,BSIN,BCOS,XORD,YORD,UM,DS,
* NDL,PI,P,D,IE,IB,IP,IN,NAK)
C*****
C THIS SUBROUTINE FORMS THE LEADVECTOR FOR EACH LOADCASE
C*****
C COMMON / SETUP / NUMP,NUMEL,NBEAM,NMAT,NRUN,NFLDL,IBANDM,NEQ,
* NBLK,NUMBLK,NEQBC,MARG,AH
COMMON / BOCCN / NEBC(2CC),ANGLE(200)
DIMENSION NP(IE,4),NBP(1B,2),MAT(1E),NAT(1E),MBAT(1B),BSIN(1B),
* BCOS(1B),XORD(1P),YORD(1P),DM(1M,3,3),DS(1M,6),
* NDL(1E),PI(1E),P(NAK),D(NAK)
DIMENSION IPE(4),ITE(4),X(5),Y(5),FORCE(5),ST(20,20),SPB(19,19)
LOGICAL TBC,TDIS
DATA IPE /4,1,2,3/, ITE /2,3,4,1/
C INITIALIZATION
REWIND 1
LDG = 3
IF (NRUN.GT.1) LCG=8
REWIND LDG
READ (1) NP,NBP
READ (1) XORD,YORD,MAT,NAT,MBAT,DP,DS,BSIN,BCOS
NN = NBLK
DO 130 I=1,NEQ
D(I) = 0.0
130 P(I) = 0.0
DO 140 I=1,NUMEL
140 PI(I) = 0.0
C*****
C LOADINPUT
C*****
C PRINT 104, NRUN
104 FORMAT (12H1 LOAD CASE I4 //)
READ 106, NLD,NLL,NDL,PLL,PDL
106 FORMAT (314,2F10.2)
PRINT 107, ALD,ALL,NLD
107 FORMAT (//40H NUMBER OF LOCAL LOADS
* 40H NUMBER OF DIFF LL FOR TCP CHECK ELEM
* 40H DEAC LOAD INCLUDED (NLD=1)
IF (NLD.EQ.0) GO TO 30C
INPUT CF CONCENTRATED LOCAL LOADS
PRINT 81
DO 120 N=1,NLD
READ 80, M,TBCT,TDIS,(FCRCE(I),I=1,5)
PRINT 82, M,TBCT,TDIS,(FCRCE(I),I=1,5)
K8 = M*10
KC = KB-1
KD = KC-1
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305 FORMAT (20I4)
DO 308 K=1,NEL
IA = NOLL(K)
308 PI(IA) = PLL
306 CONTINUE
311 PRINT 303
DO 301 I=1,NUMEL
301 PRINT 309, I,PI(I),PDI
303 FORMAT (30H) UNIFORM LOADS FOR ELEMENT //
1,10X, 8H ELEMENT,3X,13H LL INTENSITY,7X,20H UNIT WEIGHT FOR DL //
309 FORMAT (10X,15,8X,F10.3,10X,F10.3)

C
C DETERMINATION OF DL AND LL CONTRIBUTION OF EACH DECK ELEMENT
C TO VERTICAL NODAL LCADS (TRIBUTARY AREA CONCEPT)
C
310 DO 240 I=1,NUMEL
M = MAT(I)
L = NAT(I)
THU = SQRT(12.0*DM*(M,3,3)/DS(M,3))
THL = SQRT(12.0*DM*(L,3,3)/DS(L,3))
DO 220 J=1,4
NR = NP(I,J)
X(J) = XORDINB)
220 Y(J) = YORDINB)
X(5) = 0.25*(X(1)+X(2)+X(3)+X(4))
Y(5) = 0.25*(Y(1)+Y(2)+Y(3)+Y(4))
PLL = PI(I)
IF (NP(I,1).NE.NP(I,4)) GO TO 235

C
C FCR TRIANGULAR DECK ELEMENT
C
A3 = X(2)-X(1)
A2 = X(1)-X(3)
B2 = Y(3)-Y(1)
B3 = Y(1)-Y(2)
WT = (A3*B2-A2*B3)/6.0
DO 222 J=1,3
NA = NP(I,J)
KD = NA*10 - 2
LD = KD-5
P(KD) = P(KD) - WT*THU*PDL
P(LD) = P(LD) - WT*THL*PDL
P(KD) = P(KD) + WT*PLL
222 CONTINUE
GO TO 240

C
C FOR QUADRILATERAL DECK ELEMENT
C
235 DO 230 J=1,4
L = IPE(J)
K = ITE(J)
X41 = 0.5*(X(L) - X(J))
Y41 = 0.5*(Y(L) - Y(J))
X21 = 0.5*(X(K) - X(J))
Y21 = 0.5*(Y(K) - Y(J))
WT = 0.5*(X(5)-X(J))*(Y41-Y21) + (Y(5) - Y(J))*(X21 - X41)

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KE = KD-1
KF = KE-1

C
C ADD CONCENTRATED NODAL LCADS TO GLCPAL LOAD VECTOR
C
IF (FORCE(1).EG.0.0) GC TC 722
IF (TRUT) GO TO 721
IF (TCIS) DIKF = FORCE(1)
P(KF) = FORCE(1)
GO TC 720
721 IF (TCIS) DIKF-5) = FORCE(1)
P(KF-5) = FORCE(1)
720 IF (FORCE(2).EG.0.0) GC TC 722
IF (TRUT) GO TO 723
IF (TCIS) DIKEI = FORCE(2)
P(KE) = FORCE(2)
GO TO 722
723 IF (TCIS) DIKE-5) = FORCE(2)
P(KE-5) = FORCE(2)
722 IF (FORCE(3).EG.0.0) GC TC 724
IF (TRUT) GO TO 725
IF (TCIS) DIKC) = FORCE(3)
P(KC) = FORCE(3)
GO TO 724
725 IF (TCIS) DIKO-5) = FORCE(3)
P(KO-5) = FORCE(3)
724 IF (FORCE(4).EG.0.0) GC TC 726
IF (TRUT) GO TO 727
IF (TCIS) DIKC) = FORCE(4)
P(KC) = FORCE(4)
GO TO 726
727 IF (TCIS) DIKC-5) = FORCE(4)
P(KC-5) = FORCE(4)
726 IF (FORCE(5).EG.0.0) GC TO 120
IF (TRUT) GO TO 729
IF (TCIS) DIK8) = FORCE(5)
P(K8) = FORCE(5)
GO TO 120
729 IF (TCIS) DIK8-5) = FORCE(5)
P(K8-5) = FORCE(5)
120 CONTINUE
80 FORMAT (14,2L2,2X,5F10.2)
81 FORMAT (/80H NODE BOTTCM DIS-BC
*CE X-MOMNT Y-MOMNT //)
82 FORMAT (14,2L8,4X,5F10.2)

C
C INPUT OF LIVE LOAD INTENSITIES FOR DECK ELEMENTS IF NLL.GT.1
C
300 IF (NLL.LE.0.AND.NDL.LE.C) GO TC 200
DO 304 I=1,NUMEL
304 PI(I) = PLL
IF (NLL.LE.1) GO TO 311
DO 306 I=2,NLL
READ 307, I,NEL,PLL
307 FORMAT (2I4,F10.2)
READ 305, (NOLL(K),K=1,NEL)

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DO 448 KB=1,5
448 P(KB+NL) = P(KB+NL) - ST(KB+NJ,MA)*DIS
444 CONTINUE
IF (NCA.EQ.0.AND.PA.NE.C) GO TO 44C
460 READ (1) ST,SPB
IF (NCA.EQ.0.AND.PA.NE.C) GO TO 446
440 CONTINUE
C
C LOAD CONTRIBUTION FROM BEAM ELEMENTS
C
DO 470 J=1,NBEAM
READ (1) ST
DO 480 L=1,2
NA = NBP(J,L)
IF (NA.NE.NB) GO TO 480
MA = (L-1)*10 + ICO
GO TO 472
480 CONTINUE
GO TO 470
472 DO 474 KA=1,2
NK = NBP(J,KA)
NJ = (KA-1)*10
NL = (NK-1)*10
DO 476 KB=1,10
476 P(KB+NL) = P(KB+NL) - ST(KB+NJ,MA)*DIS
474 CONTINUE
470 CONTINUE
400 CONTINUE
DO 420 I=1,NEQBC
NR = NEBC(I)
420 P(NR) = C(NR)
PRINT 494
DO 490 I=1,NUMNP
KA = I*10
490 PRINT 492, I,P(KA-4),P(KA-3),P(KA-2),P(KA-1),P(KA),
* P(KA-9),P(KA-8),P(KA-7),P(KA-6),P(KA-5)
494 FORMAT (/// 30F FINAL LOAD VECTOR
* 10X,10H NODAL PT, 8X,8H U-FORCE, 8X,8H V-FORCE,
* 7X,9H X-MOMENT, 7X,9H Y-MOMENT /)
492 FORMAT (10X,15,8X,1P5E16.5/23X,1P5E16.5//)
C
C*****
C WRITE TOTAL LOAD VECTOR ON TAPE LDG
C*****
C
WRITE (LCG) P
PRINT 2019
2019 FORMAT (// 22H LINK NO 3 CCMPLTEC //)
RETURN
END

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NA = NP(I,J)
KD = NA*10 - 2
LU = KD - 5
P(KD) = PI(KD) - WT*THU*PDL
P(LD) = P(LD) - WT*THL*PDL
P(KD) = PI(KD) + WT*PLL
230 CONTINUE
240 CONTINUE
IF (INCL.NE.1) GO TO 200
C
C DETERMINATION OF DEADLOAD CONTRIBUTION OF WEB ELEMENTS
C
DO 260 I=1,NBEAM
M = MBAT(I)
THB = SQRT(12.0*CM*(M,3,3)/DS(M,3))
NA = NBP(I,1)
NB = NBP(I,2)
DL = (XORD(NB) - XORD(NA))*2 + (YORD(NB) - YORD(NA))*2
WT = SQRT(DL)*AH*THB*PDL/4.0
KA = NA*10 - 2
KB = NB*10 - 2
P(KA) = PI(KA)-WT
P(KB) = PI(KB)-WT
P(KA-5) = P(KA-5)-WT
260 P(KB-5) = P(KB-5)-WT
C
C*****
C EFFECT OF BOUNDARY CONDITIONS ON LOAD VECTOR
C*****
C
200 DO 400 I=1,NEQBC
NR = NEBC(I)
DIS = D(NR)
IF (DIS.EQ.0.0) GO TO 400
NB = (NR-1)/10 + 1
ICO = NR - (NB-1)*10
NCA = 0
IF (ICO.GT.5) NCA = 5
C
C LOAD CONTRIBUTION FROM BEAM ELEMENTS
C
DO 440 J=1,NUMEL
MA = 0
READ (1) ST,SPB
DO 450 L=1,4
NA = NP(J,L)
IF (NA.NE.NB) GO TO 45C
MA = (L-1)*5 + ICC - NCA
GO TO 442
450 CONTINUE
GO TO 460
442 IF(NCA.EQ.0) GO TO 460
446 DO 444 KA=1,4
NK = NP(J,KA)
NJ = (KA-1)*5
NL = (NK-1)*10 + NCA

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OVERLAY (RDX,4,0)
PROGRAM BANSOL
C*****
C COMPUTES AND RESETS FIELDLENGTH IF NECESSARY FOR BAND
C*****
C COMMON / SETUP / NUMP,NUMEL,NBEAR,NMAT,NRUN,NFLDL,IBANDW,NEQ,
* NBLKL,NUMBLK,NEQEC,MARG,AH
COMMON D(1)
INTEGER FWORD
C
CALL LWA (FWORD)
ND = NBLKL
NS = ND*IBANDW
NORG = 8
NRED = 3
N1 = 1
N2 = N1+NS
N3 = N2+NS
N4 = N3+NEQ
N5 = N4+ND
LWGRD = FWORD+NS
IF (LWGRD.GT.(NFLDL-MARG).AND.LWGRD.LT.NFLDL) GO TO 100
NFLDL = LWGRD
IF (NFLDL.LT.335008) NFLDL=335008
CALL REF (NFLDL)
CALL BAND (DIM1,C(IN2),C(IN3),DIM4),NEC,ND,IBANDW,NS,NUMBLK,NRUN,
* NORG,NRED)
RETURN
END
C
C*****
C SUBROUTINE BAND (A,AA,B,NBL,NEQ,NEQB,MBAND,NSB,NUMBLK,
* KKK,NOFG,NREC)
C*****
C OUT OF CORE BAND SOLVER FOR SYMMETRIC, POSITIVE DEFINITE MATRIX BAND
C RESTRICTION BANDWIDTH=MBAND,LE,NEQB=THE NUMBER OF EQUATIONS
C PER BLOCK. SPARSITY IS ACCOUNTED FOR BY SKIPPING INNERMOST DO-BAND
C LOOP EITHER IF MULTIPLIER C(I,KJ)=0 WITHIN BAND OR IF ALL J=0 BAND
C AT THE EDGE OF THE BAND BY VARYING THE BAND WIDTH WITH 45 DEGR BAND
C SHADOWS. IF PIVOT=C ALGORITHM SKIPS ASSOCIATED EQUATION
C*****
C A(NSB) LEADING BLOCK OF RECTIFIED BANDMATRIX (IN SINGLE BAND
C AA(NSB) TRAILING BLOCK OF RECTIFIED BANDMATRIX (IN SINGLE BAND
C B(NEQ) REDUCTION OF A PART OF IT IS OVERWRITTEN BY
C NBL(NEQB) TOTAL LEADVECTOR, ONE AT A TIME. IS OVERWRITTEN BAND
C NEQ SOLUTION VECTOR BY
C NSB=NEQB*MBAND MAXIMUM WIDTH OF HALF BAND
C KKK,LE,1 NUMBLK=(NEQ-1)/NEQB+1 NUMBER OF BLOCKS
C KKK,GT,1 REDUCTION OF B WITH BACKSUBSTITUTION
C TAPES REDUCTION OF B WITH BACKSUBSTITUTION
C NORG FILE FOR ORIGINAL MATRIX A, OVERWRITTEN BY
C NRED SOLUTION VECTOR AND LOAC VECTORS FOR CASES.GT.1
C DIMENICN (ANSB),AA(NSB),B(NEQ),NBL(NEQB) FILE WITH FIRST LOAD VECTOR, OVERWRITTEN BY
C INITIALIZATION REDUCED MATRIX A DURING DECOMPOSITION
C MB = MBAND-1
C INC= NEQB-1
C IF (KKK.GT.1) GO TO 800
C DECOMPSITION CF MATRIX A BLOCK BY BLOCK
C
C *****
C REWIND NORG
C REWIND NRED
C READ (NCRG) A
C READ (NREC) B
C REWIND NRED
C CALL NDBLCK (NRED)
C DO 700 N=1,NUMBLK
C *****
C DO 300 I=1,NEQB
C D = A(I)
C IF (D) 120,300,130
C 120 M= NEQB*(N-1)+1
C PRINT 1000, M,D

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1000 FORMAT (// 20F-PIVOT IS NEGATIVE /
* 26 H DIAGONAL TERM OF EQUATION 16,8H EQUALS 1PE12.5//)BAND
C ESTABLISH VARIABLE BANDWIDTH+
C 130 DO 140 J=NEQB,NSB,NEQB
140 CONTINUE
150 NBL(I) = NSB-J*I
C
C** JL = I+1
MAX = NBL(I)
JH = (MAX-1)/NEQB+1
II = I
C** DO 200 J=JL,JH
II = II+NEQB
C = A(II)/O
IF (C.EQ.0.0) GO TO 200
IF (J.GT.NEQB) GO TO 220
KK = J
C* DO 100 JJ=II,MAX,NEQB
A(KK) = A(KK)-C*A(JJ)
100 KK=KK+NEQB
220 A(II) = C
200 CONTINUE
300 CONTINUE
C C STORE REDUCED BLCK CN TAPE NRED
C 320 WRITE (NREC) A,NBL
C REDUCTION OF TRAILING BLOCK
IF (N.EQ.NUMBLK) GO TO 800
READ (NORG) AA
C*** NE = NEQB-MB
IL = NSB-MB
DO 600 I=I,MB
NE = NE+1
IL = IL+1
II = IL
C** DO 500 J=NE,NEQB
C = A(II)*A(J)
IF (C.EQ.0.0) GO TO 500
MAX = NBL(JJ)
KK = I
C* DO 400 JJ=II,MAX,NEQB
AA(KK) = AA(KK)-C*A(JJ)
400 KK = KK+NEQB
500 II = II+NEQB
600 CONTINUE

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C C C REPLACE LEADING BLOCK BY TRAILING BLOCK
C DO 650 I=1,NSB
650 A(I) = A(II)
700 CONTINUE
C C REDUCTION OF SINGLE LCADS - TOTAL LCAEVECTOR IN CORE
C 800 REWIND NRED
IF (KK.LE.1) GO TO 810
REWIND NORG
READ (NCRG) B
C C REDUCE LEADING BLOCK BY BLCK
C 810 DO 820 A=1,NUMBLK
READ (NREC) A,NBL
IN = (N-1)*NEQB
C*** DO 840 I=1,NEQB
IN = IN+1
IF (A(II).EQ.0.0) GO TO 840
JL = I+1
II = I
JH = (NBL(II)-1)/NEQB+1
IF (IN-NEQ) 830,850,840
830 JN = IN
C = B(IN)
IF (C.EQ.0.0) GO TO 840
C** DO 860 J=JL,JH
II = II+NEQB
JN = JN+1
860 B(JN) = B(JN)-C*A(II)
850 B(IN) = B(IN)/A(I)
840 CONTINUE
820 CONTINUE
C C BACKSUBSTITUTION - TOTAL SOLUTION IN CORE
C DO 900 A=1,NUMBLK
BACKSPACE NRED
READ (NREC) A,NBL
910 BACKSPACE NRED
NQB = (NUMBLK-N+1)*NEQB+1
NE = NEQB+1
C*** DO 920 I=1,NEQB
JI = NE-I
IF (A(IJ).EQ.0.0) GO TO 920
IL = JI+NEQB
MAX = NBL(JI)
KNI = NQB-I
JNI = KNI+1
IF (JNI.GT.NEQ) GC TO 920

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BAND 156
BAND 157
BAND 158
BAND 159
BAND 160
BAND 161
BAND 162
BAND 163
BAND 164
BAND 165
BAND 166
BAND 167
BAND 168

```



```

OVERLAY (BOX,5,0)
PROGRAM OUTPUT
C *****
C THIS PROGRAM READS IN INFORMATION FOR OUTPUT OPTIONS AND
C RESETS THE FIELDLENGTH IF NECESSARY
C *****
C COMMON / SETUP / NUMNP,NUMEL,NBEAM,NMAT,ARUN,NFLDL,IBANDW,NEQ,
* NBLK,NUMBLK,NEQBC,MARG,AH
COMMON / BOCCN / NEBC(200),ANGLE(200)
COMMON D(1)
INTEGER FMGRG,T1,T2,T3
C CONTROL INFORMATION OF OUTPUT OPTICNS
C
C READ 200, T1,T2,T3,NSTC,NDIA,ALF
200 FORMAT (5I4,F10.2)
PRINT 220, T1,T2,T3,NSTC,NDIA,ALF
220 FORMAT (7// 32HCONTROL PARAMETERS FOR OUTPUT ///
* 47H GLOBAL NCDAL FORCES CALCULATED AND PRINTED
* 47H STRESS RES. PRINTED AT CENTER OF DECK ELEM
* 47H OUTPUT IS PUNCHED ON CARDS
* 47H AC. OF NCDES WHERE STRESS RES. AVER. AND PRINT 14/OUTPUT
* 47H NUMBER OF WEB ELEMENTS IN Y-DIRECTION
* 40H ANGLE FROM X-AXIS TO DIR OF STRESS RES F17.5)
C
C DETERMINE REQUIRED FIELD LENGTH AND RESET IF NECESSARY
NEO = 0
NPD = 0
NND = 12*NSTO
IF (T1.EQ.1) NPD=NEQ
IF (T2.EQ.1) NEO=NUMEL
CALL LWA (FWORD)
N1 = 1
N2 = N1 + 4*NUMEL
N3 = N2 + 2*NBEAM
N4 = N3 + NUMEL
N5 = N4 + NUMEL
N6 = N5 + NBEAM
N7 = N6 + NBEAM
N8 = N7 + NBEAM
N9 = N8 + NUMNP
N10 = N9 + NUMNP
N11 = N10 + 9*NMAT
N12 = N11 + 6*NMAT
N13 = N12 + NEC
N14 = N13 + NPC
N15 = N14 + NEO*12
N16 = N15 + NSTO
N17 = N16 + NSTO
N18 = N17 + NNC
N19 = N18 + NNC
N20 = N19 + NNC
N21 = N20 + NNC

```

```

BAND 169
BAND 170
BAND 171
BAND 172
BAND 173
BAND 174
BAND 175
BAND 176
BAND 177
BAND 178
BAND 179
BAND 180
C**
C = B(KNT)
DO 930 I=IL,PAX,NEQB
C = C-A(I)*B(JNI)
930 JNI = JNI+1
B(KNI) = C
920 CONTINUE
900 CONTINUE
REWIND NDRG
WRITE (NDRG) B
RETURN
END

```

1 OUTP  
2 OUTP  
3 \*\*\*\*\*OUTP  
4 OUTP  
5 OUTP  
6 \*\*\*\*\*OUTP  
7 OUTP  
8 OUTP  
9 OUTP  
10 OUTP  
11 OUTP  
12 OUTP  
13 OUTP  
14 OUTP  
15 OUTP  
16 OUTP  
17 OUTP  
18 OUTP  
19 OUTP  
20 14/OUTP  
21 14/OUTP  
22 14/OUTP  
23 14/OUTP  
24 14/OUTP  
25 14/OUTP  
26 OUTP  
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53 OUTP  
54 OUTP  
55 OUTP

```

N22 = N21 + NNC
N23 = N22 + NKO
N24 = N23 + NNC
LWORD = FWORD+N24
IF (LWORD.GT.(INFLDL-MARG).AND.LWORD.LI.NFLDL) GO TO 100
CALL REL (NFLCL)
NFLDL = LWORD
100 CALL OTPT (D(N1),D(N2),D(N3),D(N4),D(N5),D(N6),D(N7),D(N8),D(N9),D(N10),D(N11),D(N12),D(N13),D(N14),D(N15),D(N16),
* D(N17),D(N18),D(N19),D(N20),D(N21),D(N22),D(N23),
* NUMEL,NBEAM,NUMNP,MMAT,NEQ,NSTC,NDIA,T1,T2,T3,NP0,NEQ,
* ALF)
** RETURN
END

```

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SUBROUTINE OTPT (NP,NBP,MAT,NAT,MBAT,BSIA,BCGS,XORD,YORD,DM,DS,
* D,P,SCT,NPS,NHT,XNS,YNNS,XYNS,XXMO,YYMO,XYMO,
* SNP,IE,IP,IP,IM,IG,NSTO,NDIA,T1,T2,T3,NP0,NEQ,
* ALF)
C *****
C THIS SUBROUTINE CALCULATES AND PRINTS THE GLOBAL NODAL DIS-
C PLACEMENTS AND FORCES AND DETERMINES CENTER AND NODAL
C STRESS RESULTANTS TOGETHER WITH THEIR AVERAGES AND PRINCIPAL
C VALUES FOR BOTH DECK AND WEB ELEMENTS
C *****
C COMMON / SETUP / NUMNP,NUMEL,NBEAM,MMAT,ARUN,NFELD,IBANDW,NEQ,
* COMMON / NBLK,NUMBLK,NEQBC,MARG,AH
* COMMON / EBCON / NEBC(200),ANGLE(200)
* COMMON / PLBDG / XI(5),YY(5),CM(3,3),PP(5),BME(3,5),CV(3,5),
* SPB(19,19),RA(19)
* COMMON / PLSTR / XA(4),YA(4),CS(6),SPD(12,12),RL(12),ST(5,3)
C
C DIMENSION NP(16,4),NBP(16,2),MAT(IE),NAT(IE),MBAT(16),BSIN(16),
* BCGS(16),XORD(16),YORD(16),DM(16,3),DS(16,6),D(16),
* R(NP),SCT(NEG,12),NPSINSTC,NHT(NSTO),XNS(NSTO,12),
* YNS(NSTO,12),XYNS(NSTO,12),XPC(NSTO,12),YYMO(NSTO,12),
* XMO(NSTO,12),SNP(NSTO,12)
C DIMENSION SK(20,20),ITE(2),ILE(2)
DATA ITE /1,4/, ILE /9,4/
INTEGER TI,T2,T3

C
C *****
C INITIALIZATION
C
C REWIND 1
C REWIND 2
C READ (1) NP,NBP
C READ (1) XORD,YORD,MAT,NAT,MBAT,DM,DS,BSIN,BCGS
C READ (8) D

C *****
C OUTPUT OF NODAL POINT DISPLACEMENTS
C *****
C DO 780 I=1,NEQBC
C NR=NEBC(I)
C PHI=ANGLE(1)
C IF (PHI.EQ.0.0) GO TO 780
C TRANSFORM SKEW DISPL. INTO GLOBAL X-Y COORD DISPL
C
C NRI=NR-1
C D(NR) = D(NRI)*SIN(PHI)
C D(NRI) = D(NRI)*COS(PHI)
780 CONTINUE
PRINT ,9C
900 FORMAT (21H1 DISPLACEMENT VECTOR////
* 10X,10H NODAL PT.,7X,7H U-DIS.,9X,7H V-DIS.,9X,7H W-DIS.,
* 8X,8H X-ROTAT,8H Y-ROTAT //)
DO 901 I=1,NUMNP
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KA = NBP(N,I)*10 - 10
DO 380 J=1,10
JA = (I-1)*10 + J
380 SUM = SUM + SK(A,JA)*D(J+KA)
375 CONTINUE
P(K+LA) = P(K+LA) + SUM
370 CONTINUE
365 CONTINUE
360 CONTINUE
PRINT 395
DO 390 J=1,NUMNP
KA = J*10
390 PRINT 90T, J,P(KA-4),P(KA-3),P(KA-2),P(KA-1),P(KA),
* P(KA-9),P(KA-8),P(KA-7),P(KA-6),P(KA-5)
395 FORMAT (30H GENERALIZED FORCE VECTOR
* 10X,10H NODAL PT., 6X,8H U-FORCE, 8X,8H V-FORCE,
* 7X,9H X-MOMENT, 7X,9H Y-MOMENT ////)
C *****
C DETERMINATION OF INTERNAL FORCES AT CENTER AND AT THE NODES *****
C OF EACH ELEMENT *****
C *****
C 400 IF (NSTO.LE.0.AND.T2.NE.1) GO TO 800
REWIN 1
READ (1) NP,NBP
READ (1) XORD,YORD,MAT,NAT,MBAT,DM,DS,BSIN,BCOS
IF (NSTO.LE.0) GO TO 640
IF (NSTO.NE.NUMNP) GO TO 454
DO 452 I=1,NUMNP
452 NPS(I) = 1
GO TO 460
454 READ 456, (NPS(I),I=1,NSTO)
456 FORMAT (20I4)
C
C INITIALIZATION OF ARRAYS FOR NODAL POINT STRESSES
C
460 DO 464 J=1,NSTC
DO 462 L=1,12
XNS(J,L) = 0.0
YNS(J,L) = 0.0
XMO(J,L) = 0.0
YMO(J,L) = 0.0
464 NMT(J) = 0
PRINT 457
PRINT 459, (NPS(I),I=1,NSTC)
457 FORMAT (// 65H NODES WHERE INTERNAL FORCES COMPUTED, AVERAGED AND
* OUTPUT ////)
459 FORMAT (20I4)
C *****
C DETERMINATION OF STRESS RESULTANTS AND MOMENTS FOR DECK ELEMENT *****
C *****
C *****

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K2 = I*10
K3 = K2 - 1
K4 = K2 - 2
K5 = K2 - 3
K6 = K2 - 4
PRINT 90T, I, D(K6),D(K5),D(K4),D(K3),D(K2),
* D(K6-5),D(K5-5),D(K4-5),D(K3-5),D(K2-5)
901 CONTINUE
907 FORMAT (10X,15,8X,1P5E15,5/23X,1P5E15,5//)
IF (I3.NE.1) GO TO 120
WRITE (7) D
ENDFILE 7
C *****
C DETERMINATION OF GENERALIZED FORCES AND REACTIONS *****
C *****
C *****
C 120 IF (I1.NE.1) GC TC 400
DO 140 I=1,NEQ
140 P(I) = 0.0
C *****
C CONTRIBUTION FROM DECK ELEMENTS
C *****
C *****
DO 310 N=1,NUMEL
NCT = 0
NKA = 5
325 READ (1) SK,SPB
DO 340 L=1,4
LA = NP(N,L)*10 - NKA
DO 350 K=1,5
IA = (L-1)*5 + K
SUM = 0.0
DO 320 I=1,4
KA = NP(N,I)*10 - NKA
DO 330 J=1,5
JA = (I-1)*5 + J
330 SUM = SUM + SK(A,JA)*D(J+KA)
320 CONTINUE
P(K+LA) = P(K+LA) + SUM
350 CONTINUE
340 CONTINUE
NKA = 10
NCT = NCT + 1
IF (NCT.EQ.1) GO TO 325
310 CONTINUE
C *****
C CONTRIBUTION FROM BEAM ELEMENTS
C *****
C *****
DO 360 N=1,NBEBEAM
READ (1) SK
DO 365 L=1,2
LA = NBP(N,L)*10 - 10
DO 370 K=1,10
IA = (L-1)*10 + K
SUM = 0.0
DO 375 I=1,2

```

```

640 IF (I2.EQ.1) PRINT 421
DO 450 N=1,NUMEL
NCT = 0
M = MAT(N)
NKA = 4
NKB = 3
NTRI = 4
IF (NP(N,I).EQ.NP(N,I+4)) NTRI=3
620 KKK = 0
434 DO 441 I=1,4
JA = NP(N,I)
XAI(I) = XCRD(JAI)
YAI(I) = YCRD(JAI)
XXI(I) = XCRD(JAI)
YYI(I) = YCRD(JAI)
PPI(I) = 1.0
KA = JA*10 - NKA
KB = KA + 1
I1 = 2*I - 1
I2 = 2*I
RI(I1) = C(KA)
RI(I2) = C(KB)
KC = JA*10 - NKB
DO 662 JL=1,3
KKK=KKK+1
KC = KC + 1
662 RA(KK) = D(KC)
441 CONTINUE
DO 444 I=1,6
CS(I) = DS(M,I)
DO 668 J=1,19
668 RA(J) = 0.0
DO 674 I=1,3
DO 674 J=1,3
674 CHI(I,J) = DM(M,I,J)
READ (I) SK,SPB
CALL PSOLL (NTRI)
CALL FPLATE (NTRI)
NC4 = NCT*6

```

C C C OUTPUT OF INTERNAL FORCES AT CENTER OF EACH ELEMENT

```

IF (I2.NE.1) GO TO 477
IF (NCT.EQ.0) PRINT 423,N
DO 429 J=1,3
I = J+NCT*6

```

```

429 SCT(N,I) = ST(I5,J)
PRINT 469, ST(I5,1),ST(5,2),ST(5,3),BM(I,5),BM(2,5),BM(3,5)
423 FORMAT (12H ELEMENT I5)
421 FORMAT (50H INTERNAL FORCES AT CENTER OF DECK ELEMENTS
* 25X, 12H NKX ,5X,12H NY ,5X,12H NKY
* 6X,10H HXK ,7X,10H HXK ,7X,10H HKY ///)
477 IF (NSTO.LE.0) GO TO 450

```

C C AVERAGE INTERNAL FORCES OF ADJACENT ELEMENTS AT EACH NODE

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C C
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C C C OUTPUT OF AVERAGE INTERNAL FORCES AT THE NODES

```

DO 455 K=1,4
IF (K.EQ.4.AND.NTRI.EQ.3) GO TO 455
N2 = NP(N,K)
DO 472 J=1,NSTC
IF (N2.EQ.NPS(J)) GO TO 474
472 CONTINUE
GO TO 455
474 IF (NCT.EQ.1) GO TO 478
NMT(J) = NMT(J)+1
478 XNS(J,L) = ST(K,1)
YNS(J,L) = ST(K,2)
XMO(J,L) = BM(I,K)
YMO(J,L) = BM(I,K)
XMO(J,L) = BM(2,K)
YMO(J,L) = BM(3,K)
455 CONTINUE
M = NAT(N)
NKA = 9
NKB = 8
NCT = NCT+1
IF (NCT.EQ.1) GO TO 620
450 CONTINUE

```

C C C OUTPUT OF AVERAGE INTERNAL FORCES AT THE NODES

```

IF (NSTO.LE.0) GO TO 800
ALF = ALF/57.29578
BSI = SIN(ALF)
BCO = COS(ALF)
PRINT 465
DO 467 I=1,NSTO
PRINT 468, NPS(I)
K = NPS(I)
LL = 0
KJ = K*10-2
NN = NMT(I)
NKA = 1
NKB = NN
NCT = 0
480 SNXX = 0.0
SNYY = 0.0
SNXY = 0.0
SMXX = 0.0
SMYY = 0.0
SMXY = 0.0
SMY = 0.0
DO 476 L=NKA,NKB
PRINT 469, XNS(I,L),YNS(I,L),XMO(I,L),YMO(I,L),
* XMO(I,L)
SNXX = SNXX + XNS(I,L)
SNYY = SNYY + YNS(I,L)
SNXY = SNXY + XMO(I,L)
SMXX = SMXX + XNS(I,L)
SMYY = SMYY + YMO(I,L)
SMXY = SMXY + XMO(I,L)

```

C C C OUTPUT OF AVERAGE INTERNAL FORCES AT THE NODES

```

C*****
C
240 IF (NDIA.EQ.0) NCIA=NBEAM
NTA = 1
570 IF (NTB.EQ.NBEAM) PRINT 764
IF (NTB.LI.NBEAM) PRINT 762
DO 510 J=1,NSTO
DO 505 L=1,4
XXNS(J,L) = 0.0
YYNS(J,L) = 0.0
XXMO(J,L) = 0.0
YYMO(J,L) = 0.0
505 XMO(J,L) = 0.0
510 NMT(L) = 0
DO 500 N=NTA,NTB
M = MBAT(N)
SI = BSIN(N)
CO = COS(N)
NA = NBP(N,1)
NB = NBP(N,2)
NL = (XORD(NB)-XORD(NA))*CC + (YORD(NB)-YORD(NA))*SI
DO 520 I=1,2
LA = ITE(I)
XX(LA) = 0.0
YY(I) = 0.0
YY(I+2) = A1
520 CONTINUE
DO 512 I=1,4
XA(I) = XX(I)
YA(I) = YY(I)
512 PP(I) = 1.0
DO 506 I=1,3
DO 506 J=1,3
506 CM(I,J) = DM(I,I,J)
508 CS(I) = CS(M,I)
DO 540 J=1,2
JA = ITE(J)
IB = ILE(J)
I2 = NB*10
RI(JA) = D(I1-IB)*CC + D(I1-IB+1)*SI
RI(J+1) = D(I2-IB)*CC + D(I2-IB+1)*SI
RI(JA+4) = D(I1-IB+2)
RI(J+8) = D(I2-IB+2)
RI(J+9) = D(I2-IB+3)*SI - D(I1-IB+4)*CC
540 CONTINUE
CALL PLSPI2
DO 550 J=1,2
IA = ITE(J)
IB = ILE(J)
I1 = NA*10

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476 CONTINUE
SNP(I,LL+1) = SNXX/NN
SNP(I,LL+2) = SNXY/NN
SNP(I,LL+3) = SNXX/NN
SNP(I,LL+4) = SNXX/NN
SNP(I,LL+5) = SNXY/NN
SNP(I,LL+6) = SNXY/NN
PRINT 463, (SNP(I,LL+J),J=1,6)
C
C
C
DETERMINATION OF PRINCIPAL VALUES OF INTERNAL FORCES
226 IF (BSI.NE.0.0) GO TO 272
AB = 0.5*(SMXX+SMYY)/NN
BA = 0.5*(SMXX-SMYY)/NN
AA = 0.5*(SMXX+SMYY)/NN
BB = 0.5*(SMXX-SMYY)/NN
TAU = SQRT((BA**2+(SMXY/NN)**2))
SMMAX = AB+TAU
SMMIN = AB-TAU
SMAX = AA+TAU
SMIN = AA-TAU
IF (BA.EQ.0.0.AND.SMXY.EQ.0.0) GO TO 221
BAG = 57.29578*ATAN2((SPXY/NN),BA)/2.0
GO TO 219
221 BAG = 360.0
219 IF (BB.EQ.0.0.AND.SNXY.EQ.0.0) GO TO 211
BANG = 57.29578*ATAN2((SNXY/NN),BB)/2.0
GO TO 218
211 BANG = 360.0
GO TO 218
272 SS = BSI*BSI
SC = BSI*BCO
CC = BCO*ECO
SMAX = (SMXX*CC+SMYY*SS+2.0*SMXY*SC)/NN
SMIN = (SMXX*CC+SMYY*SS+2.0*SMXY*SC)/NN
SMIN = ((SMXY-SMXX)*SC+SMXY*(CC-SS))/NN
SMIN = BSI
BANG = BCI
BAG = BCO
218 PRINT 486, SMAX,SMIN,BANG,SMAX,SMIN,BAG
IF (NCT.GT.0) GO TO 467
NKA = 7
NKB = NWT(I)+6
LL = 6
NCT = NCT+1
GO TO 480
467 CONTINUE
IF (T3.NE.1) GO TO 240
WRITE (7) SNP
ENDFILE 7
WRITE (7) YYHO
ENDFILE 7
C
C
C
DETERMINATION OF INTERNAL FORCES IN THE WEB ELEMENTS
*****
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OTPT 449
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SNVY = 0.C
SNXX = 0.0
SMXX = 0.0
SMXY = 0.0
SMZY = 0.0
DO 530 L=NKA,NKB
PRINT 469, XXNS(J,L),YYNS(J,L),XVNS(J,L),XXMO(J,L),YYMO(J,L),
*
SMXX = SNXX + XXNS(J,L)
SNVY = SNVY + YYNS(J,L)
SMXY = SMXY + XVNS(J,L)
SMZY = SMZY + XXMO(J,L)
SMXY = SMXY + YYMO(J,L)
530 CONTINUE
SNP(J,LL+1) = SNXX/NN
SNP(J,LL+2) = SNVY/NN
SNP(J,LL+3) = SMXX/NN
SNP(J,LL+4) = SMXY/NN
SNP(J,LL+5) = SMZY/NN
SNP(J,LL+6) = SPXY/NN
C
C
C
C
C
DETERMINATION OF PRINCIPAL INTERNAL FORCES IN WEBS
568 AA = 0.5*(SNXX+SNVY)/NN
BB = 0.5*(SNXX-SNVY)/NN
AB = 0.5*(SMXX-SMXY)/NN
BA = 0.5*(SMXX+SMXY)/NN
TAU = SQR(BB**2+(SNXX/NN)**2)
TAV = SQR(BA**2+(SMXY/NN)**2)
SNMAX = AA+TAU
SNMIN = AA-TAU
SMAX = AB+TAV
SMIN = AB-TAV
SMIN = AB-TAV
IF (BB.EQ.0.0.AND.SNXY.EQ.0.0) GO TO 574
IF (BA.EQ.0.0.AND.SNXY.EQ.0.0) GO TO 574
BANG = 57.29578*ATAN2((SNXY/NN),BB)/2.0
GO TO 575
574 BANG = 360.
575 IF (BA.EQ.0.0.AND.SMXY.EQ.0.0) GO TO 576
IF (BA.EQ.0.0.AND.SMXY.EQ.0.0) GO TO 576
GO TO 577
576 BAG = 360.0
577 PRINT 463, (SNP(J,LL+1),I=1,6)
PRINT 486, SNMAX,SNMIN,BANG,SMHAX,SMHIN,BAG
IF (NKA.EQ.3) GO TO 580
NKA = 3
NKB = 4
LL=6
GO TO 572
580 CONTINUE
571 IF (NTB.EQ.NBEAM) GO TO 484
NTA = NTB*I
NTB = NBEAM
GO TO 570
484 IF (I3.NE.1) GO TO 600

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OTPT 448

I2 = NB*IC
I3 = IA*3 - 2
I4 = (J+1)*3 - 2
RA(I3) = D(I1-IB)*SI - D(I1-IB+1)*CC
RA(I4) = D(I2-IB)*SI - D(I2-IB+1)*CC
RA(I3+1) = D(I1-IB+3)*CO + D(I1-IB+4)*SI
RA(I4+1) = D(I2-IB+3)*CO + D(I2-IB+4)*SI
RA(I3+2) = 0.0
RA(I4+2) = 0.0
CALL FONEH
550 CONTINUE
C
C
C
C
C
DETERMINATION AND AVERAGING OF INTERNAL FORCES OF WEB
ELEMENTS AT THE NODES
C
DO 560 K=1,2
KA = 5 - K
K2 = NBP(N,K)
L = K
DO 544 J=1,NSTO
IF (N2.EQ.NPS(J)) GO TO 546
544 CONTINUE
GO TO 560
546 XXNS(J,L) = ST(KA,1)
YYNS(J,L) = ST(KA,2)
XVNS(J,L) = ST(KA,3)
XXMO(J,L) = BM(I,KA)
YYMO(J,L) = BM(2,KA)
XVMO(J,L) = BM(3,KA)
XXNS(J,L+2) = ST(K,1)
YYNS(J,L+2) = ST(K,2)
XVNS(J,L+2) = ST(K,3)
XXMO(J,L+2) = BM(I,K)
YYMO(J,L+2) = BM(2,K)
XVMO(J,L+2) = BM(3,K)
NWT(J) = NWT(J)+1
560 CONTINUE
500 CONTINUE
C
C
C
AVERAGING OF INTERNAL FORCES AT THE NODES OF WEB ELEMENTS
C
DO 571 N=NTA,NTB
DO 580 M=1,2
I = NBP(N,M)
DO 582 J=1,NSTO
IF (I.EQ.NPS(J)) GO TO 584
582 CONTINUE
GO TO 580
584 IF (N.EQ.NBEAM) GO TO 586
IF (NBP(N,I).EQ.NBP(N+1,I)) GO TO 586
IF (NBP(N,M).EQ.NBP(N+1,I)) GO TO 580
586 PRINT 468, I
562 NKA = 1
NKB = 2
LL = 0
NN = NWT(J)
572 SNXX = 0.C

```

```
WRITE (7) SNP
ENDFILE 7
WRITE (7) YYP
ENDFILE 7
600 PRINT 2019
C
C   FORMAT STATEMENTS
C
463 FORMAT (5X,12H NP. AVERAGE,3X,1P6E17.5/)
464 FORMAT (2X,17F PRIN NP QUANT,1X,1P6E17.5///)
465 FORMAT (12H NODAL POINT 15/)
466 FORMAT (48H1 INTERNAL FORCES AT NODES OF DECK PLATES
  * 25X,12H MXX 7X,10H MY 7X,10H MXY
  * 6X,10H MXX 7X,10H MY 7X,10H MXY 7X,10H MXY 7X,10H MXY
  * 25X,12H MXX 7X,10H MY 7X,10H MXY 7X,10H MXY 7X,10H MXY
  * 6X,10H MXX 7X,10H MY 7X,10H MXY 7X,10H MXY 7X,10H MXY
  * 25X,12H MXX 7X,10H MY 7X,10H MXY 7X,10H MXY 7X,10H MXY
  * 6X,10H MXX 7X,10H MY 7X,10H MXY 7X,10H MXY 7X,10H MXY)
2019 FORMAT (// 23H LINK NO 5 COMPLETED //)
800 RETURN
END
```

```
PSDI 1
PSDI 2
PSDI 3
PSDI 4
PSDI 5
PSDI 6
PSDI 7
PSDI 8
PSDI 9
PSDI 10
PSDI 11
PSDI 12
PSDI 13
PSDI 14
PSDI 15
PSDI 16
PSDI 17
PSDI 18
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PSDI 21
PSDI 22
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PSDI 44
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PSDI 46
PSDI 47
PSDI 48
PSDI 49
PSDI 50
PSDI 51
PSDI 52
PSDI 53
PSDI 54
PSDI 55
PSDI 56

SUBROUTINE PSDI1 (NTRI)
C *****
C THIS SUBROUTINE COMPLETES THE STRESS RESULTANTS FOR THE PLANE
C STRESS ELEMENTS Q809 AND Q8011(2) AT NODES AND AT CENTER *****PSDI
C *****
C INPUT
C
C D(1)
C I=1,6
C XAI(4)
C YAI(4)
C RL(I)
C
C CONSTITUTIVE LAW RELATING STRESS-RES TO STRAINS
C ELEMENTS 11,22,33,12,13,23
C GLOBAL X-COORDINATES OF NODES
C GLOBAL Y-COORDINATES OF NODES
C GLOBAL U-V DISPLACEMENTS AT THE NODES
C
C OUTPUT
C
C ST(I,J)
C
C INTERNAL STRESS RESULTANTS AT CENTER AND
C CORNER NODES
C CCRNER NODES
C CENTER NODE
C NXX,NYY,NXY STRESS RESULTANTS
C
C CUMMCN / PLSTR / XAI(4),YAI(4),D(6),S(12),RI(12),ST(5,3)
C
C *****
C DIMENSION P(4,2),CC(4,2),A(2,2),ETA(2),IPERM(2),IT(5),
C AX(5),AY(5),X(4),Y(4),DIS(8),U(4),V(4)
C
C DATA DC / -1., 1., 1., -1., -1., -1., -1., 1., 1., 1. /
C DATA AX / 0., -1., 1., 1., -1., -1., -1., 1. /
C DATA AY / 0., -1., -1., 1., 1., 1., 1., 1. /
C DATA IT / 5,1,2,3,4,1,PERM / 2,1/
C EQUIVALENCE (A11,A(1)), (A21,A(2)), (A12,A(3)), (A22,A(4))
C
C *****
C INITIALIZATION
C
C DO 100 J=1,3
C DO 100 I=1,5
C 100 ST(I,J)=0.0
C EXY = 0.0
C
C *****
C TRANSFORMATION INTO LOCAL CONNECTED COORDINATES
C
C
C DX = XAI(2)-XAI(1)
C DY = YAI(2)-YAI(1)
C AL = SQRT(DX**2+DY**2)
C CO = DX/AL
C SI = DY/AL
C DO 520 I=1,6
C X(I) = (XAI(I)-XAI(1))*CO + (YAI(I)-YAI(1))*SI
C 520 Y(I) = -(XAI(I)-XAI(1))*SI + (YAI(I)-YAI(1))*CO
C CC = CO*CC
C SS = SI*SI
C SC = SI*CC
C DO 540 I=1,4
C I2 = I*I
C I1 = I2-1
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PSDI 113
PSDI 114
PSDI 115
PSDI 116
PSDI 117
PSDI 118
PSDI 119
PSDI 120
PSDI 121
PSDI 122
PSDI 123
PSDI 124
PSDI 125
PSDI 126
PSDI 127
PSDI 128
PSDI 129

SXY = ZZ/DET+D(3)*EXY
TRANSFORM STRESS RESULTANTS INTO GLOBAL COORDINATES
ST(JJ,1) = SXX*CC+SY*SS-2.0*SXY*SC
ST(JJ,2) = SXX*SS+SY*CC+2.0*SXY*SC
ST(JJ,3) = (SXX-SY)*SC+SXY*(CC-SS)
300 CONTINUE
IF (NTRI.NE.3) GO TO 4CC
C
C DETERMINATION OF LOCAL STRESSES FOR CONSTANT STRAIN TRIANGLE
DO 700 I=1,4
DO 420 J=1,3
420 ST(I,J) = ST(I5,J)
400 RETURN
END

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PSDI 57
PSDI 58
PSDI 59
PSDI 60
PSDI 61
PSDI 62
PSDI 63
PSDI 64
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PSDI 68
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PSDI 106
PSDI 107
PSDI 108
PSDI 109
PSDI 110
PSDI 111
PSDI 112

DIS(I1) = RI(I1)*CO + RI(I2)*SI
540 DIS(I2) = -RI(I1)*SI + RI(I2)*CC
C
C COMPUTE STRESS RESULTANTS AT SAMPLING POINTS
DO 300 I=1,5
JJ = IT(I)
IF (II.GT.II.ANC.NTRI.EQ.3) GC TO 70C
ETA(I) = AX(I)
ETA(2) = AY(I)
C
C FORMATION OF LOCAL DERIVATIVES
DO 150 I = 1,2
J = I*PERM(I)
A(I,1) = 0.
A(I,2) = 0.
DO 140 L = 1,4
C=0.250*DC(L,I)*(1.0+DC(L,J)*ETA(I))
P(L,I)=C
A(I,1)=A(I,1)+C*X(L)
A(I,2)=A(I,2)+C*Y(L)
140 CONTINUE
150 CONTINUE
DET = A11*A22 - A12*A21
IF (DET.GT.0.0) GC TO 250
PRINT 1010, DET, ETA(1), ETA(2)
PRINT 1020
PRINT 1030, (X(I),Y(I)), I=1,4
1010 FORMAT (//30H DETERMINATE OF JACOBIAN E15.5/
* 30H SAMPLING AT LOCATION X E15.5/
* 30H SAMPLING AT LOCATION Y E15.5/
1030 FORMAT (10X,2E15.5)
C
C FORMATION OF GLOBAL DERIVATIVES
250 DO 200 J=1,4
U(J) = A22*p(J,1) - A12*p(J,2)
200 V(J) = -A21*p(J,1) + A11*p(J,2)
C
C DETERMINATION OF STRESS RESULTANTS AT NODES AND CENTER
XX=0.0
YY=0.0
ZZ=0.0
DO 500 I=1,4
K = I + I
J = K - 1
IF (II.NE.1) GC TO 560
EXY = EXY + (V(I)*DIS(J)+U(I)*DIS(K))/DET
560 XX = XX + (V(I)*DIS(J) + U(I)*DIS(K))
YY = YY + (U(I)*DIS(J) + V(I)*DIS(K))
ZZ = ZZ + (U(I)*DIS(J) + V(I)*DIS(K))
500 CONTINUE
SXX = XX/DET+D(5)*EXY
SYY = YY/DET+D(6)*EXY

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SUBROUTINE FPLATE (INTRJ)
C *****
C THIS SUBROUTINE DETERMINES THE MOMENT RESULTANTS FOR THE
C QUADRILATERAL PLATE BENDING ELEMENT LCCT-9 AND THE TRIANGULAR
C PLATE BENDING ELEMENT LCCT-9
C *****
C INPUT
C GM(3,3) CONSTITUTIVE LAW RELATING MOMENTS TO CURVATURES
C X(4) GLOBAL X-COORDINATES OF NODES
C PL(4) NODAL LOAD INTENSITIES OF DISTRIBUTED LOADS
C *****
C OUTPUT
C *****
C BM(I,J) INTERNAL MOMENTS AT CORNER AND CENTER NODES
C J=1,4 CENTER NODES
C J=5 CENTER NODE
C I=1,3 MXX,MYX,MYX MOMENT COMPONENTS
C *****
C COMMON / PLBDG / X(5),Y(5),CM(3,3),P(5),BM(3,5),CV(3,5),S(19,19),
C *****
C COMMON / TRIAG / R(3),A(3),CMT(3,3),PT(3),CVT(3,3),ST(12,12),R(12)
C *****
C DIMENSION IPERM(4),NC(3),FAC(3)
C DATA IPERM /2,3,4,1/, FAC /5,5,5,25/
C *****
C LOGICAL TRIG
C *****
C INITIALIZE PARAMETERS
C *****
C NTR = 4
C TRIG = NTRI.EQ.3
C IF (.NOT. TRIG) NTR = 1
C DO 130 I=1,3
C DO 130 J=1,5
C 130 CV(I,J) = 0.0
C X(5) = 0.25*(X(1)+X(2)+X(3)+X(4))
C Y(5) = 0.25*(Y(1)+Y(2)+Y(3)+Y(4))
C P(5) = 0.25*(P(1)+P(2)+P(3)+P(4))
C NBF = 11
C L1 = 13
C L2 = 19
C IF (.NOT. TRIG) GO TO 120
C NBF = 9
C GO TO 160
C *****
C RECOVER DISPLACEMENTS FOR DCF ELIMINATED BY STAT COND
C *****
C 120 DO 140 L=L1,L2
C M = L - 1
C DO 140 K = 1,M
C 140 RI(L) = RI(L) - S(K,L)*RI(K)
C *****
C PREPARE INPUT FOR MOMENT CALCULATION IN EACH LCCT TRIANGLE
C *****
C 160 NC(3) = NBF - 6

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PLSP 113
PLSP 114
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PLSP 155
PLSP 156
PLSP 157
PLSP 158
PLSP 159
PLSP 160
PLSP 161

*
* 30H OUTPUT AT LOCATION X E15.5/
* 30H OUTPUT AT LOCATION Y E15.5//
* 1030 FORMAT (1//30H- ELEMENT COORDINATES /)
* 1030 FORMAT (10X,2E15.2)
* GO TO 200
C *****
C FORMATION OF LOCAL DERIVATIVES
C *****
C 450 DO 400 I=1,4
C VY(I) = 0.125*FY4(I)*(2.0+3.0*FX4(I)*X-FX4(I)*XB)
C VX(I) = 0.375*(1.0+FY4(I)*Y)*(FX4(I)-FX4(I)*XB)
C VY(I+4) = 0.125*FY4(I)*(-FX4(I)-X*FX4(I)*XA+XB)
C 400 VX(I+4) = 0.125*(1.0+FY4(I)*Y)*(-1.0+2.0*FX4(I)*X+3.0*XA)
C *****
C FORMATION OF DERIVATIVES IN GLOBAL COORDINATES
C *****
C DO 420 I=1,4
C EX(I) = X22*EX(I)-X12*FY(I)
C 420 EY(I) = -X21*FX(I)+X11*FY(I)
C *****
C K=IT(I)
C AA = VX(I)+FY4(I)*FT1(K)*VX(K+4) + FY4(I)*FT1(J)*VY(J+4)
C BB = VY(I)+FY4(I)*FT1(K)*VY(K+4) + FY4(I)*FT1(J)*VX(J+4)
C EY(I+4) = -X21*FT2(I)*VX(I+4)+X11*FT2(I)*VY(I+4)
C EY(I+4) = X22*FT2(I)*VX(I+4)-X12*FT2(I)*VY(I+4)
C EY(I) = -X21*AA + X11*BB
C 440 EY(I) = X22*AA - X12*BB
C *****
C COMPUTATION OF NODAL POINT STRESSES
C *****
C XX=0.0
C YY=0.0
C ZZ=0.0
C DO 500 I=1,4
C XX=XX+D(1)*EX(I)*V(I)+D(5)*EY(I)*V(I)
C YY=YY+D(4)*EX(I)*V(I)+D(6)*EY(I)*V(I)
C 500 ZZ=ZZ+D(5)*EX(I)*V(I)+D(3)*EY(I)*V(I)
C DO 510 I=1,8
C XX=XX+D(4)*EY(I)*V(I+4)+D(5)*EX(I)*V(I+4)
C YY=YY+D(2)*EY(I)*V(I+4)+D(6)*EX(I)*V(I+4)
C 510 ZZ=ZZ+D(6)*EY(I)*V(I+4)+D(3)*EX(I)*V(I+4)
C ST(I,1) = XX/DET
C ST(I,2) = YY/DET
C ST(I,3) = ZZ/DET
C 200 CONTINUE
C RETURN
C END

```



```

SUBROUTINE FONEH
C*****
C THIS SUBROUTINE COMPUTES THE MY-MOMENTS AT EACH NODE
C ASSUMING ONE WAY BENDING IN THE Y - DIRECTION
C INPUT AND OUTPLT ARE THE SAME AS FOR FPLATE
C*****
COMMON / PLBDG / X(5),Y(5),CM(3,3),P(5),EM(3,5),CV(3,5),S(19,19),
* R(19)
C
C DIMENSION FA(4,2), ITE(4), ILE(4), IPE(2), FB(4), FC(4)
C DATA FA /-3.0,-4.0,3.0,-2.0, 3.C, 2.C,-3.0,6.0 /
C DATA ITE /1,2,10,11/, ILE /4,5,7,8/, IPE /1,4/
C
C INITIALIZATION
C DO 100 I=1,3
C DO 100 J=1,5
C 100 BM(I,J) = 0.0
C FCT = CM(1,1)/2.0
C XL = (Y(4) - Y(1))/2.0
C XR = (Y(3) - Y(2))/2.0
C FAC = 0.25*(X(2)-X(1))+X(3)-X(4))
C
C DETERMINE MY-MOMENTS AT THE NODES ASSUMING BEAM BEHAVIOR
C
DO 200 I=1,2
DO 210 J=1,3,2
FB(J) = FA(J,1)/XL/XL
FC(J) = FA(J,1)/XR/XR
210 FC(J) = FA(J,1)/XR/XR
DO 220 J=2,4,2
FB(J) = FA(J,1)/XL
FC(J) = FA(J,1)/XR
IA = IPE(I)
SUM = 0.0
SUN = 0.0
DO 300 J=1,4
JA = ITE(J)
JB = ILE(J)
SUM = SUM + FB(J)*R(JA)
SUN = SUN + FC(J)*R(JB)
300 SUN = SUN + FCT*SUN*FAC
BM(2,IA) = FCT*SUN*FAC
BM(2,IB) = FCT*SUN*FAC
200 CONTINUE
RETURN
END

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```

FLCC 57
FLCC 58
FLCC 59
FLCC 60
FLCC 61
FLCC 62
FLCC 63
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FLCC 68
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FLCC 70
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FLCC 87
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FLCC 89
FLCC 90
FLCC 91
FLCC 92
FLCC 93
FLCC 94
FLCC 95
FLCC 96
FLCC 97
FLCC 98

U2 = U(J)
U3 = U(K)
W2 = 1.-U2
W3 = 1.-U3
C21 = -(2.*W2)*B2-(2.*+U3)*B3
C22 = B2*W2-B3*U3
C31 = -(2.*W2)*A2-(2.*+U3)*A3
C32 = A2*W2-A3*U3
C51 = 4.*B3-B2+B3*W3
C52 = B2-B3*W3
C61 = 4.*A3-A2+A3*W3
C62 = A2-A3*W3
C81 = B3-4.*B2-B2*U2
C82 = B2*U2-B3
C91 = A3-4.*A2-A2*U2
C92 = A2*U2-A3
C021 = -B2-(3.*+U3)*B3
C022 = B3*(3.*+W2)*B2
C031 = -A2-(3.*+U3)*A3
C032 = A3*(3.*+W2)*A2
DO 200 N = 1,3
Q11 = Q(N,I)
Q22 = Q(N,J)
Q33 = Q(N,K)
Q12 = Q(N,I+3)
Q23 = Q(N,I+3)
Q31 = Q(N,K+3)
Q1 = Q22-C33
Q2 = Q22-C33
Q3 = C33-C23
Q4 = C23+C1
Q5 = C23-C1
200 CVT(IN,I) = (-6.*C11+3.*(L3-W2)*Q1+(L3+W2)*Q23))*R(I-2)
+ (6.*Q22+3.*W3*Q4)*R(JJ-2) + (6.*C33+3.*U2*Q5))*R(KK-2)
+ ((C21*Q1+C22*Q23+4.*(B2*Q31-B3*Q12)) *R(I-1)
+ (C31*Q1+C32*Q23+4.*(A2*Q31-A3*Q12)) *R(I)
+ (C51*Q22+C52*Q3) *R(JJ-1) + (C61*Q22+C62*Q3) *R(JJ)
+ (C81*Q33+C82*Q2) *R(KK-1) + (C91*Q33+C92*Q2) *R(KK)
+ FT(K)*Q4*R(K+5) + HT(I)*C5*R(J+5)))/2.
300 CONTINUE
RETURN
END

```

APPENDIX B

Listing of Data for Example 5 -  
Two Cell Box Girder Bridge on  
Skewed Supports.

START  
ANALYSIS OF 45 DEGREE P CELL PHX

168 189 72 4 0 112 3  
 1 1 2 4 5 3 1 1 3  
 2 2 4 5 3 1 2 3  
 3 3 5 6 3 1 2 3  
 4 4 7 8 5 1 3 3  
 5 5 8 9 6 1 2 3  
 6 6 9 10 6 1 2 3  
 7 7 11 12 9 1 3 3  
 8 8 12 13 9 1 3 3  
 10 10 14 15 10 1 3 3  
 11 11 16 17 12 1 3 3  
 12 12 17 18 13 1 2 3  
 15 15 23 21 15 1 2 3  
 16 16 22 23 17 1 3 3  
 17 17 23 24 18 1 2 3  
 21 21 27 28 21 1 2 3  
 22 22 29 30 23 1 3 3  
 23 23 30 31 24 1 2 3  
 28 28 35 34 28 1 2 3  
 29 29 37 38 30 1 3 3  
 30 30 38 39 31 1 2 3  
 36 36 44 45 36 1 3 3  
 37 37 45 47 38 1 3 3  
 38 38 47 49 39 1 2 3  
 44 44 53 54 45 1 3 3  
 45 46 55 56 47 1 2 3  
 46 47 56 57 48 1 2 3  
 52 53 62 53 54 1 3 3  
 133 145 154 146 145 1 3 3  
 134 146 154 155 147 1 2 3  
 140 152 160 161 153 1 2 3  
 141 154 162 155 154 1 2 3  
 142 155 162 163 156 1 2 3  
 147 160 167 168 161 1 2 3  
 148 162 169 163 162 1 2 3  
 149 163 169 170 164 1 2 3  
 153 167 173 174 168 1 2 3  
 154 169 175 176 169 1 2 3  
 155 170 175 176 171 1 2 3  
 158 173 178 179 174 1 3 3  
 159 175 180 178 175 1 2 3  
 160 176 180 181 177 1 2 3  
 162 179 182 183 179 1 2 3  
 163 180 184 181 180 1 2 3  
 164 181 184 185 182 1 3 3  
 165 182 185 186 183 1 2 3  
 166 184 187 185 184 1 2 3  
 167 185 187 189 186 1 3 3  
 168 187 189 188 187 1 3 3  
 1 3 6 4  
 2 6 10 4  
 3 10 15 4  
 4 15 21 4  
 5 21 28 4

9 132

6 28 36 4  
 7 154 162 4  
 8 162 169 4  
 9 166 175 4  
 10 175 180 4  
 11 180 184 4  
 12 184 187 4  
 13 3 5 4  
 14 5 8 4  
 15 8 12 4  
 16 12 17 4  
 17 17 23 4  
 18 23 30 4  
 19 30 38 4  
 20 38 47 4  
 32 146 154 4  
 33 15 20 4  
 34 20 26 4  
 35 26 33 4  
 36 33 41 4  
 37 41 50 4  
 49 149 157 4  
 50 157 164 4  
 51 164 170 4  
 52 170 175 4  
 53 36 44 4  
 54 44 53 4  
 66 152 160 4  
 67 160 167 4  
 68 167 173 4  
 69 173 178 4  
 70 178 182 4  
 71 182 185 4  
 72 185 187 4  
 1 23.  
 3.  
 33.  
 46.  
 66.  
 79.  
 109.  
 1112.  
 1512.  
 1615.  
 2115.  
 2218.  
 2815.  
 2921.  
 3621.  
 3724.  
 4524.  
 4627.  
 5427.  
 5530.  
 6330.  
 13463.  
 3.  
 6.  
 9.  
 12.  
 15.  
 18.  
 21.  
 24.  
 24.  
 24.  
 3.

16163. 24.  
 16266. 6.  
 16866. 24.  
 16969. 9.  
 17469. 24.  
 17572. 12.  
 17972. 24.  
 18075. 15.  
 18375. 24.  
 18478. 18.  
 186 78. 24.  
 18781. 21.  
 18881. 24.  
 18984. 24.

1432000. 432000. 187826.087.15 .542  
 2432000. 432000. 187826.087.15 .458  
 3432000. 432000. 187826.087.15  
 4432000. 432000. 187826.087.15 .667

3 T T T  
 15 T T  
 36 T T T T  
 154 T T  
 175 T T  
 187 T T

LOAD

1  
 65 -1.0  
 1 49 12 -45.0  
 3 5 8 12 17 23 30 38 47 56 74 83 92 101 110 119 128 137 146 154  
 36 44 53 62 71 80 89 98 107 116 134 143 152 160 167 173 178 182 185 187  
 55 65 75 85 95 105 115 125 135